

# Asymptotic Models of Navier-Stokes Equations:

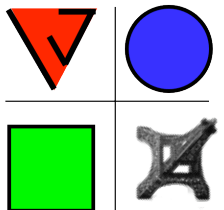
## Applications in Biomechanics

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# Aim

- simplification of Navier Stokes equations
- thanks to asymptotic theory:  
“Boundary Layer”

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- simplification of Navier Stokes equations
- thanks to asymptotic theory:  
“Boundary Layer”

Starting from Navier Stokes (Axi)

- we simplify NS to a Reduced set of equations
  - which contains the physical scales,
  - the most important phenomena
- much more simple set of equations: Integral equations (1D)
- cross comparisons in some cases of NS/ RNSP/ Integral

Prandtl 04  
Golstein 48

*paradox of upstream influence*

- *Triple Deck*

Lighthill  
Stewartson Neiland Messiter 69  
Smith

- *Interactive Boundary Layer / Viscous Inviscid Interactions*

Le Balleur 78, Carter 79, Cebeci 70s  
Veldman 81

- *Boundary layer Asymptotics*  
Sychev, Ruban, Sychev, Korelev, 98  
Sobey 00  
Cebeci Cousteix 01  
Mauss Cousteix 07 (SCEM)

3

full NS 3D

2

NS 2D/Axi

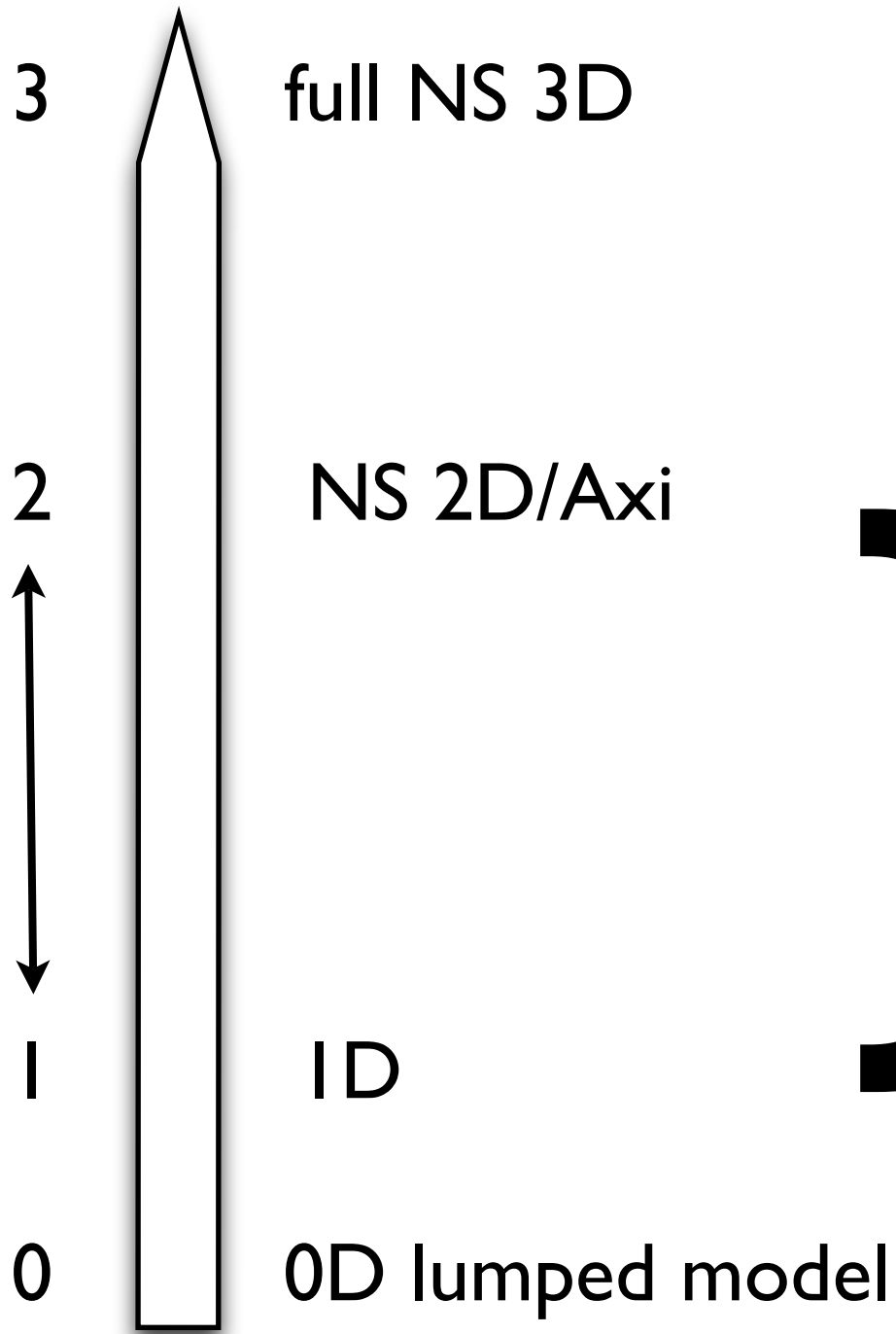
1

1D

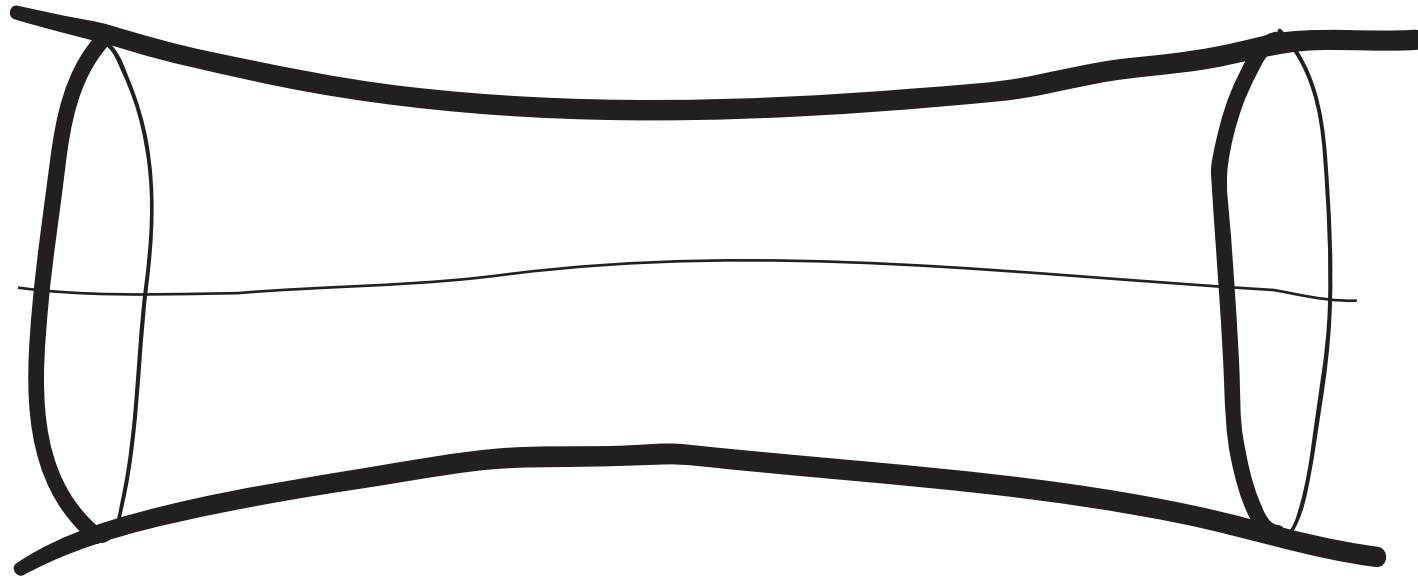
0

0D lumped model

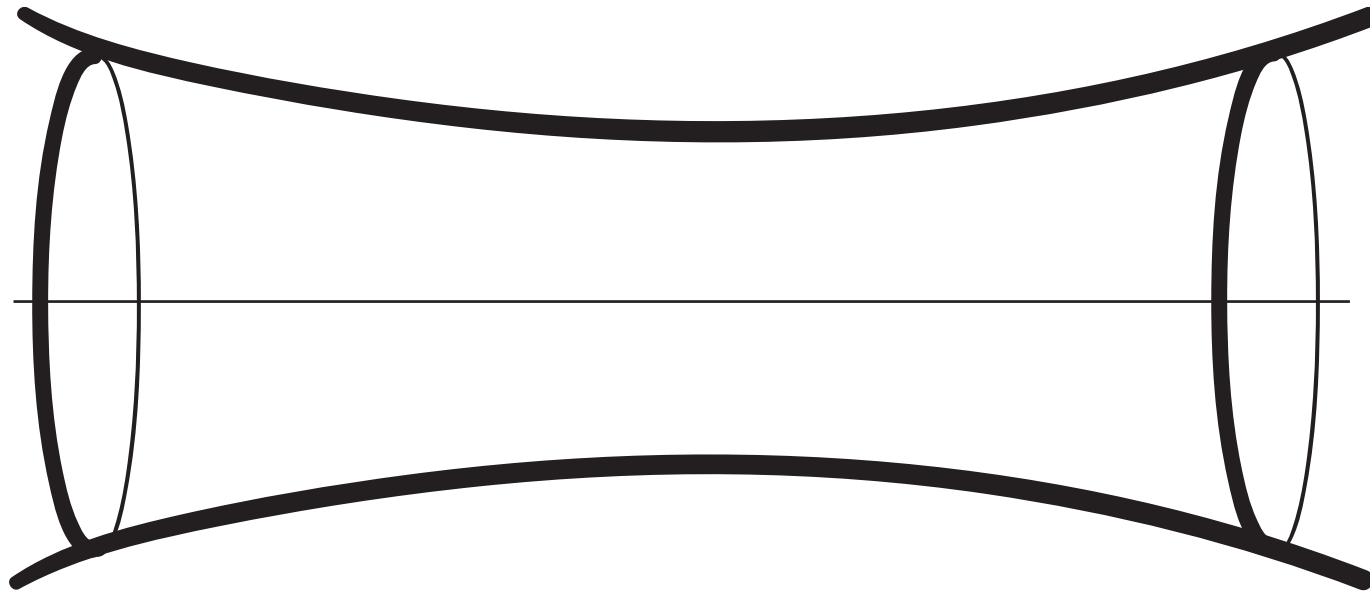




Our model equations

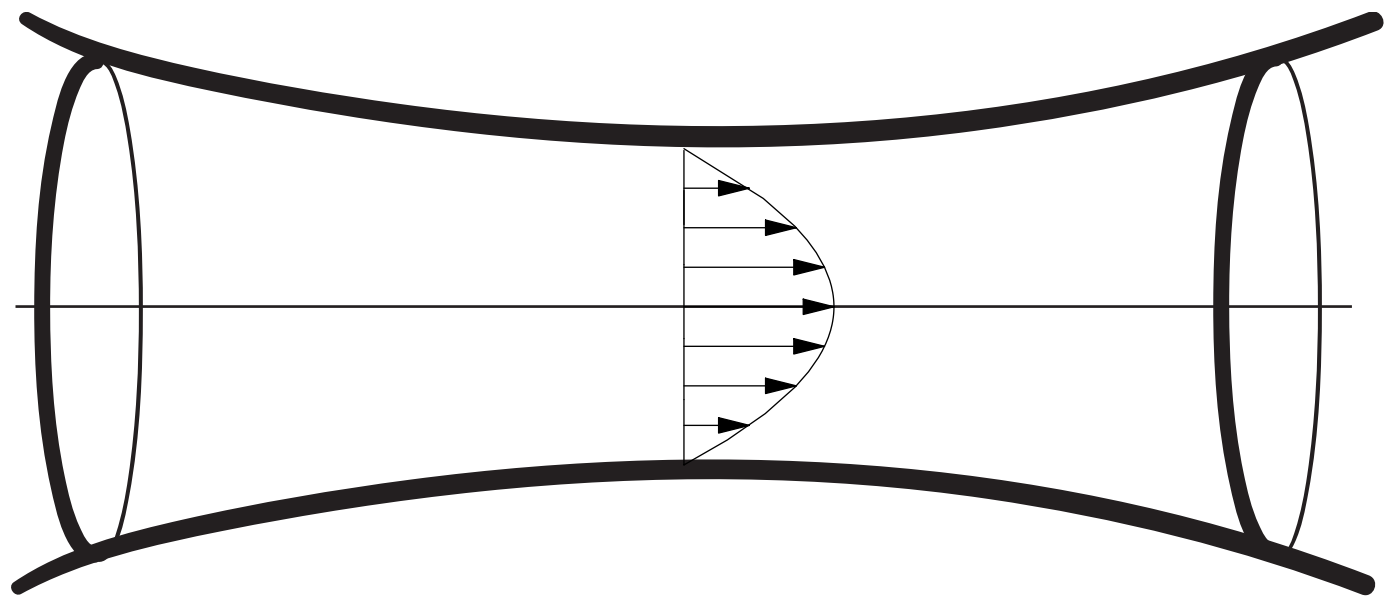


reality?

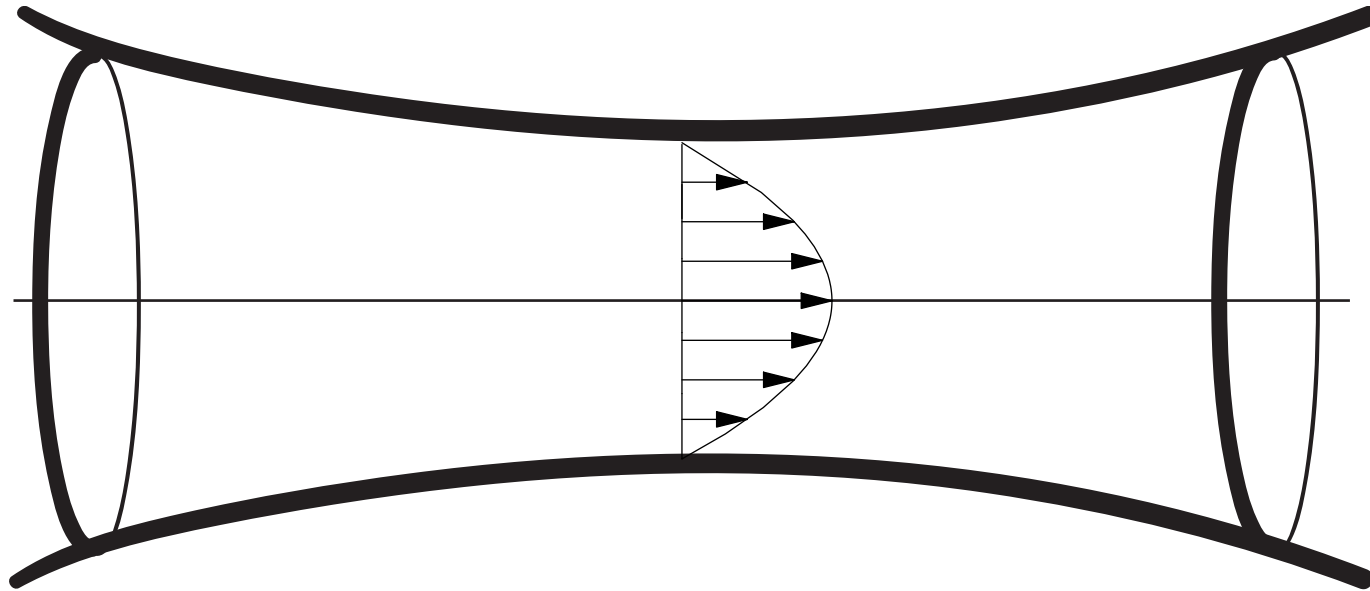


straight pipe, smooth walls, symmetry

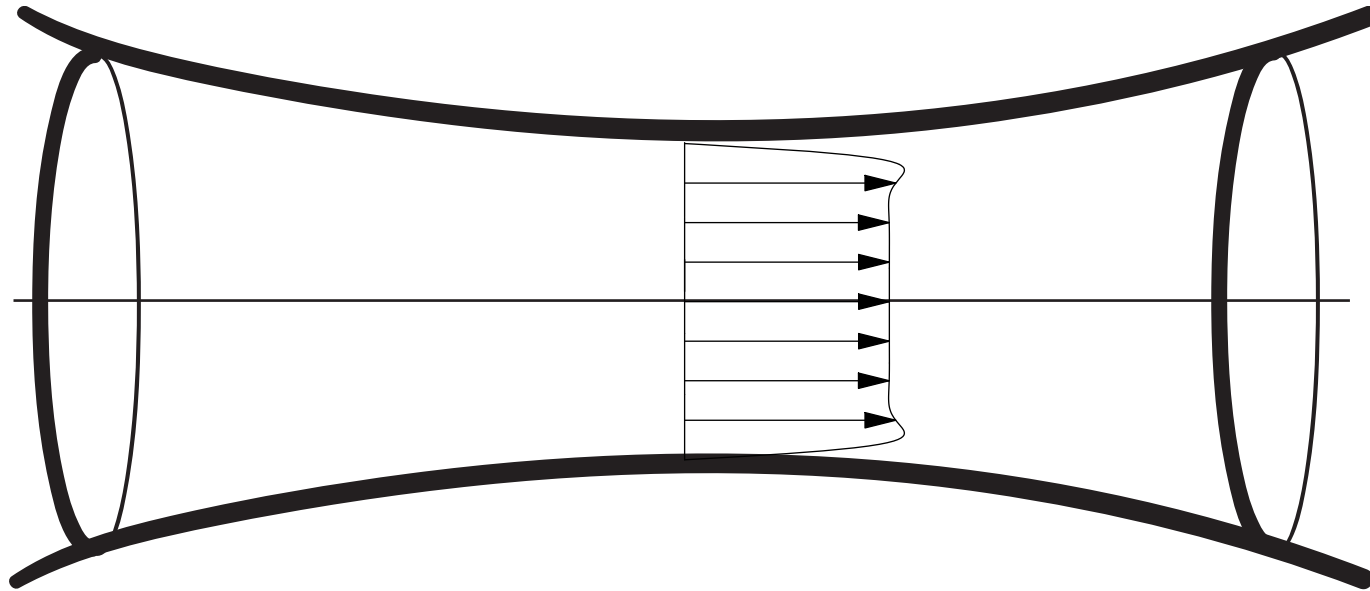




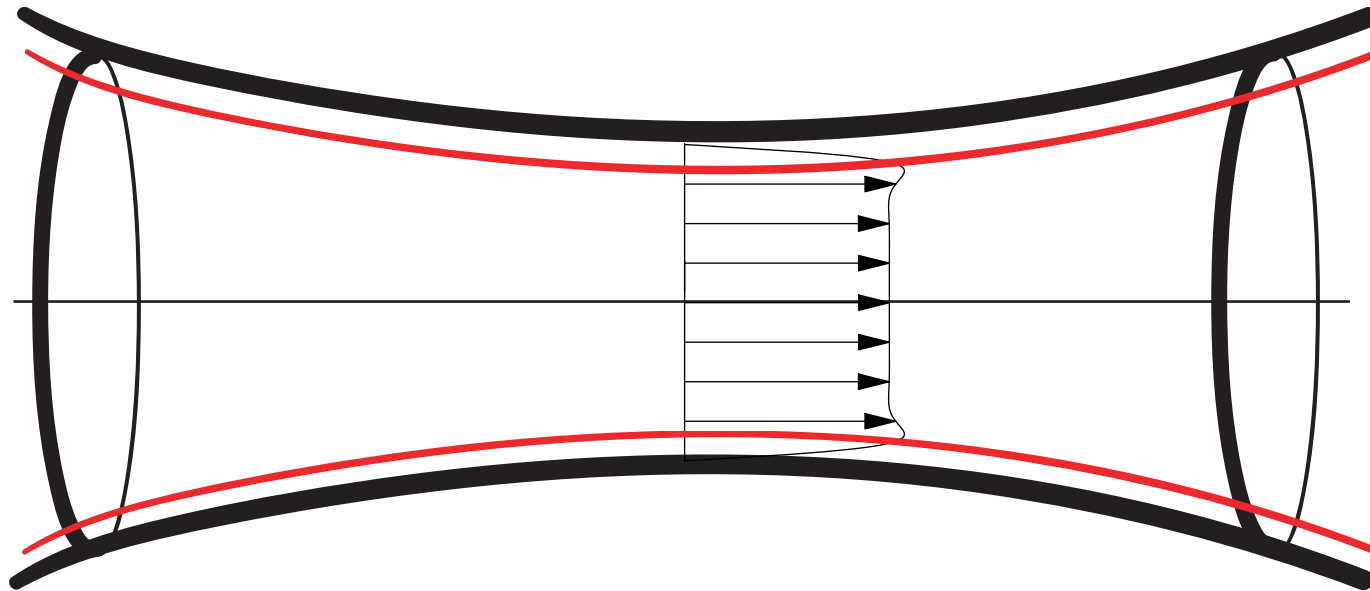
# Interactive Boundary Layer



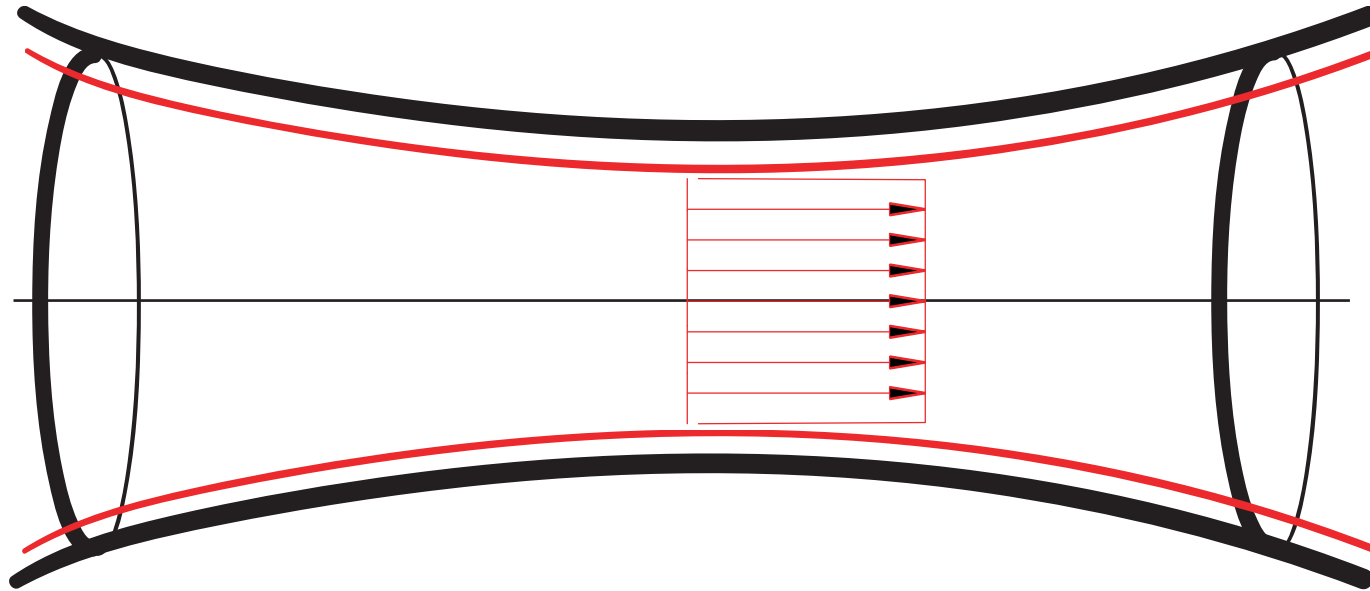
# Interactive Boundary Layer



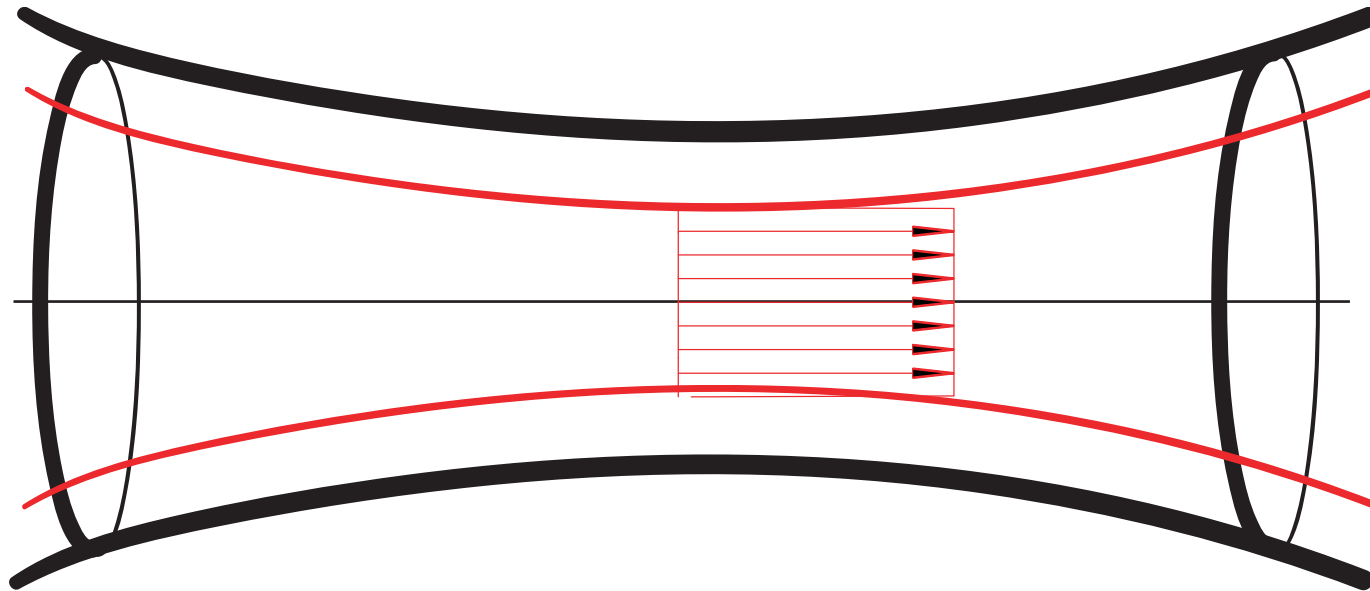
# Interactive Boundary Layer



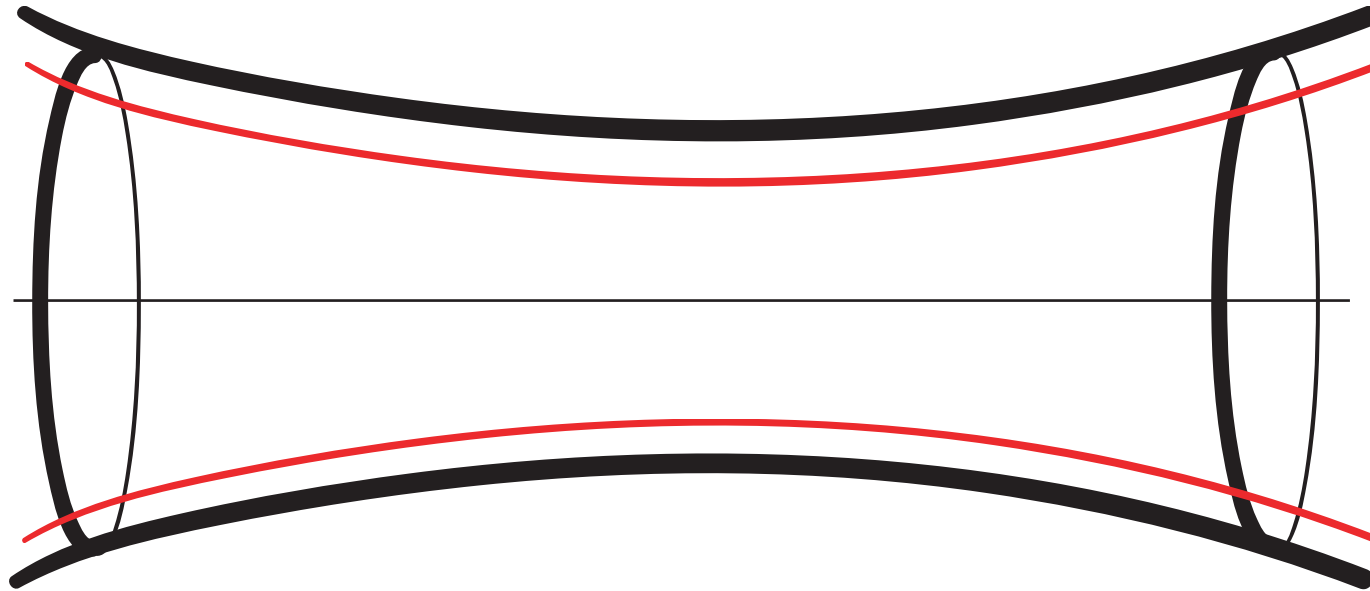
# Interactive Boundary Layer



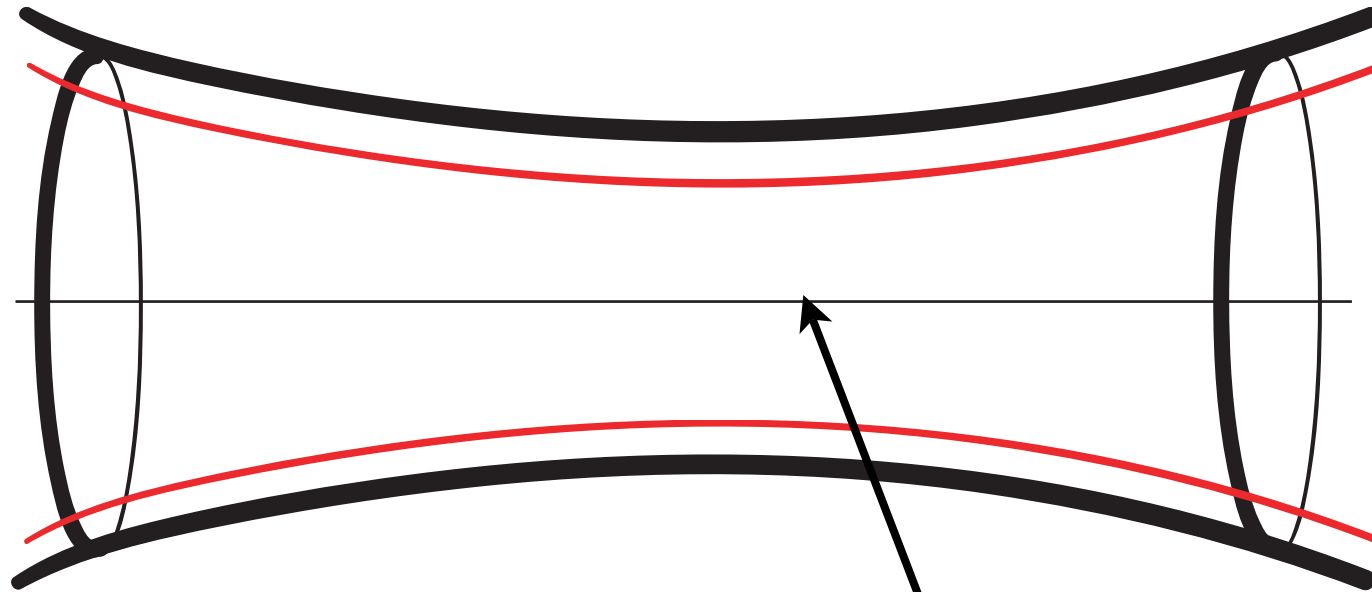
# Interactive Boundary Layer



# Interactive Boundary Layer



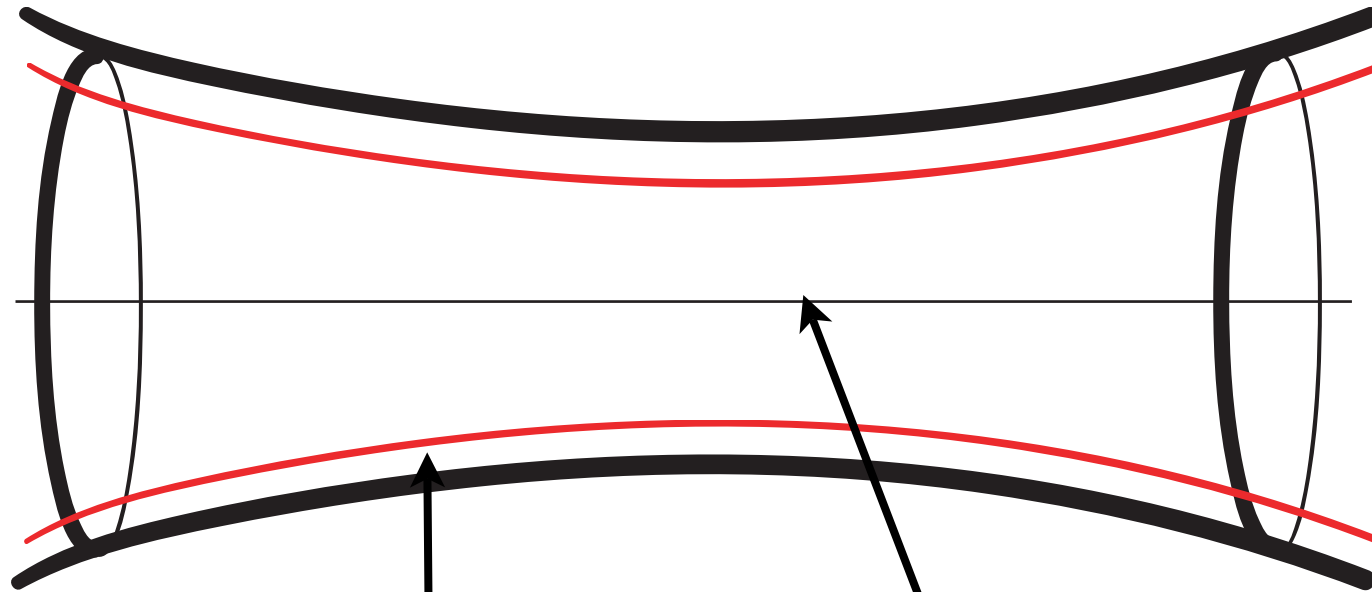
# Interactive Boundary Layer



Ideal fluid region  
flat profile



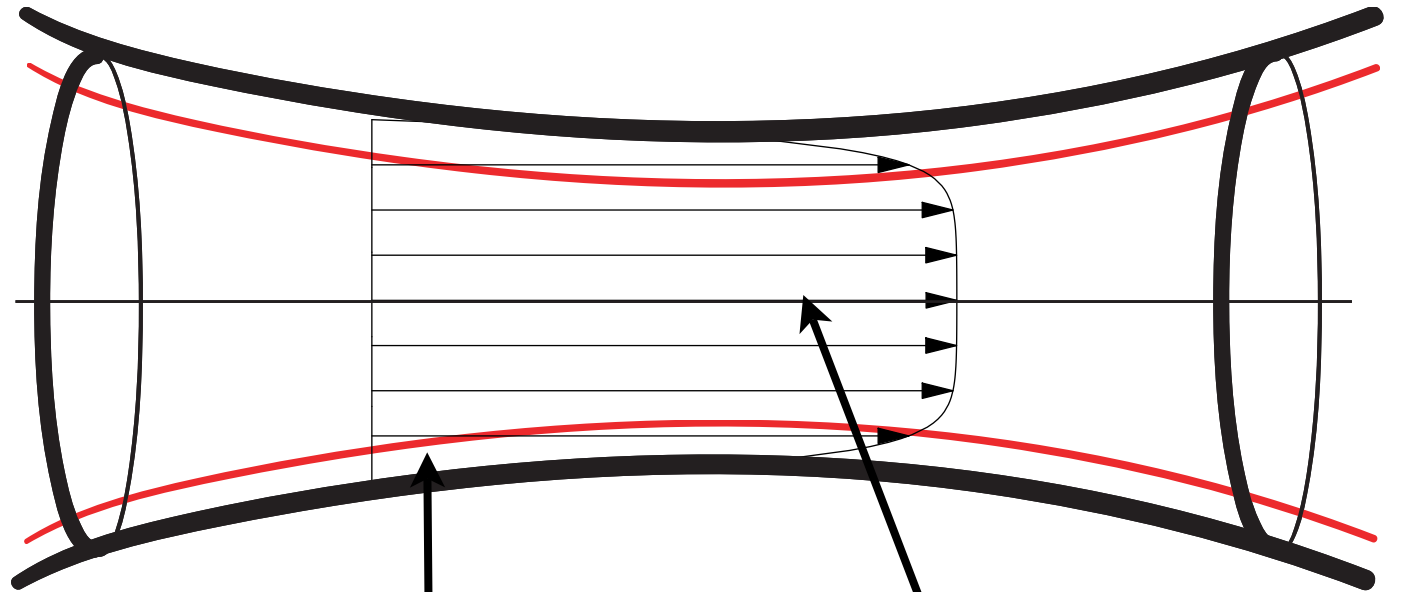
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Ideal fluid region  
flat profile

Viscous region: boundary layer

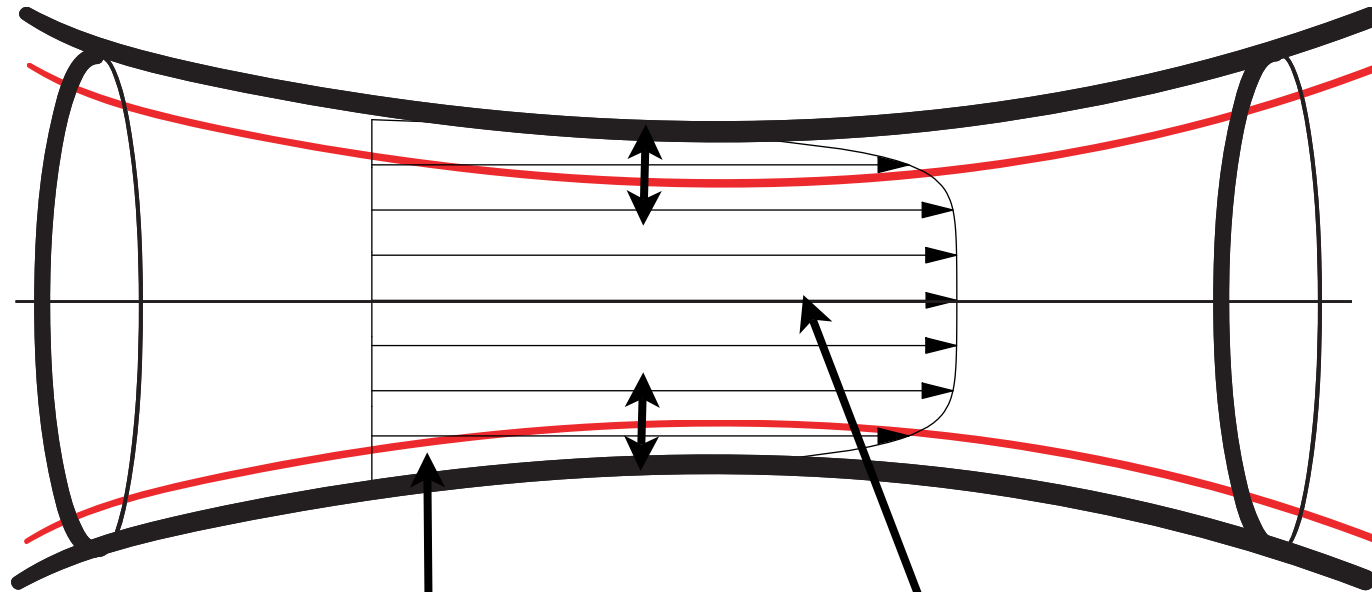
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flat profile

Viscous region: boundary layer

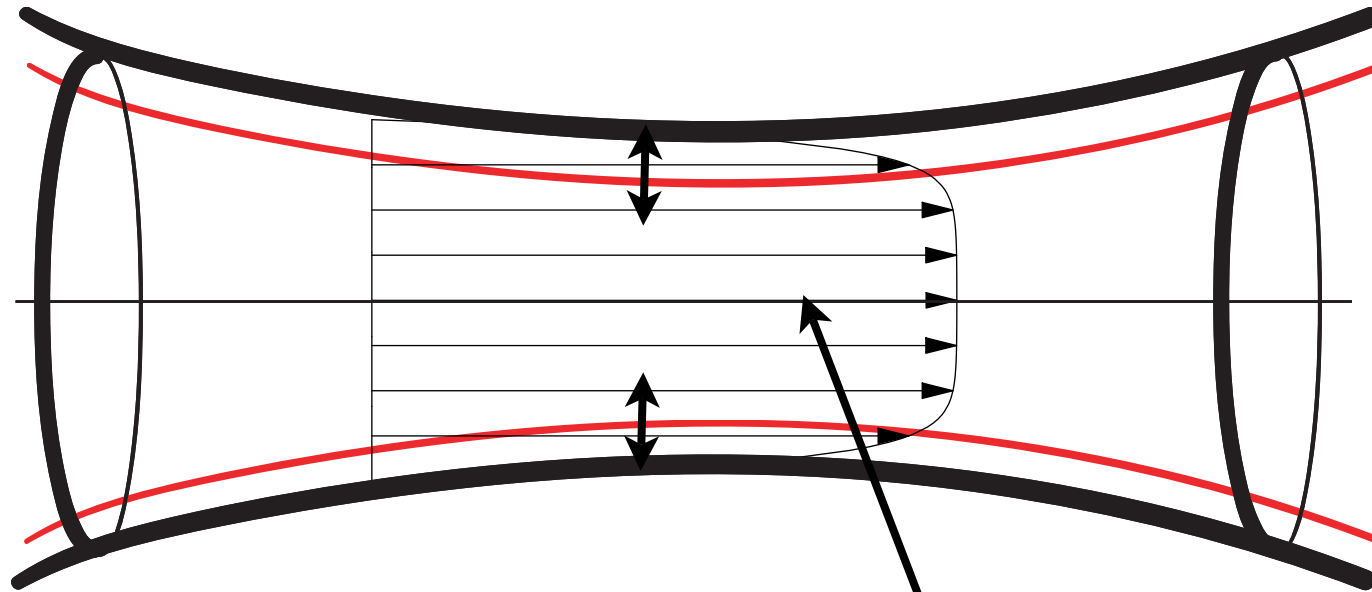
# Interactive Boundary Layer



Ideal fluid region  
flat profile

Viscous region: boundary layer

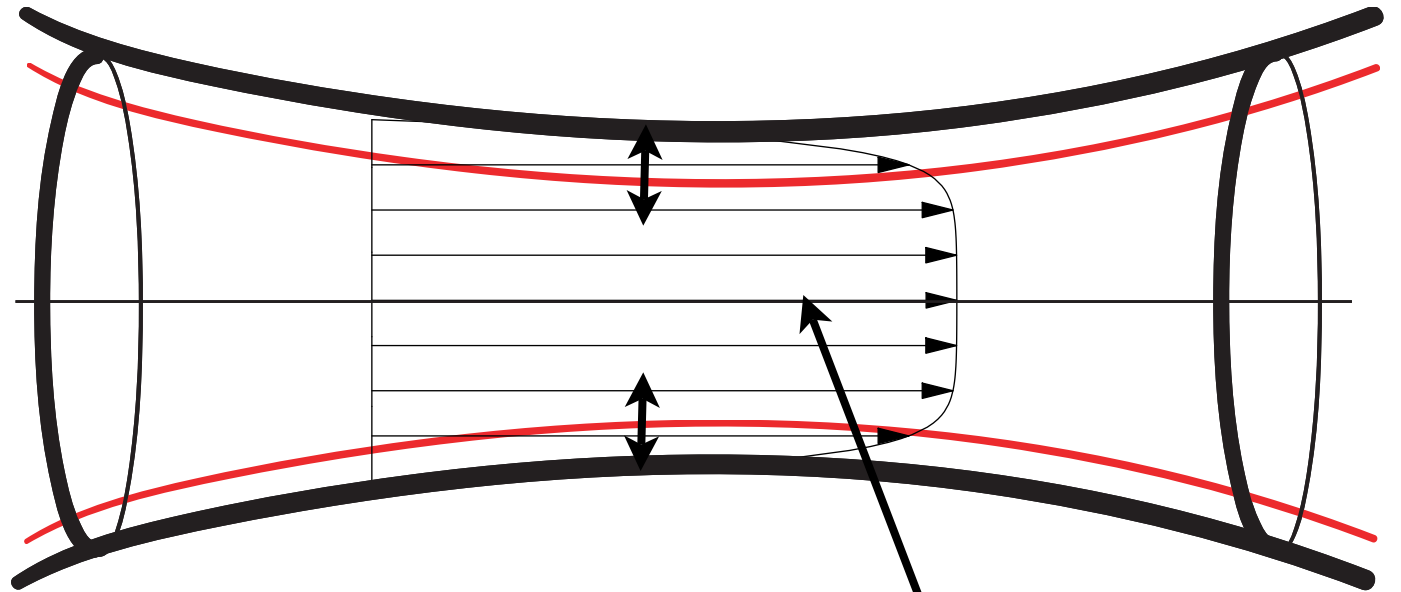
# Interactive Boundary Layer



Ideal fluid region  
flat profile

steady/ or large convective acceleration

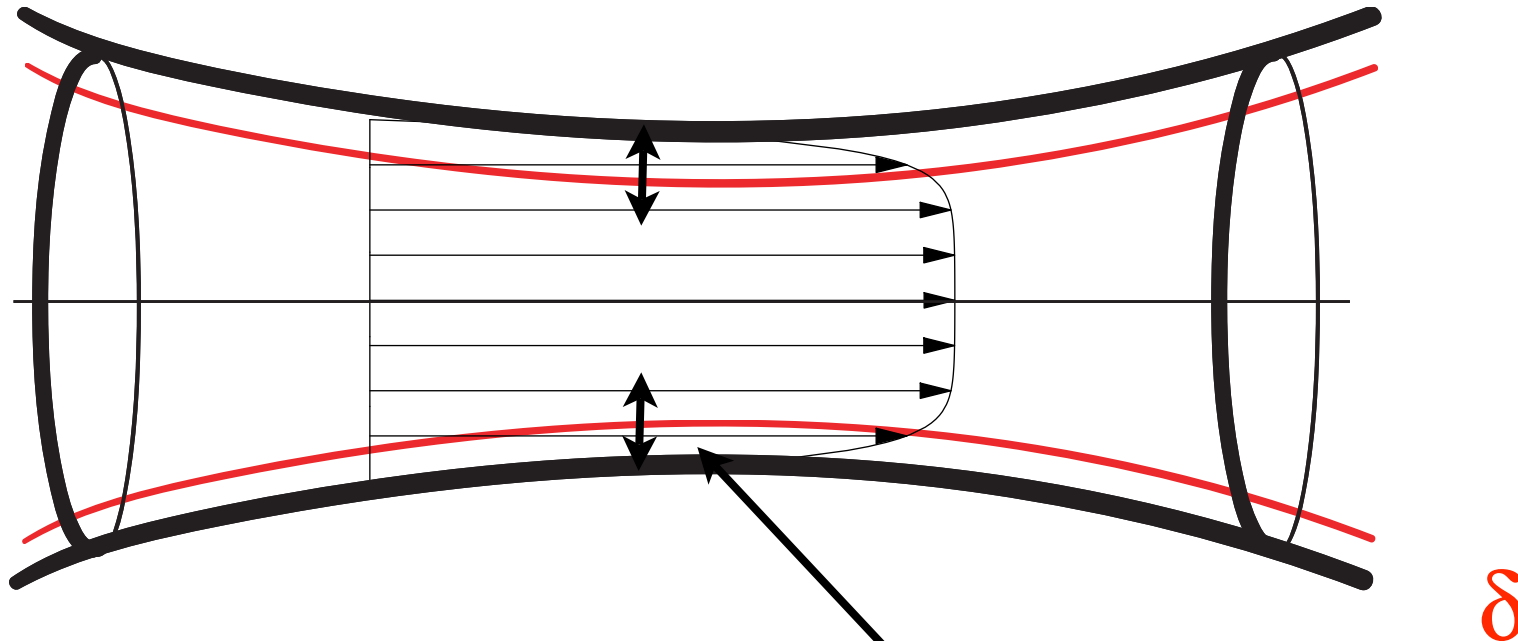
# Interactive Boundary Layer



Ideal fluid region  
flat profile

$$U_e S = cst$$

# Interactive Boundary Layer



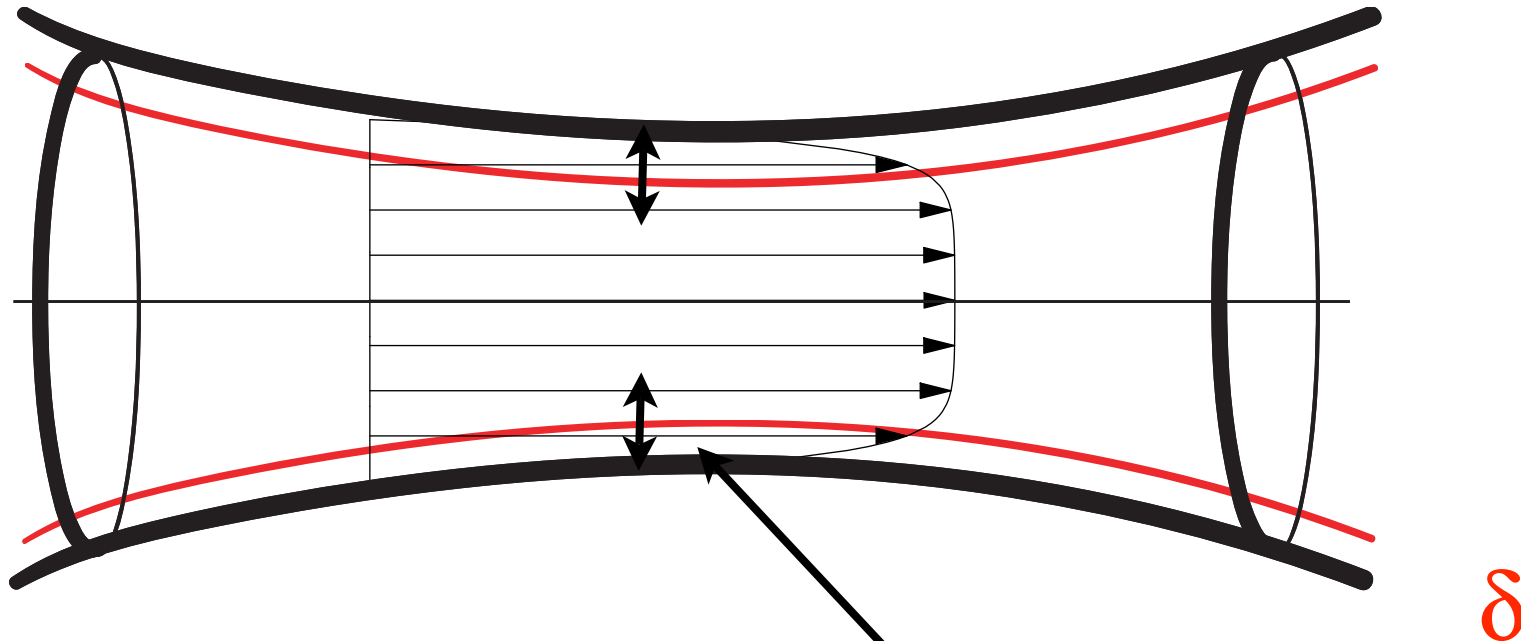
$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$

Viscous region: boundary layer

$$\cancel{\frac{\partial u}{\partial t}} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + \nu \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r} + \dots \cancel{X}$$

steady/ or large convective acceleration

# Interactive Boundary Layer



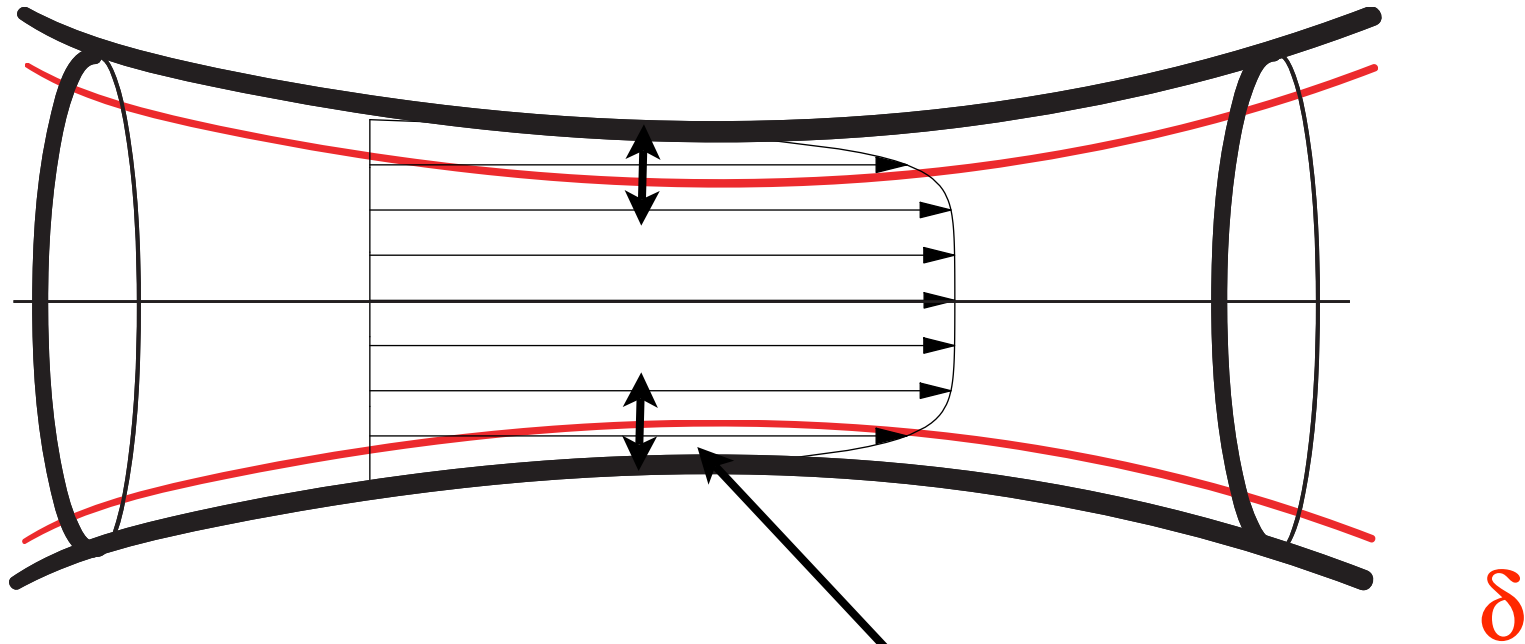
$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$

Viscous region: boundary layer

$$\boxed{\frac{U_0^2}{\lambda}} = -\frac{\partial p}{\rho \partial x} + \boxed{\frac{v}{U_0 \lambda} \frac{\lambda^2}{\delta^2} \frac{U_0^2}{\lambda}} = -\frac{\partial p}{\rho \partial r}$$

steady/ or large convective acceleration

# Interactive Boundary Layer



$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$

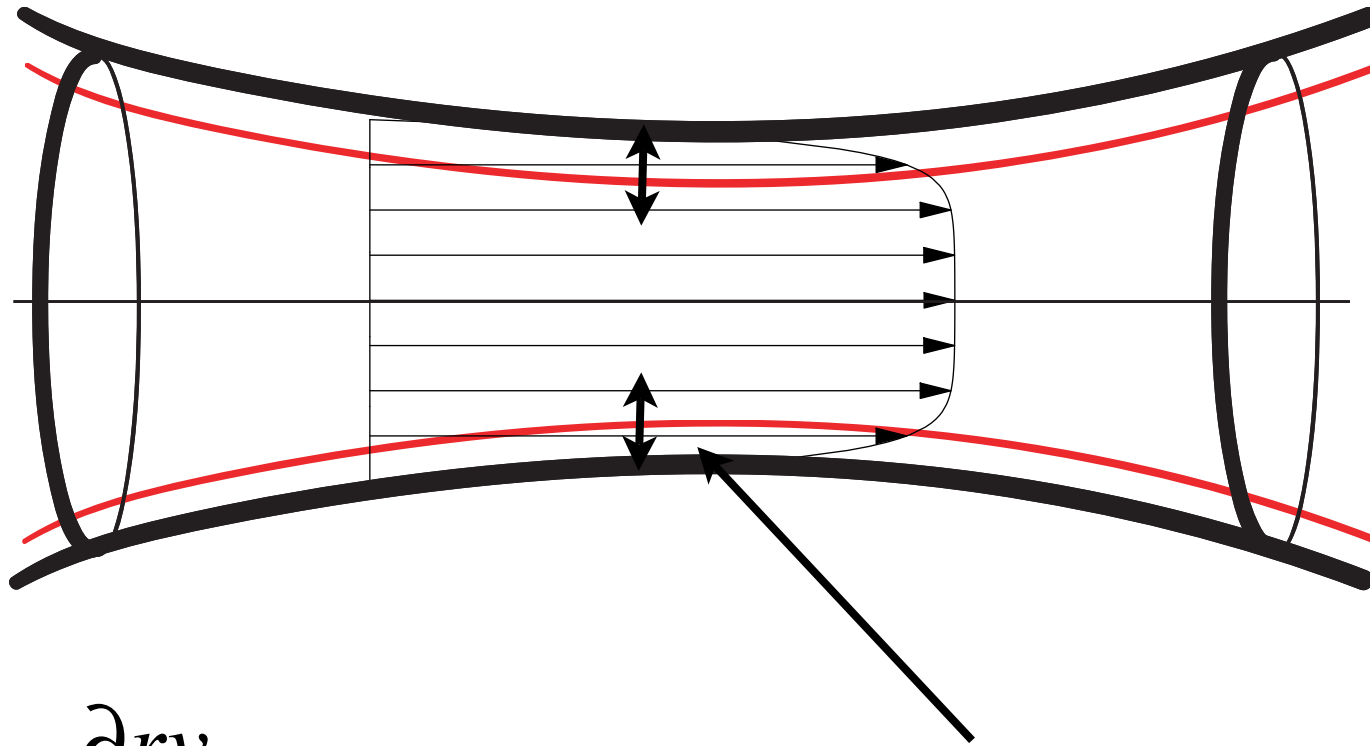
Viscous region: boundary layer

$$\boxed{\frac{U_0^2}{\lambda}} = -\frac{\partial p}{\rho \partial x} + \boxed{\frac{1}{Re} \frac{\lambda^2 U_0^2}{\delta^2 \lambda}} = -\frac{\partial p}{\rho \partial r}$$

steady/ or large convective acceleration



# Interactive Boundary Layer



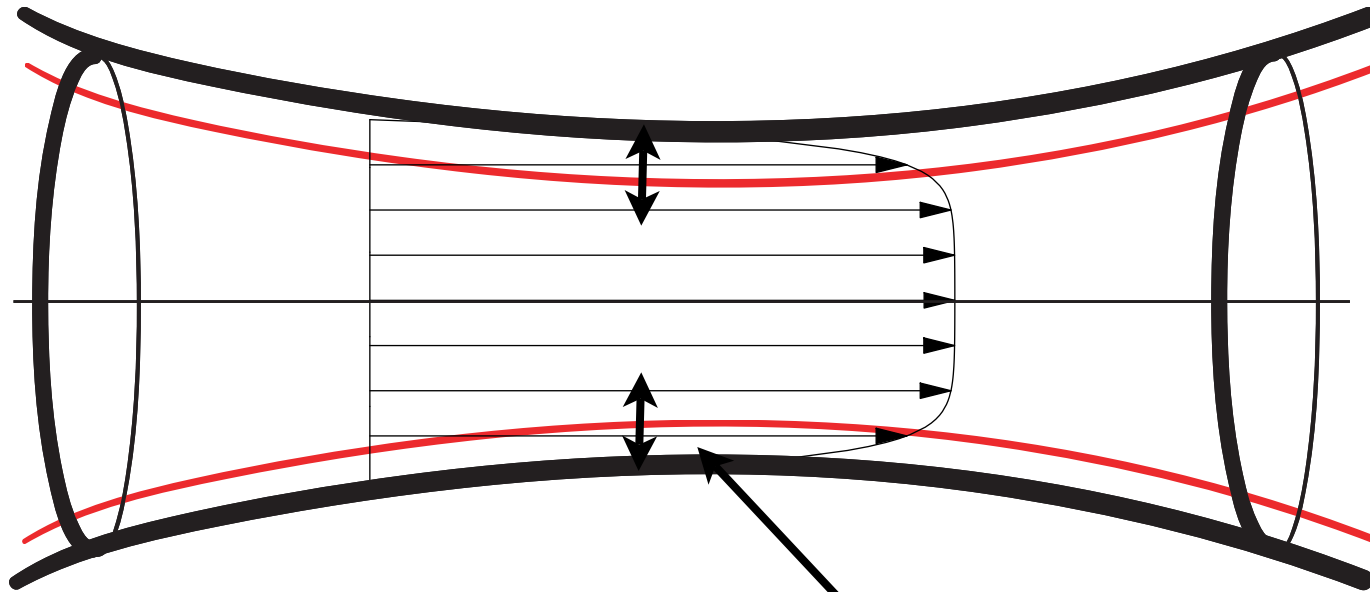
$$\delta \sim \frac{\lambda}{Re^{1/2}}$$

$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r \partial r} = 0$$

Viscous region: boundary layer

$$\frac{U_0^2}{\lambda} = -\frac{\partial p}{\rho \partial x} + \frac{1}{Re} \frac{\lambda^2 U_0^2}{\delta^2 \lambda} = -\frac{\partial p}{\rho \partial r}$$

# Interactive Boundary Layer



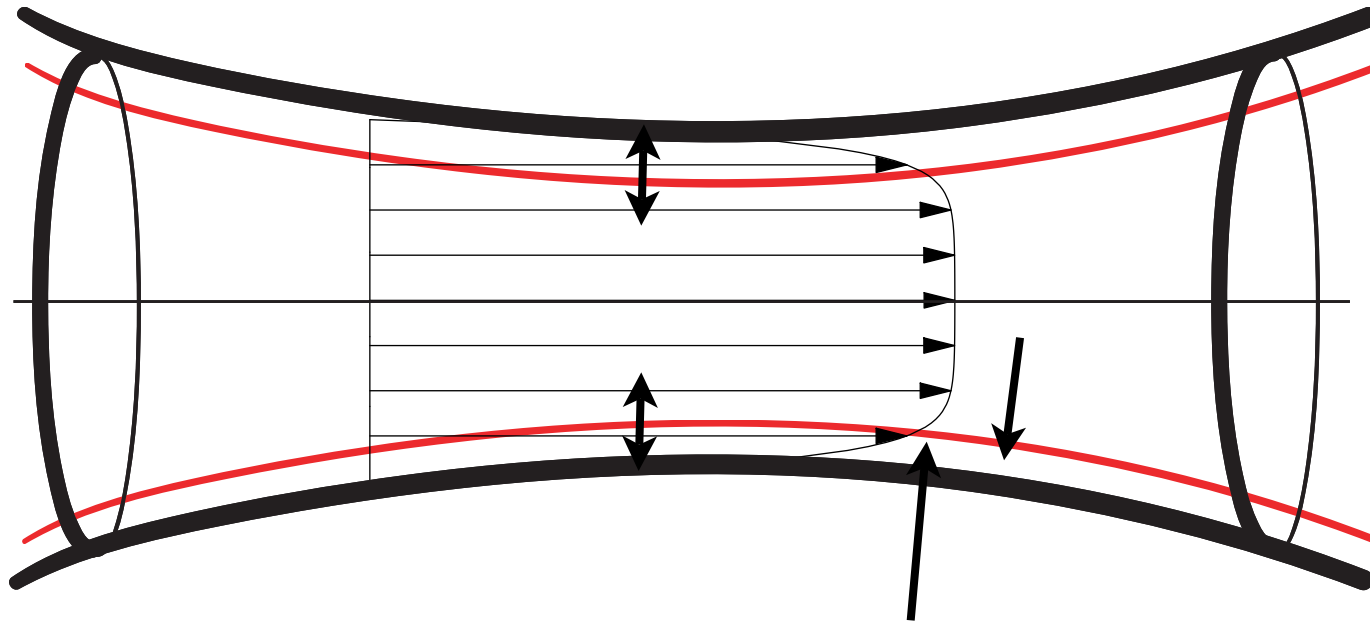
$$\delta \sim \frac{\lambda}{Re^{1/2}}$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial n} = 0$$

Viscous region: boundary layer

$$\boxed{u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial n} u} = -\frac{\partial p}{\partial x} + \boxed{\frac{\partial^2}{\partial n^2} u} \quad 0 = -\frac{\partial p}{\partial n}$$

# Interactive Boundary Layer

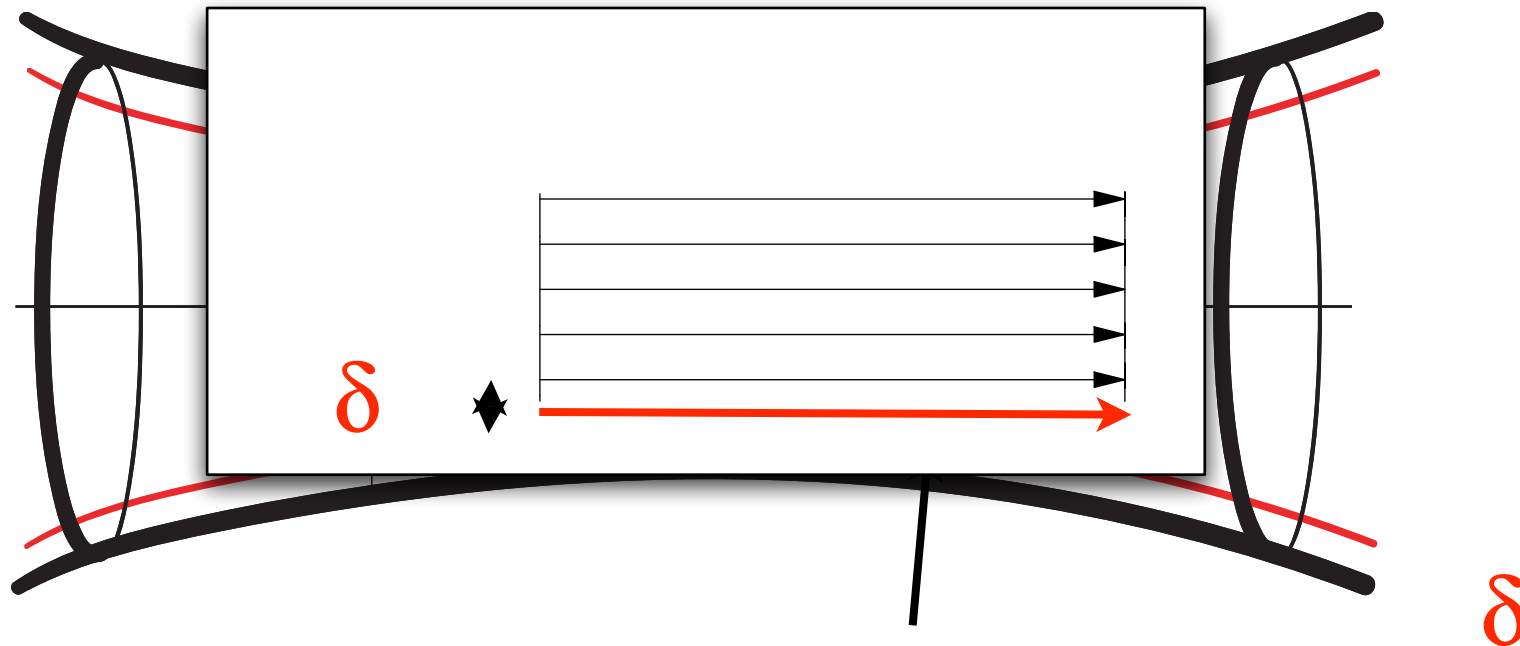


$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial n} = 0$$

Matching of velocity  
from invicid/ viscous

$$u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial n} u = -\frac{\partial p}{\partial x} + \frac{\partial^2}{\partial n^2} u \quad 0 = -\frac{\partial p}{\partial n}$$

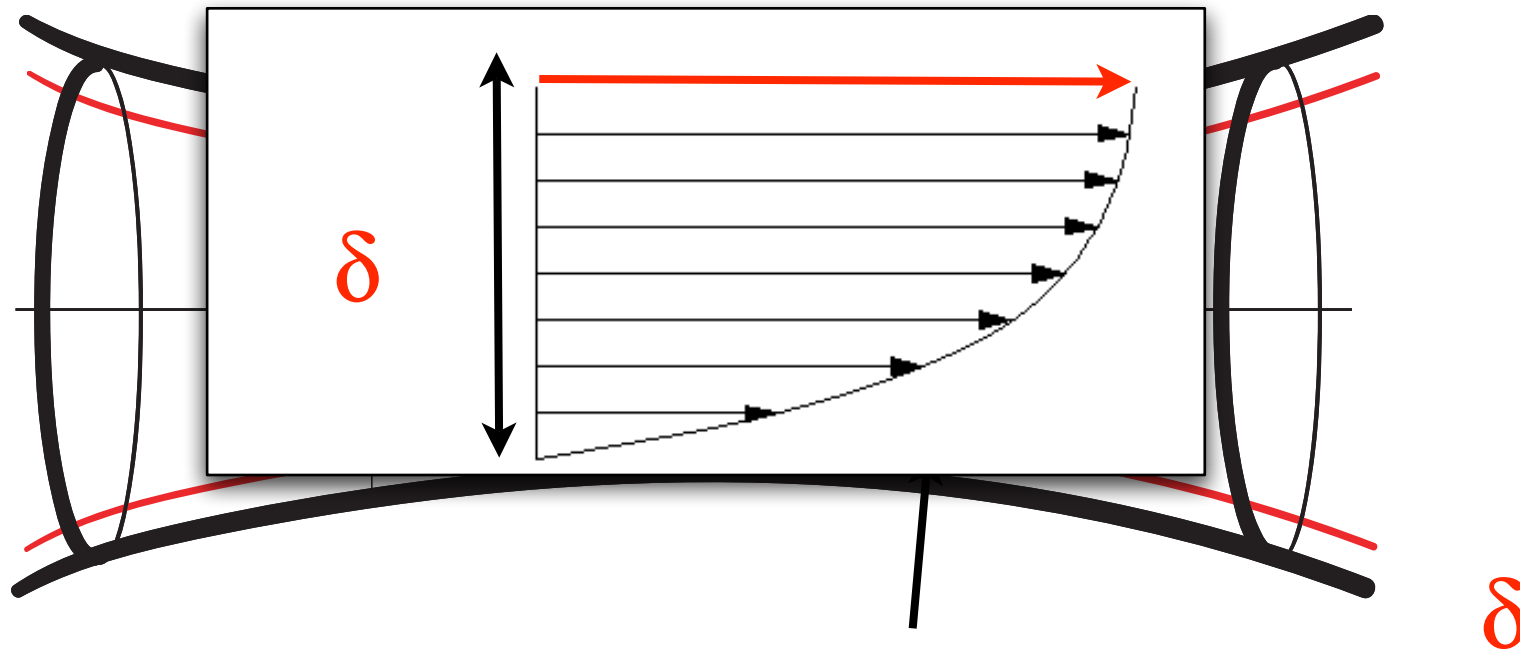
# Interactive Boundary Layer



Matching of velocity  
from invicid/ viscous

$U_e$  at the wall

# Interactive Boundary Layer

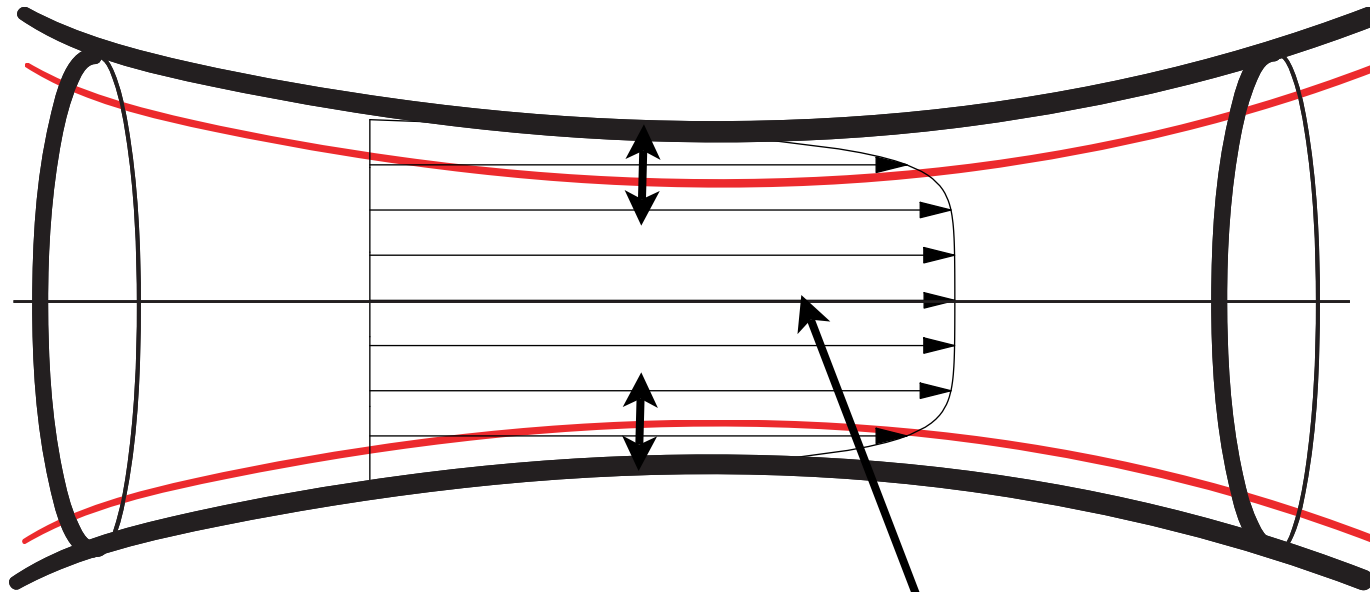


Matching of velocity  
from invicid/ viscous

$U_e$  at the wall

is the velocity at the edge of the boundary layer at "infinity"  $u(x, \infty)$

# classical Boundary Layer

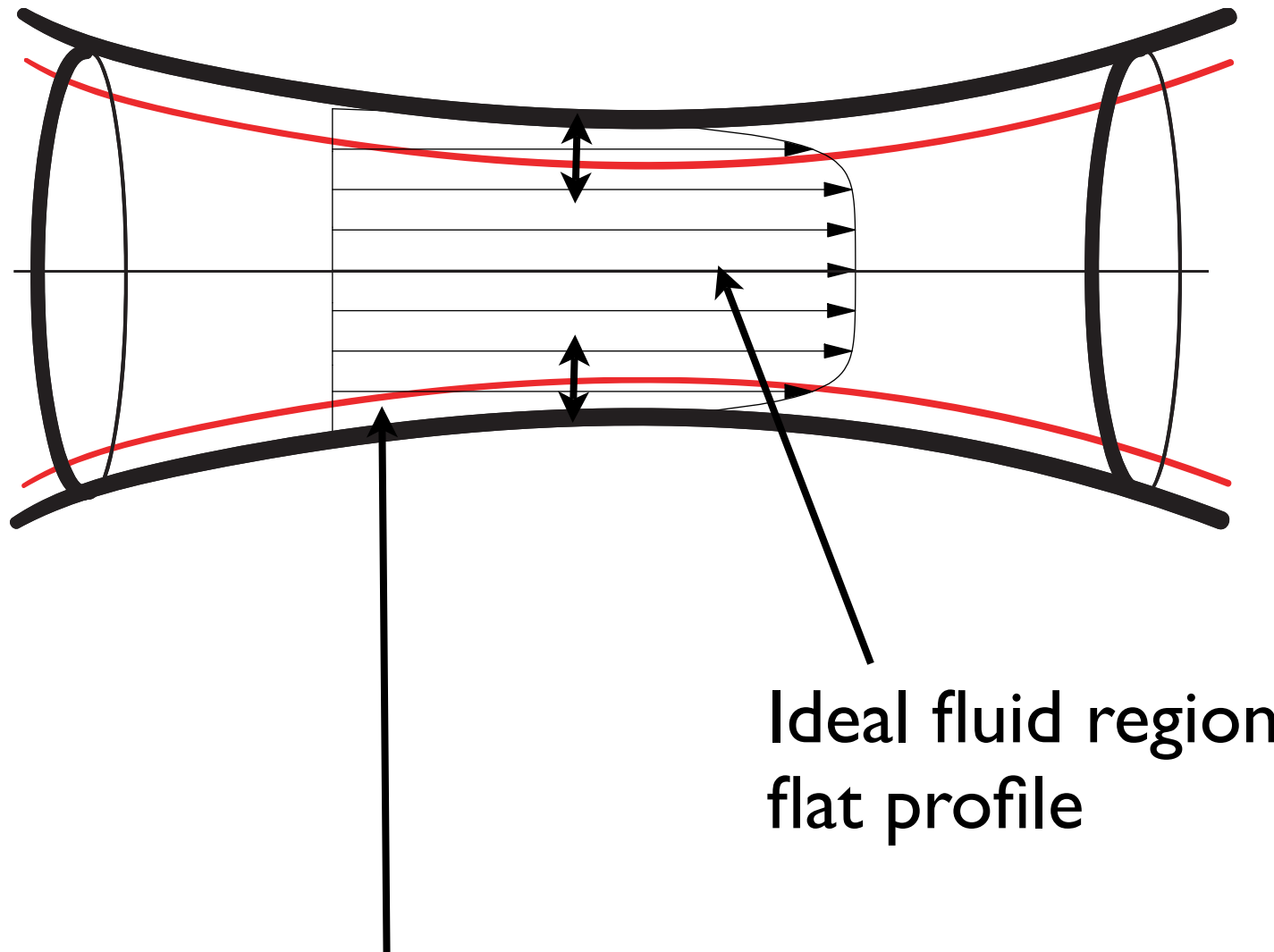


$$U_e(1 - (f'''))^2 = 1$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial n} = 0 \quad u(x, \infty) = U_e$$

$$u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial n} u = \frac{dU_e}{dx} + \frac{\partial^2}{\partial n^2} u$$

# Interactive Boundary Layer

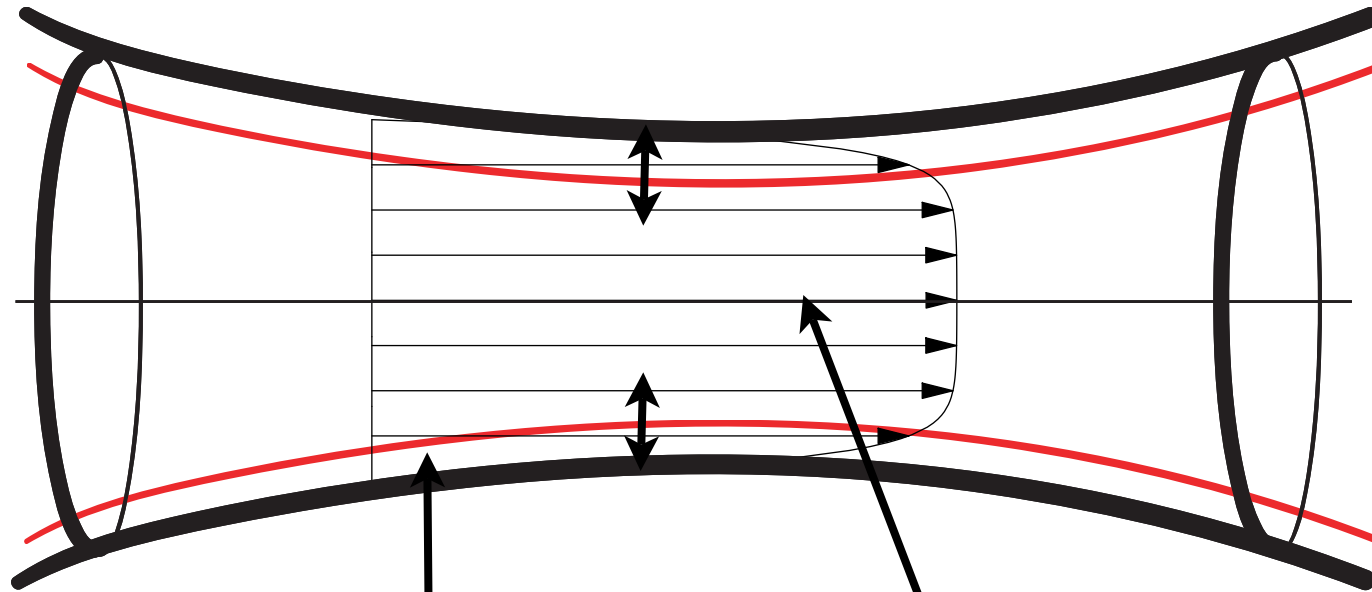


Ideal fluid region  
flat profile

$$\delta_1 = \int_0^{\infty} \left(1 - \frac{u}{U_e}\right) dn$$

displacement of stream lines

# Interactive Boundary Layer

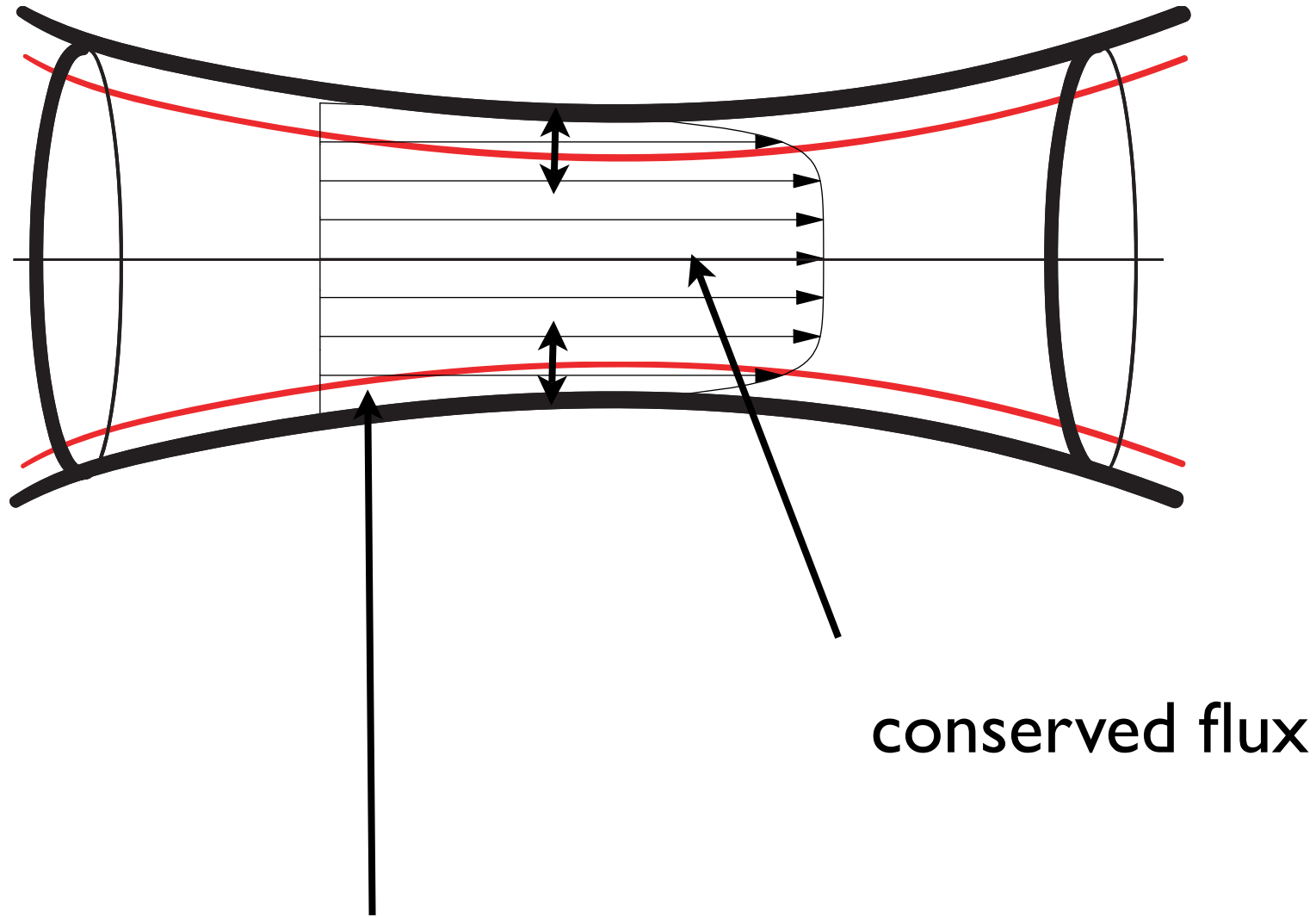


Ideal fluid region  
flat profile perturbed by the  
boundary layer thickness

$$\delta_1 = \int_0^{\infty} \left(1 - \frac{u}{U_e}\right) dn$$



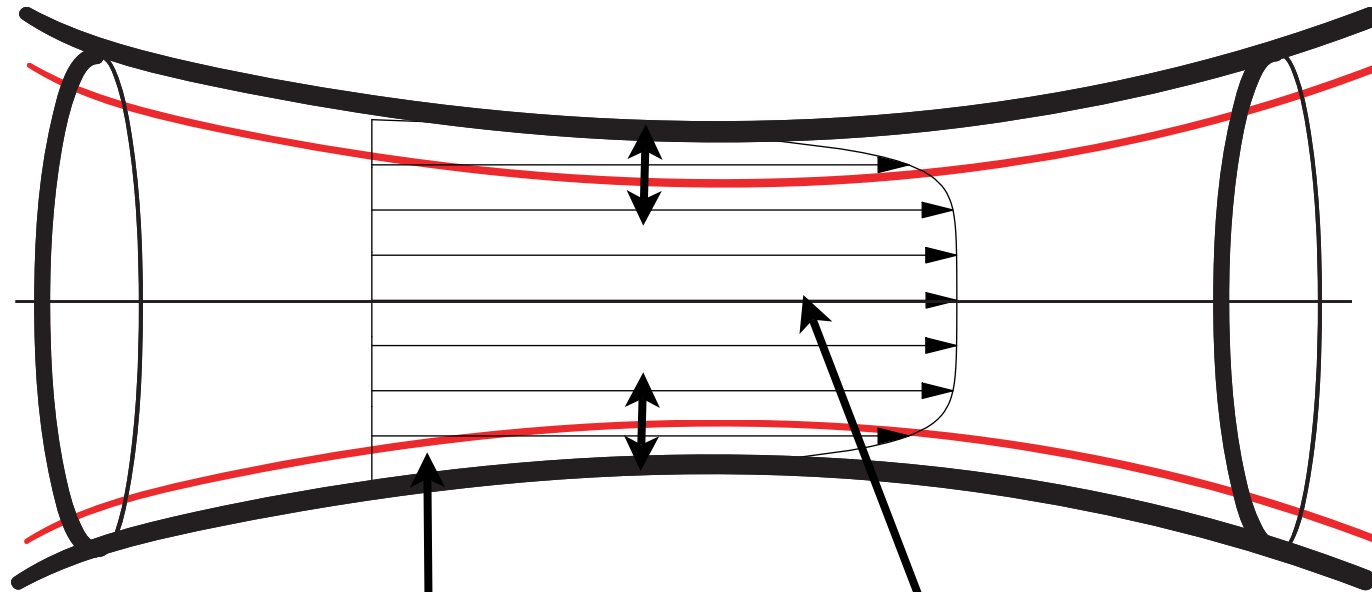
# Interactive Boundary Layer



conserved flux

$$\delta_1 = \int_0^{\infty} \left(1 - \frac{u}{U_e}\right) dn$$

# Interactive Boundary Layer

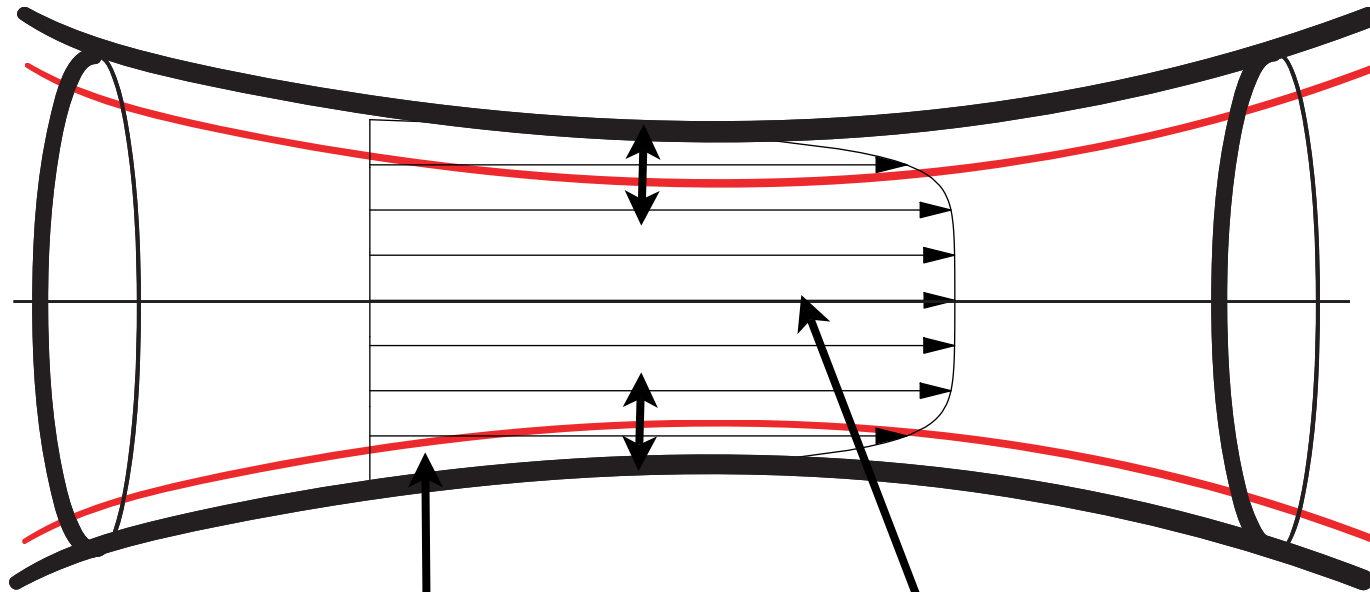


$$U_e \left( 1 - \left( f' \right)^2 \right) = 1$$

$$\delta_1 = \int_0^\infty \left( 1 - \frac{u}{U_e} \right) dn$$



# Interactive Boundary Layer



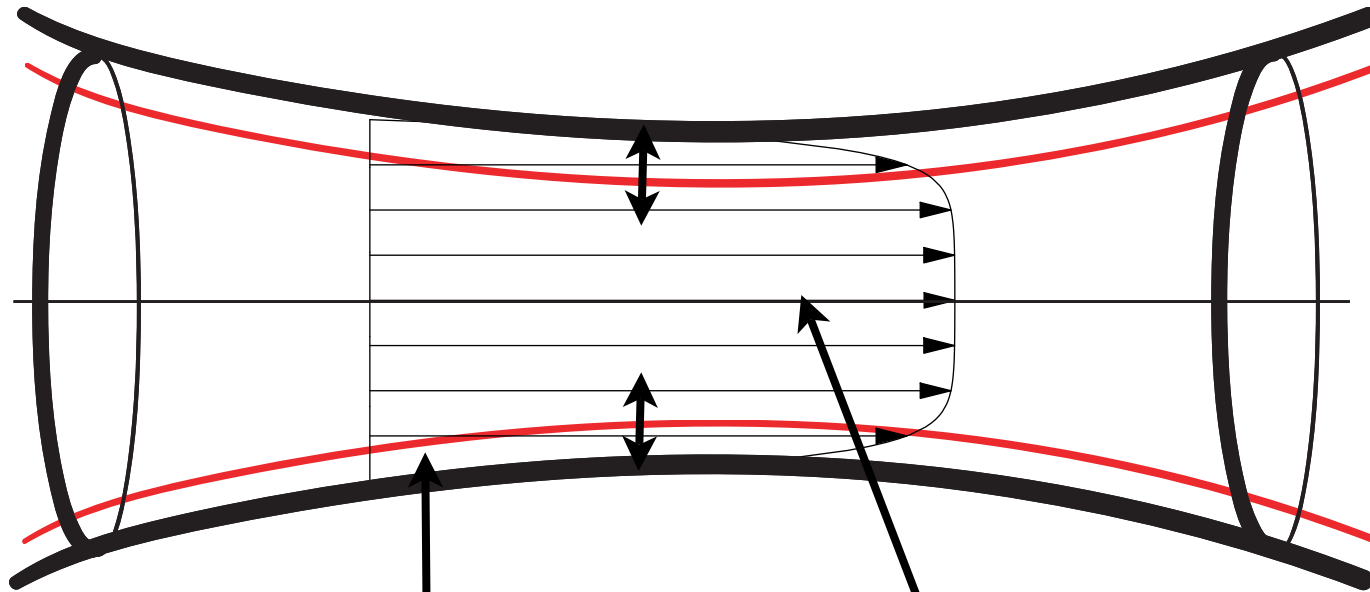
$$U_e(1 - (f + \delta_1))^2 = 1$$

$$\delta_1 = \int_0^\infty \left(1 - \frac{u}{U_e}\right) dn$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial n} = 0$$

$$u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial n} u = \frac{dU_e}{dx} + \frac{\partial^2 u}{\partial n^2}$$

# Interactive Boundary Layer



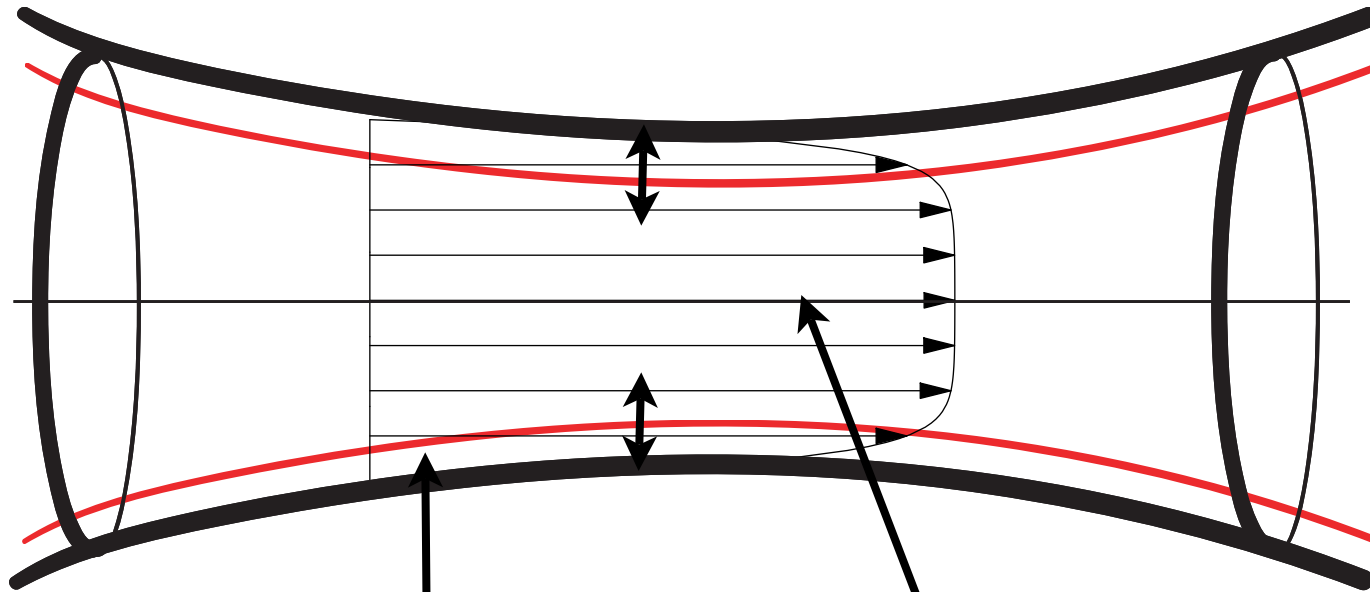
$$\delta_1 = \int_0^{\infty} \left(1 - \frac{u}{U_e}\right) dn$$

$$U_e(1 - (f + \delta_1))^2 = 1$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial n} = 0 \quad u(x, \infty) = U_e$$

$$u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial n} u = \frac{dU_e}{dx} + \frac{\partial^2}{\partial n^2} u$$

# Interactive Boundary Layer



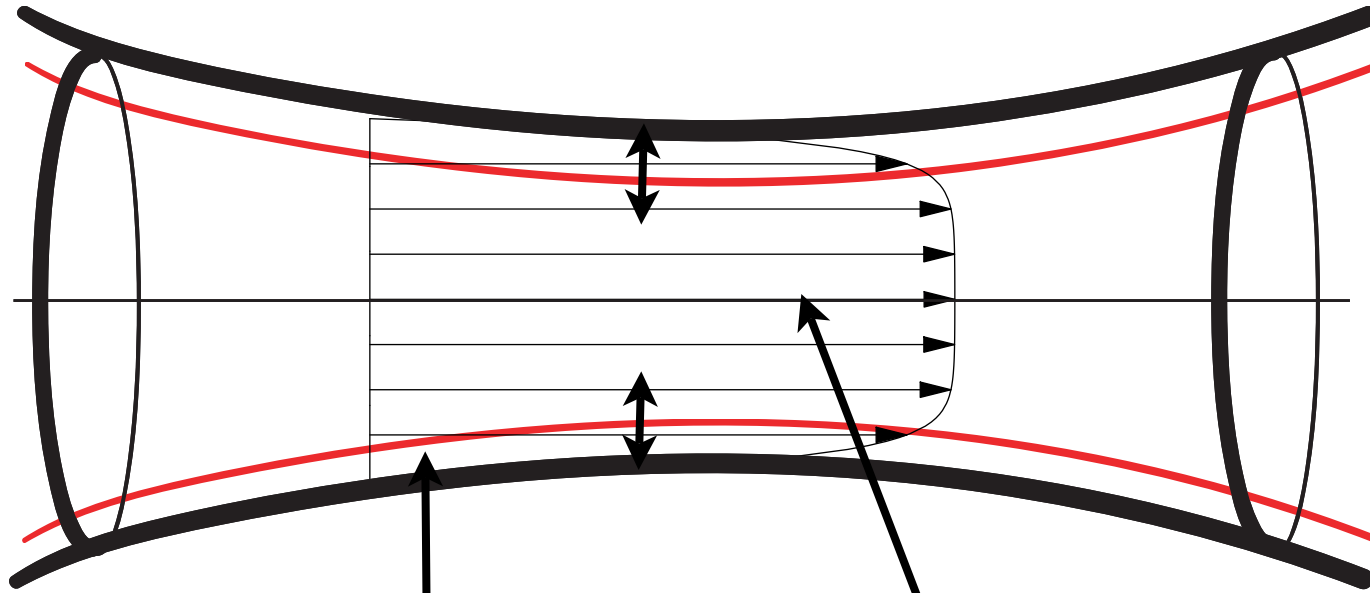
$$\delta_1 = \int_0^{\infty} \left(1 - \frac{u}{U_e}\right) dn$$

$$U_e(1 - (f + \delta_1))^2 = 1$$

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# Interactive Boundary Layer



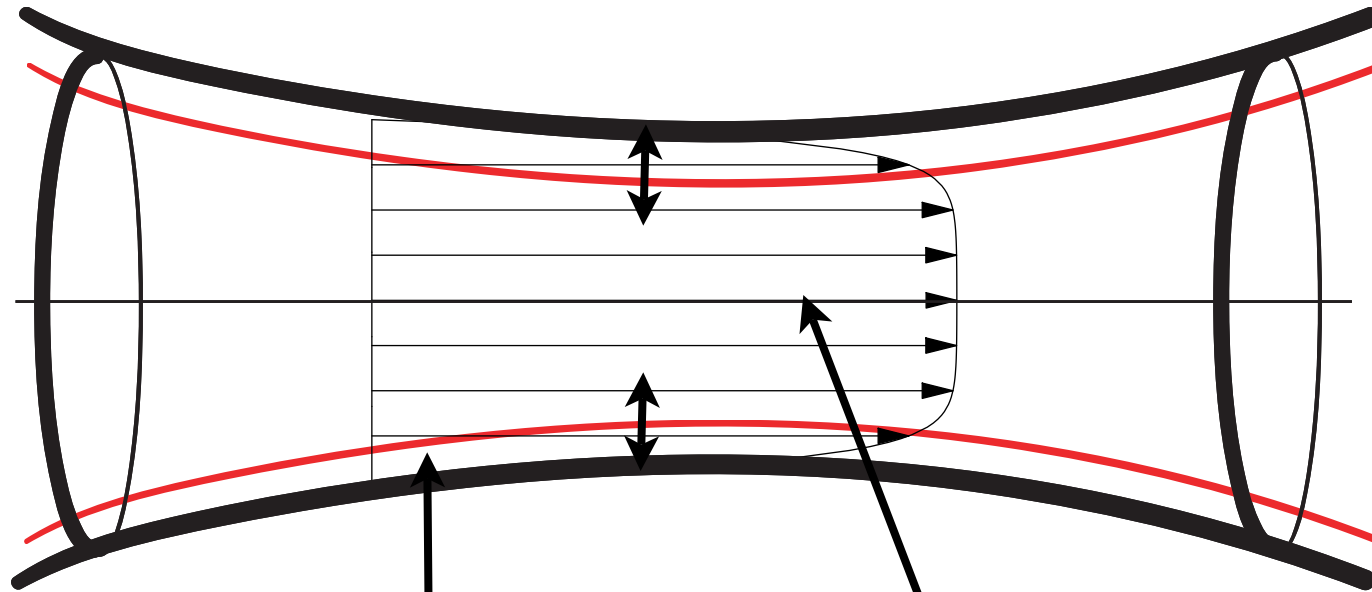
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$$u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial n} u = \frac{dU_e}{dx} + \frac{\partial^2}{\partial n^2} u$$

# Interactive Boundary Layer



Coupled System to solve

$$U_e(1 - (f + \delta_1))^2 = 1$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial n} = 0$$

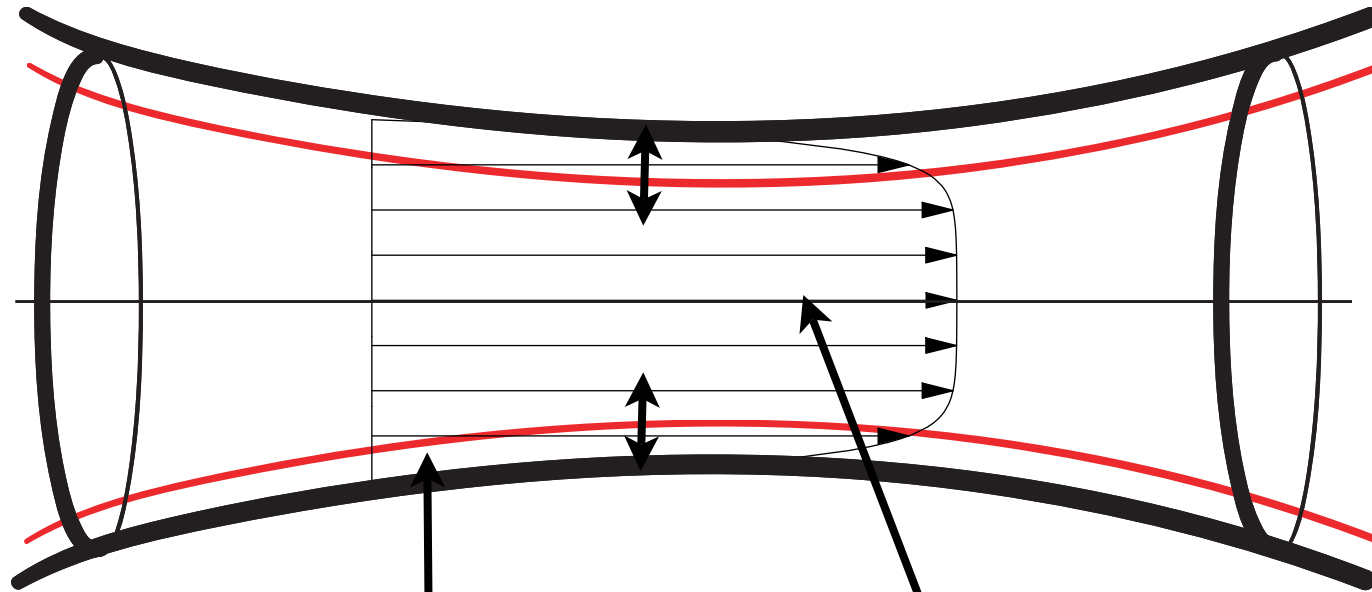
$$u(x, \infty) = U_e$$

$$u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial n} u = \frac{dU_e}{dx} + \frac{\partial^2}{\partial n^2} u$$

$$\delta_1 = \int_0^\infty \left(1 - \frac{u}{U_e}\right) dn$$



# Interactive Boundary Layer



Coupled System to solve

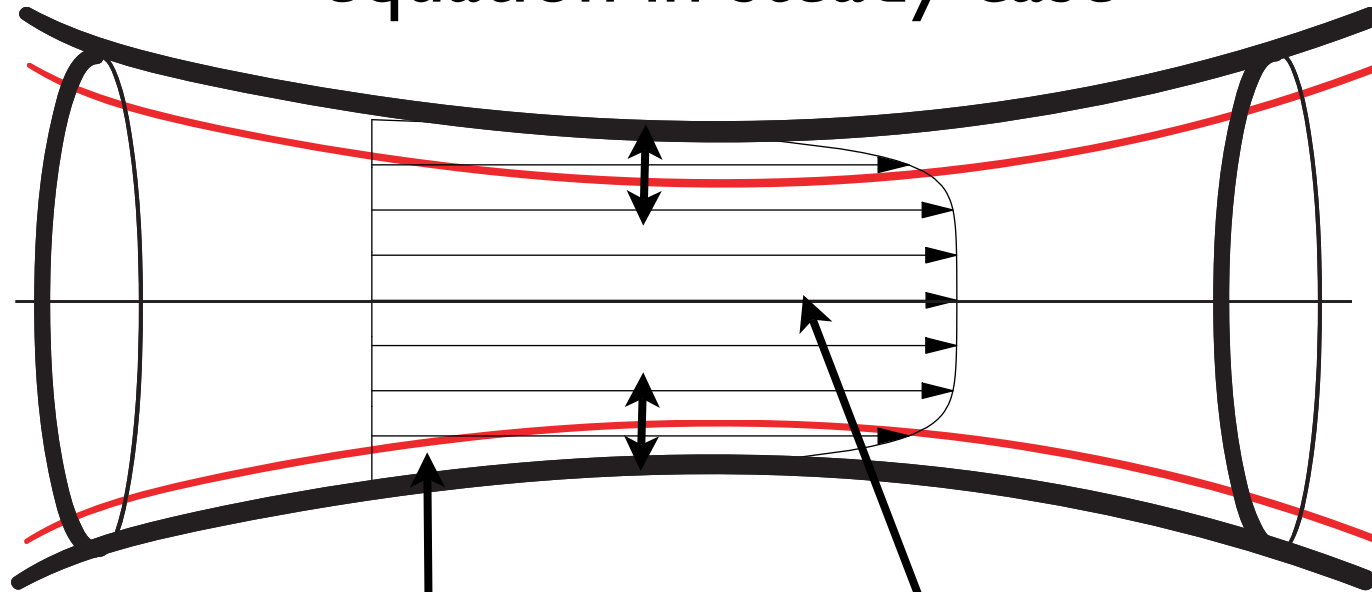
$$U_e(1 - (f + \delta_1))^2 = 1$$

$$\delta_1 = \int_0^\infty \left(1 - \frac{u}{U_e}\right) dn$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial n} = 0 \quad u(x, \infty) = U_e$$

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial n} = \frac{dU_e}{dx} + \frac{\partial^2 u}{\partial n^2}$$

# Integral resolution equation in steady case

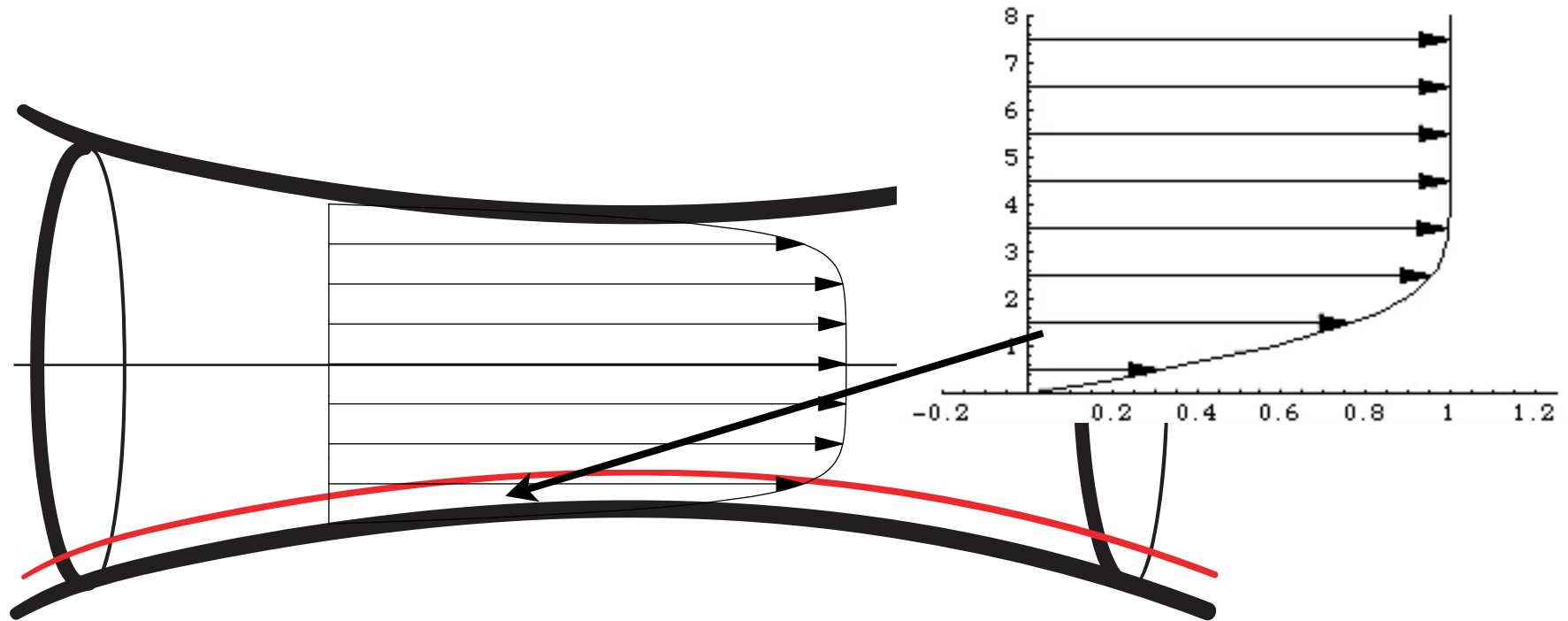


Coupled System to solve

$$U_e(1 - (f + \delta_1))^2 = 1$$

$$\delta_1 = \int_0^\infty \left(1 - \frac{u}{U_e}\right) dn$$

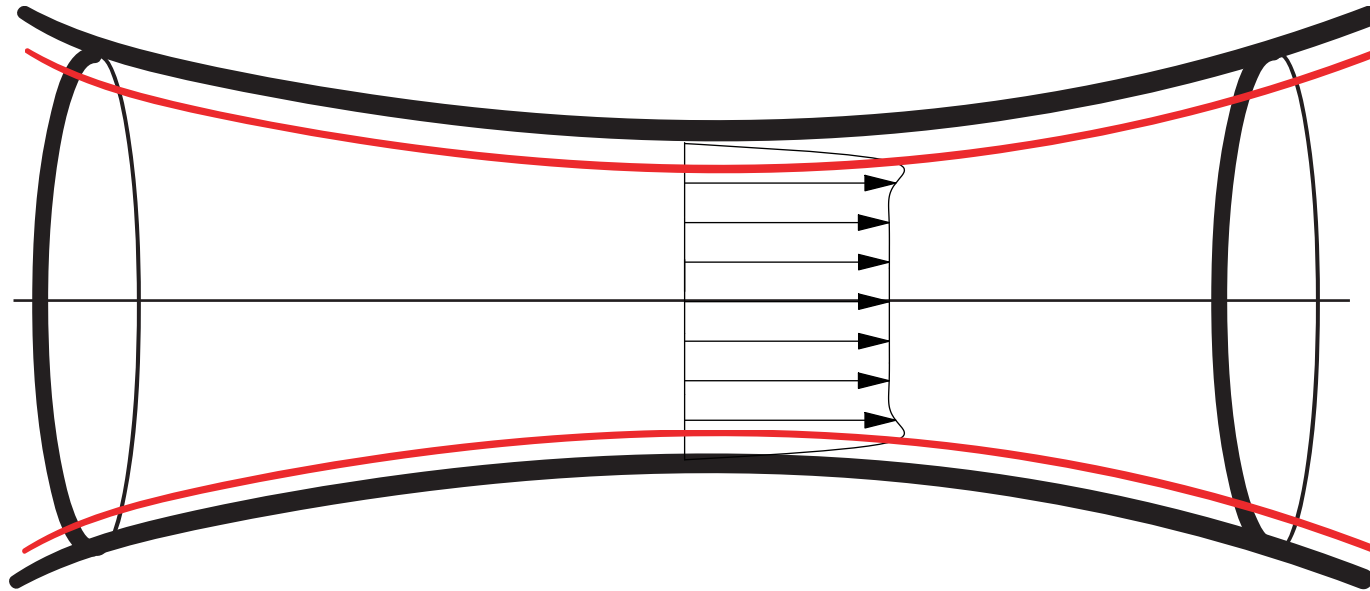
$$\frac{d}{dx} \left( \frac{\delta_1}{H} \right) + \frac{\delta_1}{U_e} \left( 1 + \frac{2}{H} \right) \frac{dU_e}{dx} = \frac{f_2 H}{\delta_1 U_e}$$



Choice of the family of simple profiles

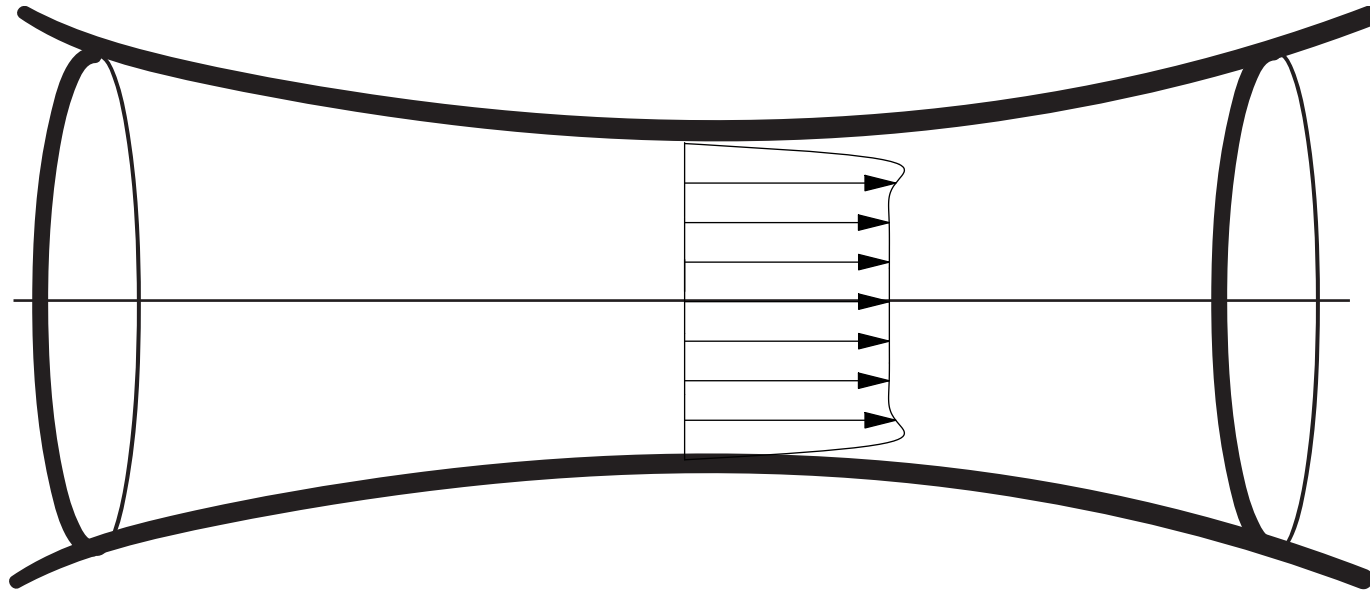
In a steady flow it is natural to use Falkner Skan

# Interactive Boundary Layer

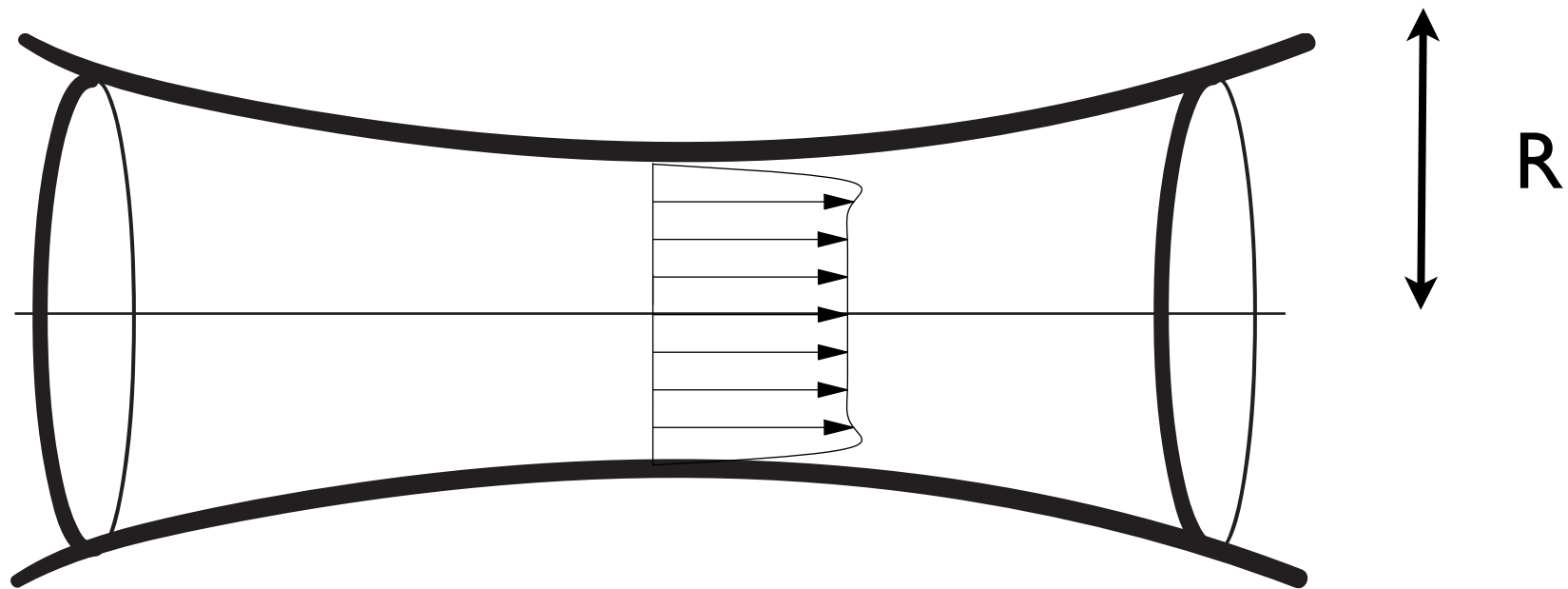


IBL is included in a larger system: RNSP

# RNSP Equations

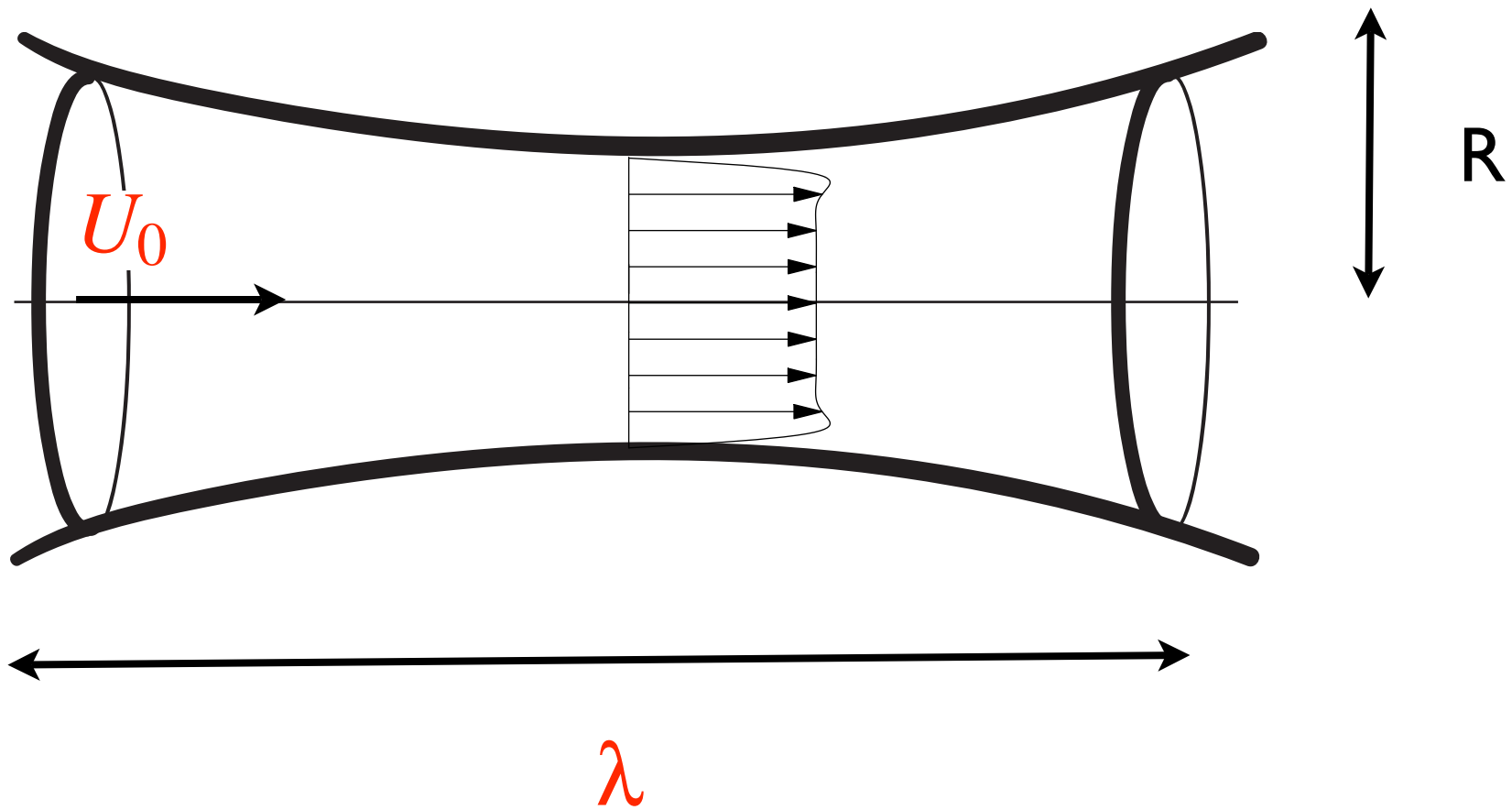


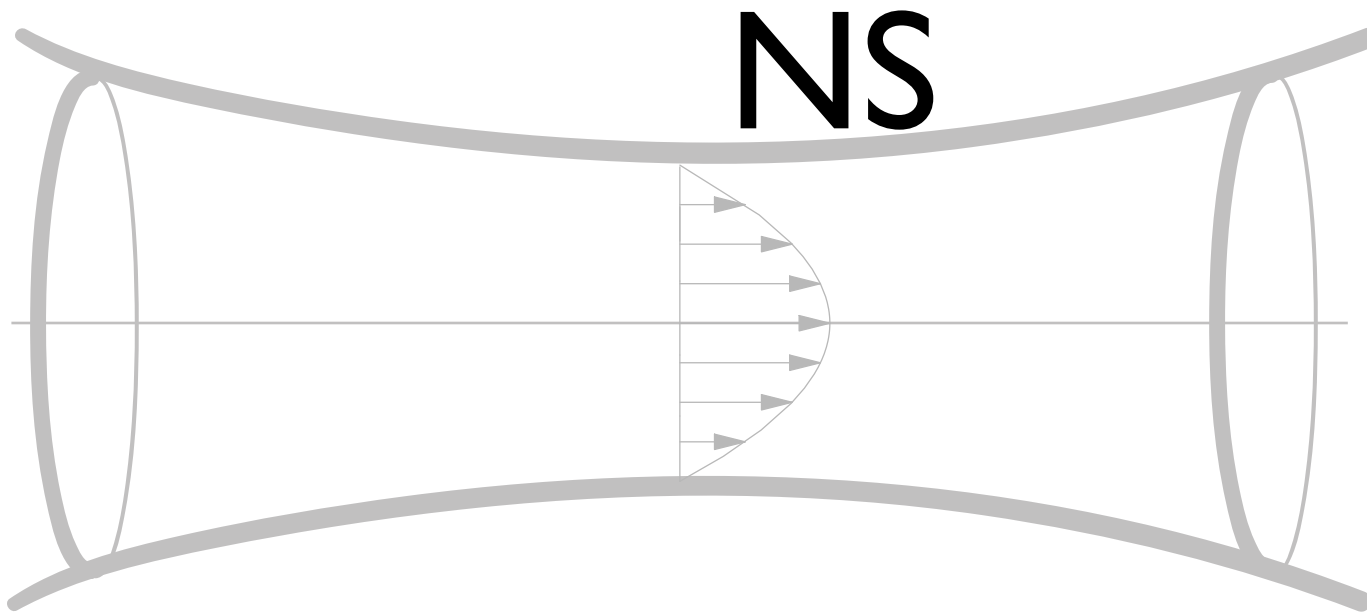
- simplified set
- deduced from orders of magnitude



$\lambda$

$$R \ll \lambda$$



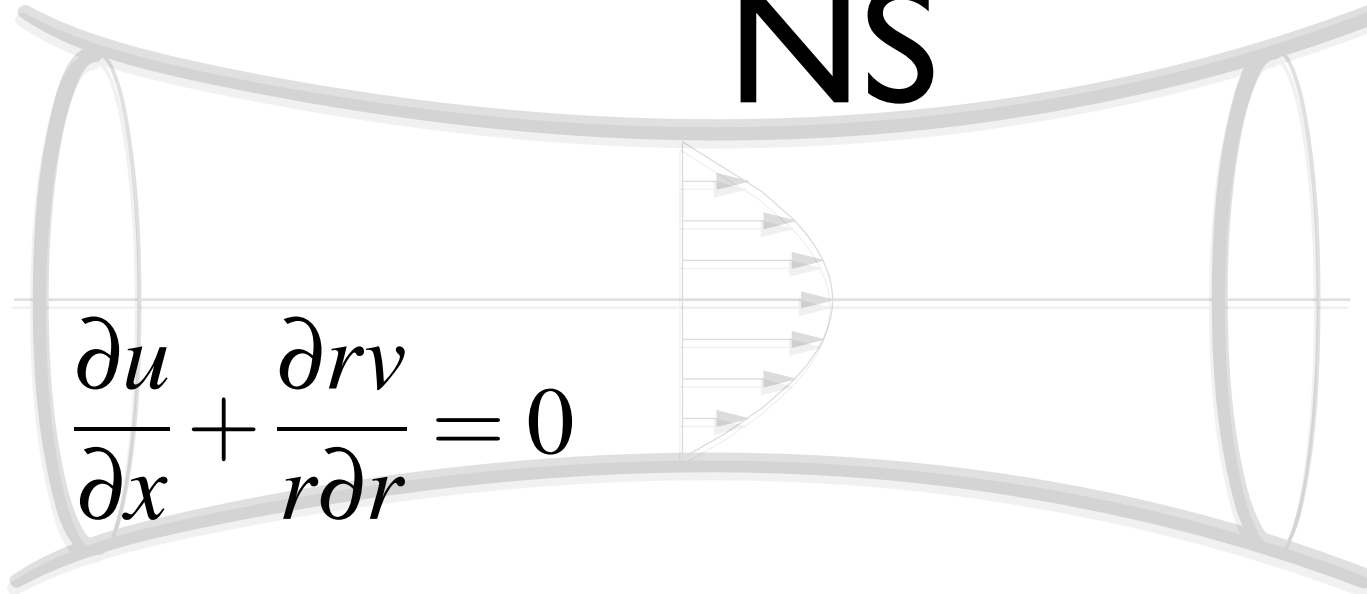


$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + \nu \frac{\partial^2}{\partial x^2} u + \nu \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r}$$

$$\frac{\partial v}{\partial t} + u \frac{\partial}{\partial x} v + v \frac{\partial}{\partial r} v = -\frac{\partial p}{\rho \partial r} + \nu \frac{\partial^2}{\partial x^2} v + \nu \frac{\partial}{r \partial r} r \frac{\partial v}{\partial r}$$



# NS

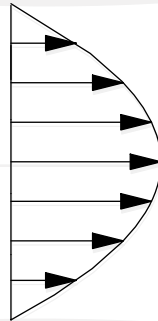


$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + \nu \frac{\partial^2}{\partial x^2} u + \nu \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r}$$

$$\frac{\partial v}{\partial t} + u \frac{\partial}{\partial x} v + v \frac{\partial}{\partial r} v = -\frac{\partial p}{\rho \partial r} + \nu \frac{\partial^2}{\partial x^2} v + \nu \frac{\partial}{r \partial r} r \frac{\partial v}{\partial r}$$

# Reduced NS

$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$



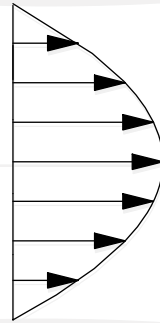
$$R \ll \lambda$$

$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + v \cancel{\frac{\partial^2}{\partial x^2}} u + v \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r}$$

$$\frac{\partial v}{\partial t} + u \frac{\partial}{\partial x} v + v \frac{\partial}{\partial r} v = -\frac{\partial p}{\rho \partial r} + v \cancel{\frac{\partial^2}{\partial x^2}} v + v \frac{\partial}{r \partial r} r \frac{\partial v}{\partial r}$$

# RNS/P

$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$



$$R \ll \lambda$$

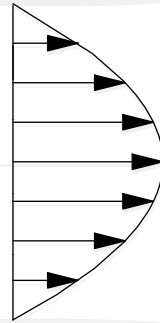
$$V \sim U_0 \frac{R}{\lambda}$$

$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + v \cancel{\frac{\partial^2}{\partial x^2}} u + v \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r}$$

$$\frac{\partial v}{\partial t} + u \frac{\partial}{\partial x} v + v \frac{\partial}{\partial r} v = -\frac{\partial p}{\rho \partial r} + v \cancel{\frac{\partial^2}{\partial x^2}} v + v \frac{\partial}{r \partial r} r \frac{\partial v}{\partial r}$$

# RNS/P

$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$



$$R \ll \lambda$$

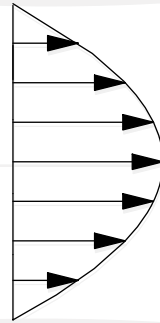
$$V \sim U_0 \frac{R}{\lambda}$$

$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + v \cancel{\frac{\partial^2}{\partial x^2}} u + v \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r}$$

$$\frac{\partial v}{\partial t} + u \frac{\partial}{\partial x} v + v \frac{\partial}{\partial r} v = -\frac{\partial p}{\rho \partial r} + v \cancel{\frac{\partial^2}{\partial x^2}} v + v \frac{\partial}{r \partial r} r \frac{\partial v}{\partial r}$$

# RNS/P

$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$



$$R \ll \lambda$$

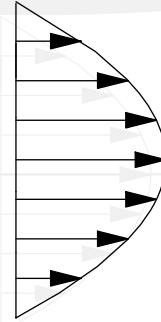
$$V \sim U_0 \frac{R}{\lambda}$$

$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + v \frac{\partial^2}{\partial x^2} u + v \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r}$$

$$\frac{\partial v}{\partial t} + u \frac{\partial}{\partial x} v + v \frac{\partial}{\partial r} v = -\frac{\partial p}{\rho \partial r} + v \frac{\partial^2}{\partial x^2} v + v \frac{\partial}{r \partial r} \frac{\partial v}{\partial r}$$

# RNS/P

$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$

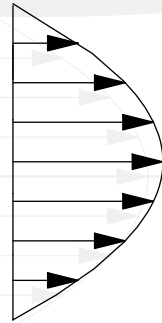


$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + v \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r}$$

$$0 = -\frac{\partial p}{\rho \partial r}$$

# RNS/P

$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$



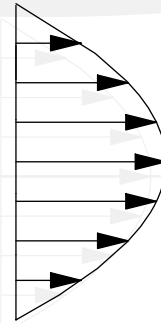
$$v \frac{1}{\omega R^2}$$

$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + v \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r}$$

$$0 = -\frac{\partial p}{\rho \partial r}$$

# RNS/P

$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$



$$\alpha = R \sqrt{\frac{\omega}{\nu}}$$

$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + \nu \frac{\partial}{\partial r} \left( \frac{\partial u}{r \partial r} \right)$$

$$0 = -\frac{\partial p}{\rho \partial r}$$

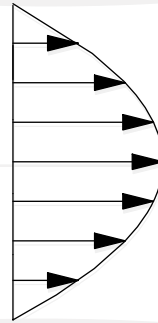
1/(Womersley)<sup>2</sup>



# RNS/P

Prandtl

$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$



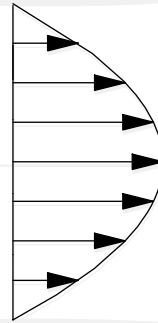
$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + v \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r}$$

$$0 = -\frac{\partial p}{\rho \partial r}$$

# RNS/P

Prandtl

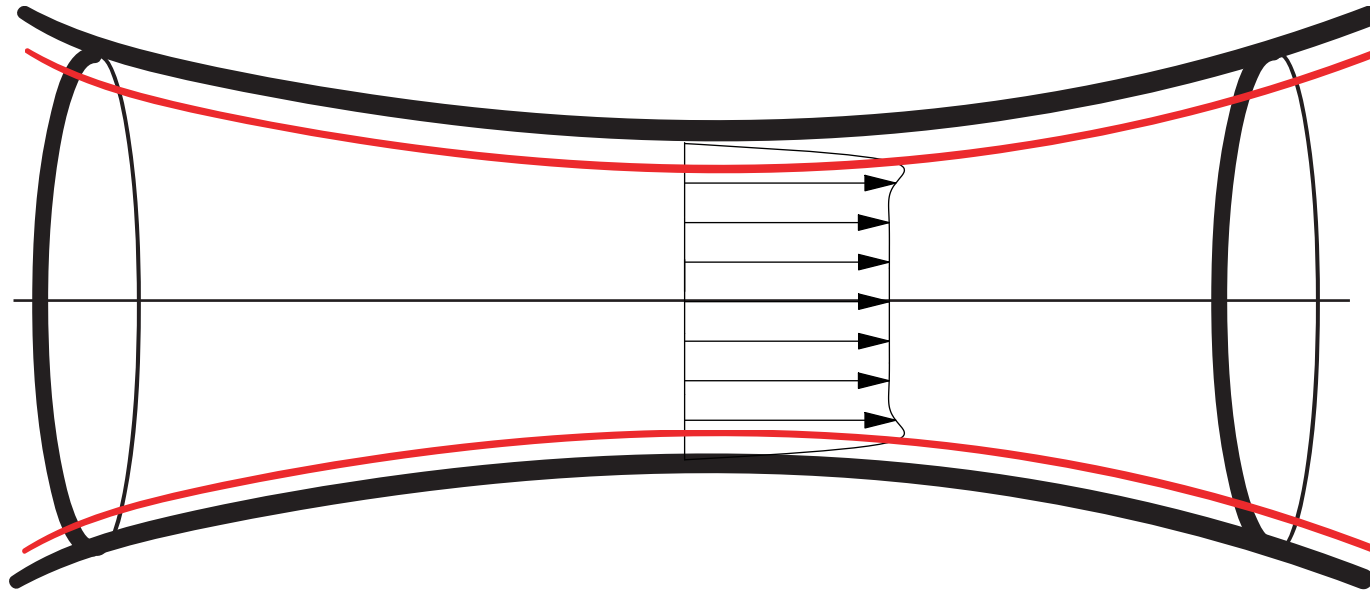
$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$



$$u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + v \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r}$$

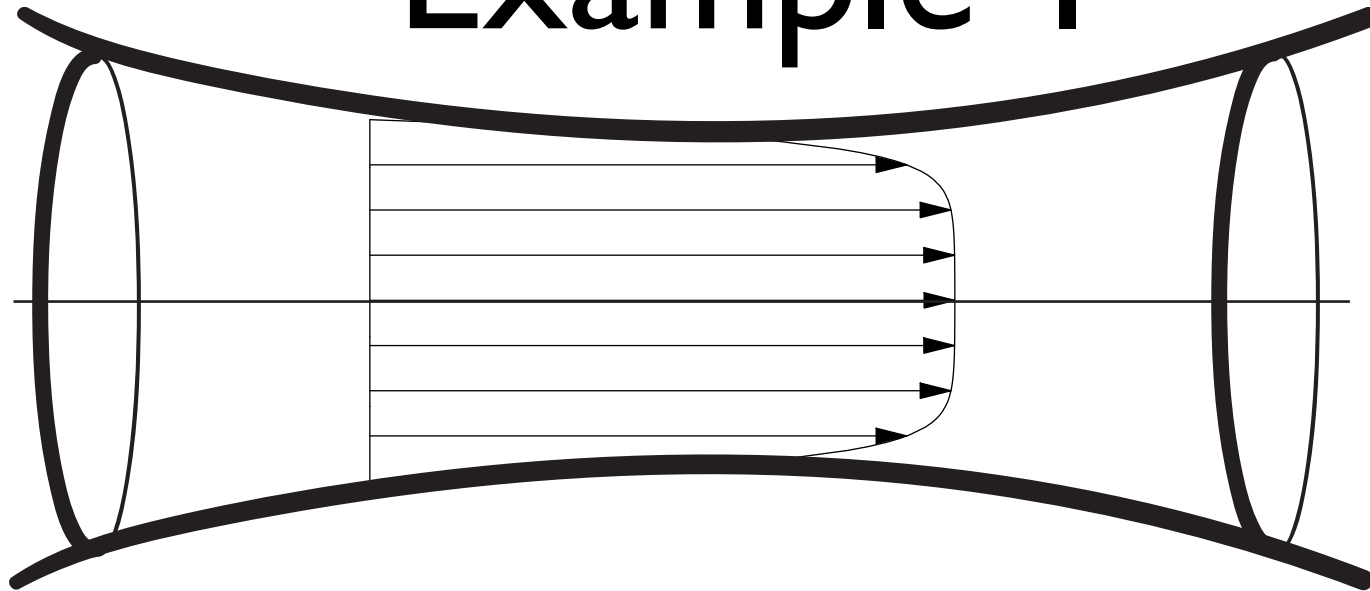
$$0 = -\frac{\partial p}{\rho \partial r}$$

# Interactive Boundary Layer

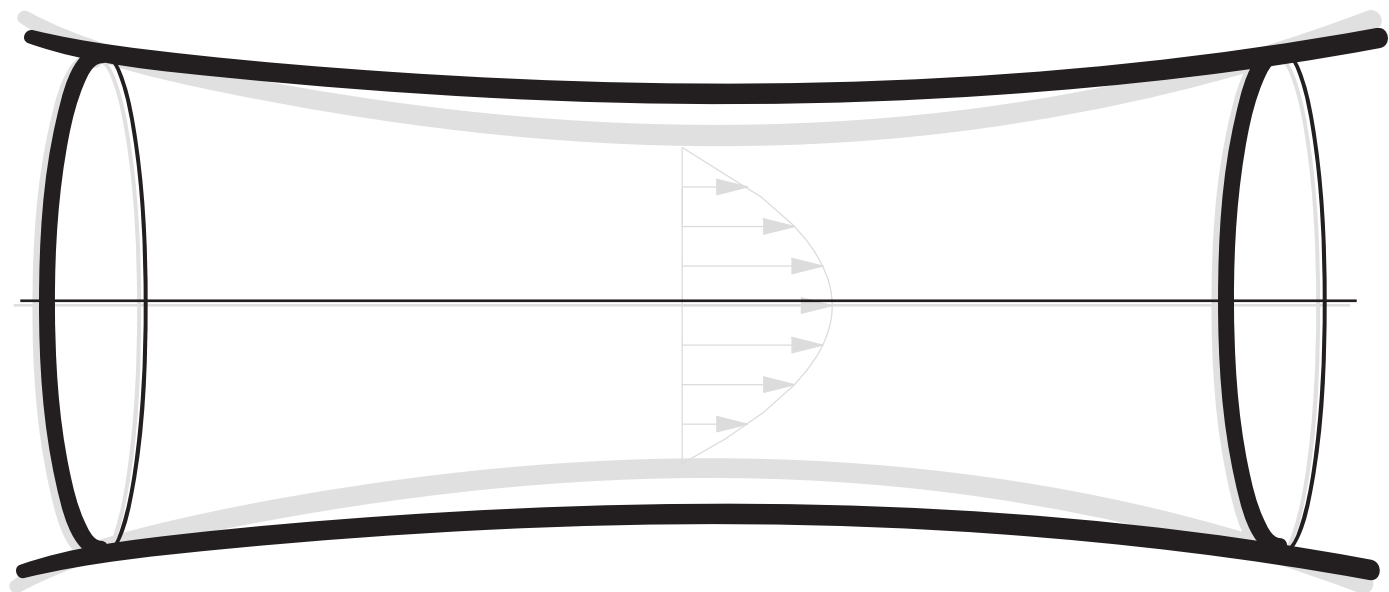


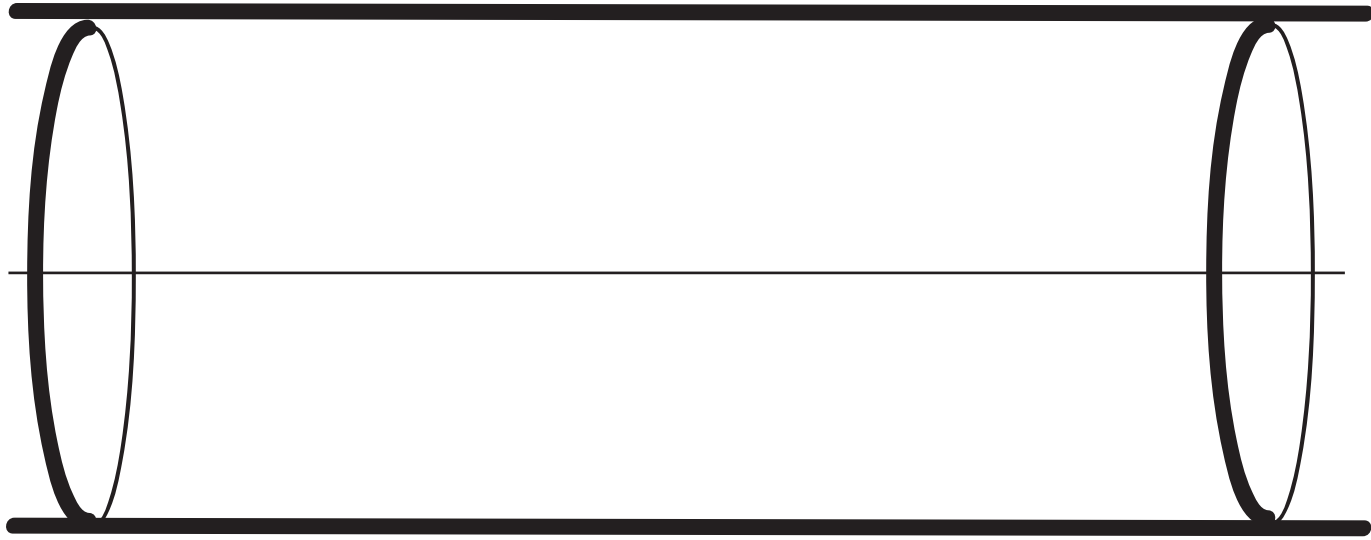
IBL is included in RNSP

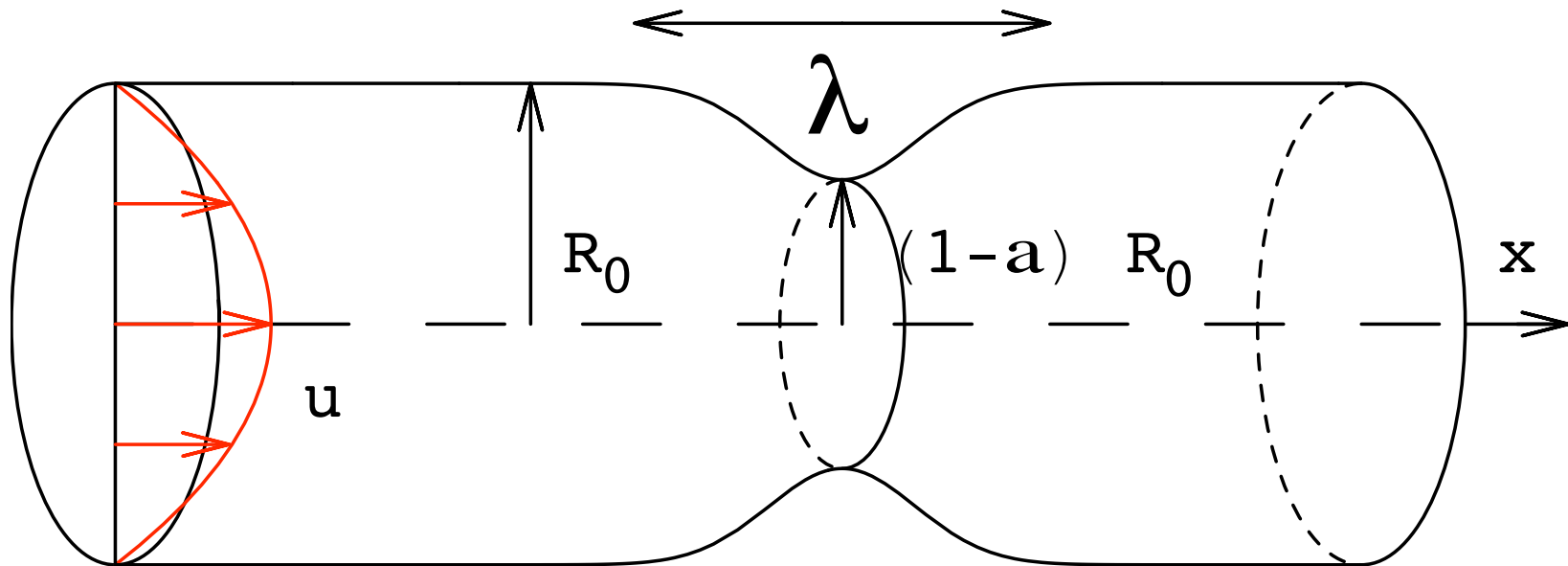
# Example I



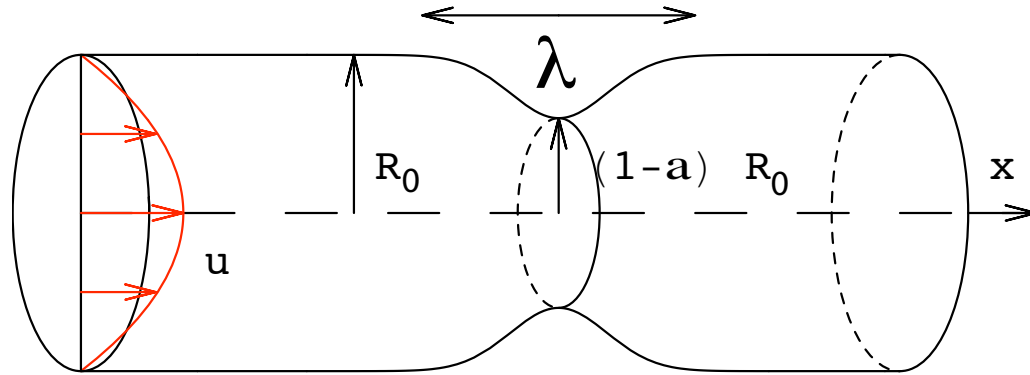
- Flow in a stenosed vessel
- steady, rigid wall







# RNSP Scales



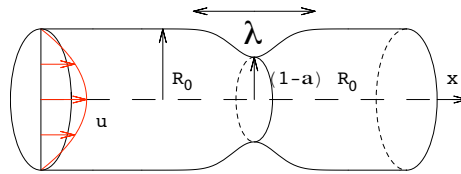
Using:

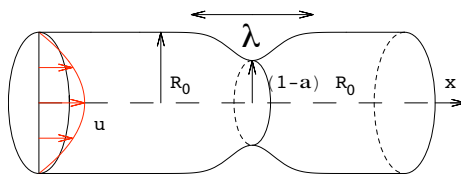
$$x^* = xR_0Re, \quad r^* = rR_0, \quad u^* = U_0u, \quad v^* = \frac{U_0}{Re}v,$$
$$p^* = p_0^* + \rho_0U_0^2p \quad \text{and} \quad \tau^* = \frac{\rho U_0^2}{Re}\tau$$

the following partial differential system is obtained from Navier Stokes as  $Re \rightarrow \infty$ :



# RNSP: Reduced Navier Stokes/ Prandtl System



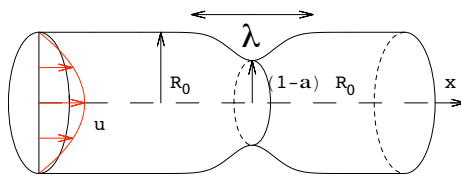


## RNSP: Reduced Navier Stokes/ Prandtl System

$$\frac{\partial}{\partial x}u + \frac{\partial}{r\partial r}rv = 0,$$

$$\left(u\frac{\partial}{\partial x}u + v\frac{\partial}{\partial r}u\right) = -\frac{\partial p}{\partial x} + \frac{\partial}{r\partial r}\left(r\frac{\partial}{\partial r}u\right),$$

$$0 = -\frac{\partial p}{\partial r}.$$



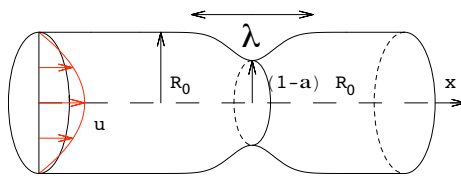
## RNSP: Reduced Navier Stokes/ Prandtl System

$$\frac{\partial}{\partial x}u + \frac{\partial}{r\partial r}rv = 0,$$

$$\left(u\frac{\partial}{\partial x}u + v\frac{\partial}{\partial r}u\right) = -\frac{\partial p}{\partial x} + \frac{\partial}{r\partial r}\left(r\frac{\partial}{\partial r}u\right),$$

$$0 = -\frac{\partial p}{\partial r}.$$

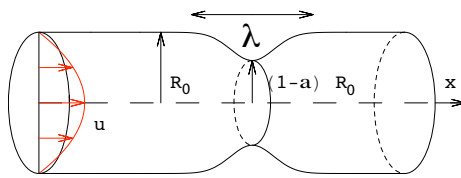
+ The boundary conditions.



## RNSP: Reduced Navier Stokes/ Prandtl System

$$\begin{aligned} \frac{\partial}{\partial x}u + \frac{\partial}{r\partial r}rv &= 0, \\ \left(u\frac{\partial}{\partial x}u + v\frac{\partial}{\partial r}u\right) &= -\frac{\partial p}{\partial x} + \frac{\partial}{r\partial r}\left(r\frac{\partial}{\partial r}u\right), \\ 0 &= -\frac{\partial p}{\partial r}. \end{aligned}$$

- axial symmetry ( $\partial_r u = 0$  and  $v = 0$  at  $r = 0$ ),
- no slip condition at the wall ( $u = v = 0$  at  $r = 1 - f(x)$ ),
- the entry velocity profiles ( $u(0, r)$  and  $v(0, r)$ ) are given
- *no* output condition in  $x_{out} = \frac{x_{out}^*}{R_0 Re}$
- streamwise marching, even when flow separation.

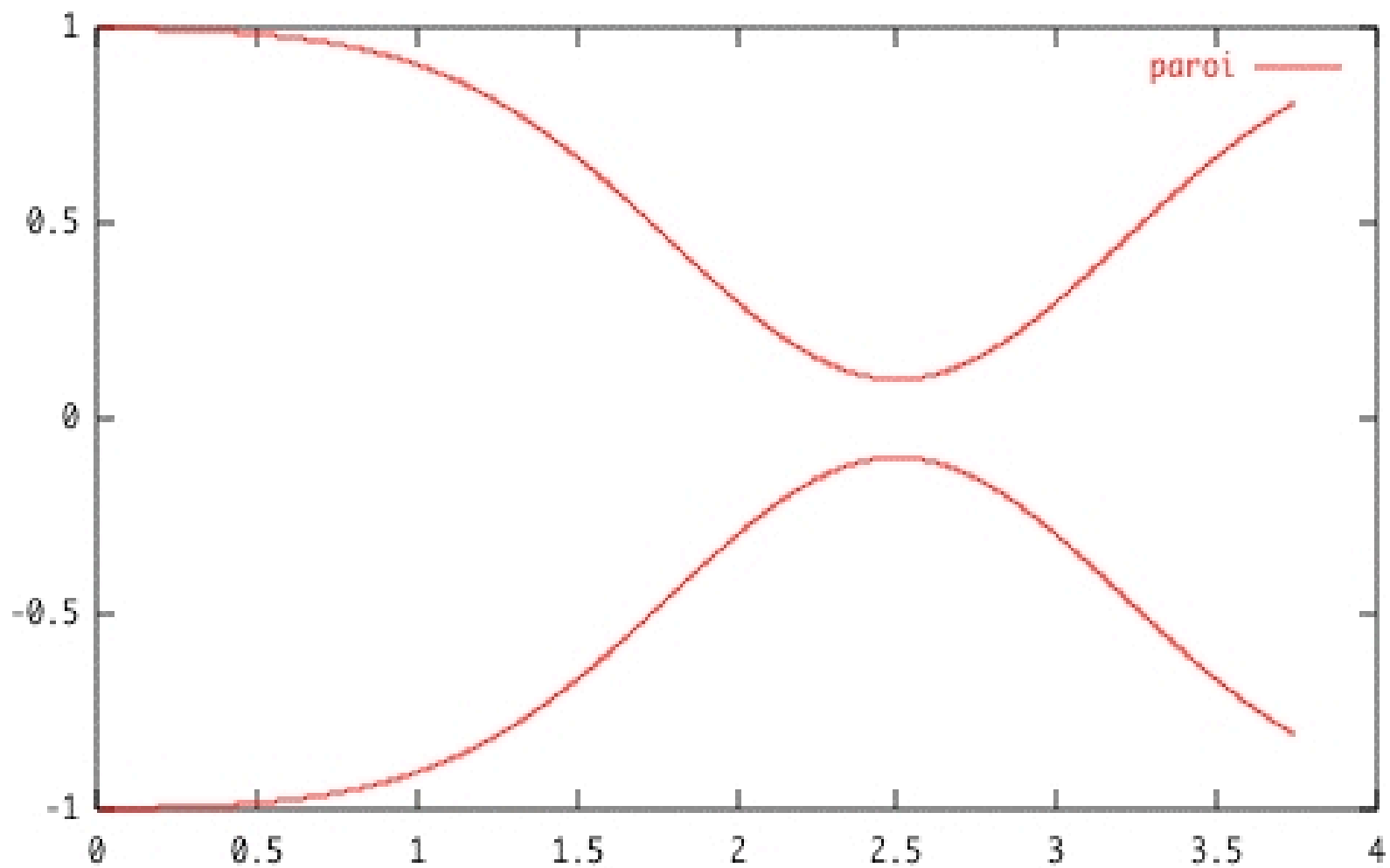
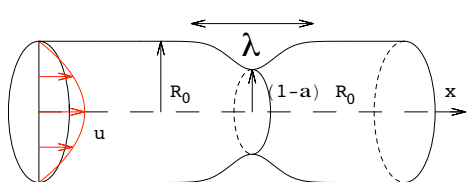


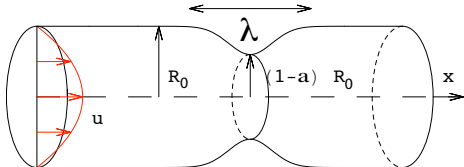
## RNSP: Reduced Navier Stokes/ Prandtl System

$$\begin{aligned} \frac{\partial}{\partial x}u + \frac{\partial}{r\partial r}rv &= 0, \\ (u\frac{\partial}{\partial x}u + v\frac{\partial}{\partial r}u) &= -\frac{\partial p}{\partial x} + \frac{\partial}{r\partial r}(r\frac{\partial}{\partial r}u), \\ 0 &= -\frac{\partial p}{\partial r}. \end{aligned}$$

### Parabolic Problem - Marching Problem

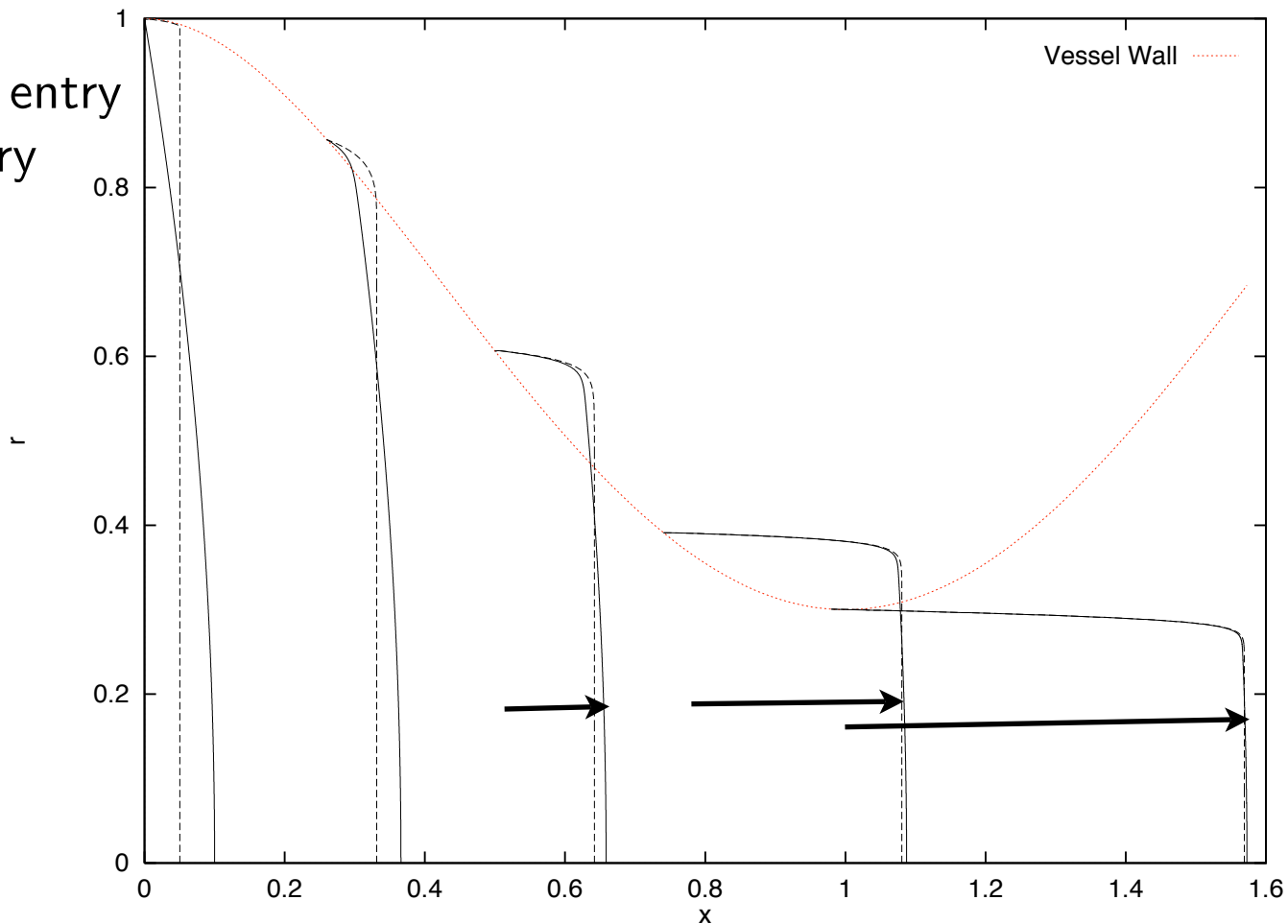
- axial symmetry ( $\partial_r u = 0$  and  $v = 0$  at  $r = 0$ ),
- no slip condition at the wall ( $u = v = 0$  at  $r = 1 - f(x)$ ),
- the entry velocity profiles ( $u(0, r)$  and  $v(0, r)$ ) are given
- *no* output condition in  $x_{out} = \frac{x_{out}^*}{R_0 Re}$
- streamwise marching, even when flow separation.

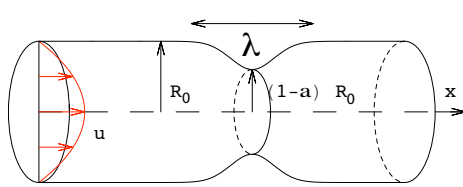




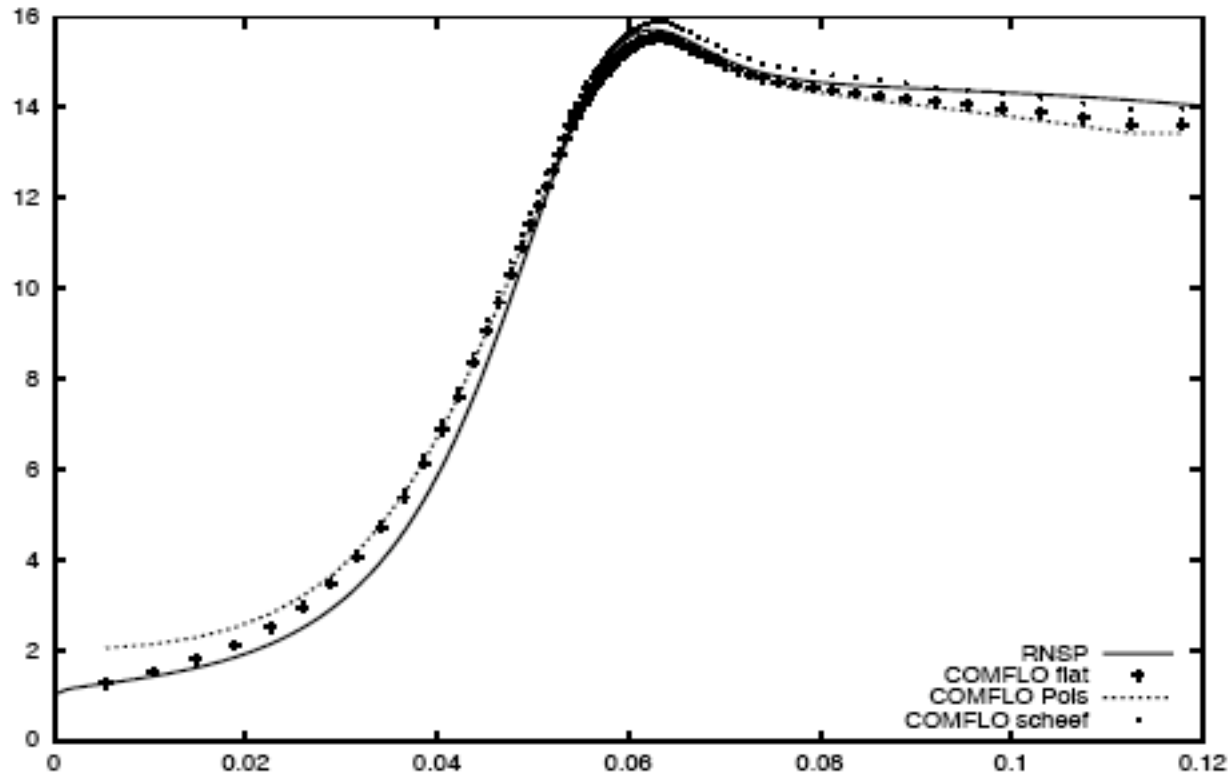
Evolution of the velocity profile along the convergent part of a 70% stenosis ( $Re = 500$ );

solid line: Poiseuille entry  
 broken line: flat entry





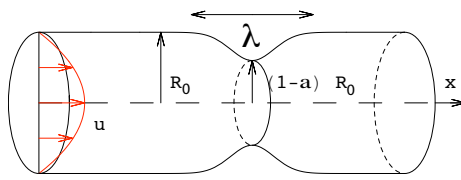
## Testing asymmetry in the entry profile



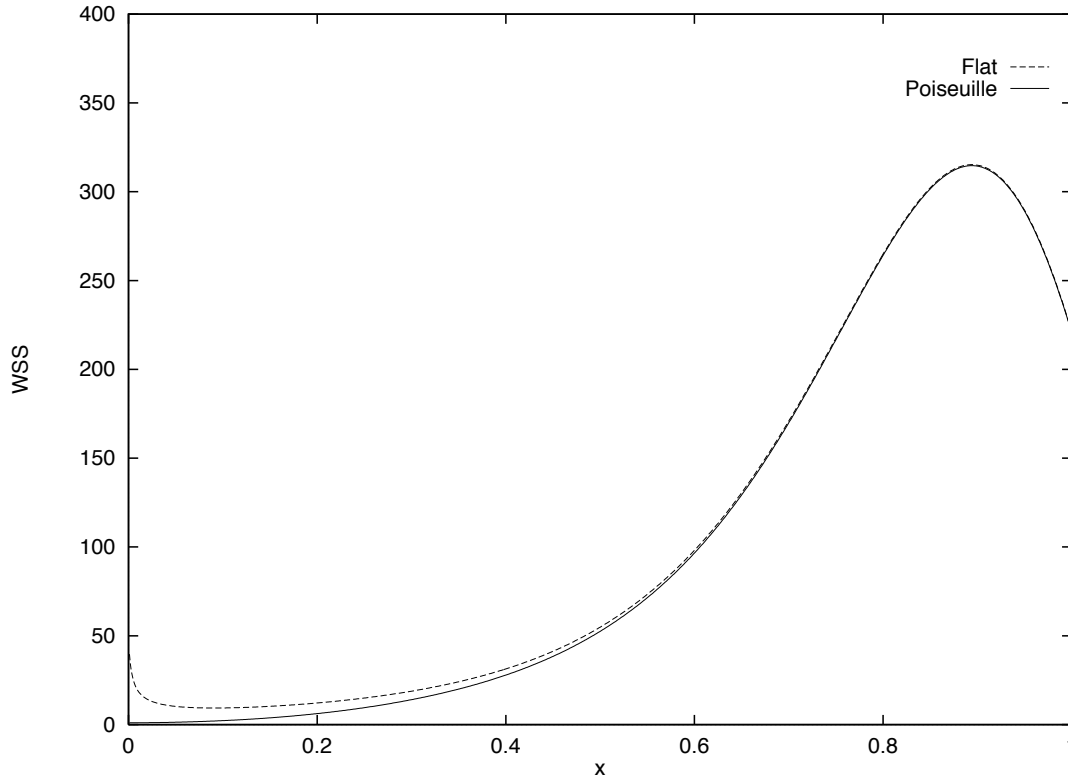
The velocities in the middle for Comflo and RNS.

Comflo uses here 50X50X100 points. Dimensionless scales!

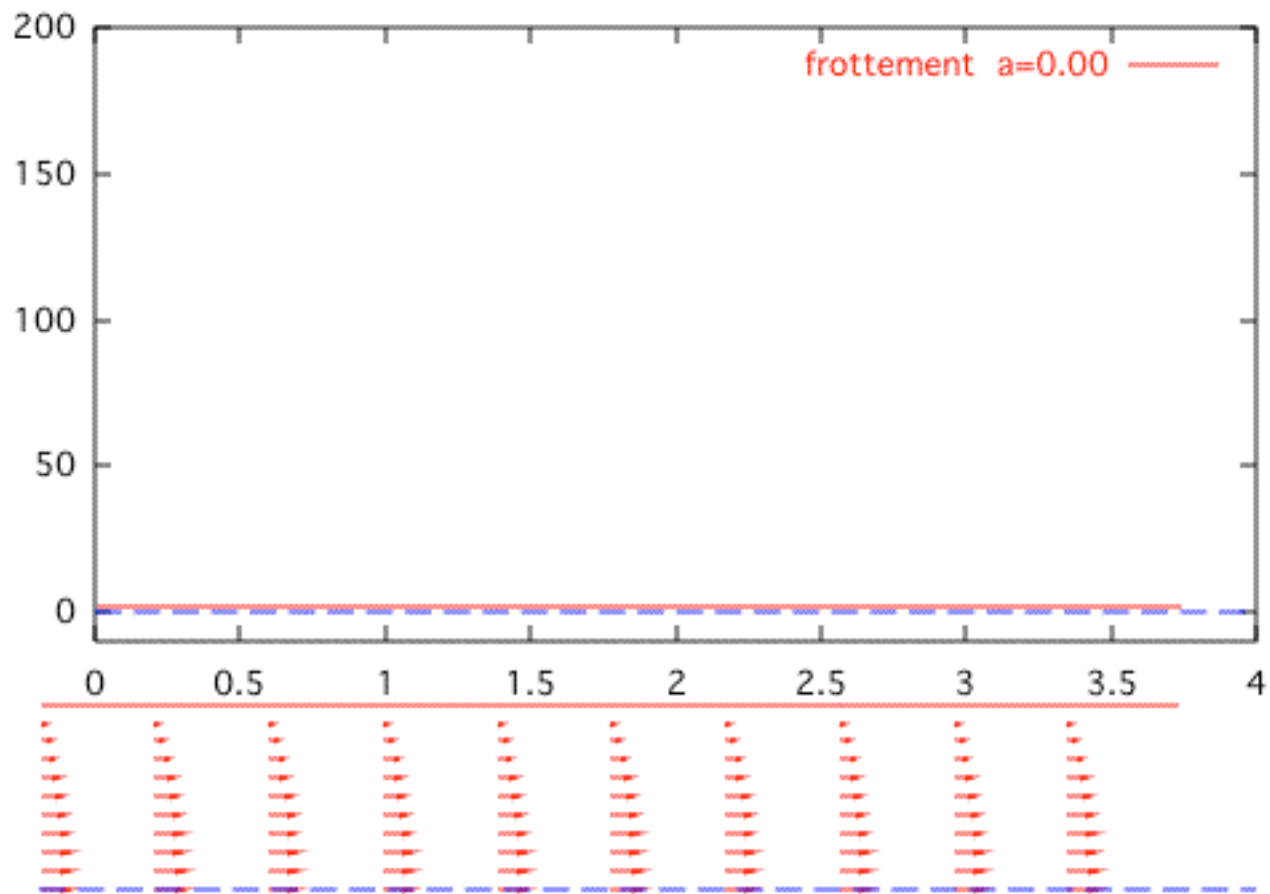
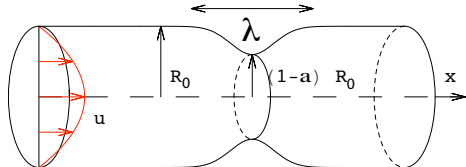




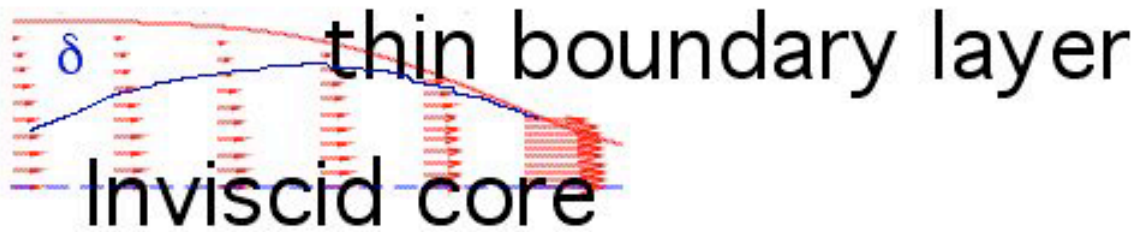
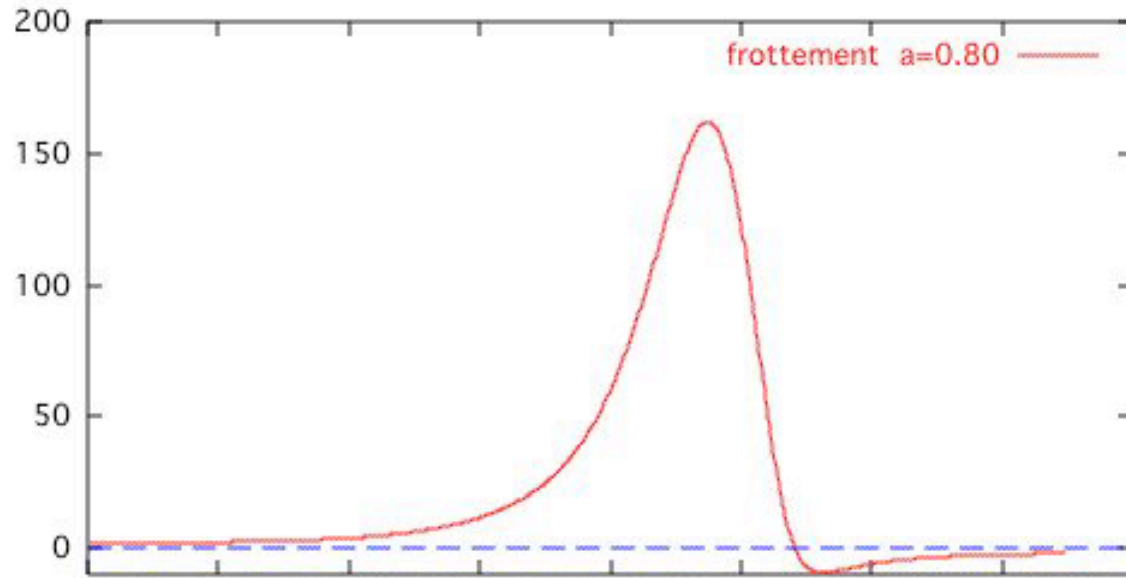
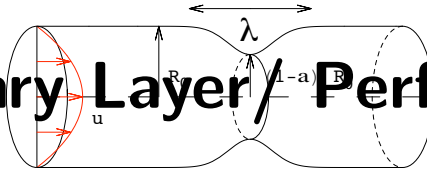
## Wall Shear Stress

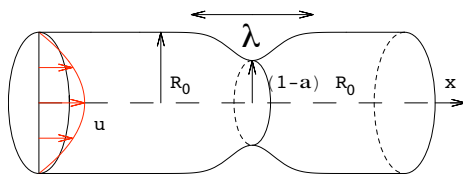


Evolution of the WSS distribution along the convergent part of a 70% stenosis ( $Re = 500$ ) ; solid line: Poiseuille entry profile ; broken line: flat entry profile.

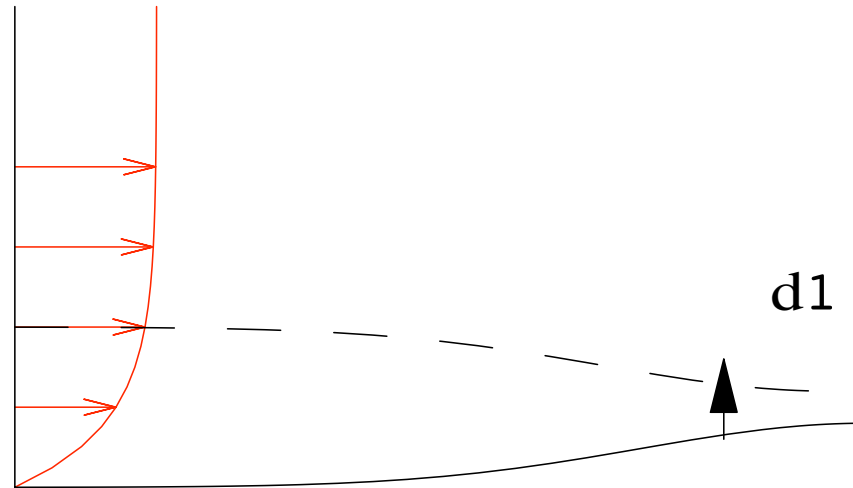


# Boundary Layer / Perfect Fluid

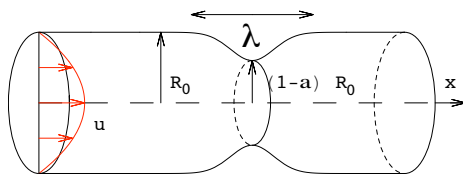




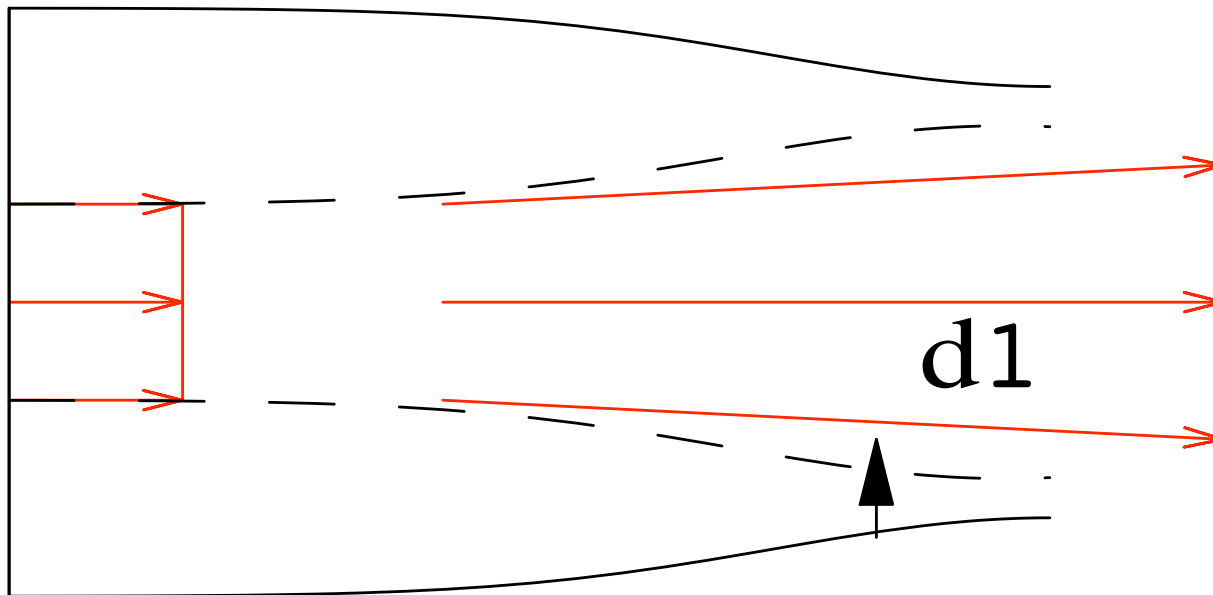
## Boundary Layer/ Perfect Fluid



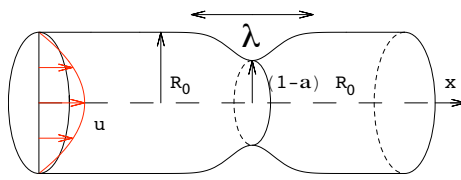
The boundary layer is generated near the wall  
 $\delta_1$  is the displacement thickness.



## Boundary Layer/ Perfect Fluid



The displacement thickness acts as a "new" wall!  
 → Interacting Boundary Layer (IBL)



## RNSP/ IBL

After rescaling:

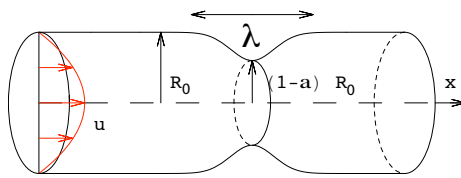
$r = R(\bar{x}) - (\lambda/Re)^{-1/2}\bar{y}$ ,  $u = \bar{u}$ ,  $v = (\lambda/Re)^{1/2}\bar{v}$  and  $x - x_b = (\lambda/Re)\bar{x}$ ,  $p = \bar{p}$ , where  $x_b$  is the position of the bump, the RNSP( $x$ ) set gives the final IBL (interacting Boundary Layer) problem as follows:

$$\frac{\partial \bar{u}}{\partial \bar{x}} + \frac{\partial \bar{v}}{\partial \bar{n}} = 0$$

$$\left( \bar{u} \frac{\partial \bar{u}}{\partial \bar{x}} + \bar{v} \frac{\partial \bar{u}}{\partial \bar{n}} \right) = \bar{u}_e \frac{d\bar{u}_e}{d\bar{s}} + \frac{\partial}{\partial \bar{n}} \frac{\partial \bar{u}}{\partial \bar{n}}$$

with:  $\bar{u}(\bar{x}, 0) = 0$ ,  $\bar{v}(\bar{x}, 0) = 0$ ,  $\bar{u}(\bar{x}, \infty) = u_e$ , where  $\bar{\delta}_1 = \int_0^\infty \left(1 - \frac{\bar{u}}{u_e}\right) d\bar{n}$ , and

$$\bar{u}_e = \frac{1}{(R^2 - 2((\lambda/Re)^{-1/2})\bar{\delta}_1)}.$$



## IBL integral: 1D equation

$$\frac{d}{d\bar{x}}\left(\frac{\bar{\delta}_1}{H}\right) = \bar{\delta}_1\left(1 + \frac{2}{H}\right)\frac{d\bar{u}_e}{d\bar{x}} + \frac{f_2 H}{\bar{\delta}_1 \bar{u}_e},$$

$$\bar{u}_e = \frac{1}{(R^2 - 2(\lambda/Re)^{-1/2}\bar{\delta}_1)}.$$

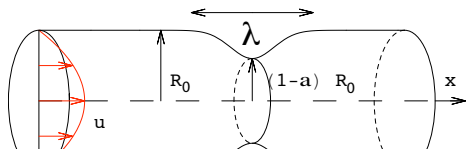
To solve this system, a closure relationship linking  $H$  and  $f_2$  to the velocity and the displacement thickness is needed.

Defining  $\Lambda_1 = \bar{\delta}_1^2 \frac{d\bar{u}_e}{d\bar{x}}$ ,

the system is closed from the resolution of the Falkner Skan system as follows:

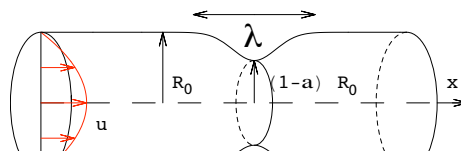
if  $\Lambda_1 < 0.6$  then  $H = 2.5905 \exp(-0.37098\Lambda_1)$ , else  $H = 2.074$ .

From  $H$ ,  $f_2$  is computed as  $f_2 = 1.05(-H^{-1} + 4H^{-2})$ .



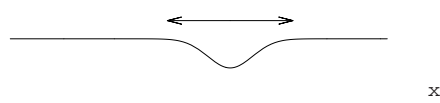
**IBL integral: 1D equation Simplified Shear Stress**





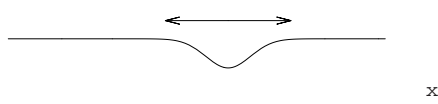
## IBL integral: 1D equation Simplified Shear Stress

- variation of velocity (flux conservation)



## IBL integral: 1D equation Simplified Shear Stress

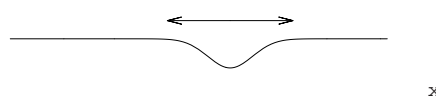
- variation of velocity (flux conservation)  $U_0 \rightarrow U_0 / (1 - \alpha - \delta_1)^2$



## IBL integral: 1D equation Simplified Shear Stress

- variation of velocity (flux conservation)  $U_0 \rightarrow U_0/(1 - \alpha - \delta_1)^2$

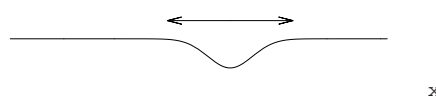
- acceleration: boundary layer  $\delta_1 \simeq \frac{\lambda}{\sqrt{Re_\lambda}}$ ,



## IBL integral: 1D equation Simplified Shear Stress

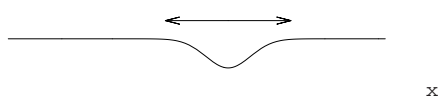
- variation of velocity (flux conservation)  $U_0 \rightarrow U_0 / (1 - \alpha - \delta_1)^2$

- acceleration: boundary layer  $\delta_1 \simeq \frac{\lambda}{\sqrt{Re_\lambda}}$ , with  $Re_\lambda = \frac{\lambda U_0}{(1 - \alpha)^2 \nu} = \frac{Re \lambda}{(1 - \alpha)^2}$



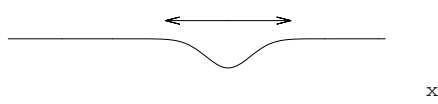
## IBL integral: 1D equation Simplified Shear Stress

- variation of velocity (flux conservation)  $U_0 \rightarrow U_0/(1 - \alpha - \delta_1)^2$
- acceleration: boundary layer  $\delta_1 \simeq \frac{\lambda}{\sqrt{Re_\lambda}}$ , with  $Re_\lambda = \frac{\lambda U_0}{(1-\alpha)^2 \nu} = \frac{Re \lambda}{(1-\alpha)^2}$
- WSS = (variation of velocity)/(boundary layer thickness)



## IBL integral: 1D equation Simplified Shear Stress

- variation of velocity (flux conservation)  $U_0 \rightarrow U_0/(1 - \alpha - \delta_1)^2$
- acceleration: boundary layer  $\delta_1 \simeq \frac{\lambda}{\sqrt{Re_\lambda}}$ , with  $Re_\lambda = \frac{\lambda U_0}{(1-\alpha)^2 \nu} = \frac{Re \lambda}{(1-\alpha)^2}$
- WSS = (variation of velocity)/(boundary layer thickness) =  $\frac{(Re/\lambda)^{1/2}}{(1-\alpha)^3}$



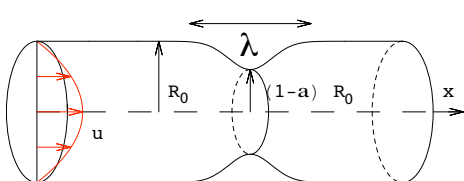
## IBL integral: 1D equation Simplified Shear Stress

- variation of velocity (flux conservation)  $U_0 \rightarrow U_0/(1 - \alpha - \delta_1)^2$
- acceleration: boundary layer  $\delta_1 \simeq \frac{\lambda}{\sqrt{Re_\lambda}}$ , with  $Re_\lambda = \frac{\lambda U_0}{(1-\alpha)^2 \nu} = \frac{Re \lambda}{(1-\alpha)^2}$
- WSS = (variation of velocity)/(boundary layer thickness) =  $\frac{(Re/\lambda)^{1/2}}{(1-\alpha)^3}$

A simple formula as been settled:

$$WSS = \left( \mu \frac{\partial u^*}{\partial y^*} \right) / \left( \mu \frac{4U_0}{R} \right) \sim .22 \frac{((Re/\lambda)^{1/2} + 3)}{(1 - \alpha)^3}$$

Reynolds number is no longer  $Re$  but  $Re\lambda$  and  $(Re/\lambda)^{1/2}$  is the inverse of the relative boundary layer thickness.

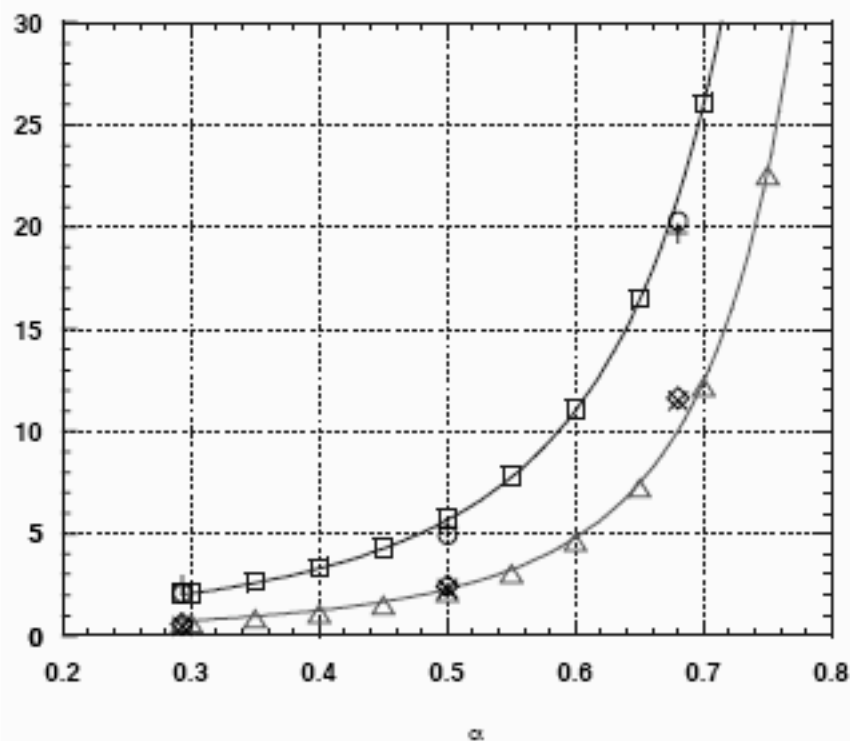


## IBL integral: Comparison with Navier Stokes (Siegel et al. 1994)

$$WSS = aRe^{1/2} + b$$

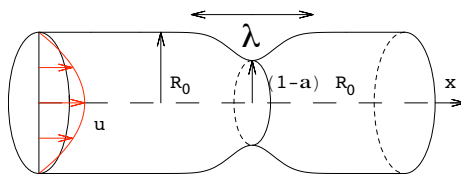
Coefficient  $a$  and  $b$  for the maximum WSS.  
 solid lines with  $\triangle$  and "square" : coefficient  $a$  and  $b$   
 obtained using the IBL integral method ;

- $\diamond$  : coefficient  $a$  derived from Siegel for  $\lambda = 3$  ;
- $\times$  : coefficient  $a$  derived from Siegel for  $\lambda = 6$  ;
- $\circ$  : coefficient  $b$  derived from Siegel for  $\lambda = 3$  ;
- $+$  : coefficient  $b$  derived from Siegel for  $\lambda = 6$ .



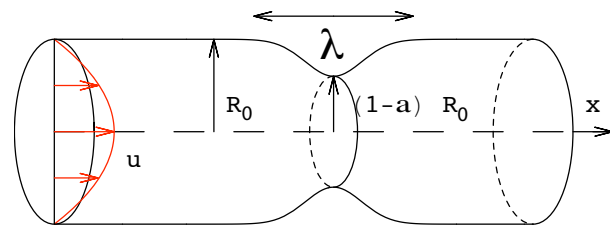
$$WSS = \left( \mu \frac{\partial u}{\partial y} \right) / \left( \mu \frac{4U_0}{R} \right) \simeq 0.22 \frac{(Re/\lambda)^{1/2} + 3}{(1-\alpha)^3}$$

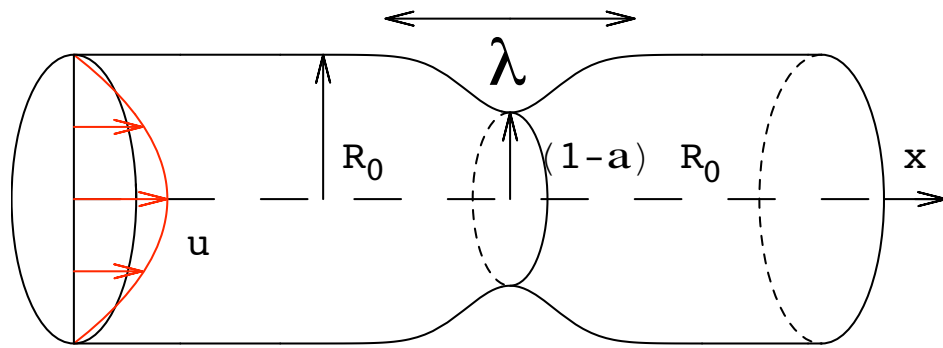


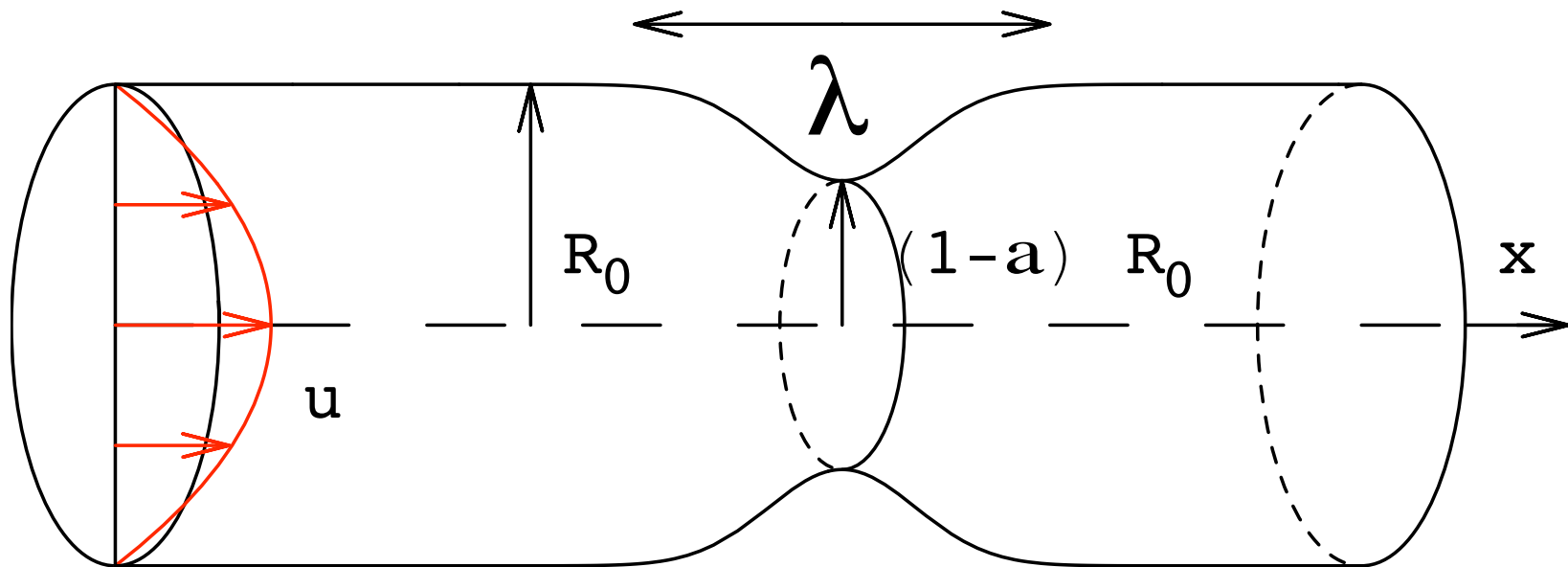


S. Lorthois, P.-Y. Lagrée, J.-P. Marc-Vergnes & F. Cassot. (2000):  
"Maximal wall shear stress in arterial stenoses: Application to the internal carotid arteries",  
Journal of Biomechanical Engineering, Volume 122, Issue 6, pp. 661-666.

Lorthois S. & Lagrée P.-Y. (2000):  
"Flow in a axisymmetric convergent: evaluation of maximum wall shear stress",  
C. R. Acad. Sci. Paris, t328, Série II b, p33-40, 2000



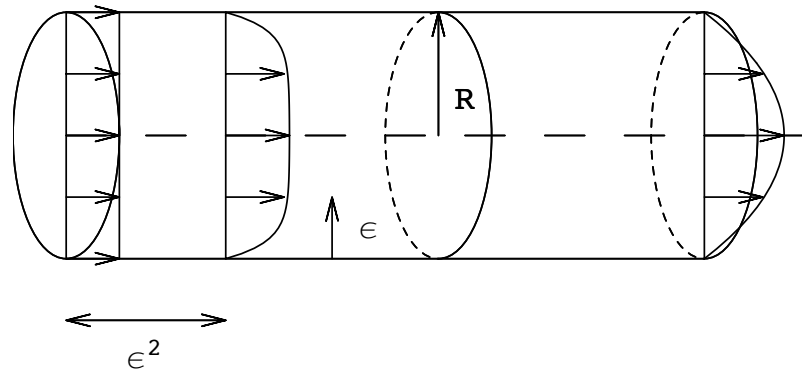




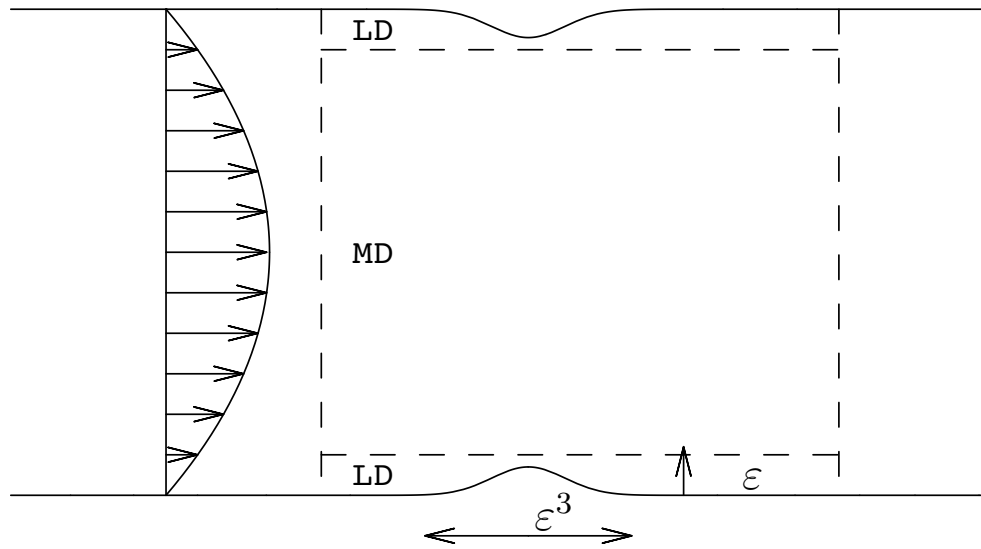
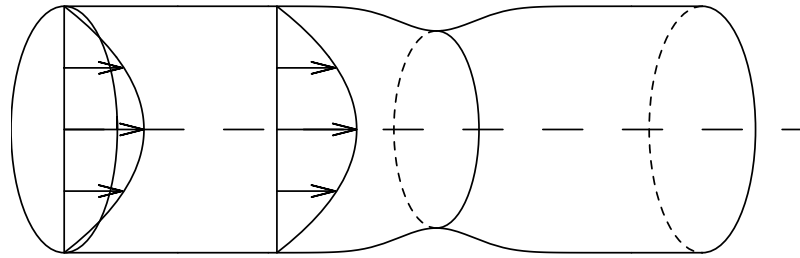






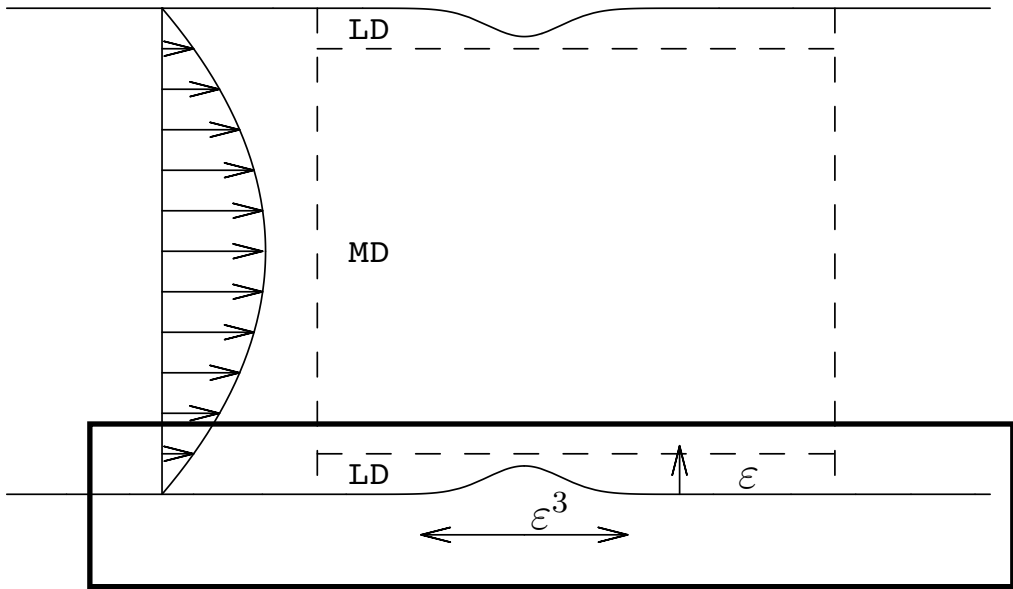
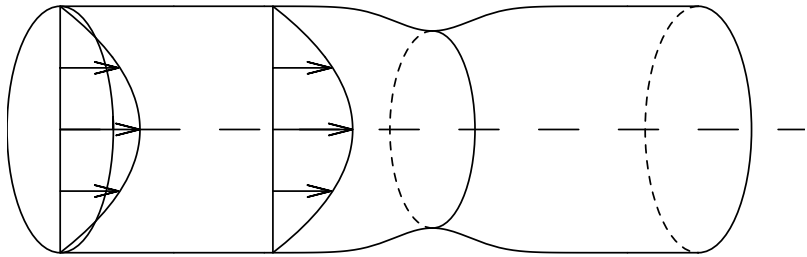






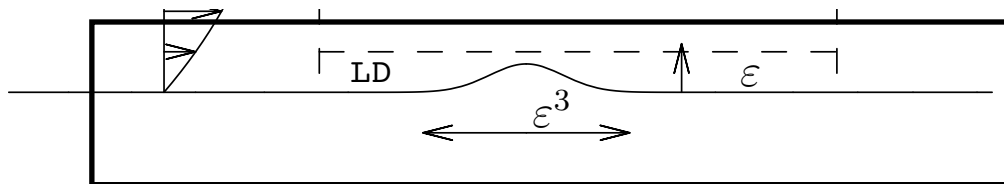
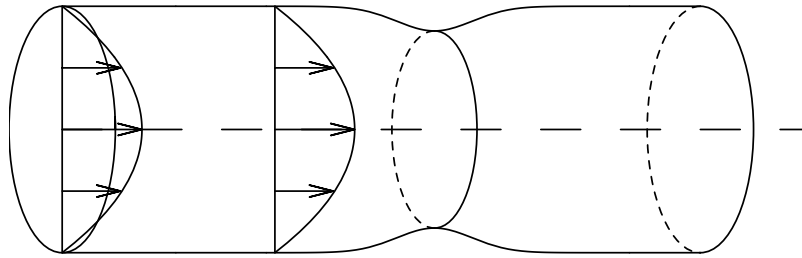
**Double Deck**

$$A = 0$$

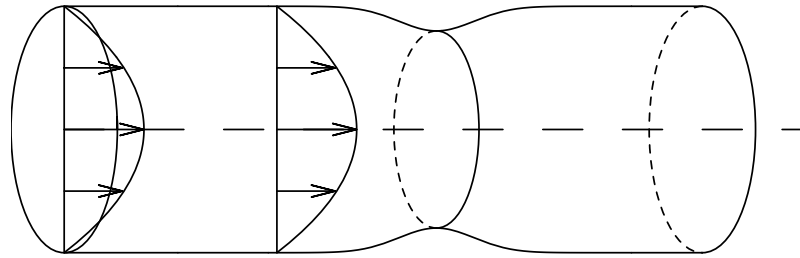


Double Deck

$$A = 0$$

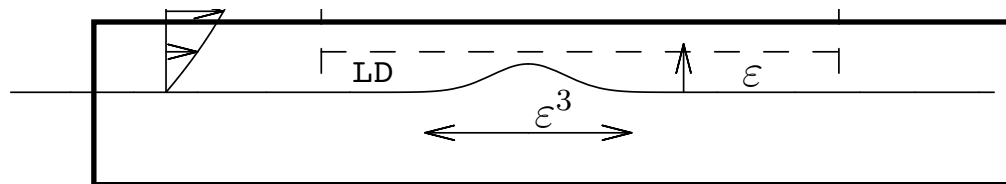


**Double Deck**

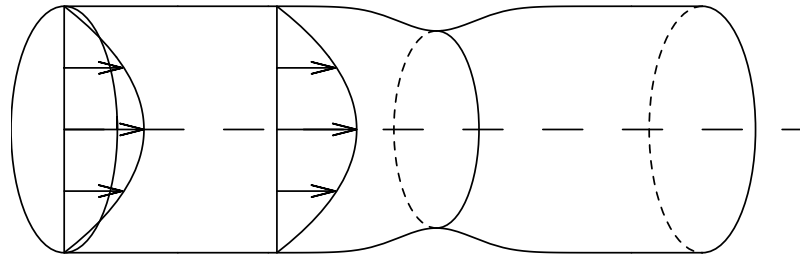


$$u = y \quad u \frac{\partial}{\partial x} u \sim \frac{\partial^2}{\partial y^2} u$$

$$\frac{\varepsilon}{x_3} \bar{u} \frac{\partial}{\partial \bar{x}} \bar{u} \sim \frac{1}{\varepsilon^2} \frac{\partial^2}{\partial \bar{y}^2} \bar{u}$$



Double Deck



$$\frac{\partial}{\partial x}u + \frac{\partial}{\partial y}v = 0,$$

$$u\frac{\partial}{\partial x}u + v\frac{\partial}{\partial y}u = -\frac{d}{dx}p + \frac{\partial^2}{\partial y^2}u$$

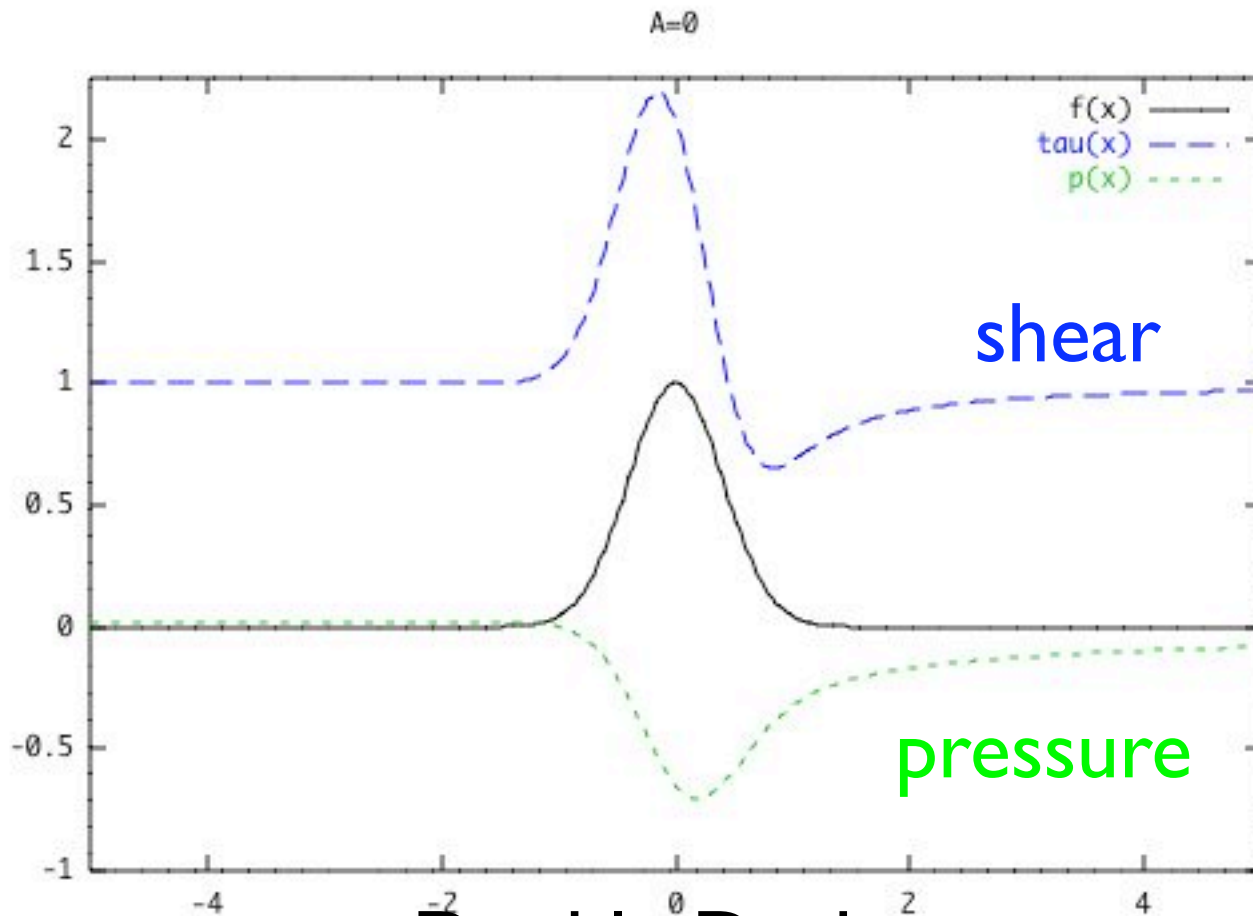
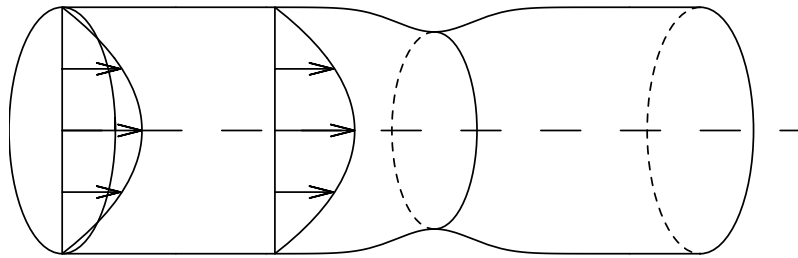
$$u(x, y = f(x)) = 0, \quad v(x, y = f(x)) = 0$$

$$\lim_{y \rightarrow \infty} u(x, y) = y.$$

again the same equations

with different scales and different boundary conditions

## Double Deck



**Double Deck**

$A = 0$

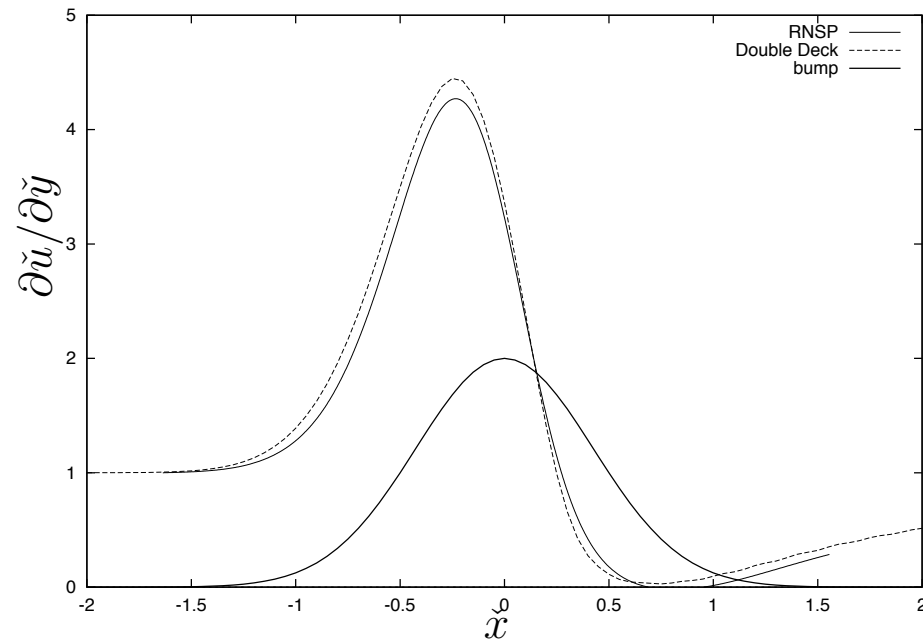
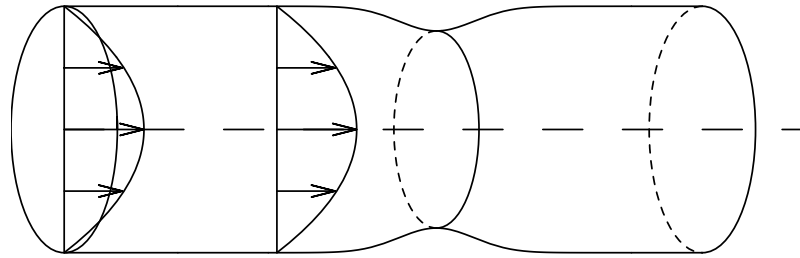
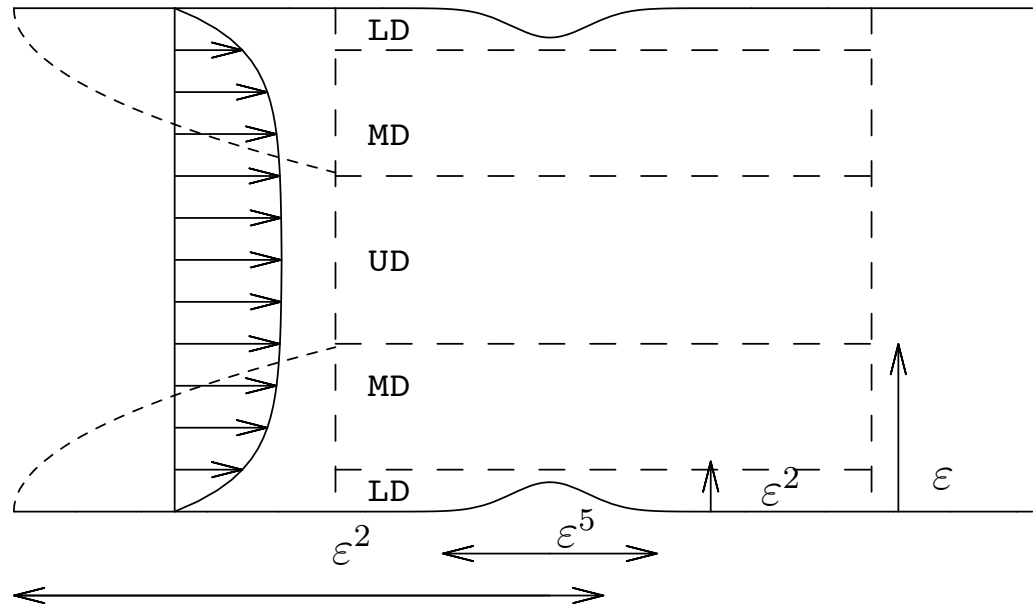
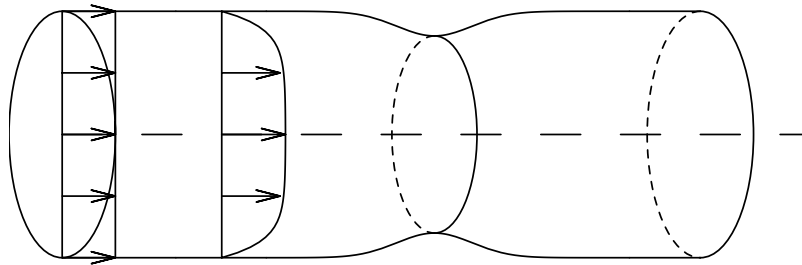


Fig. 12. Longitudinal evolution of the WSS near the incipient separation case for  $x_l = 0.0125$ . D.D. : Double Deck resolution ; RNSP : RNSP resolution rescaled in Double Deck scales.

Double Deck

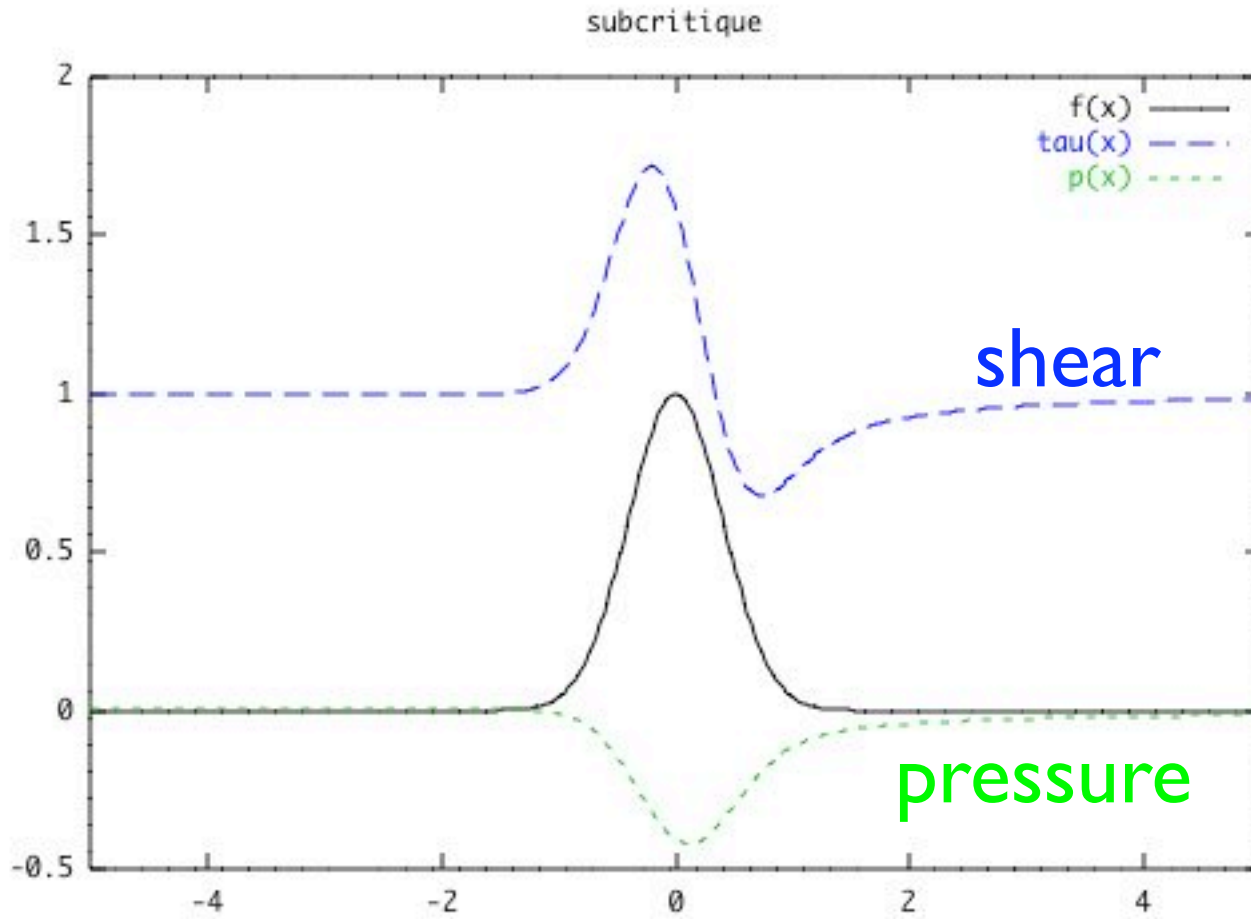
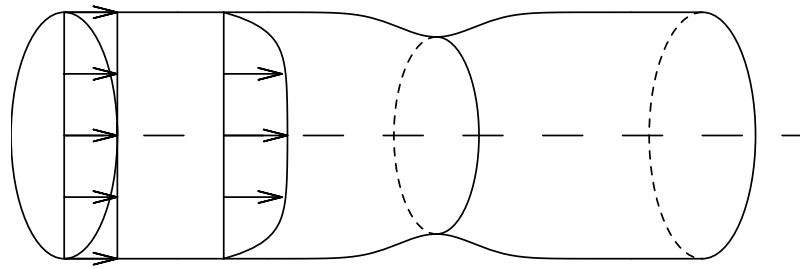
$A = 0$



**Triple Deck**

$$p = A$$





Triple Deck

$$p = A$$

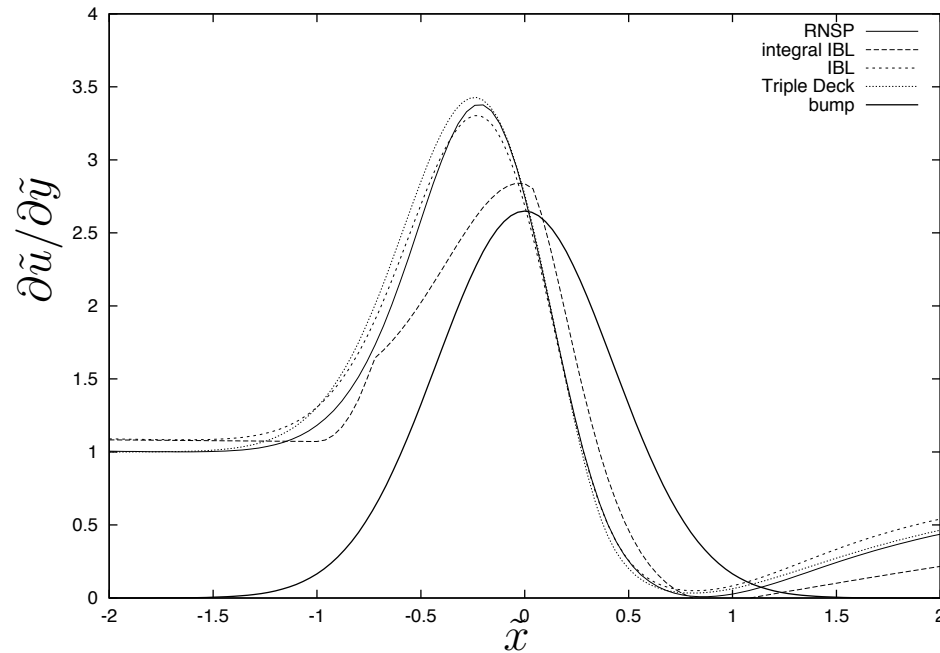
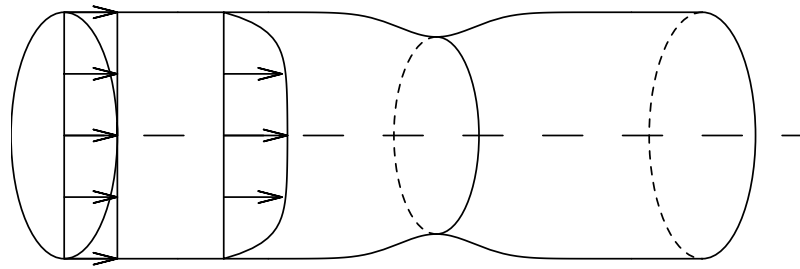
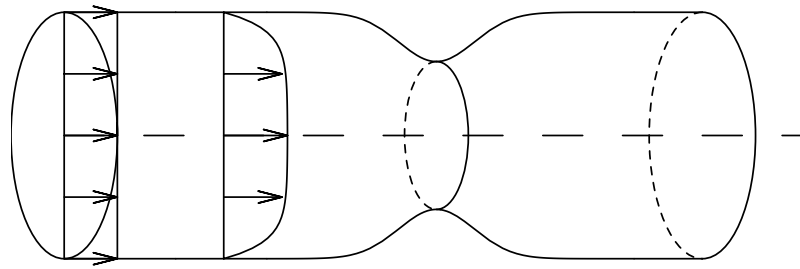


Fig. 9. Longitudinal evolution of the WSS near the incipient separation case RNSP, integral IBL, full IBL resolution (in RNSP variables, the bump is located in  $x = 0.02$ , and its width is 0.00125), and Triple Deck resolution. All the curves are rescaled in Triple Deck scales.

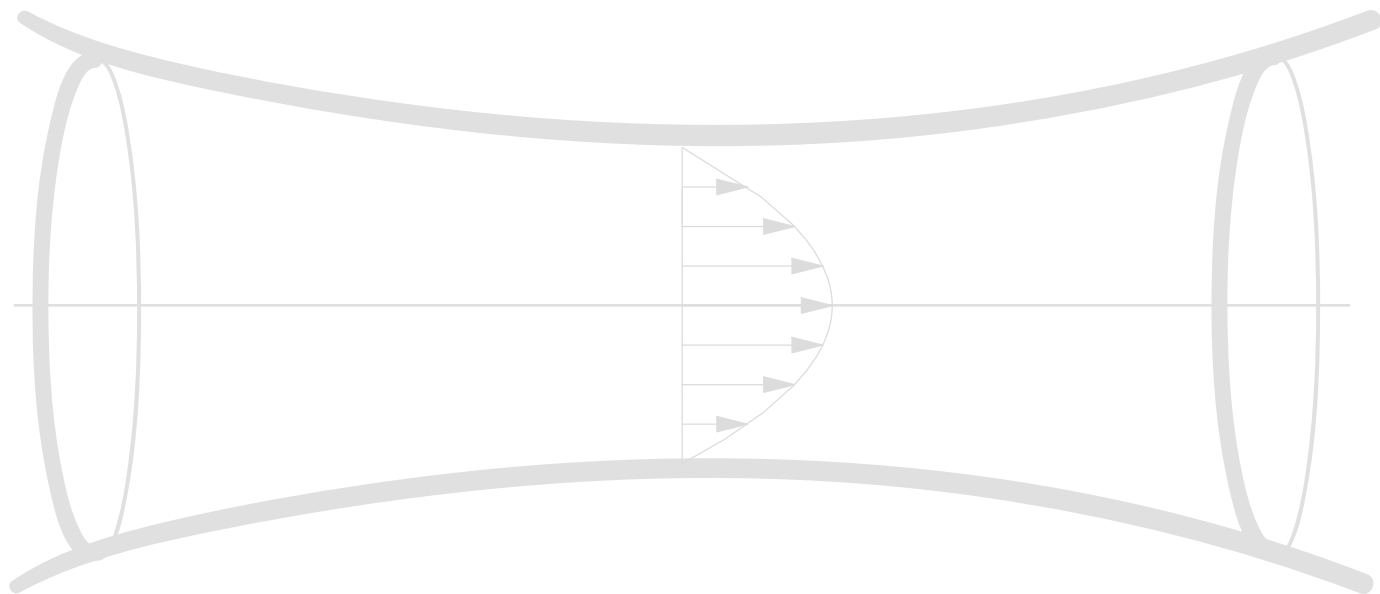
**Triple Deck**

$$p = A$$

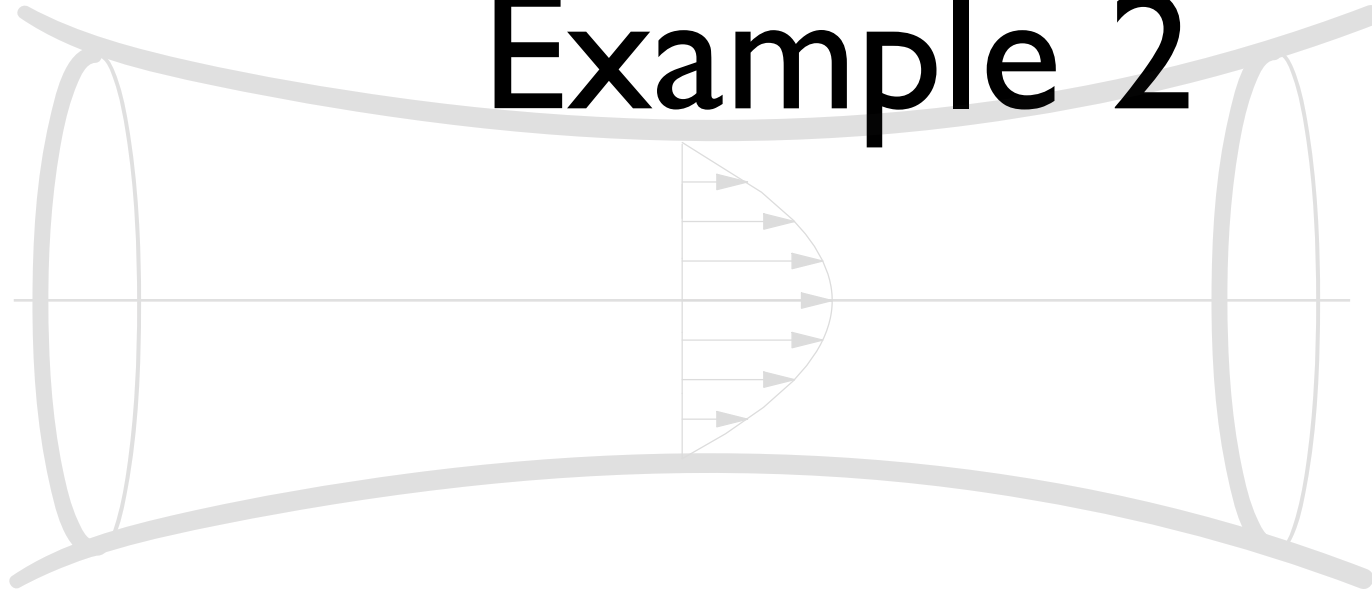


••••

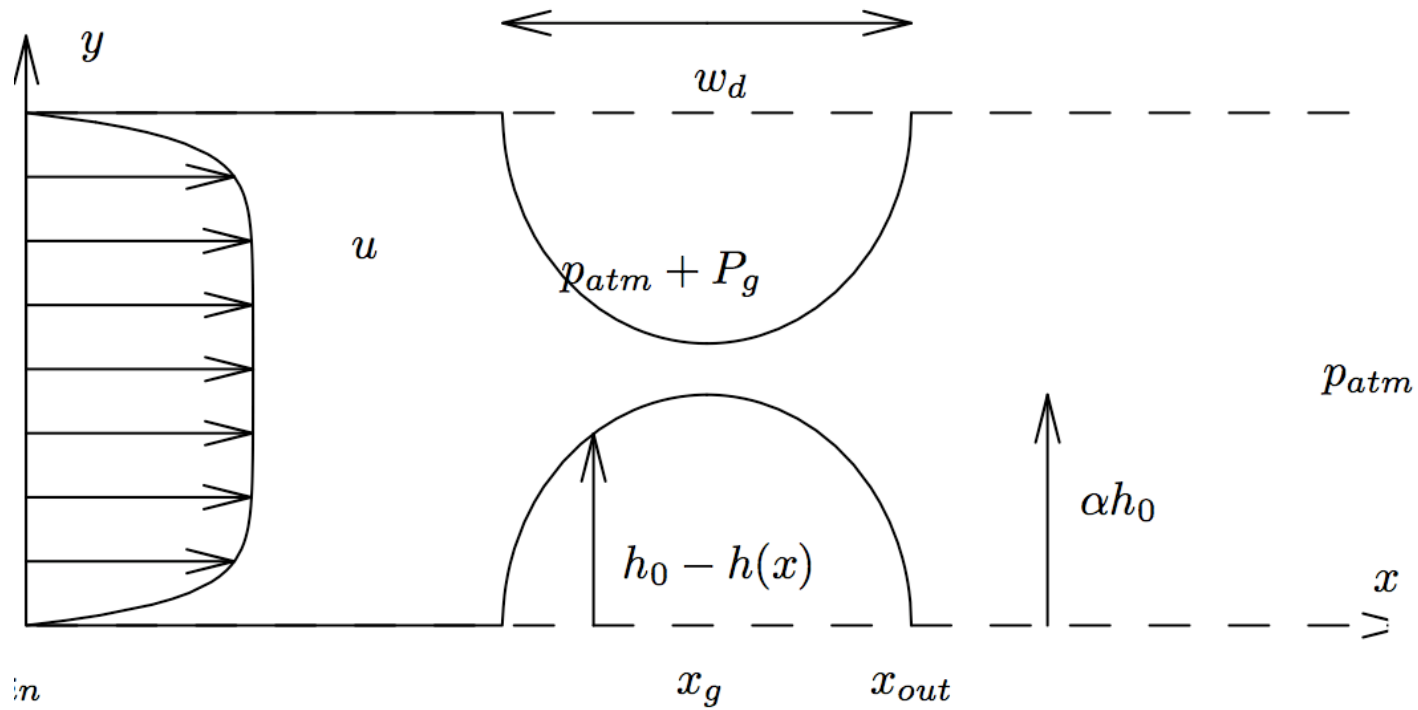
P.-Y. Lagrée & S. Lorthois (2005):  
"The RNS/Prandtl equations and their link with other asymptotic descriptions.  
Application to the computation of the maximum value of the Wall Shear Stress in a pipe",  
Int. J. Eng Sci., Vol 43/3-4 pp 352-378.



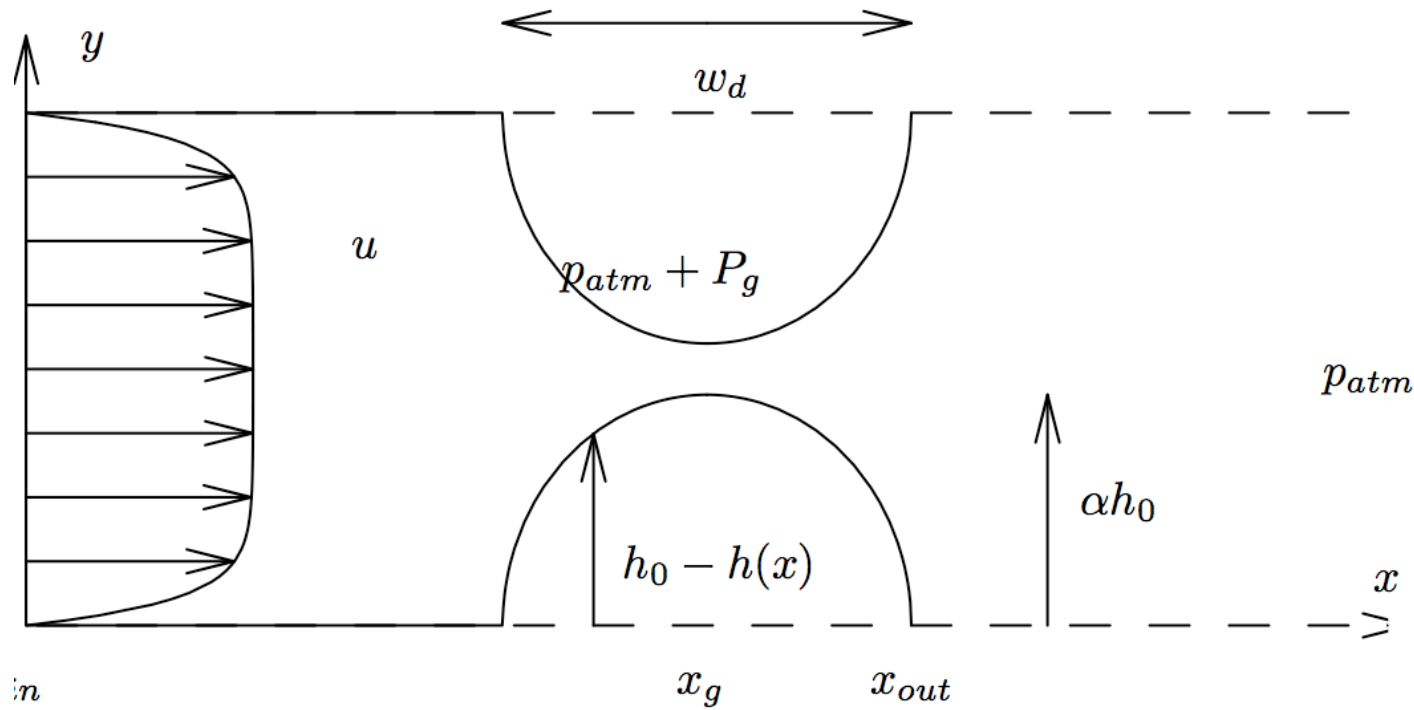
# Example 2



- Flow in a 2D stenosed vessel
- steady, rigid wall



- Flow in a stenosed vessel
- steady, rigid wall

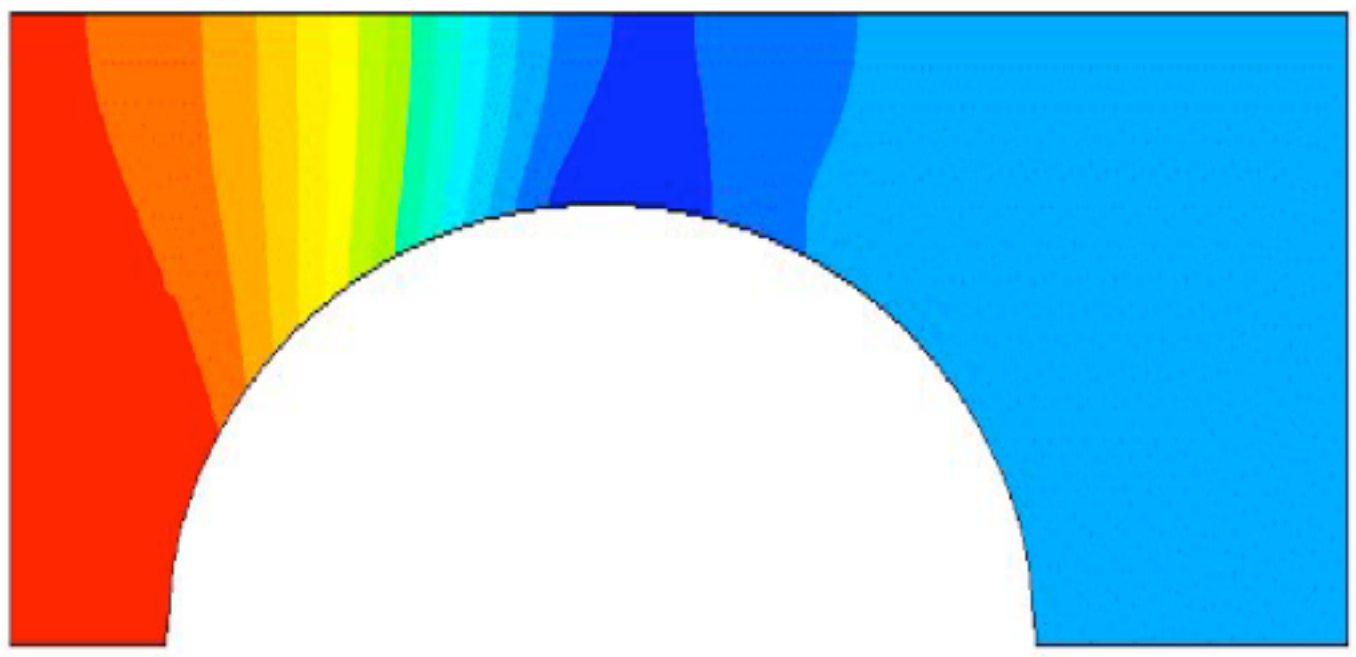
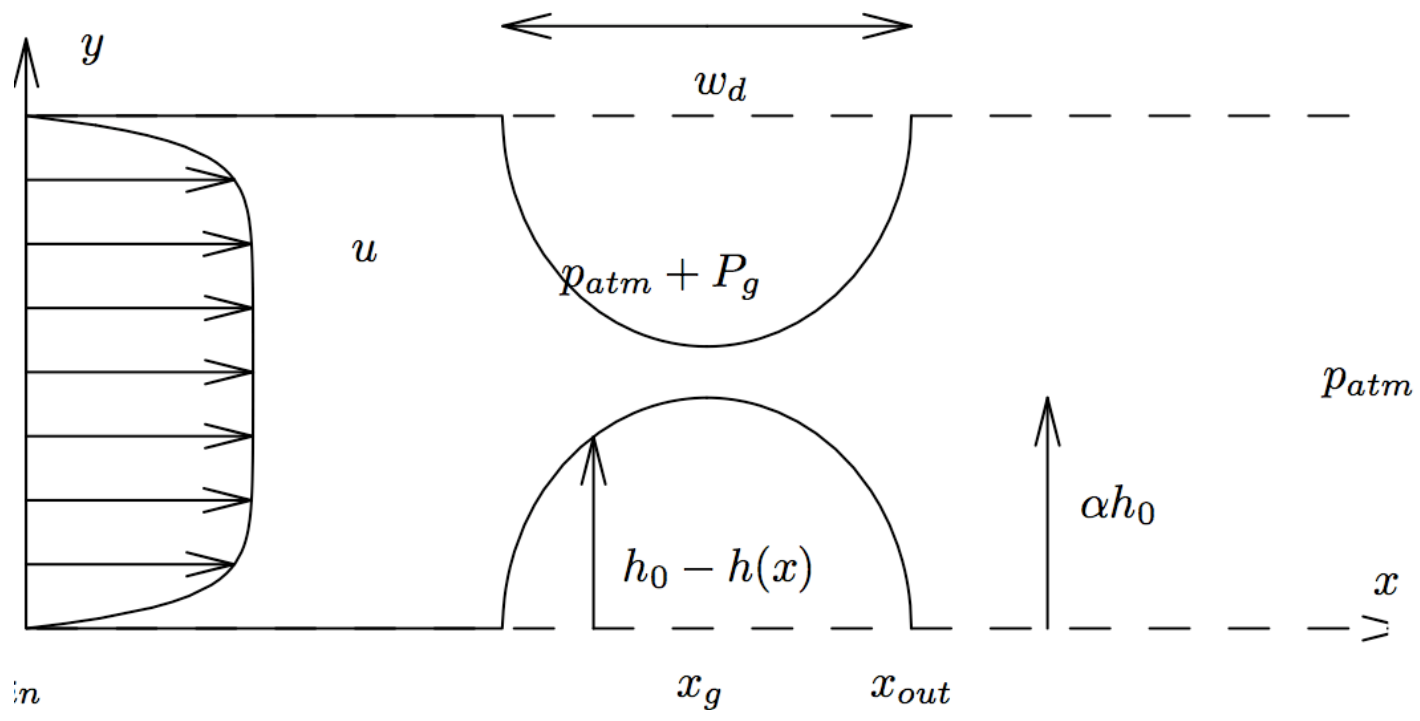


$$\frac{\partial}{\partial x}u + \frac{\partial}{\partial y}v = 0$$

$$u \frac{\partial}{\partial x}u + v \frac{\partial}{\partial y}u = -\frac{\partial}{\partial x}p + \frac{\partial^2}{\partial y^2}u$$

$$0 = -\frac{\partial}{\partial y}p$$

RNSP non dimensional





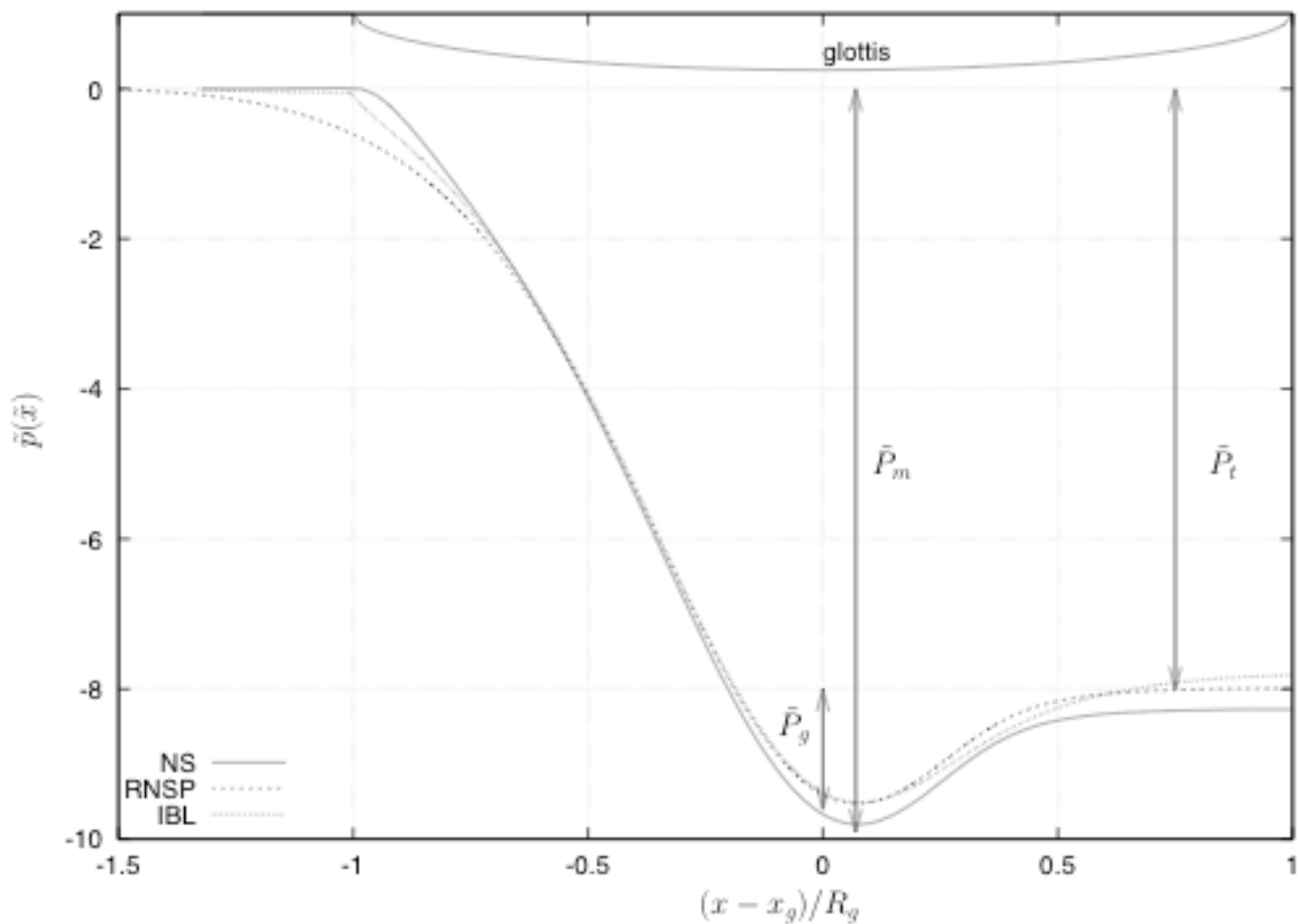
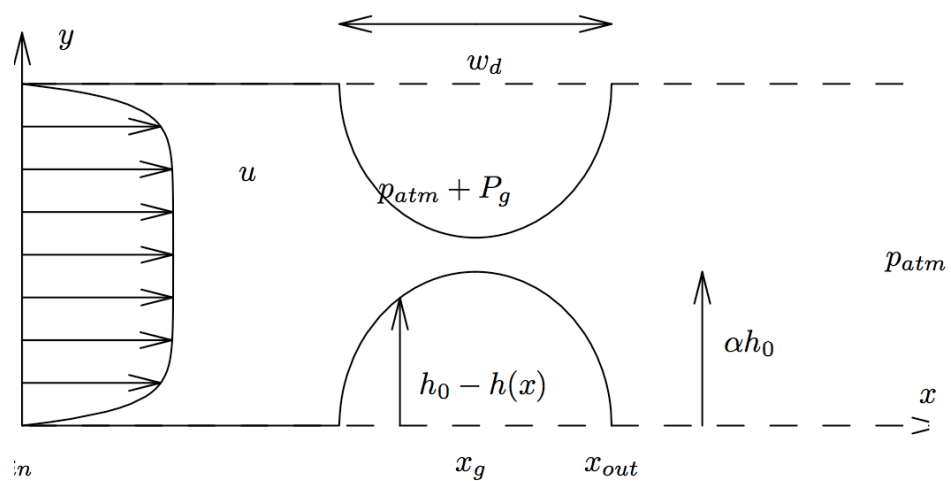


Fig. 2 A comparison between computed non-dimensional pressure for the three models (NS, IBL and RNSP, in this last case the wall has

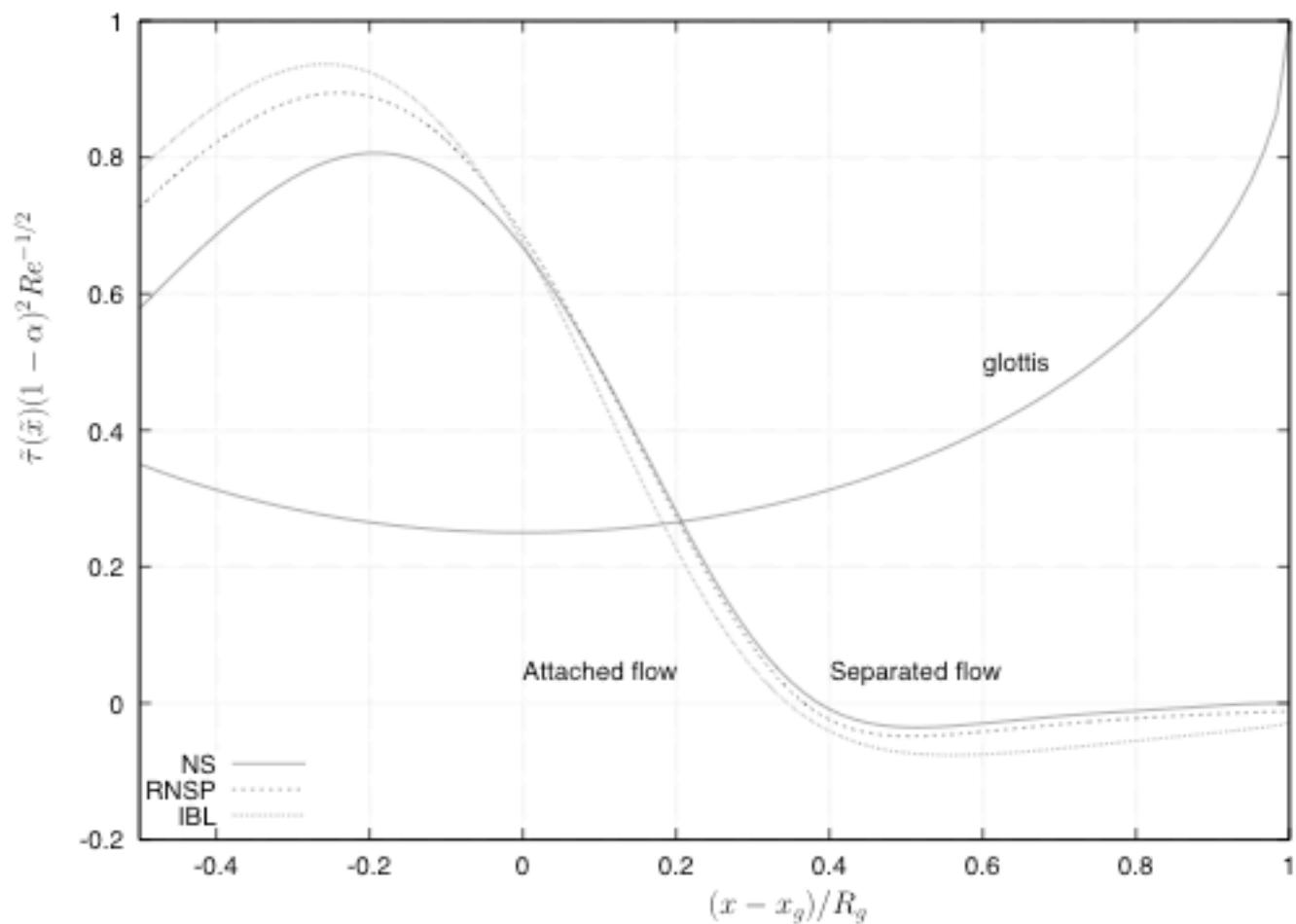
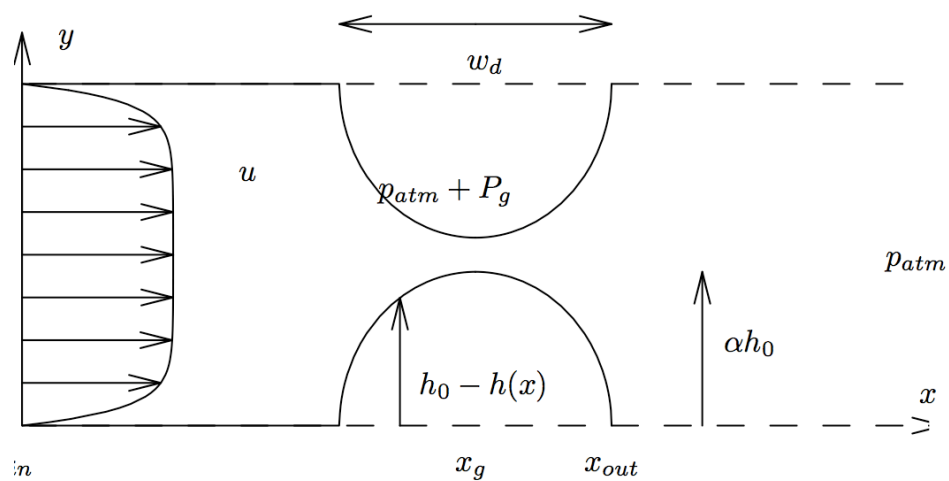
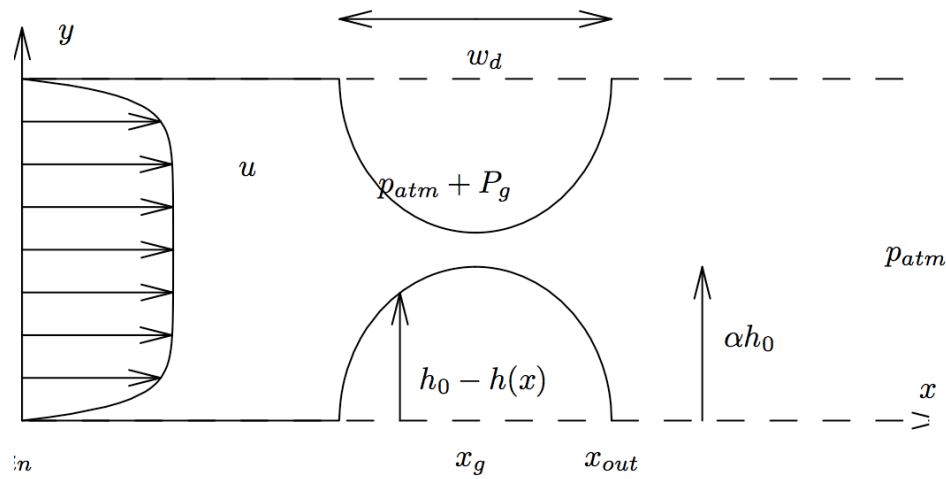


Fig. 4. A comparison between computed skin friction divided by  $(0.47 + 2.07)(1 - \alpha)^{-2}/\bar{x} \approx (1 - \alpha)^{-2} Re^{1/2}$  for the three models



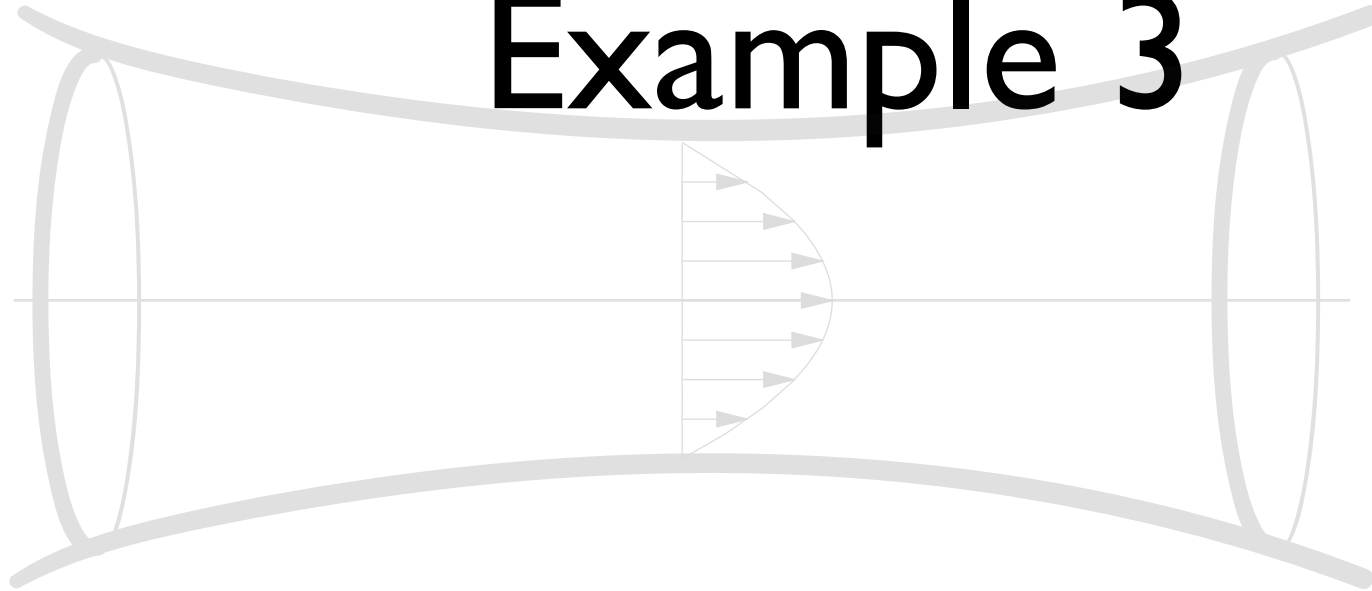
P.-Y. Lagrée, E. Berger, M. Deverge, C. Vilain & A. Hirschberg (2005):

"Characterization of the pressure drop in a 2D symmetrical pipe: some asymptotical, numerical and experimental comparisons",  
 ZAMM: Z. Angew. Math. Mech. 85, No. 2, pp. 141-146.

M. Deverge, X. Pelorson, C. Vilain, P.-Y. Lagrée, F. Chentouf, J. Willems & A. Hirschberg (2003):

"Influence of the collision on the flow through in-vitro rigid models of the vocal folds".  
 J. Acoust. Soc. Am. 114, pp. 3354 - 3362.

# Example 3



- Flow in a stenosed vessel
- steady, rigid wall
- non symmetrical case

# non symmetrical case



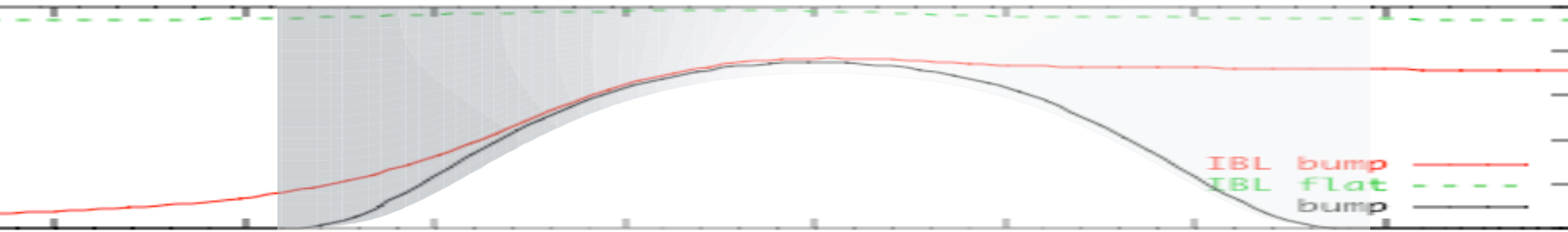
- RNSP
- modified integral method to take into account the transverse variation of pressure
- NS

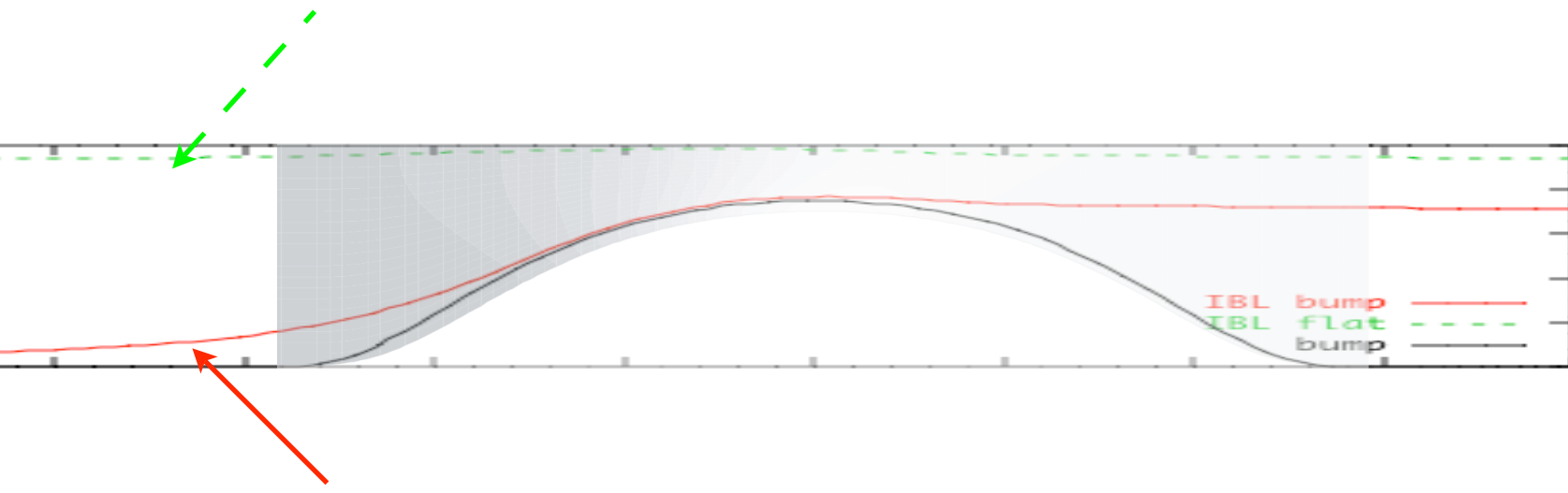
# non symmetrical case



- RNSP
- modified integral method to take into account the transverse variation of pressure
- NS

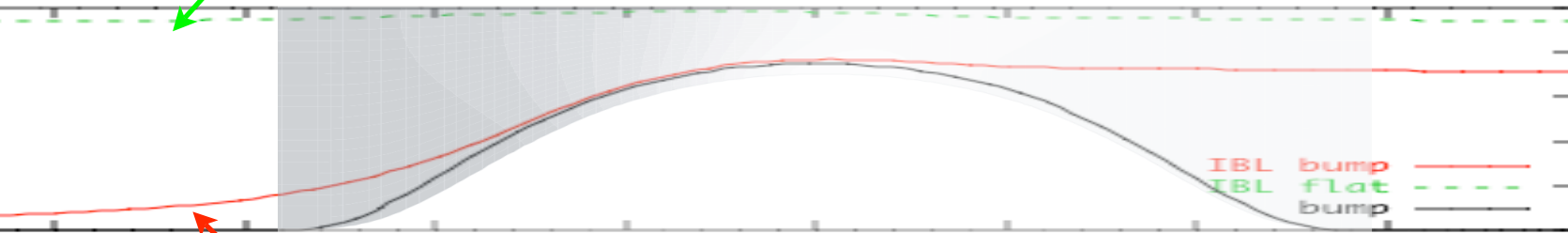
# non symmetrical case





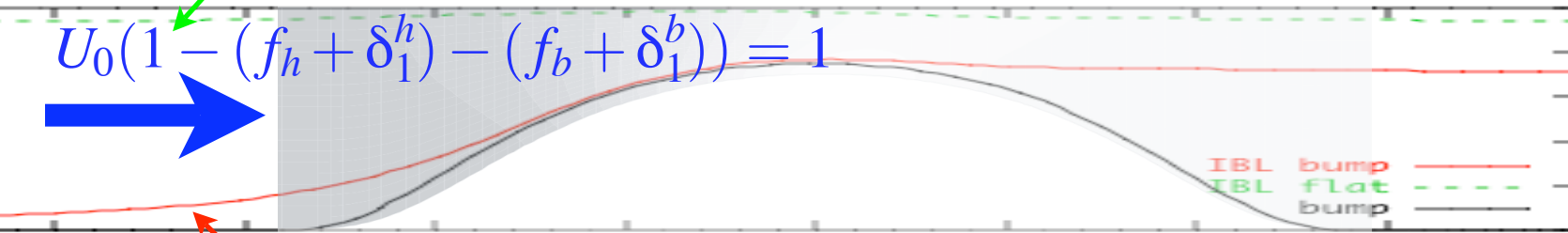


$$\frac{d}{dx} \left( \frac{\delta_1^h}{H} \right) + \frac{\delta_1^h}{u_e^h} \left( 1 + \frac{2}{H} \right) \frac{du_e^h}{dx} = \frac{f_2 H}{\delta_1^h u_e^h}, \quad \delta_1^h = F(p_e^h)$$



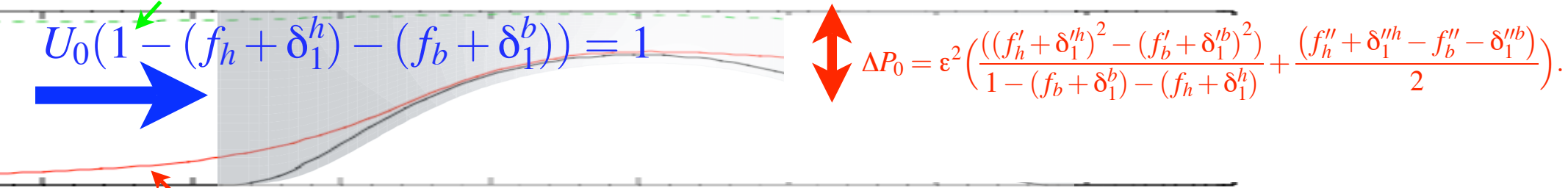
$$\frac{d}{dx} \left( \frac{\delta_1^b}{H} \right) + \frac{\delta_1^b}{u_e^b} \left( 1 + \frac{2}{H} \right) \frac{du_e^b}{dx} = \frac{f_2 H}{\delta_1^b u_e^b}, \quad \delta_1^b = F(p_e^b)$$

$$\frac{d}{dx} \left( \frac{\delta_1^h}{H} \right) + \frac{\delta_1^h}{u_e^h} \left( 1 + \frac{2}{H} \right) \frac{du_e^h}{dx} = \frac{f_2 H}{\delta_1^h u_e^h}, \quad \delta_1^h = F(p_e^h)$$

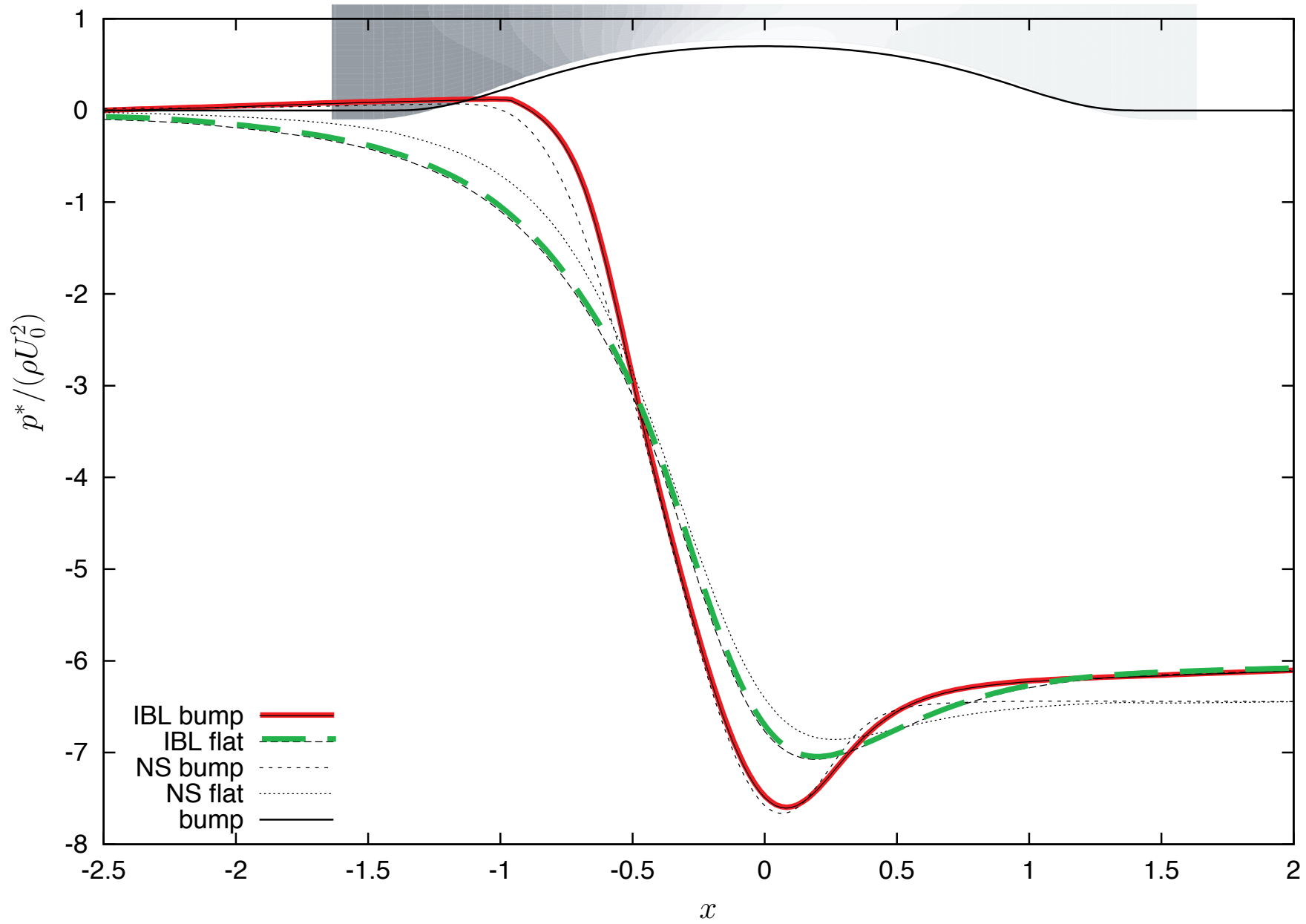


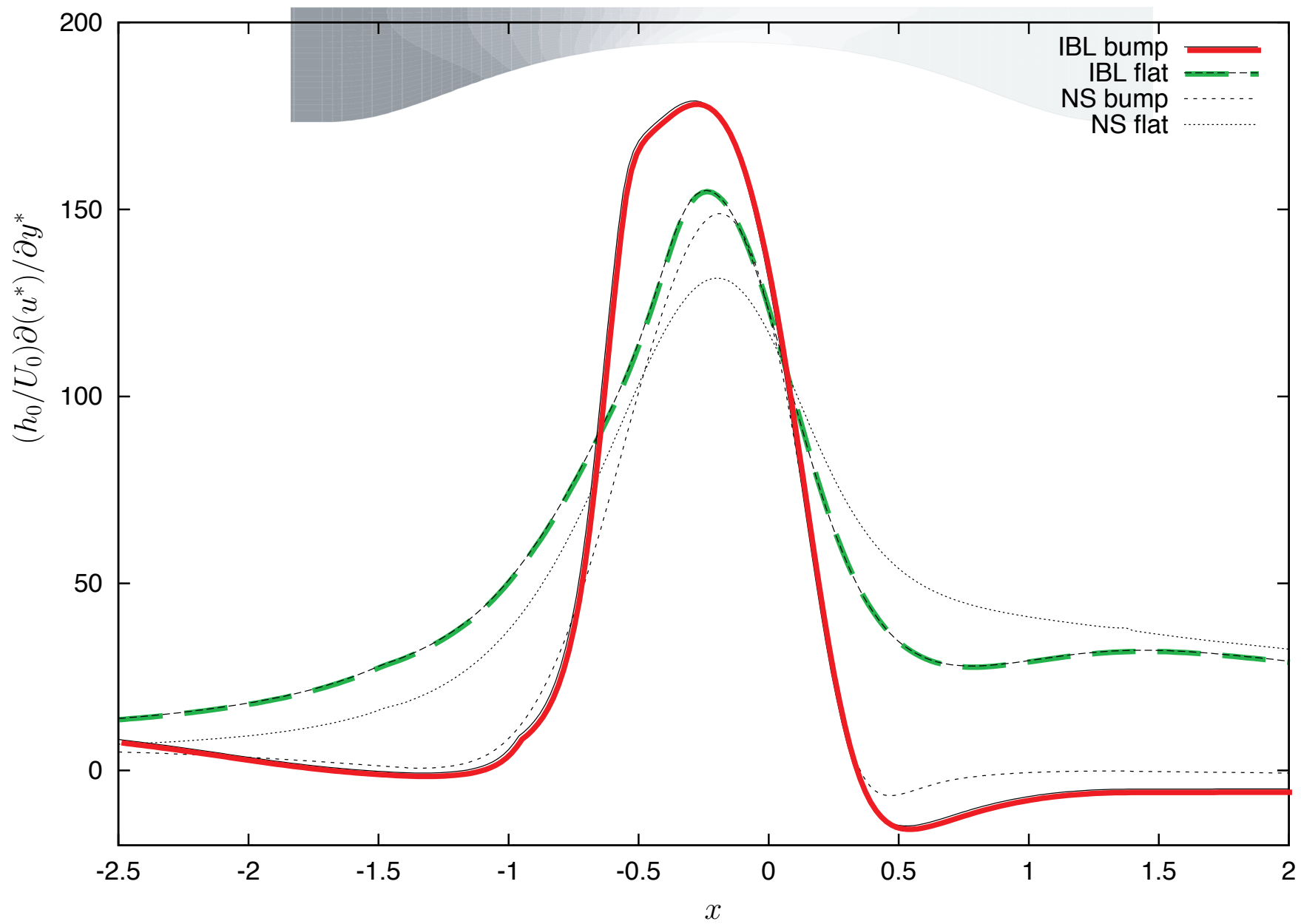
$$\frac{d}{dx} \left( \frac{\delta_1^b}{H} \right) + \frac{\delta_1^b}{u_e^b} \left( 1 + \frac{2}{H} \right) \frac{du_e^b}{dx} = \frac{f_2 H}{\delta_1^b u_e^b}, \quad \delta_1^b = F(p_e^b)$$

$$\frac{d}{dx}\left(\frac{\delta_1^h}{H}\right) + \frac{\delta_1^h}{u_e^h}\left(1 + \frac{2}{H}\right)\frac{du_e^h}{dx} = \frac{f_2 H}{\delta_1^h u_e^h}, \quad \delta_1^h = F(p_e^h)$$



$$\frac{d}{dx}\left(\frac{\delta_1^b}{H}\right) + \frac{\delta_1^b}{u_e^b}\left(1 + \frac{2}{H}\right)\frac{du_e^b}{dx} = \frac{f_2 H}{\delta_1^b u_e^b}, \quad \delta_1^b = F(p_e^b)$$

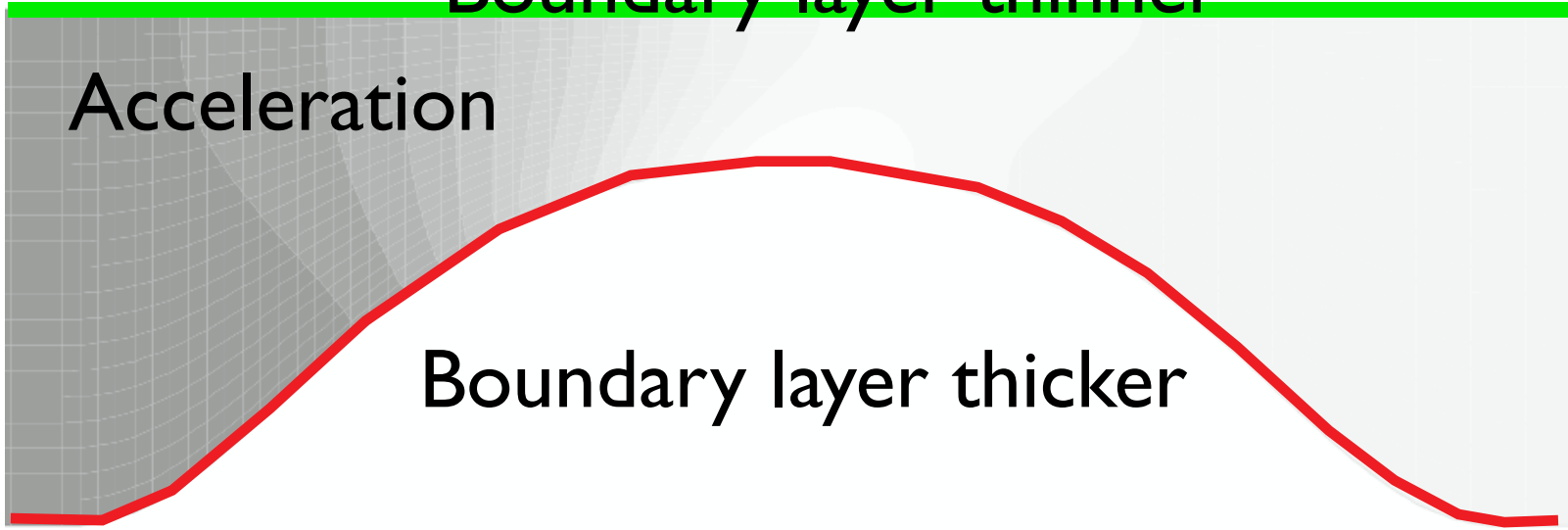




Boundary layer thinner

Acceleration

Boundary layer thicker



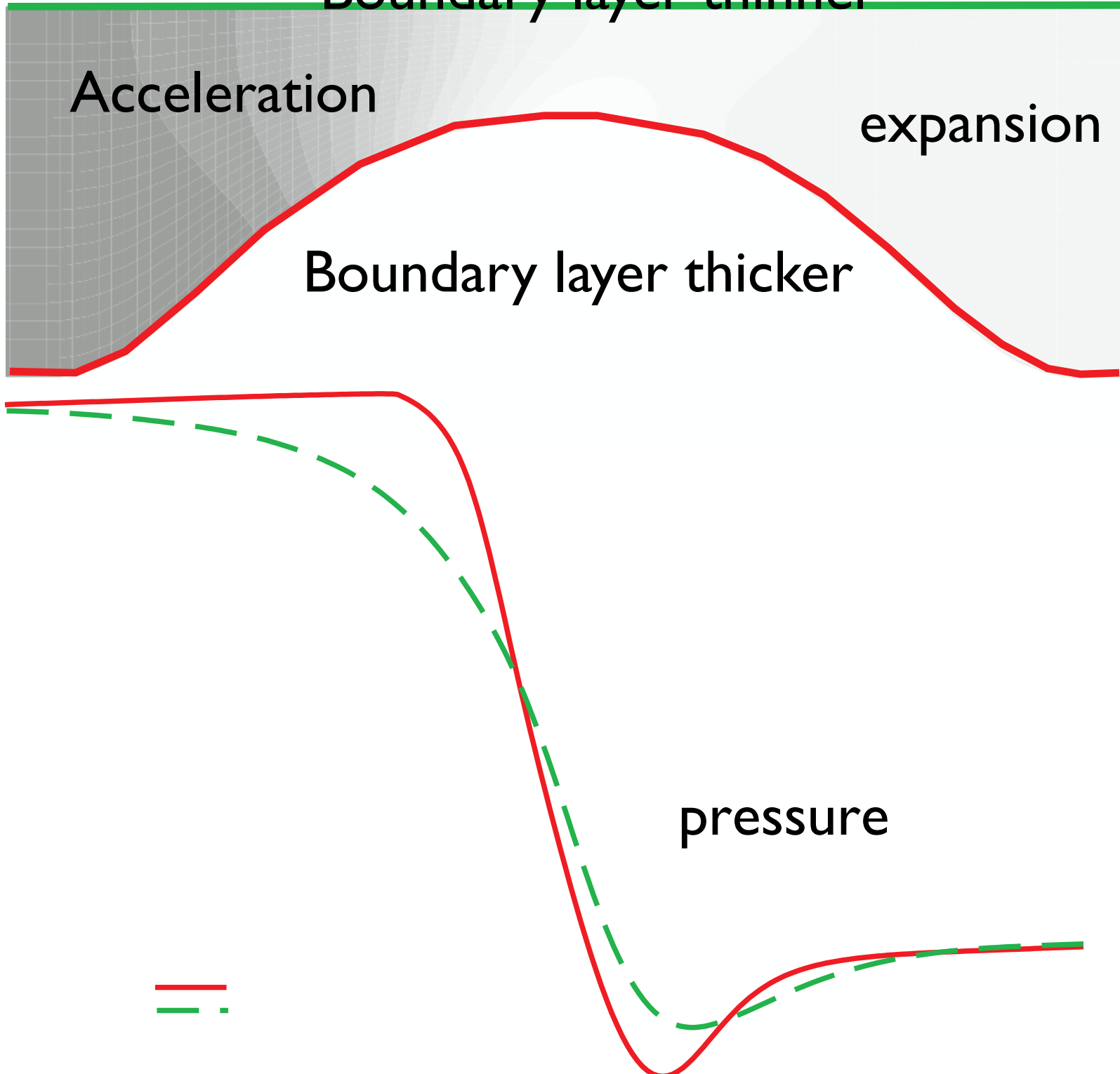
Boundary layer thinner

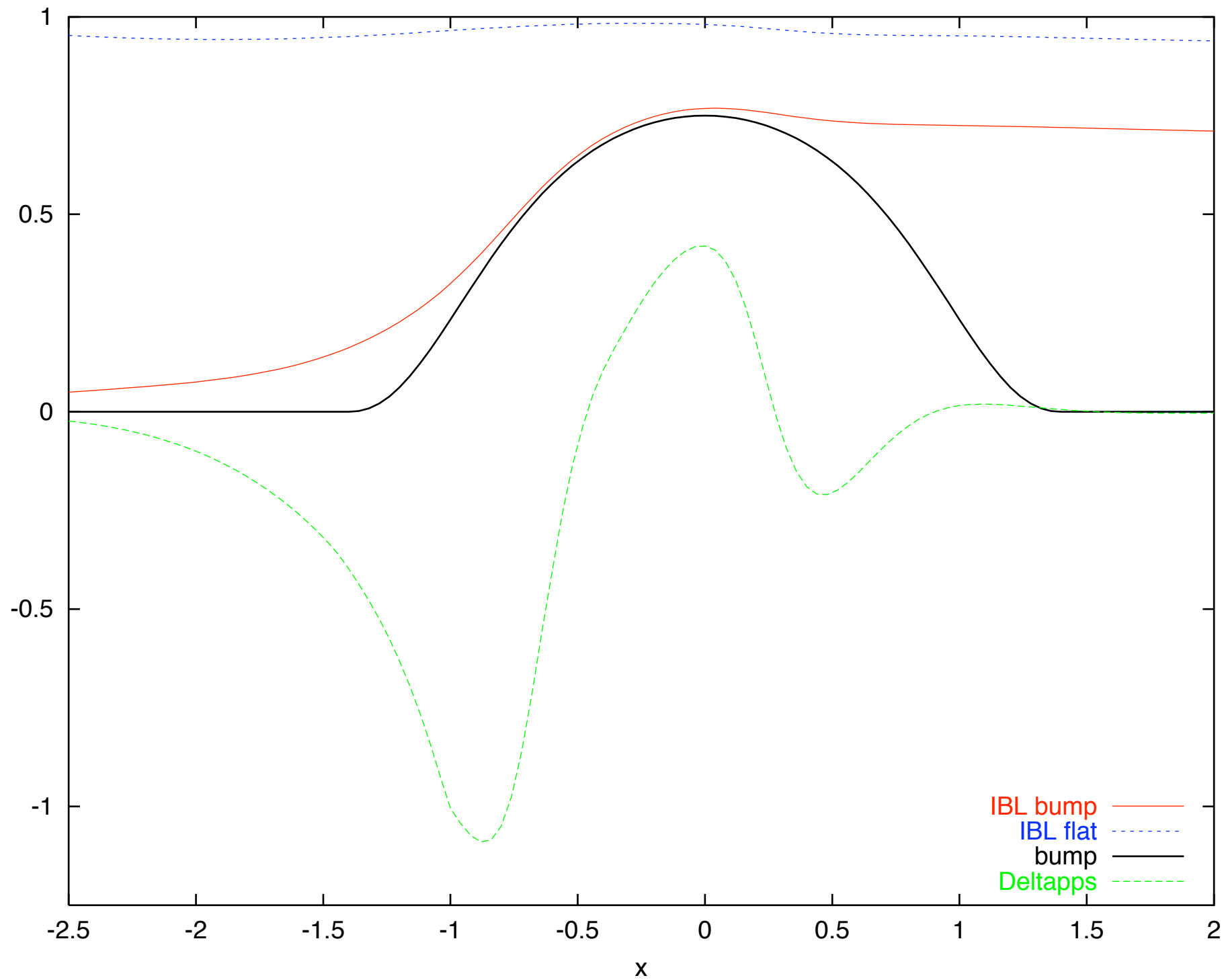
Acceleration

expansion

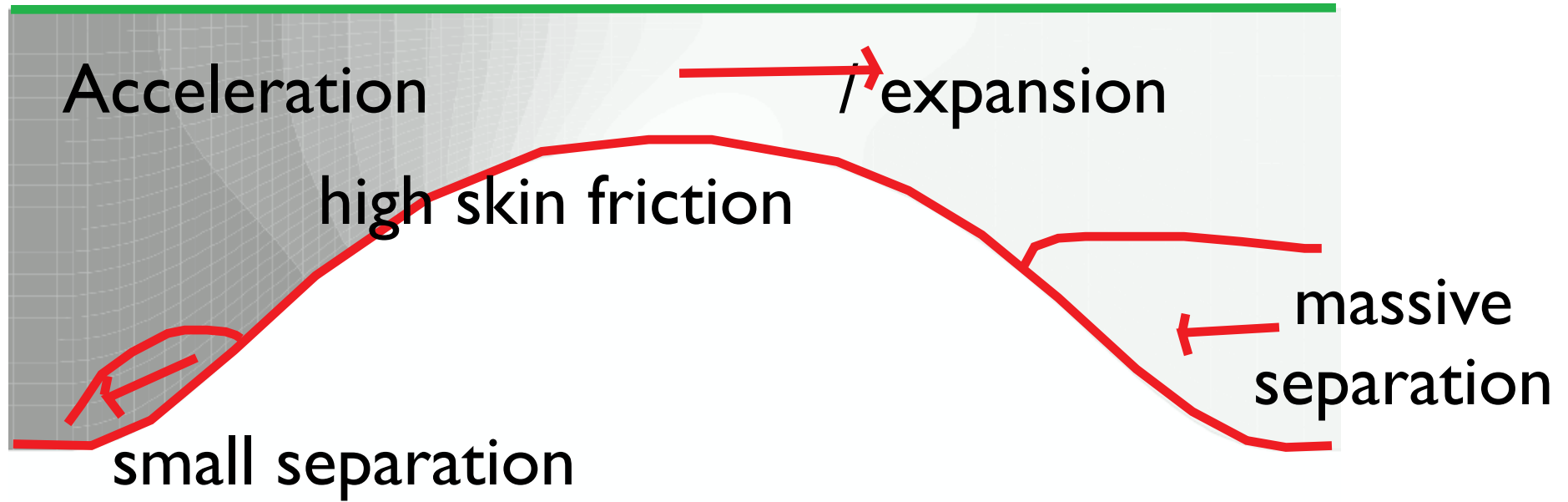
Boundary layer thicker

pressure





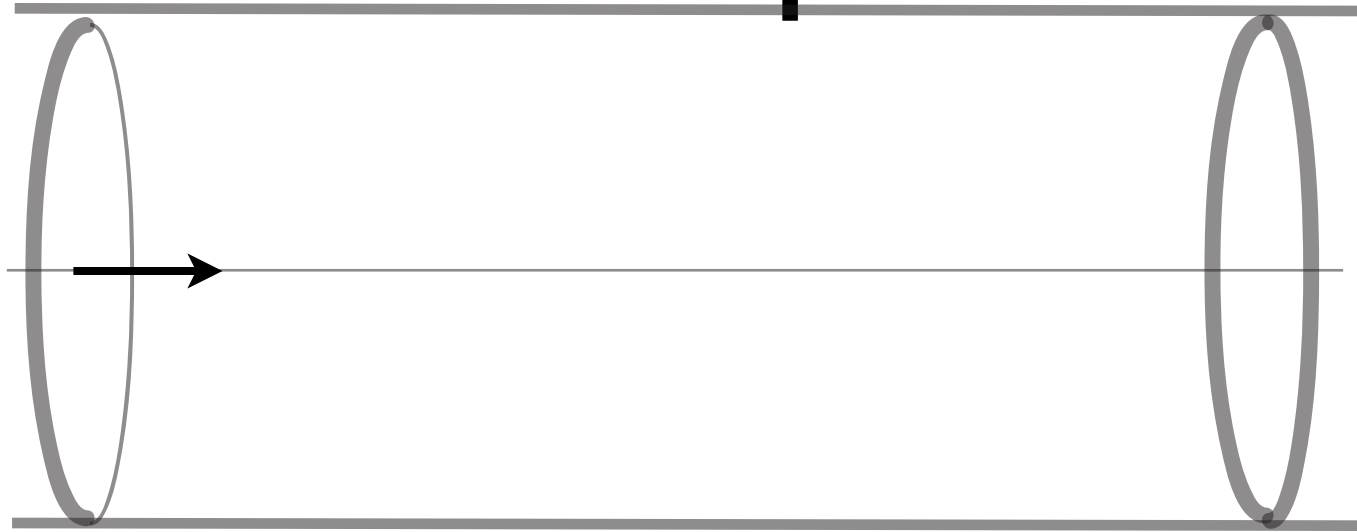




P.-Y. Lagrée, A. Van Hirtum & X. Pelorson (2007):  
"Asymmetrical effects in a 2D stenosis".

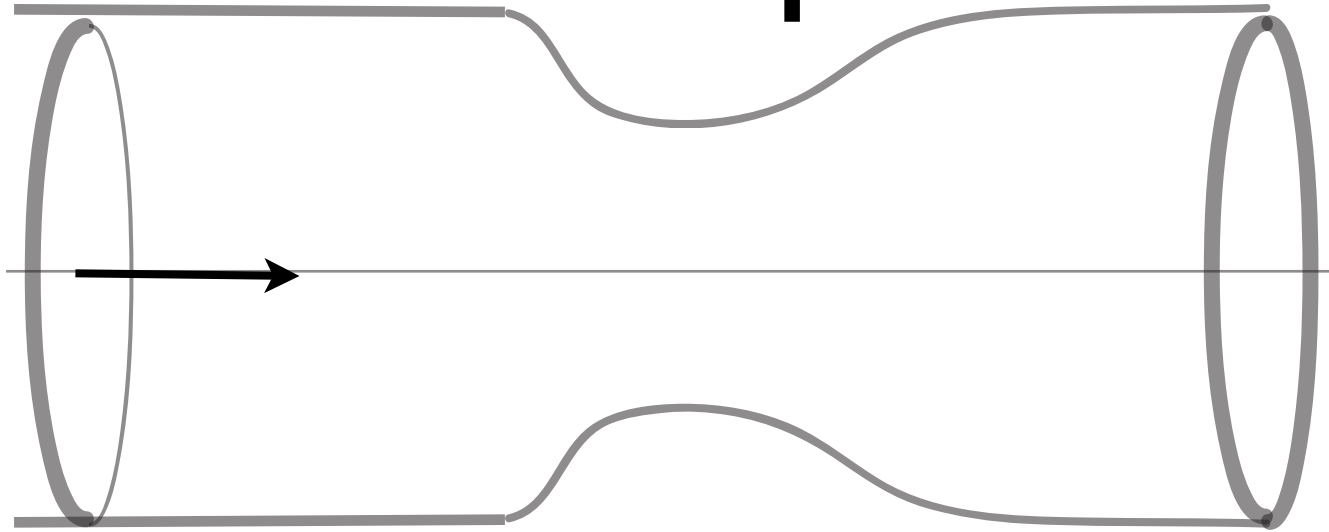
European Journal of Mechanics - B/Fluids, Volume 26, Issue 1, January-February 2007, Pages 83-92

# Example 4

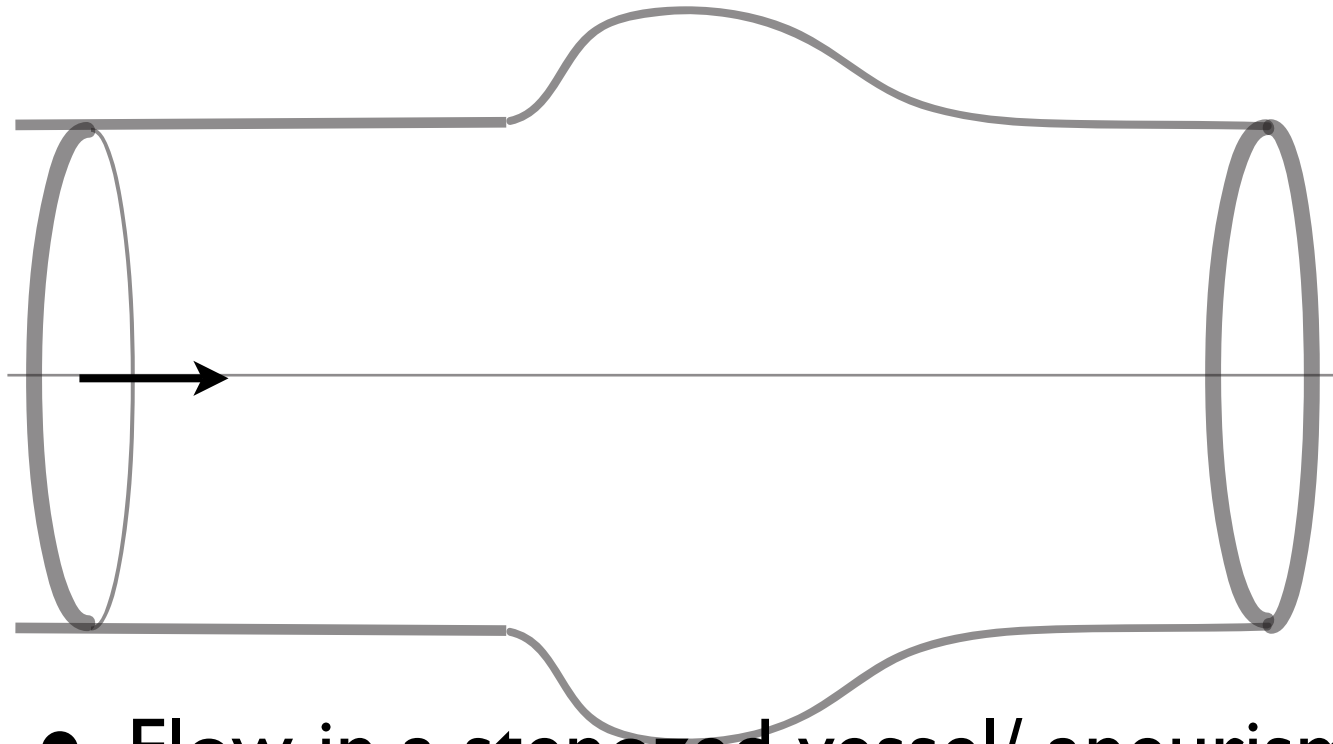


- Flow in a stenosed vessel/ aneurism
- unsteady, rigid wall

# Example 2

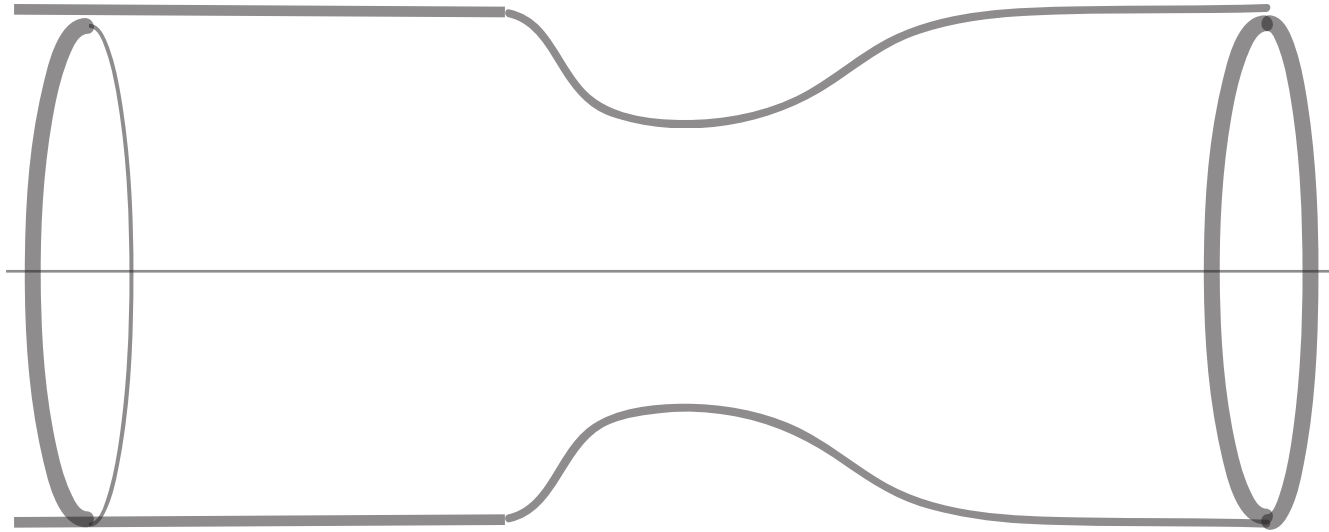


- Flow in a stenosed vessel/
- unsteady, rigid wall

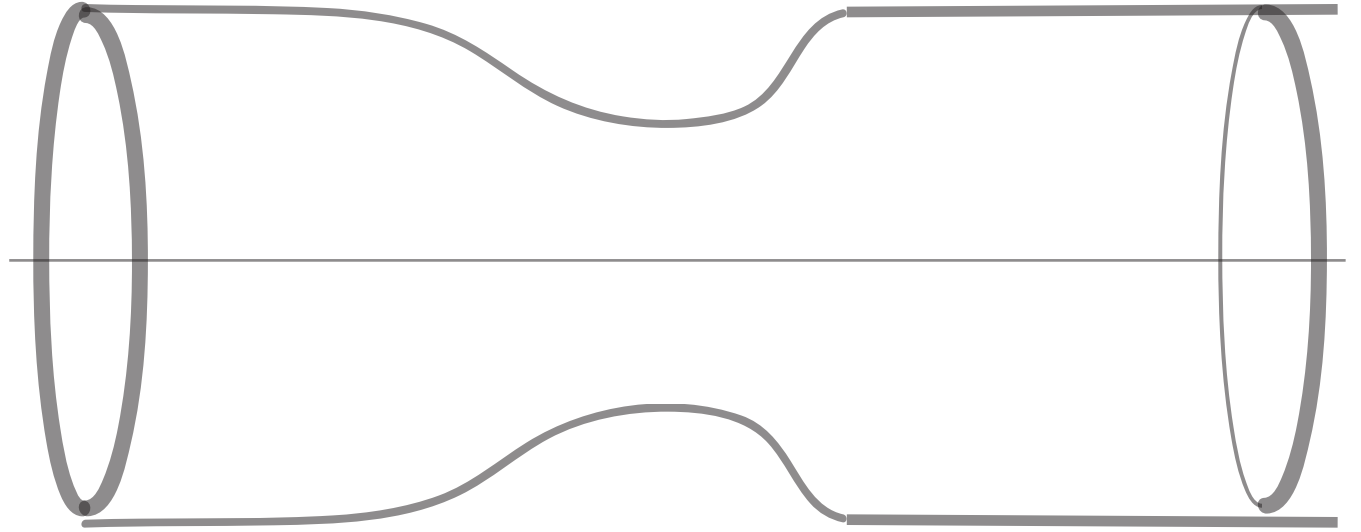


- Flow in a stenosed vessel/ aneurysm
- unsteady, rigid wall

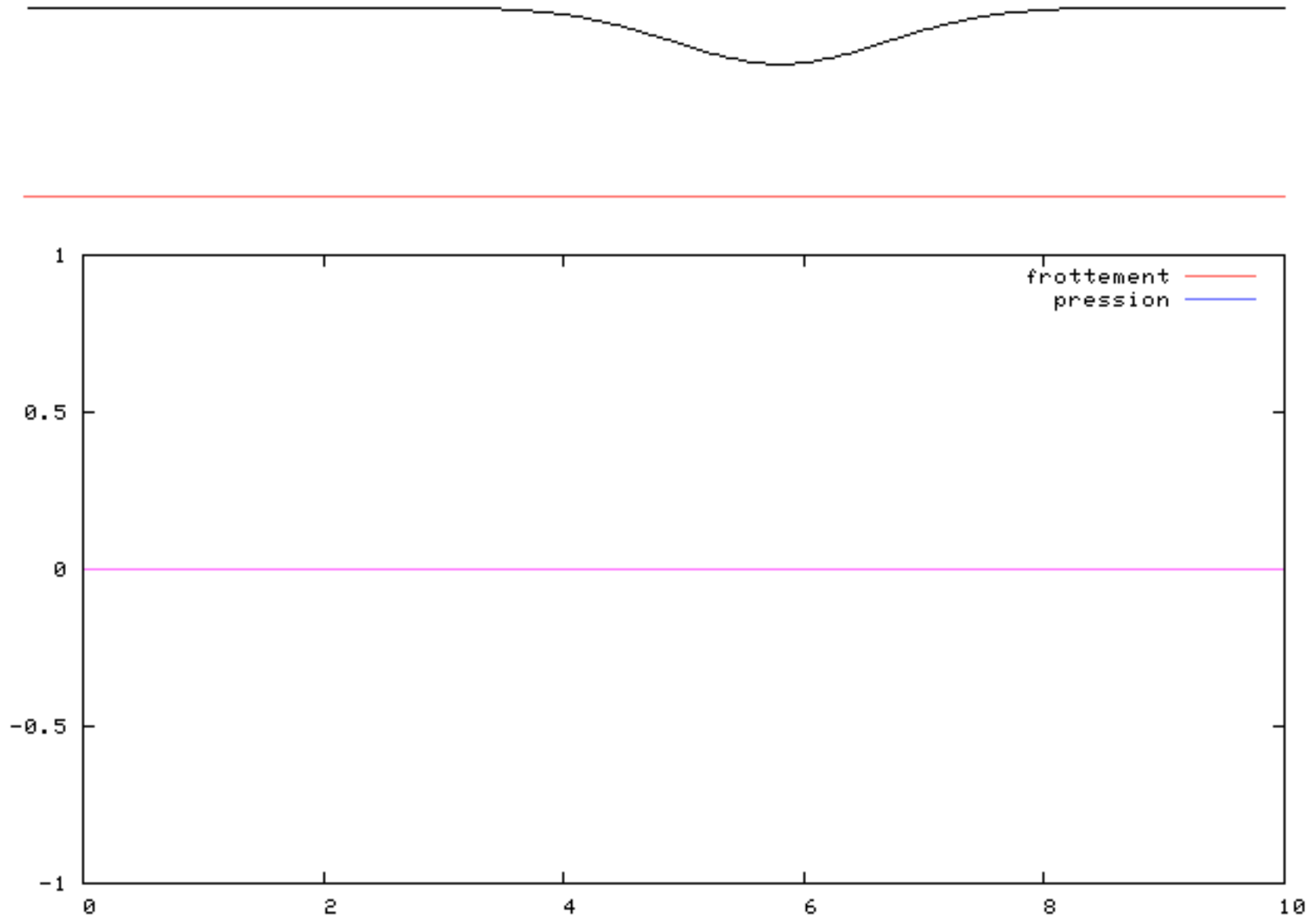
- Stenosis



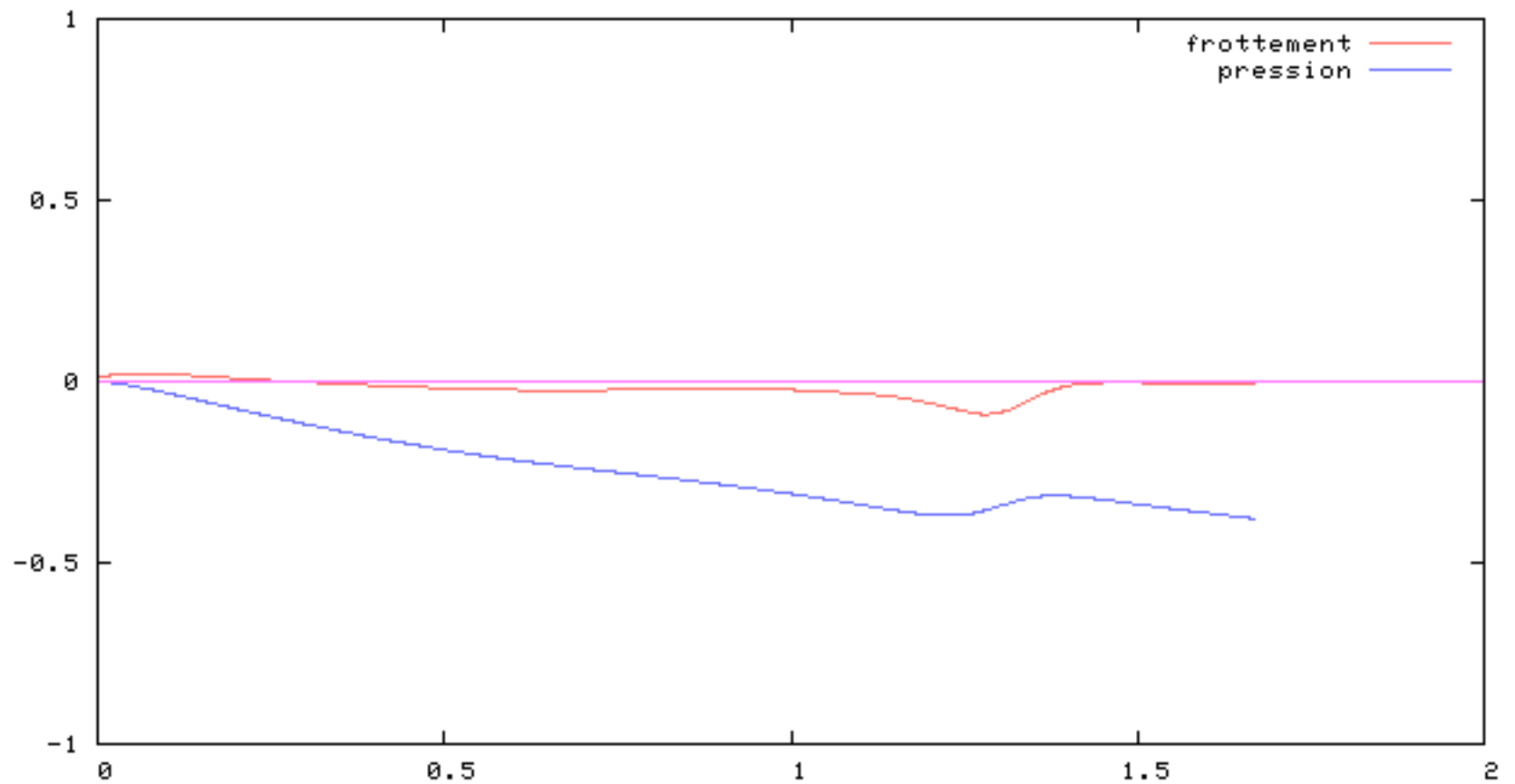
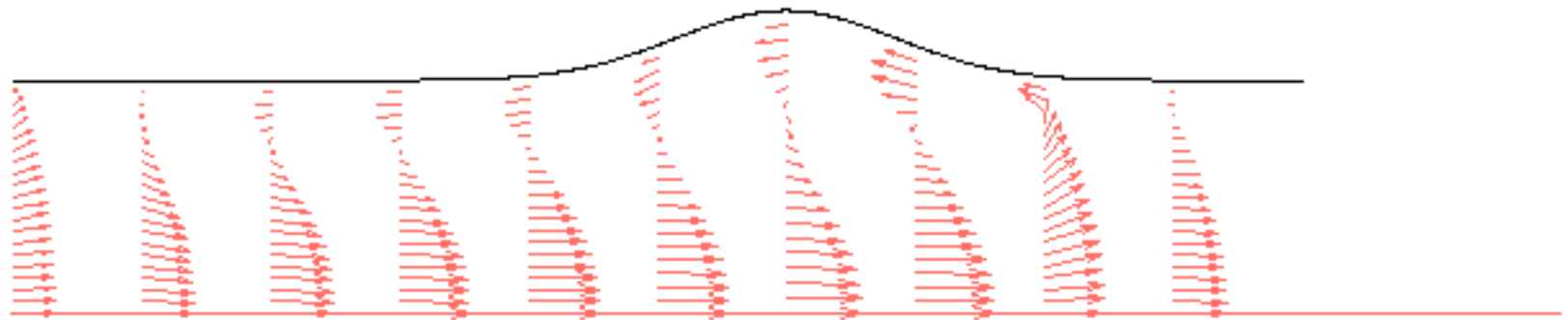
- Stenosis



- Stenosis



- Aneurism



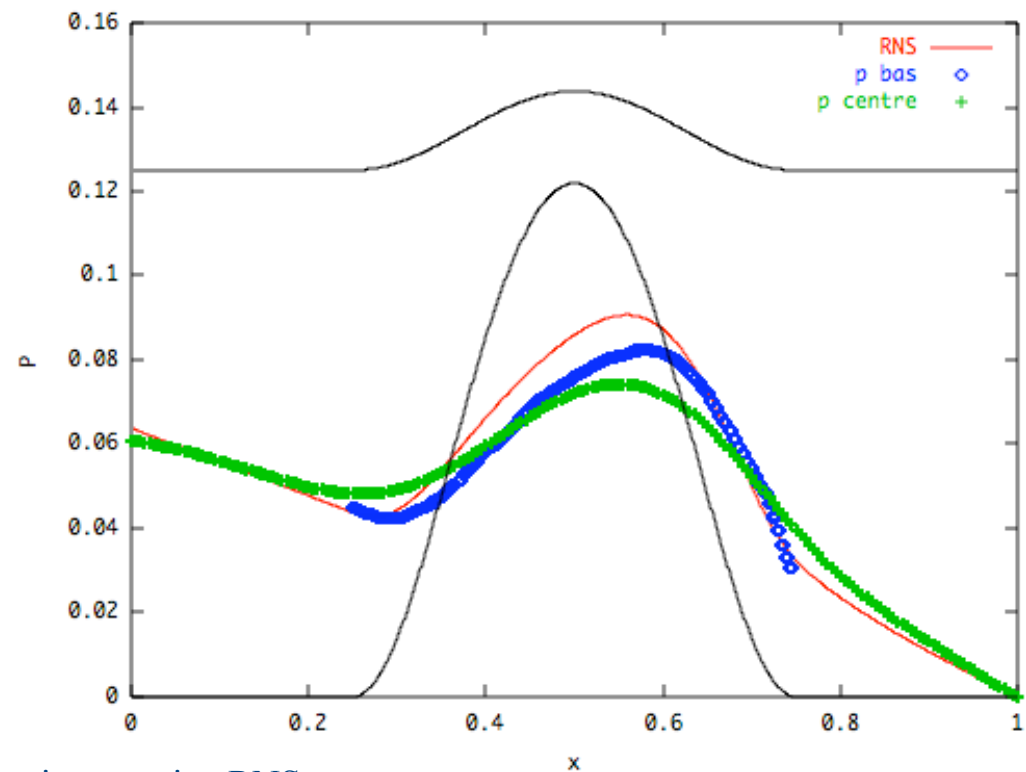
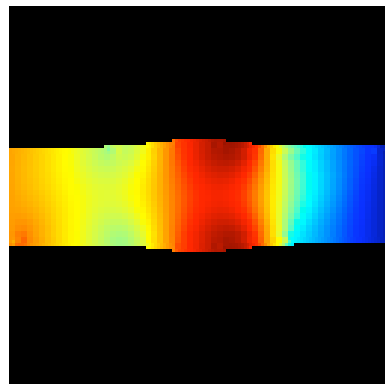


- Aneurism



pressure distribution

Steady 2D



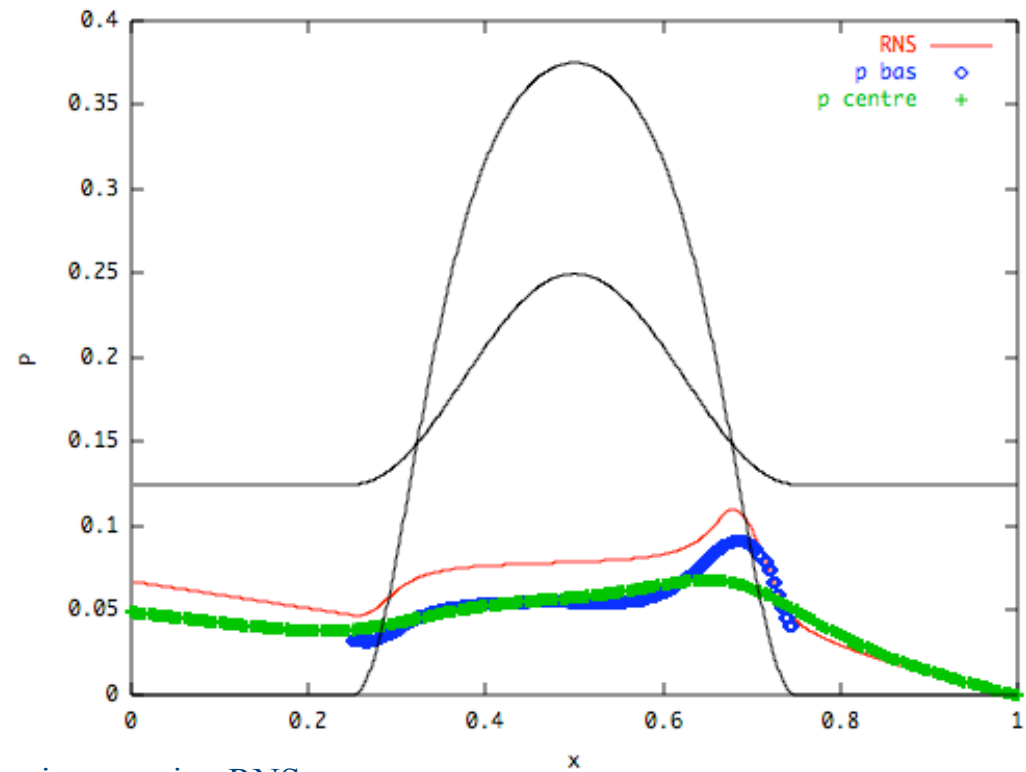
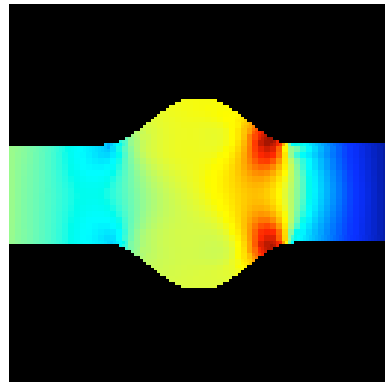
Comparaison gerris - RNS

- Aneurism



pressure distribution

Steady 2D



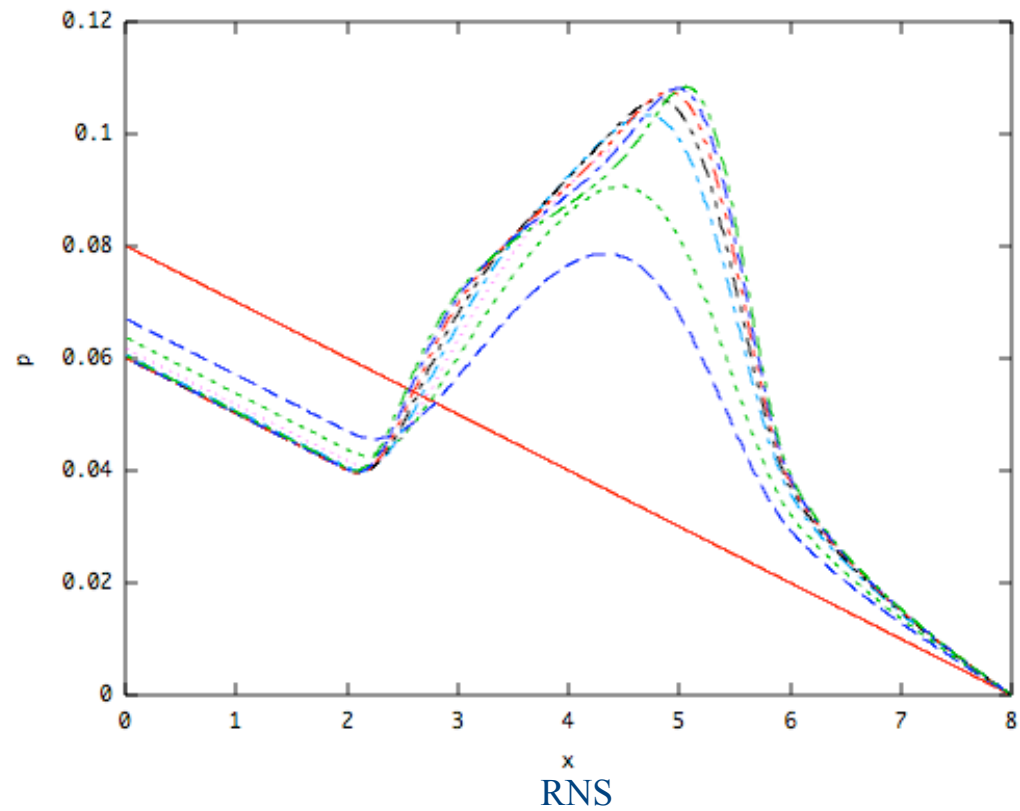
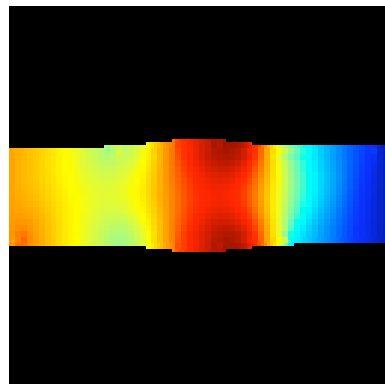
Comparaison gerris - RNS

- Aneurism



pressure distribution

Steady 2D

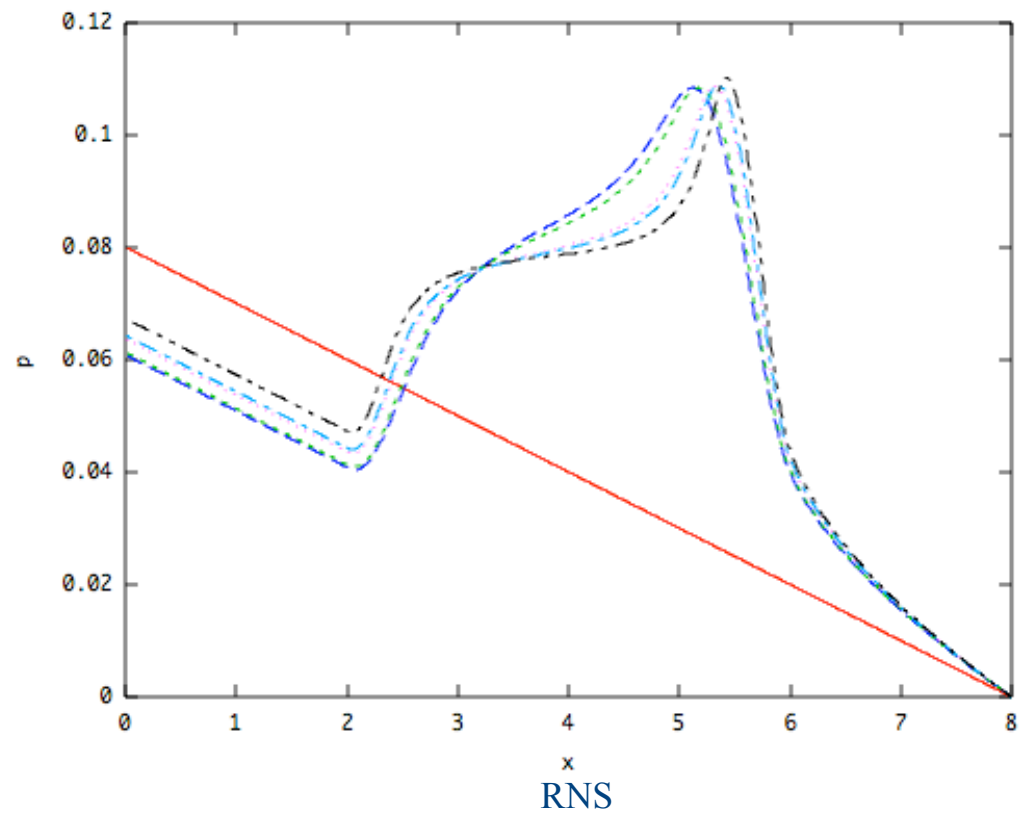
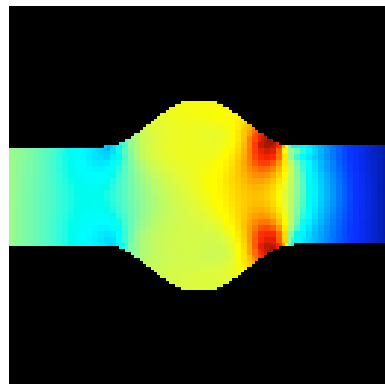


- Aneurism



pressure distribution

Steady 2D

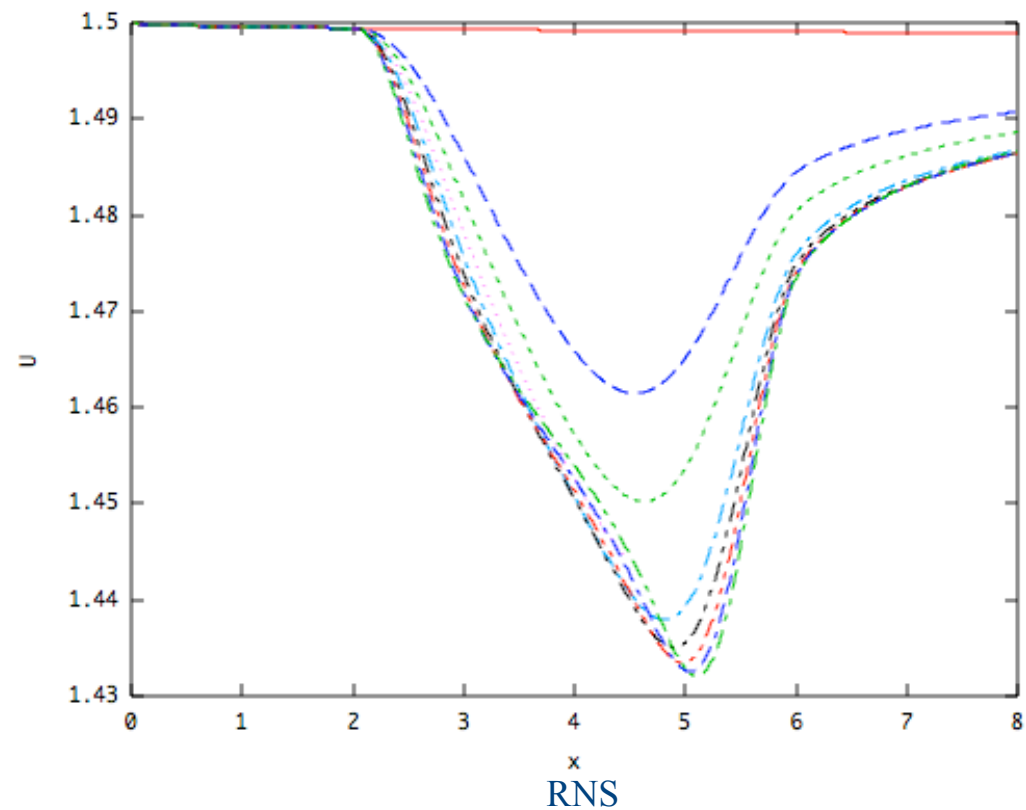
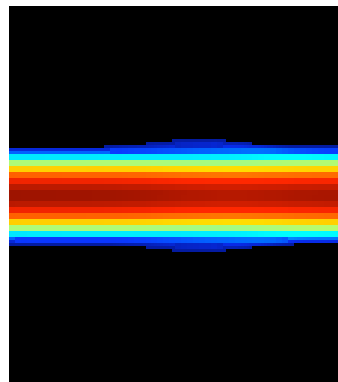


- Aneurism



### velocity distribution

Steady 2D

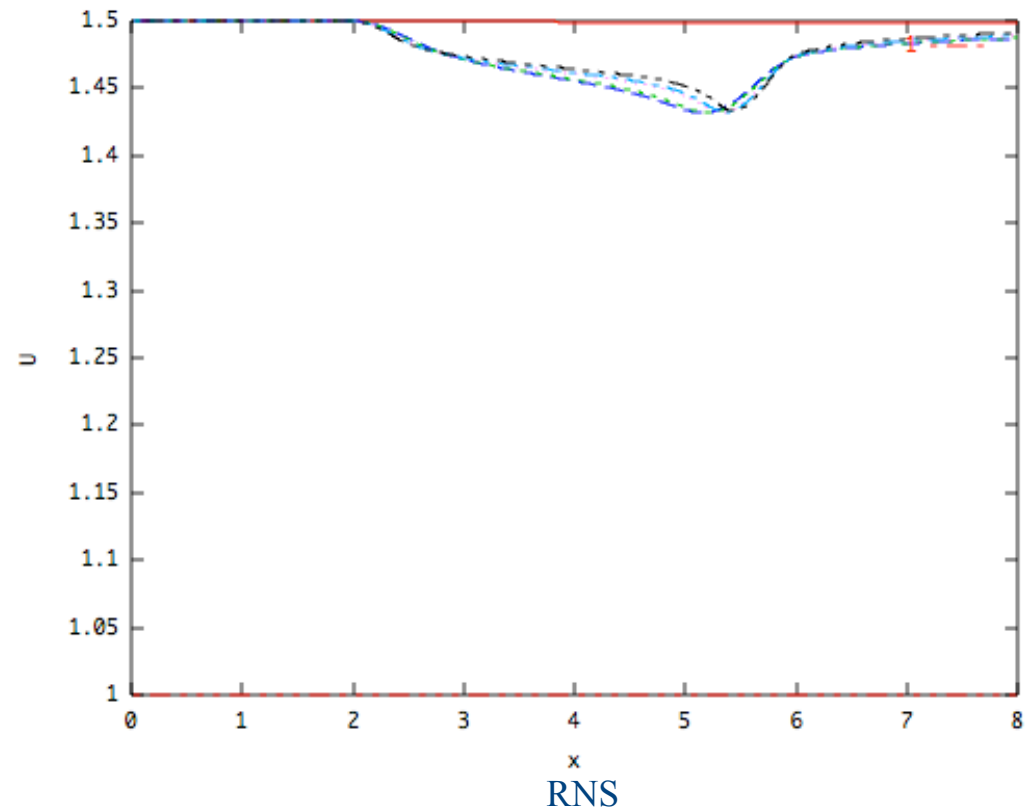
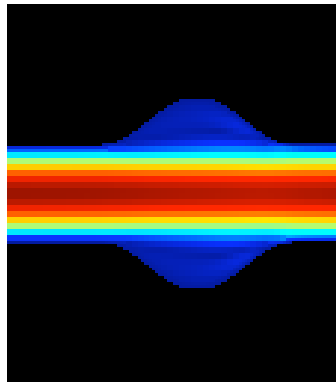


- Aneurism



## velocity distribution

Steady 2D

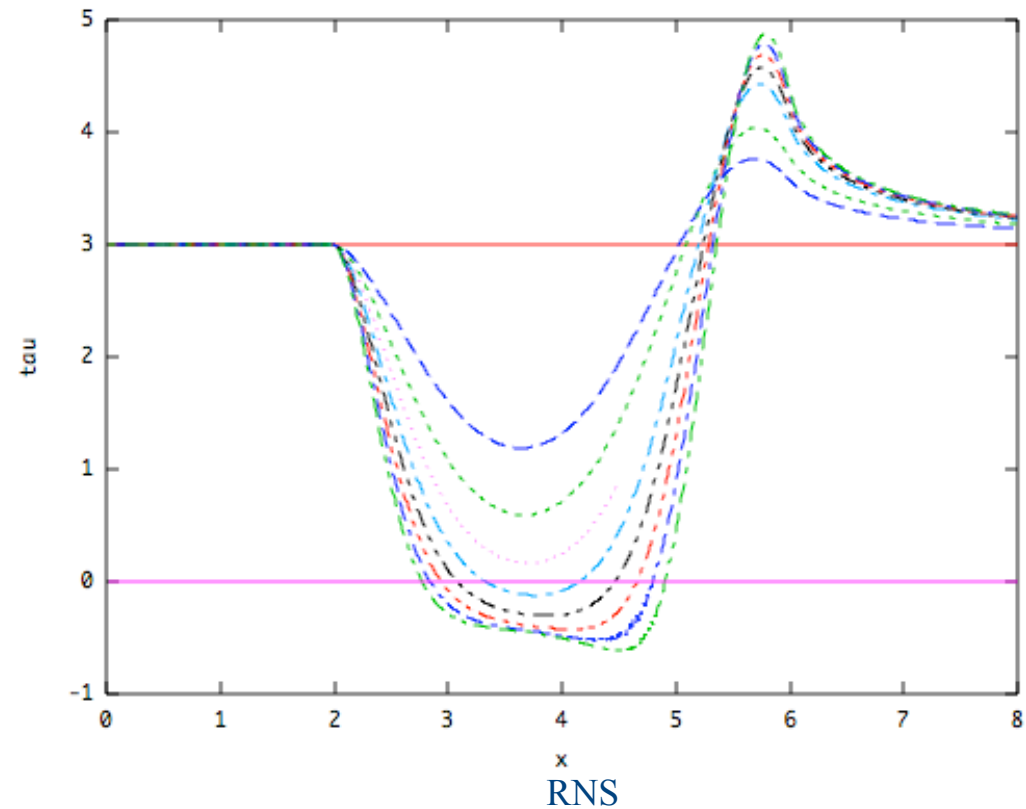


- Aneurism



shear stress distribution

Steady 2D

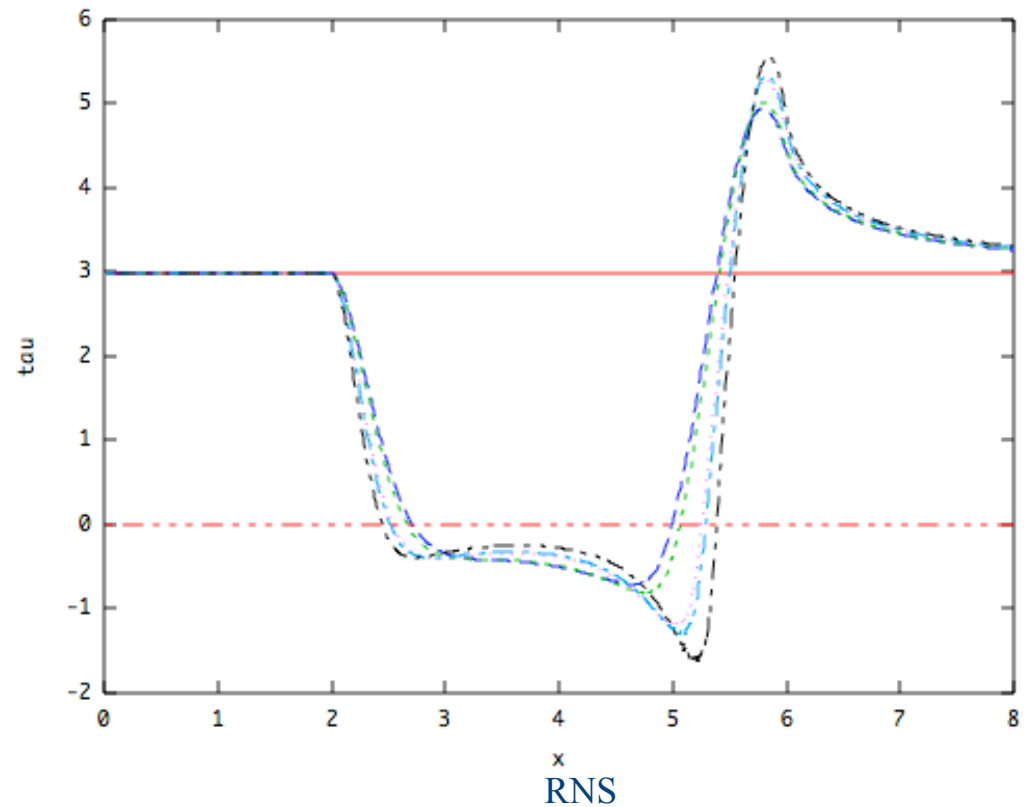
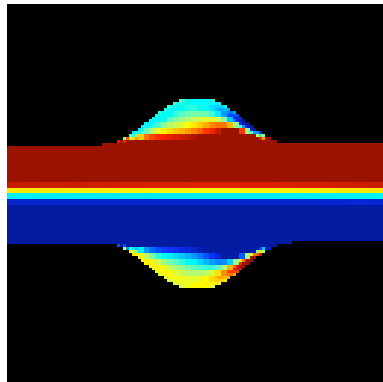


- Aneurism



### shear stress distribution

Steady 2D



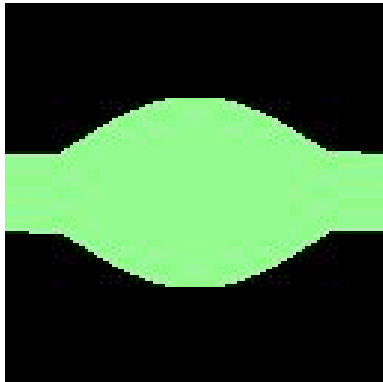


- Aneurism

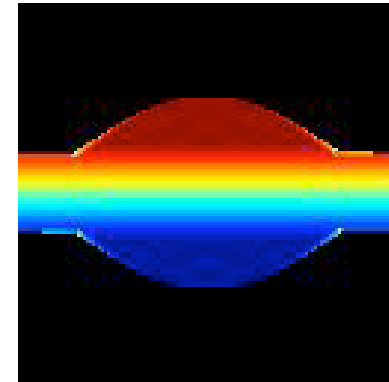


unsteady

but still rigid



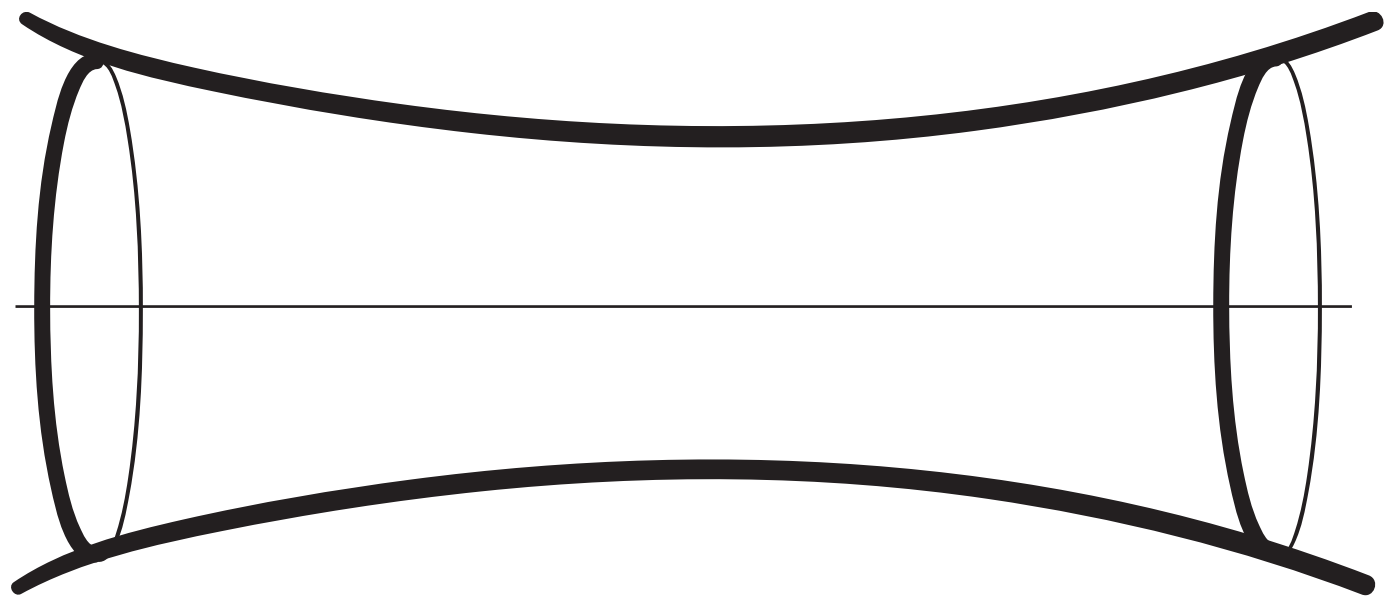
gerris imposed flux

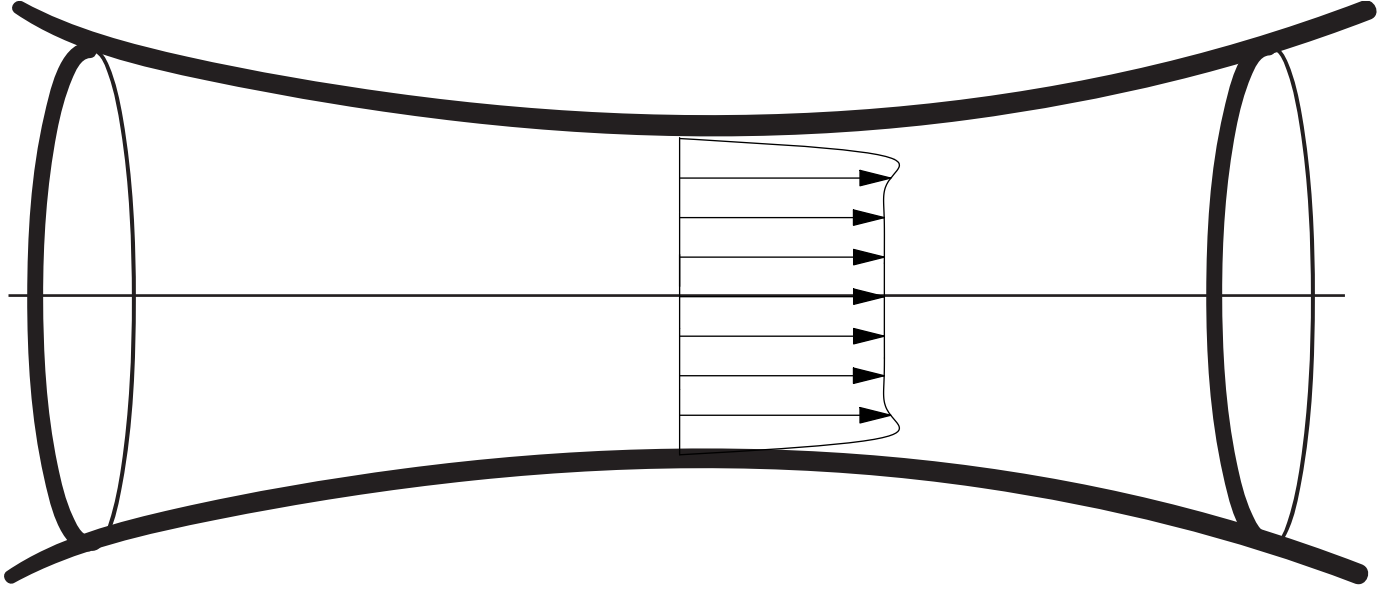


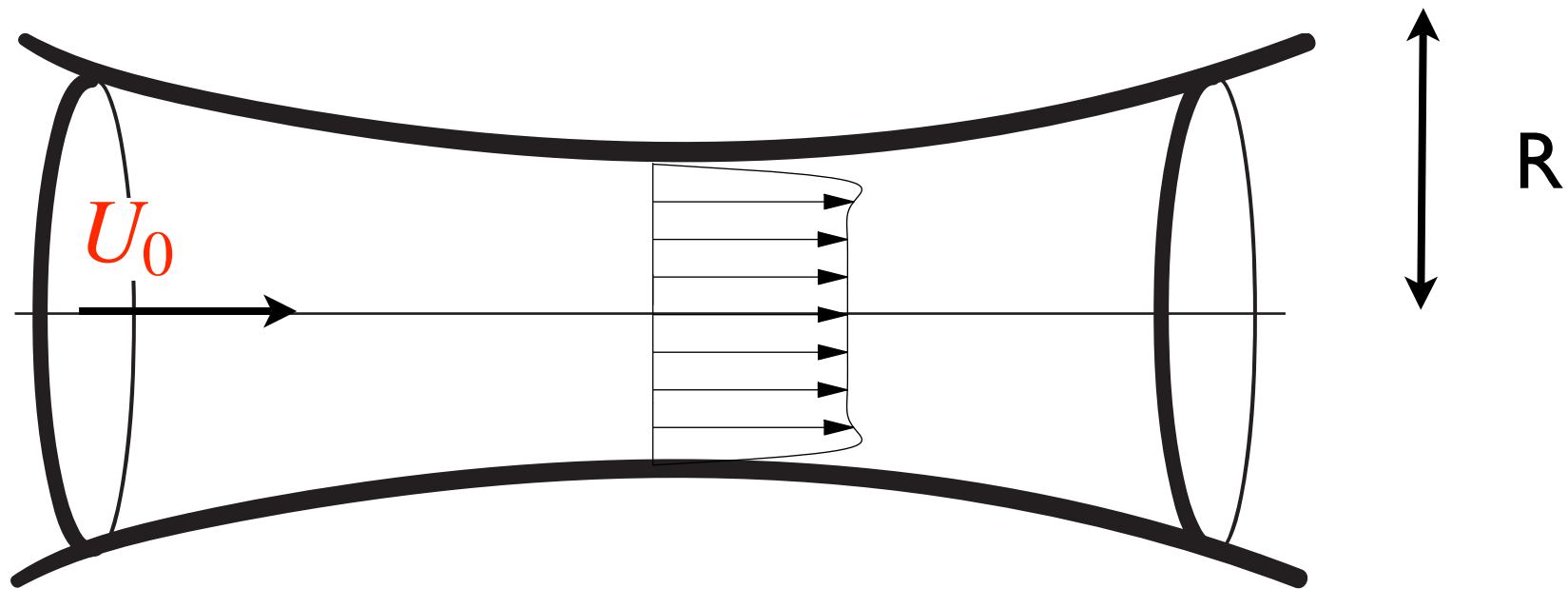
gerris imposed pressure





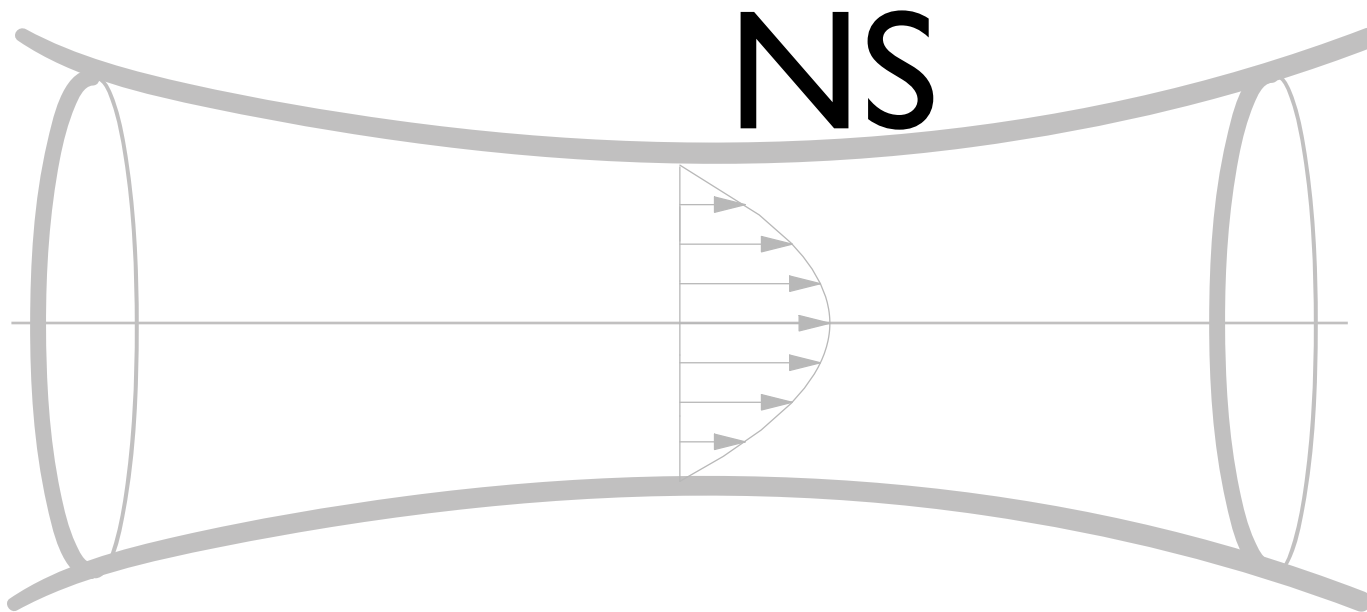






$\lambda$

$$R \ll \lambda$$



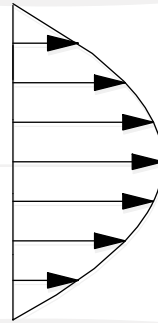
$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + \nu \frac{\partial^2}{\partial x^2} u + \nu \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r}$$

$$\frac{\partial v}{\partial t} + u \frac{\partial}{\partial x} v + v \frac{\partial}{\partial r} v = -\frac{\partial p}{\rho \partial r} + \nu \frac{\partial^2}{\partial x^2} v + \nu \frac{\partial}{r \partial r} r \frac{\partial v}{\partial r}$$

# RNS/P

Prandtl

$$\frac{\partial u}{\partial x} + \frac{\partial rv}{r\partial r} = 0$$

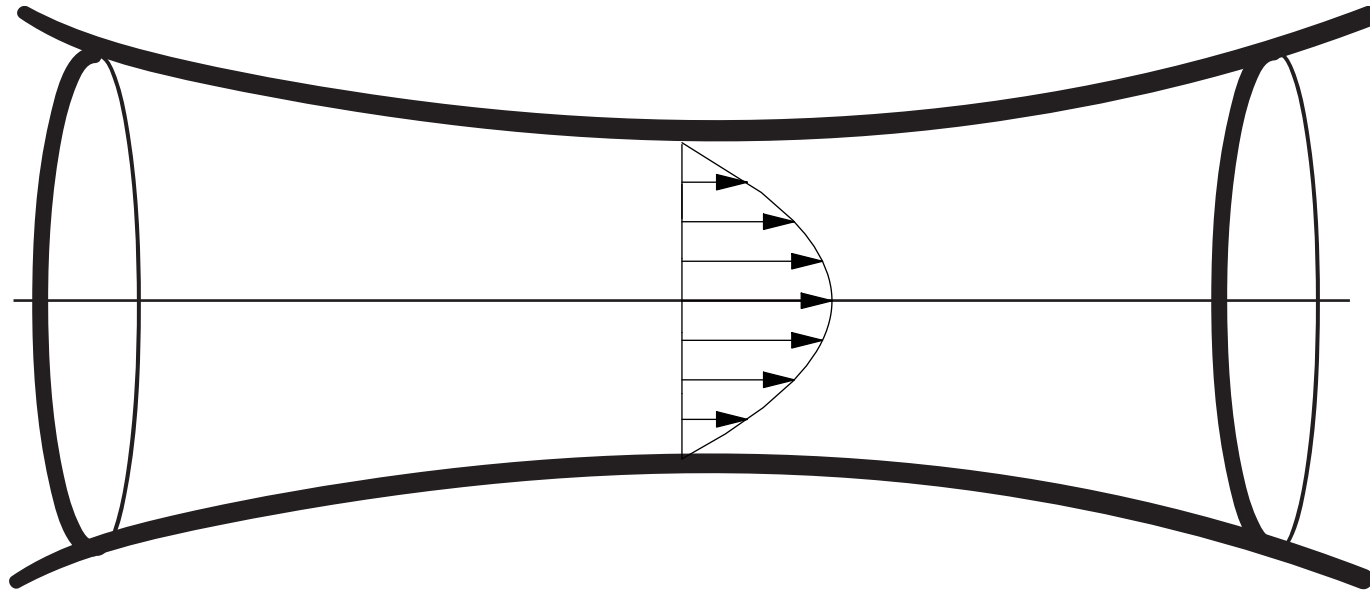


$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + \nu \frac{\partial}{\partial r} r \frac{\partial u}{\partial r}$$

$$0 = -\frac{\partial p}{\rho \partial r}$$

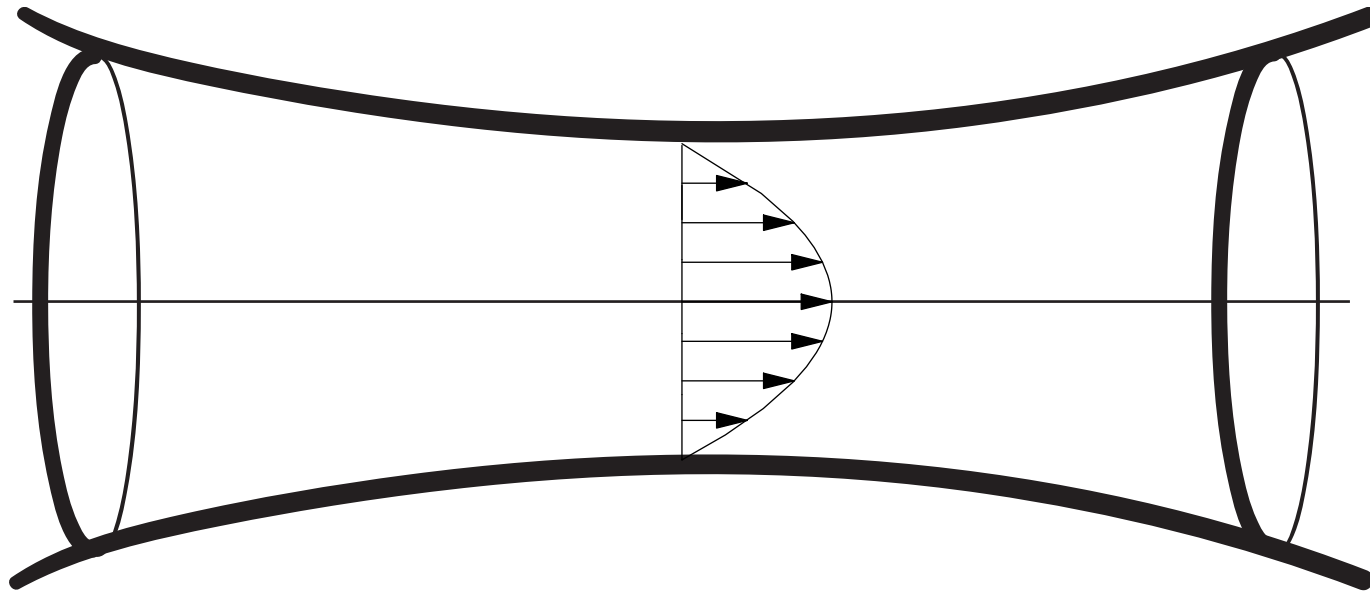


# Boundary conditions



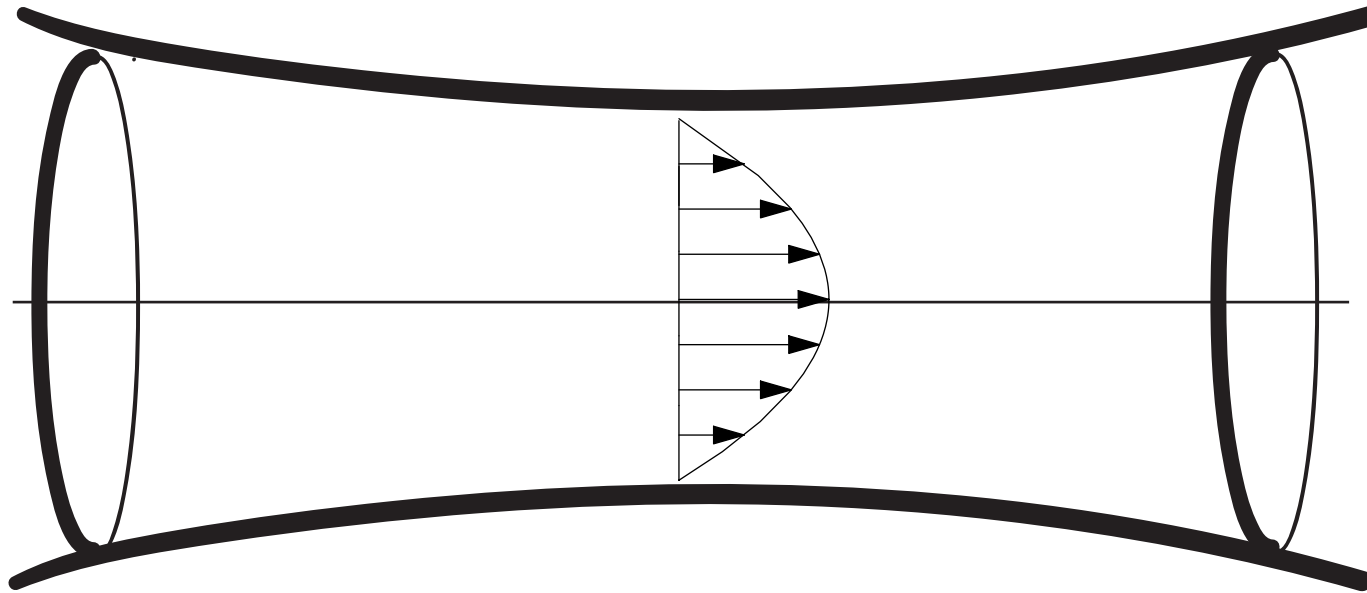
Rigid wall:  $u = v = 0$

# Boundary conditions



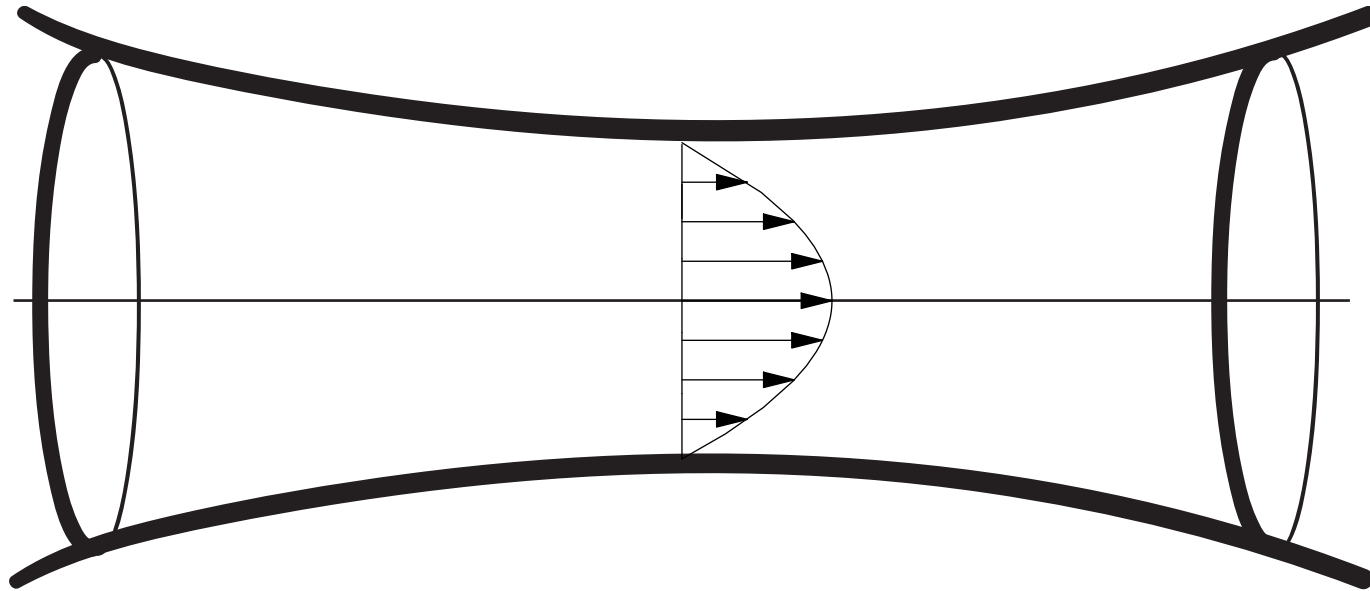
moving wall

# Boundary conditions



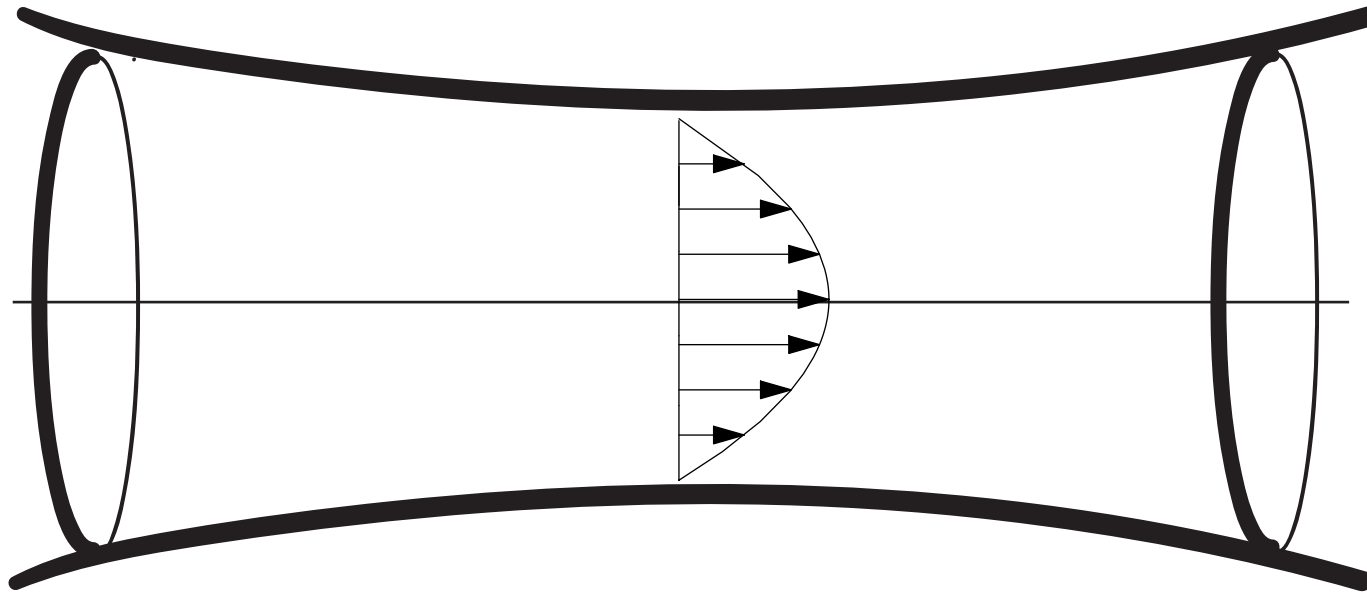
moving wall

# Boundary conditions



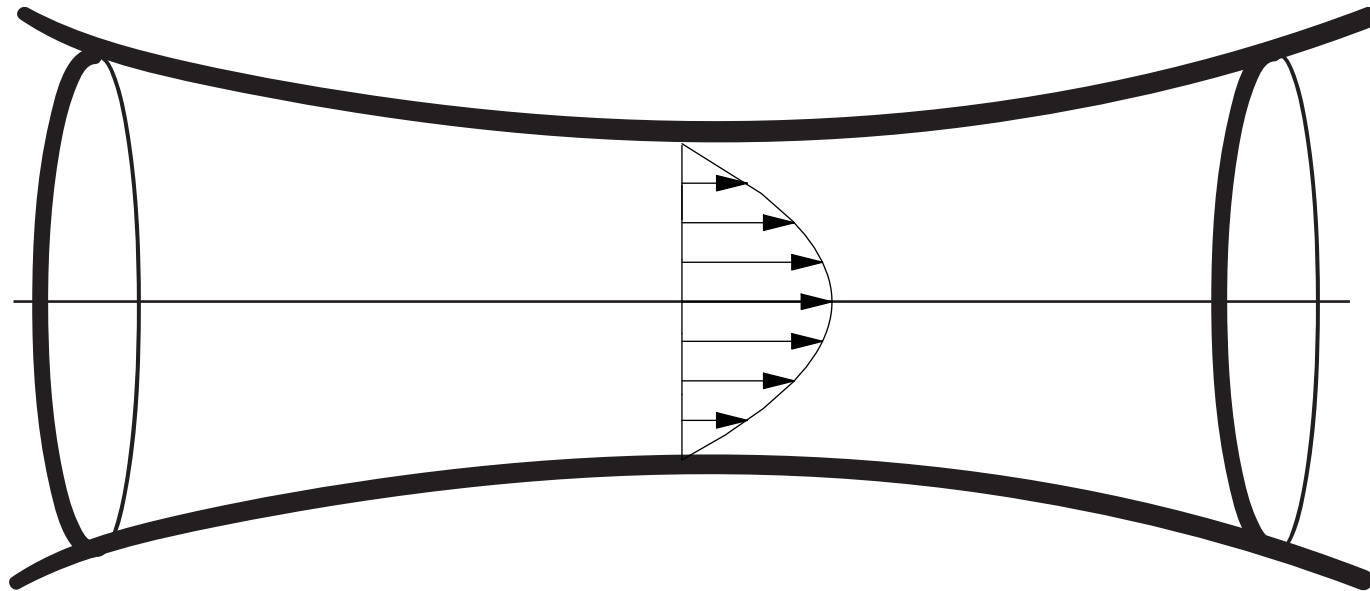
moving wall

# Boundary conditions



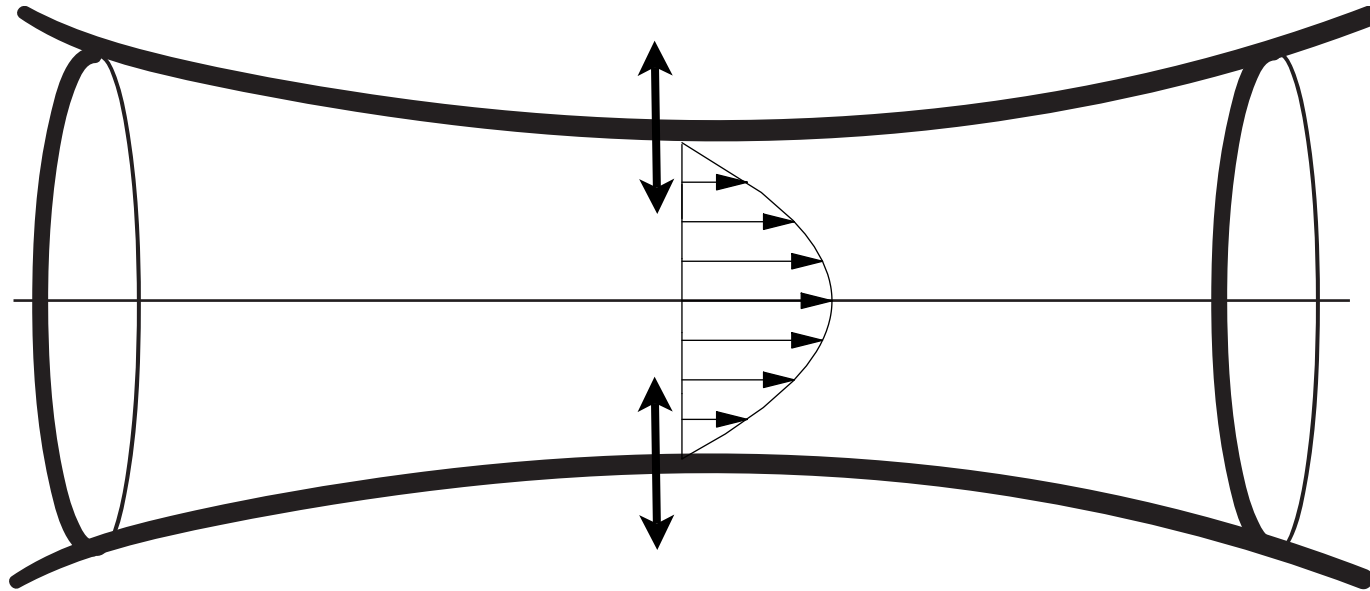
moving wall

# Boundary conditions



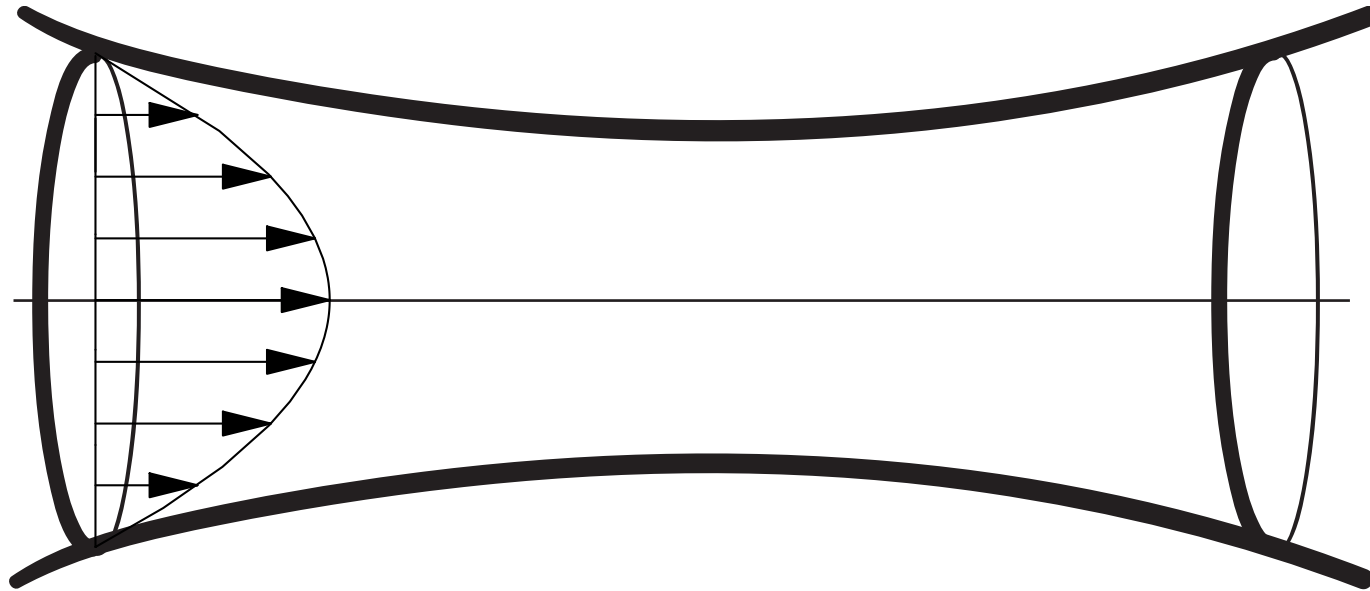
moving wall

# Boundary conditions



moving wall  $v = \frac{\partial R}{\partial t}$

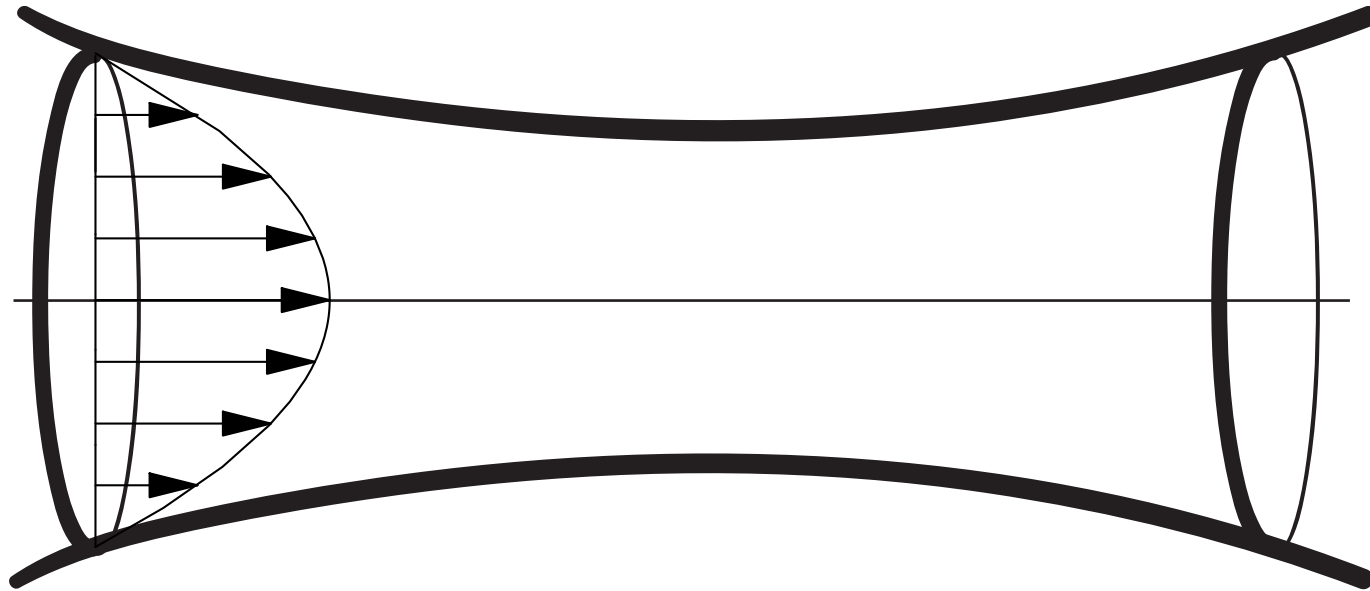
# Boundary conditions



First given profile:



# Boundary conditions

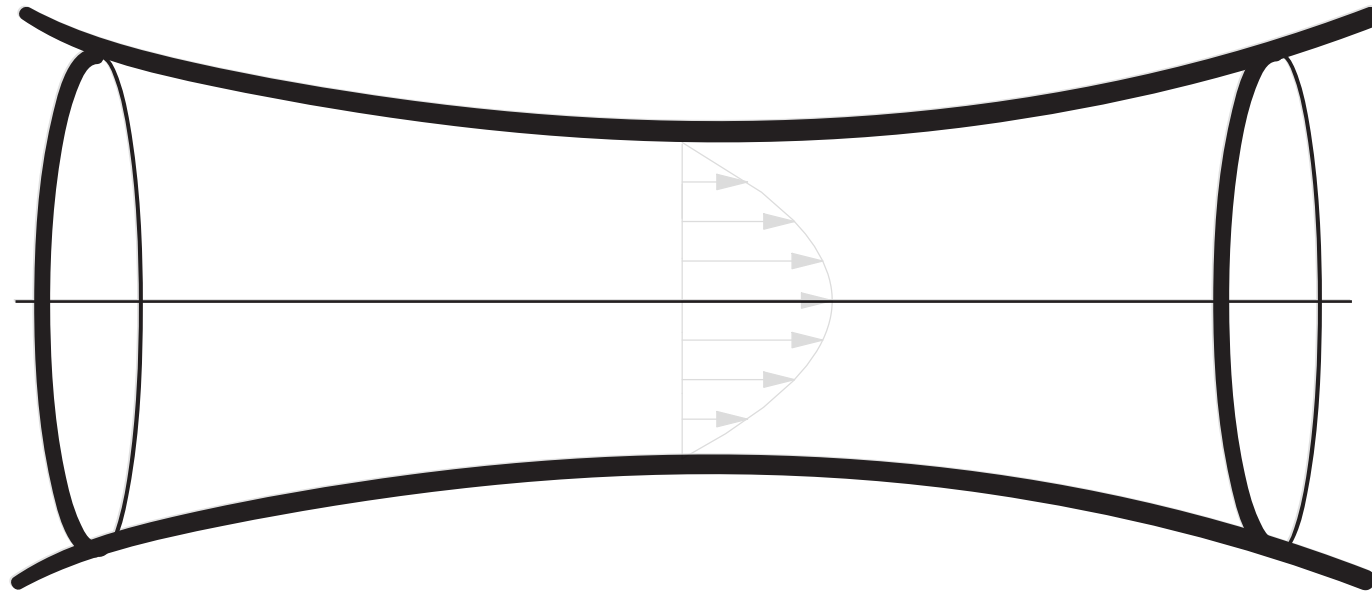


First given profile:

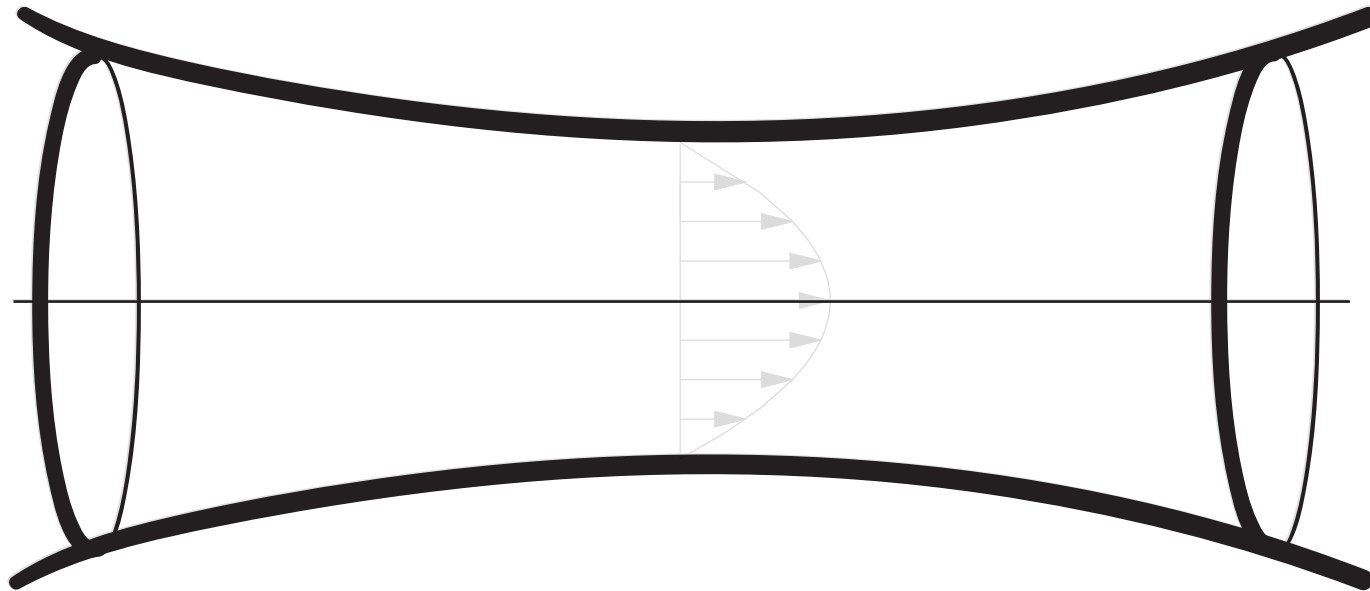
marching procedure



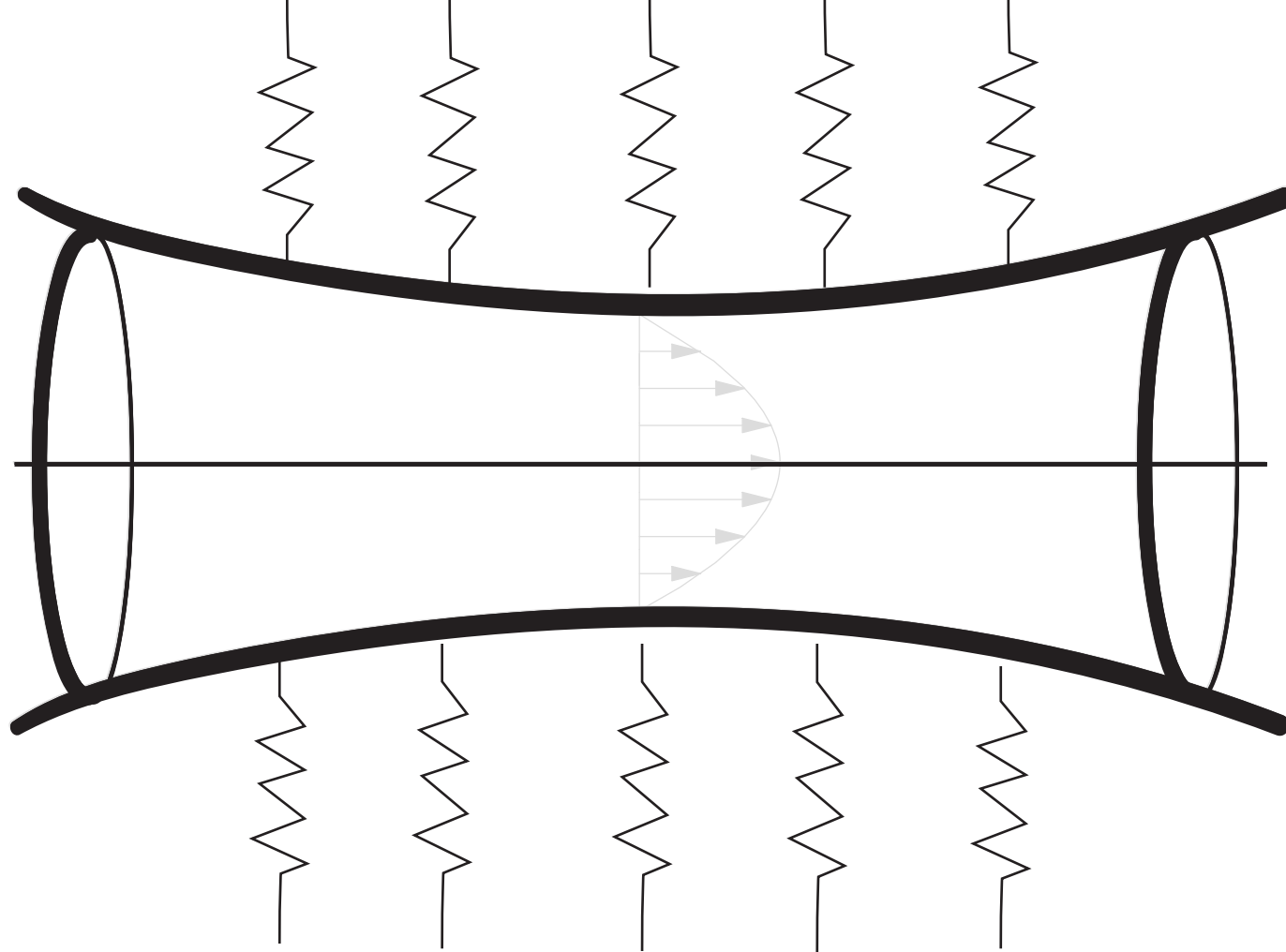
distribution of pressure is a result



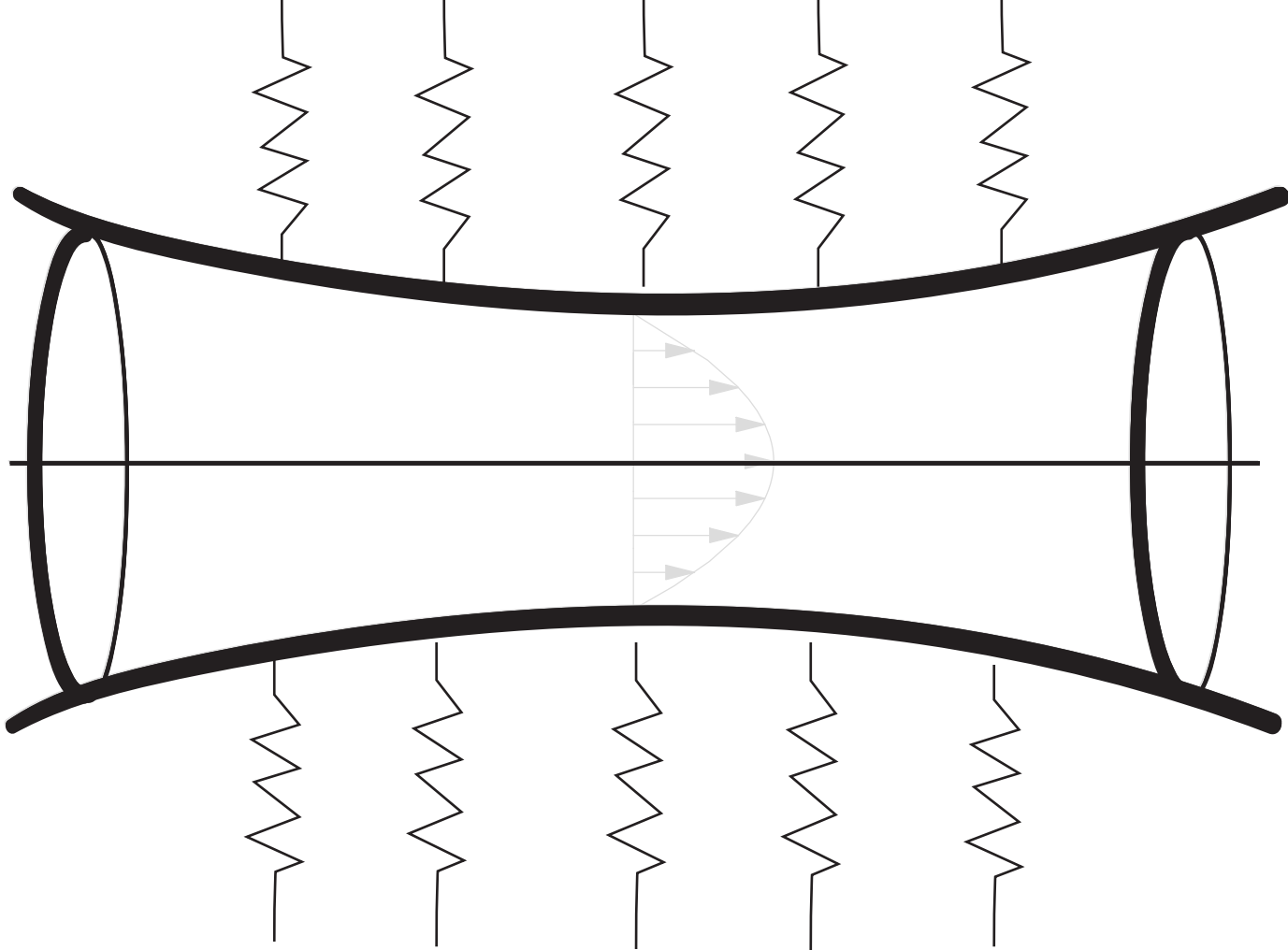
Up to now, the wall was rigid

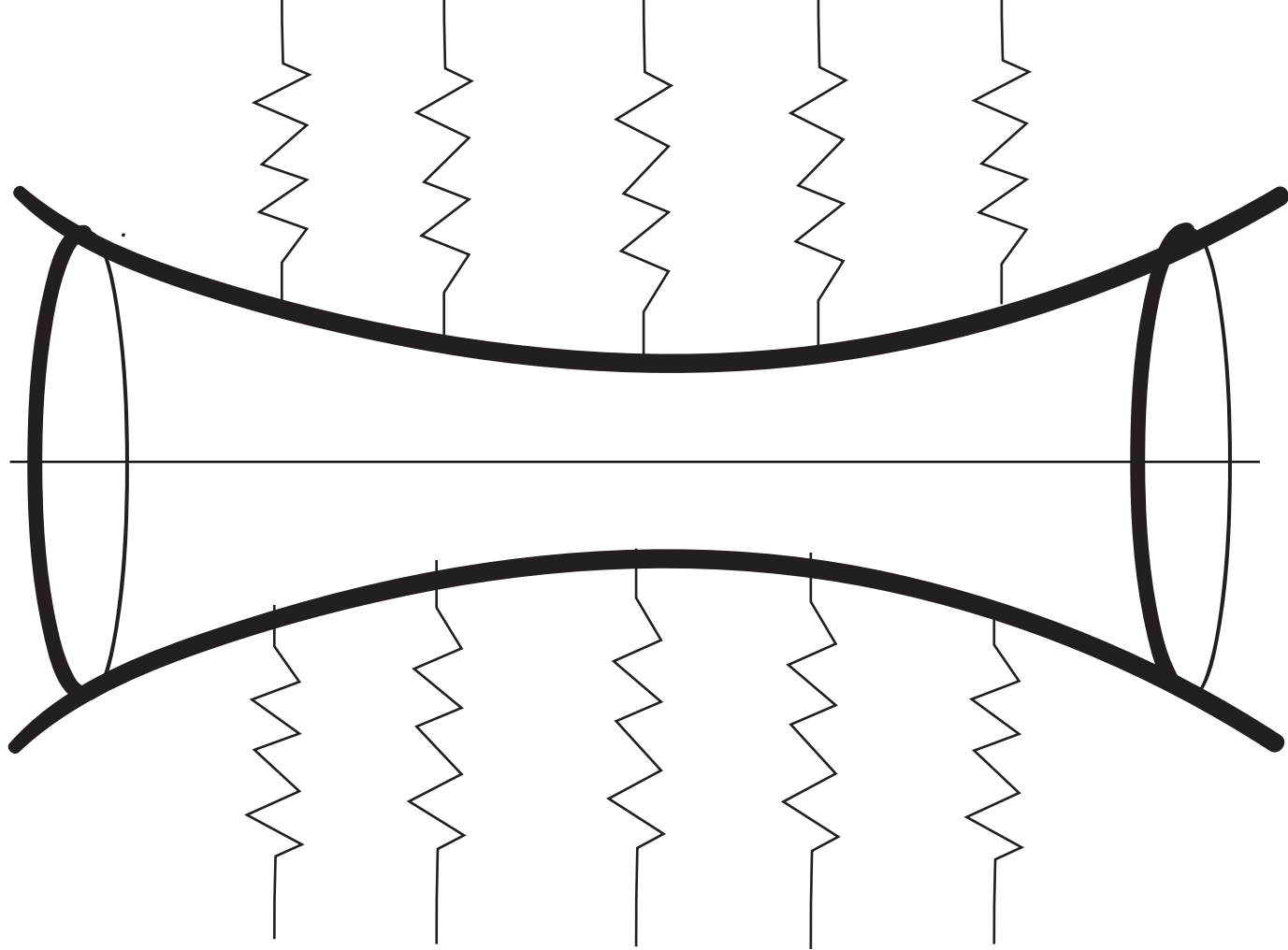


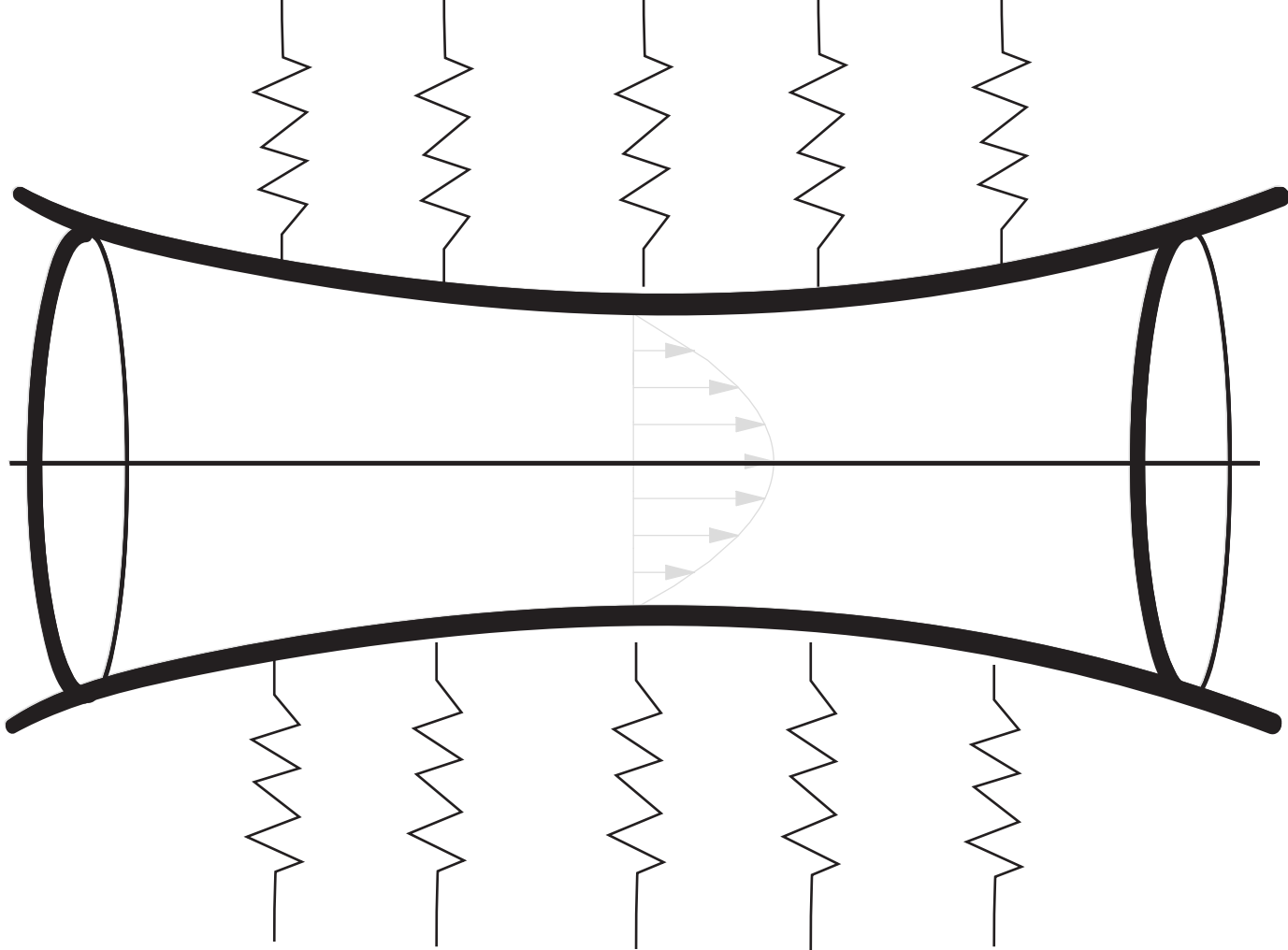
we use a simple elastic model

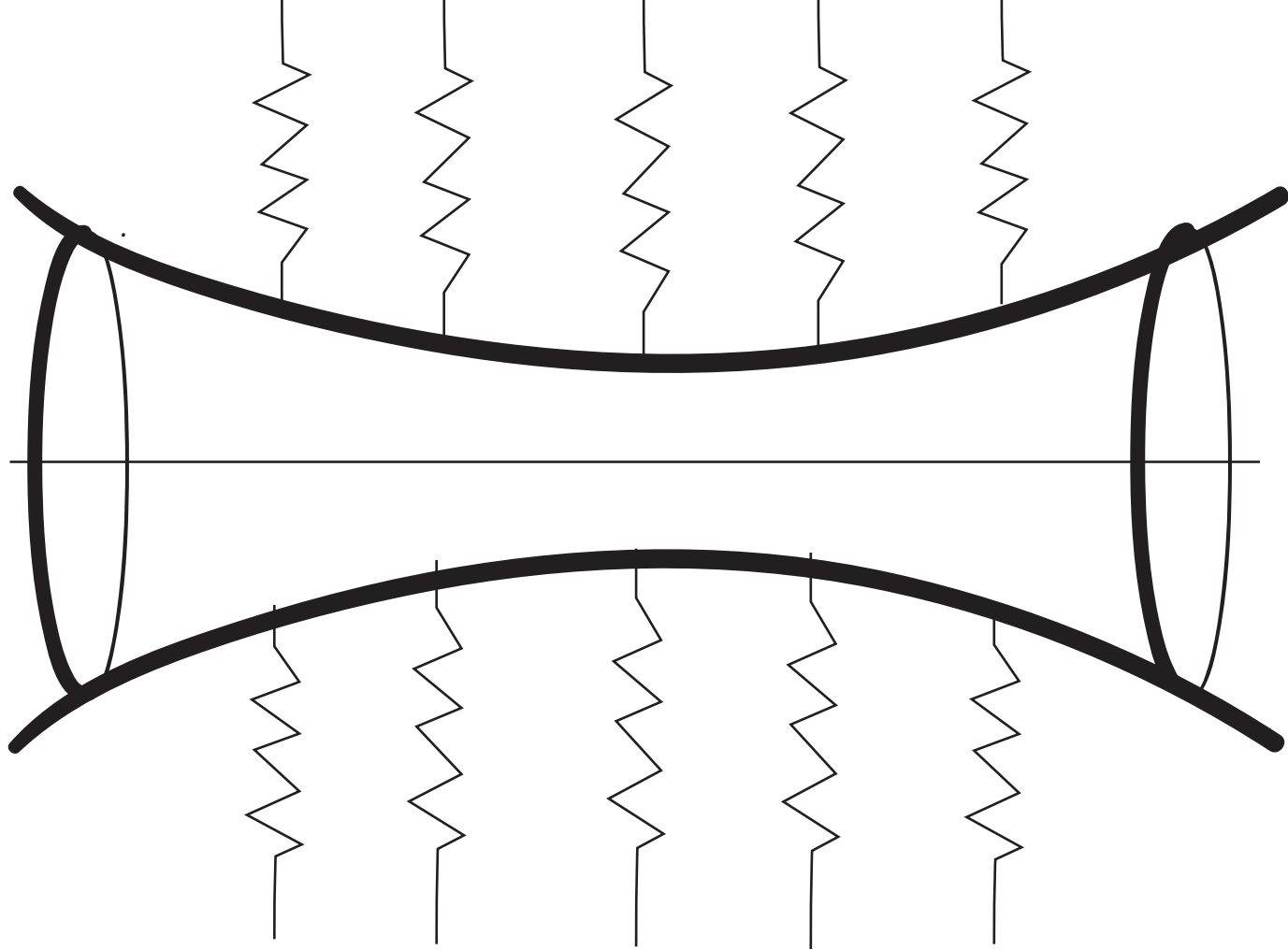


**we use a simple elastic model**

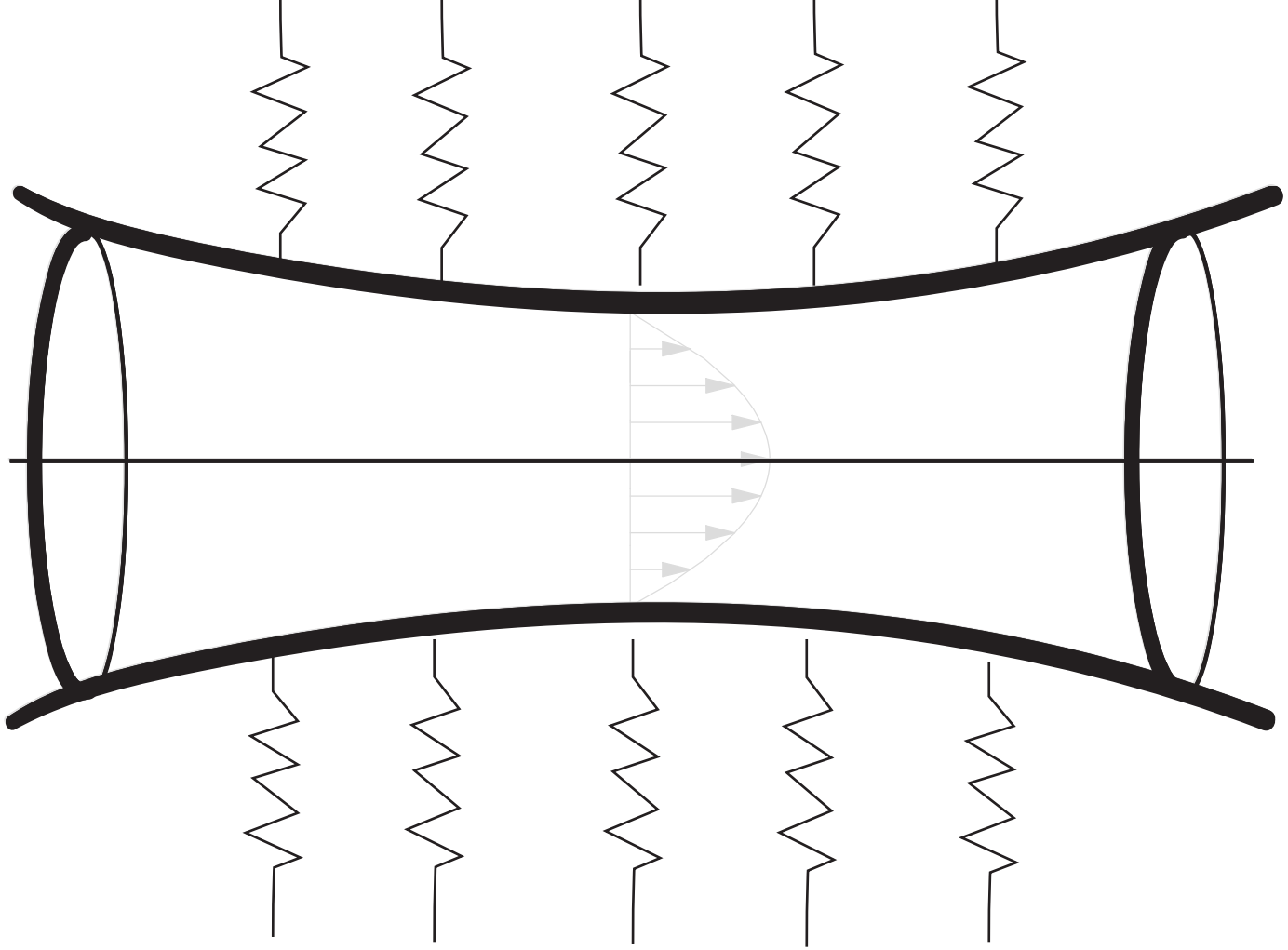




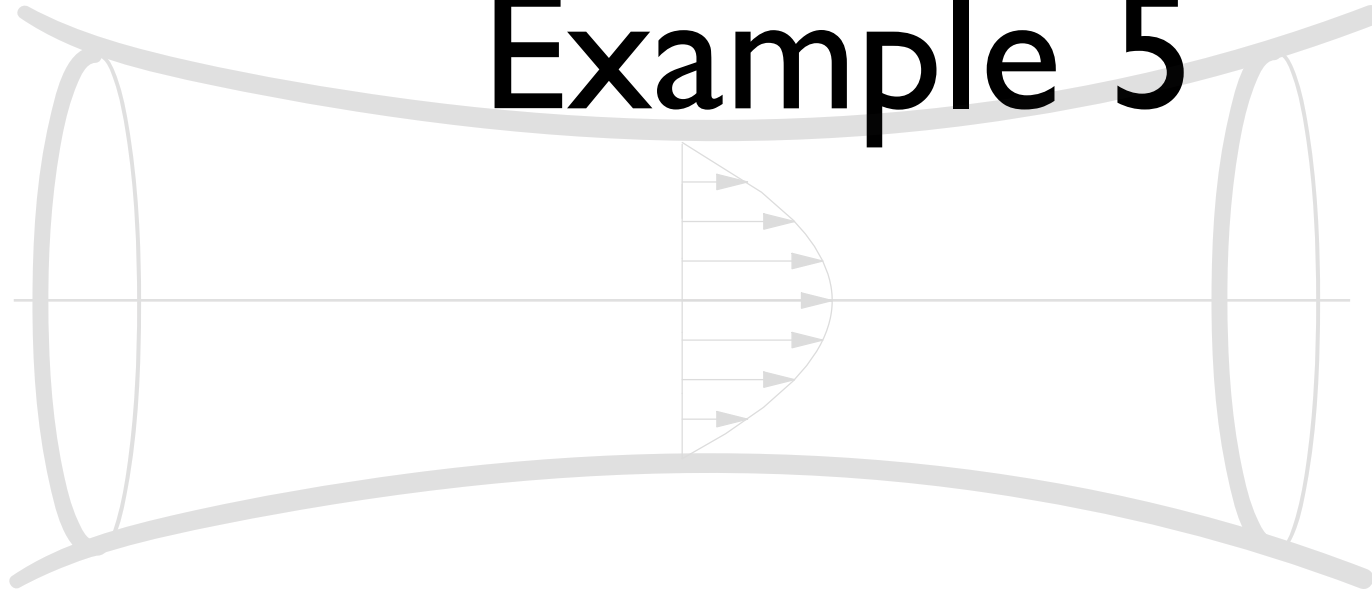




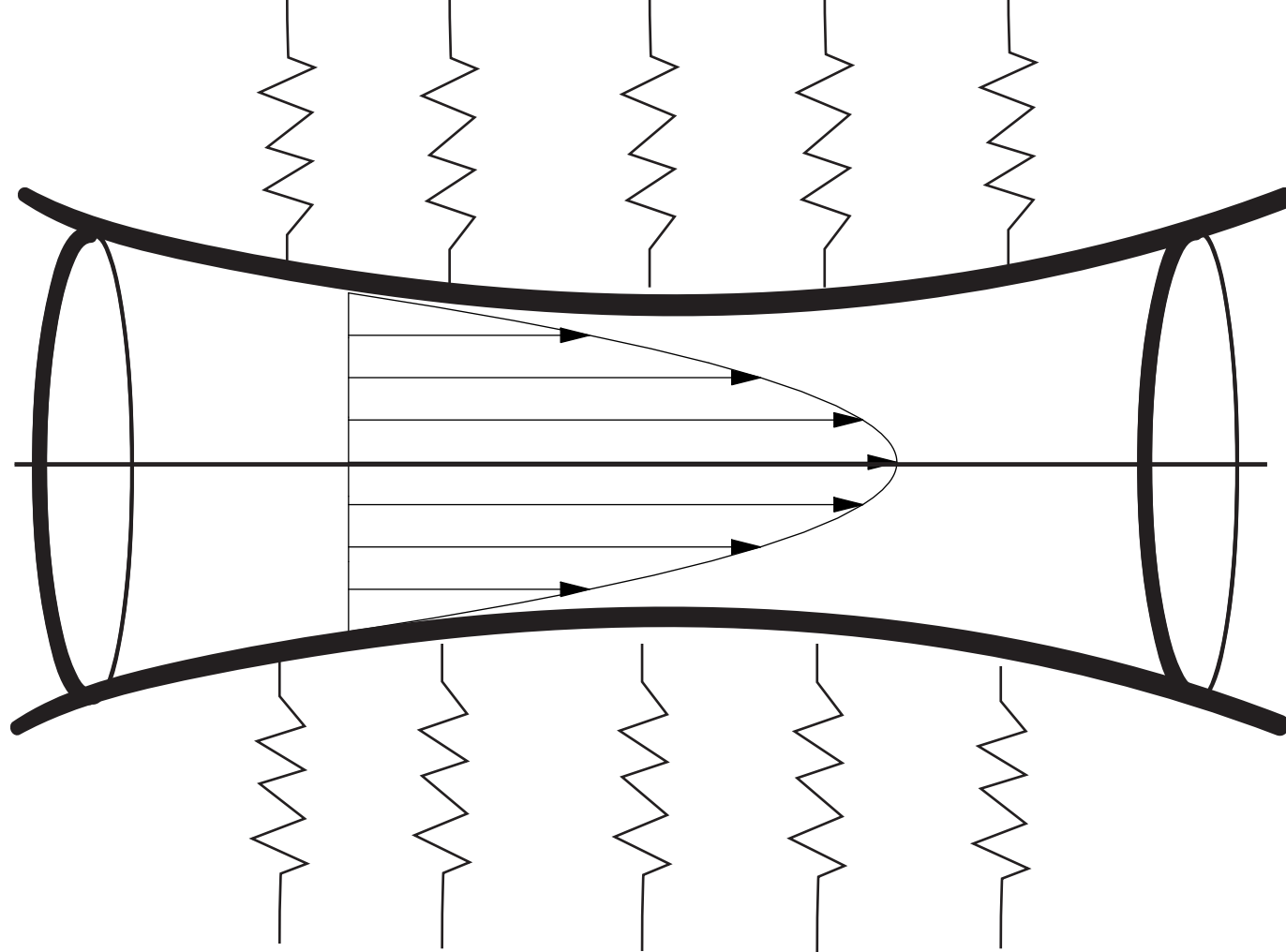




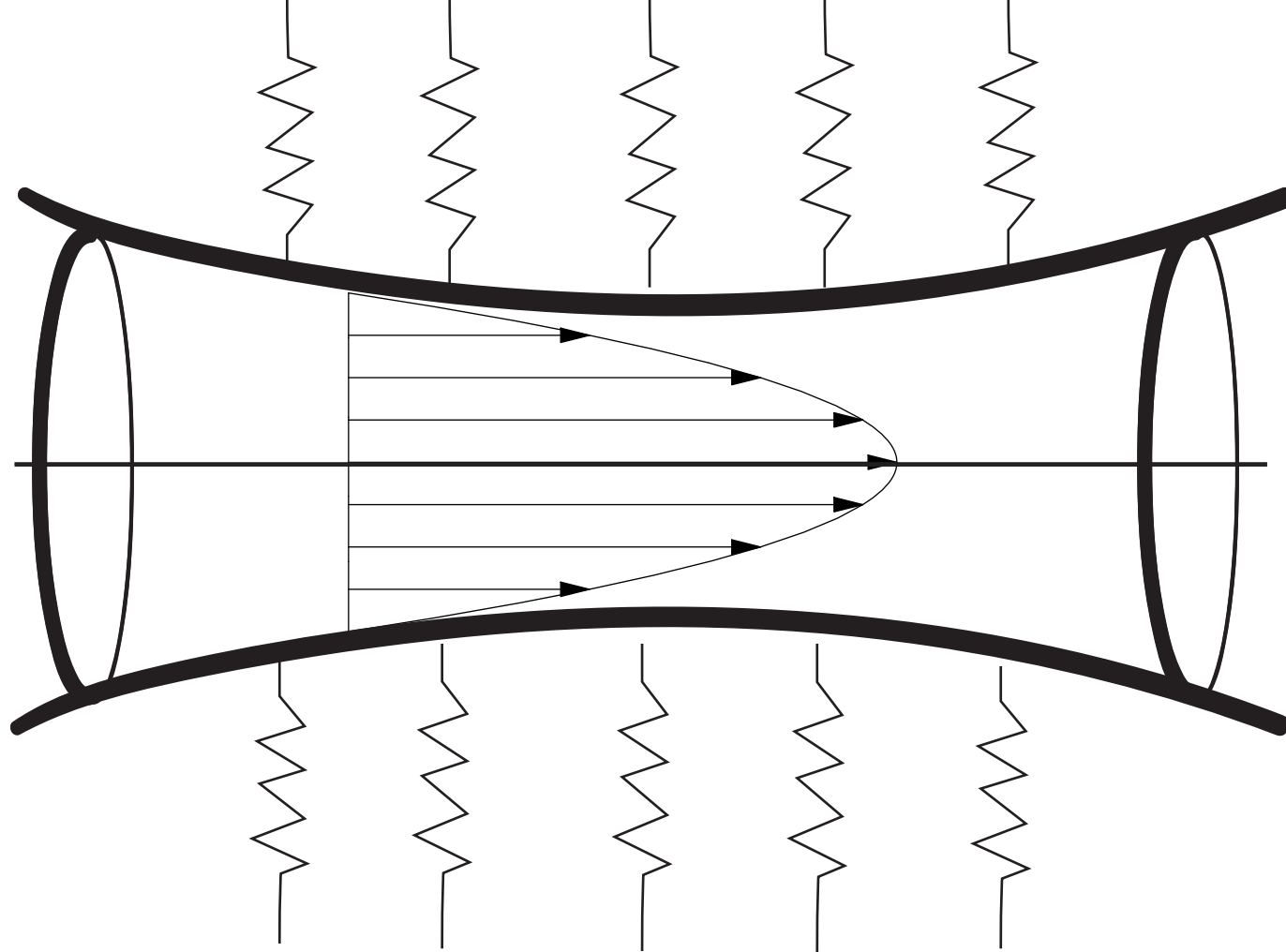
# Example 5



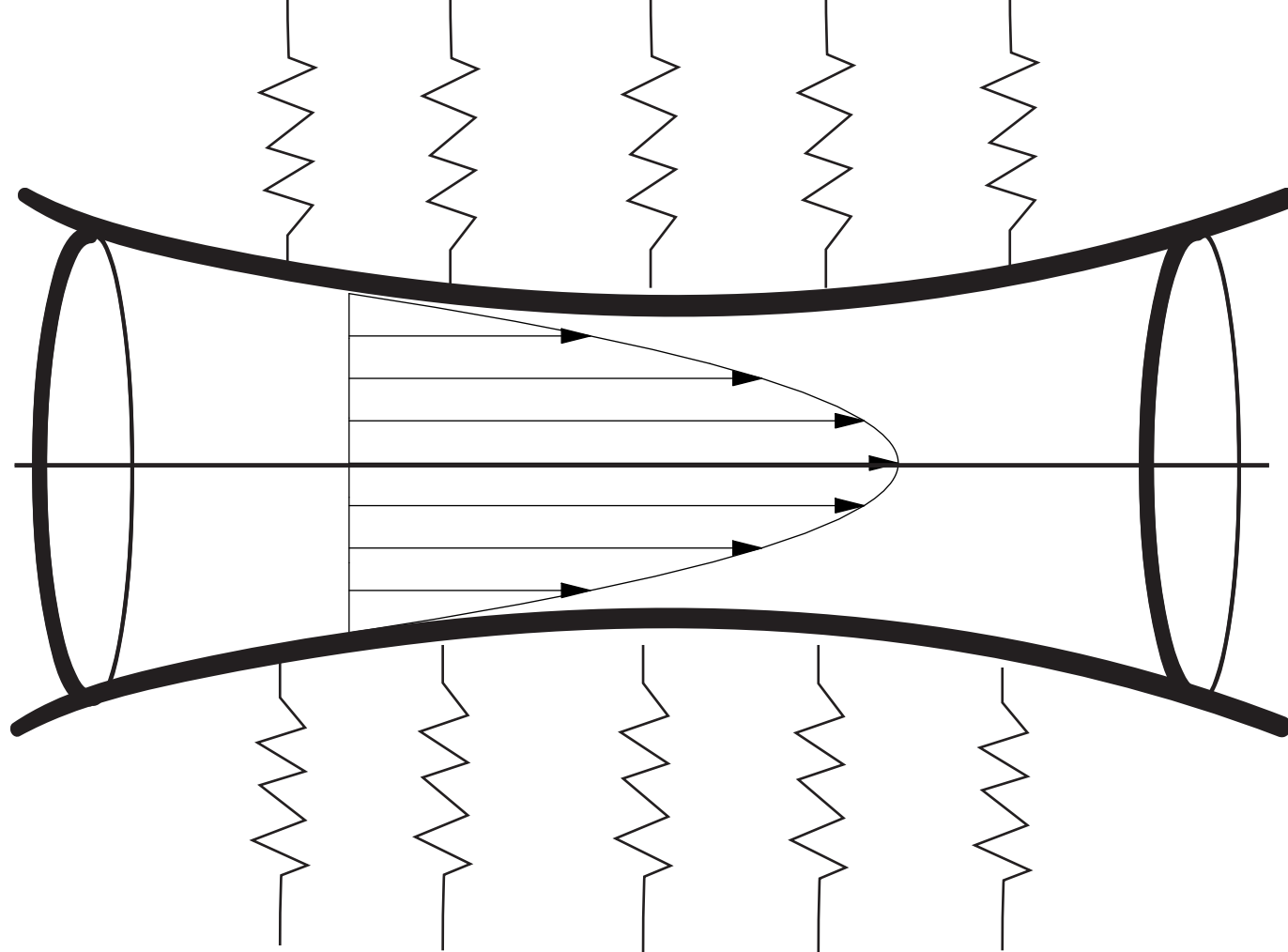
- Flow in a collapsible tube
- unsteady, elastic wall, no inertia



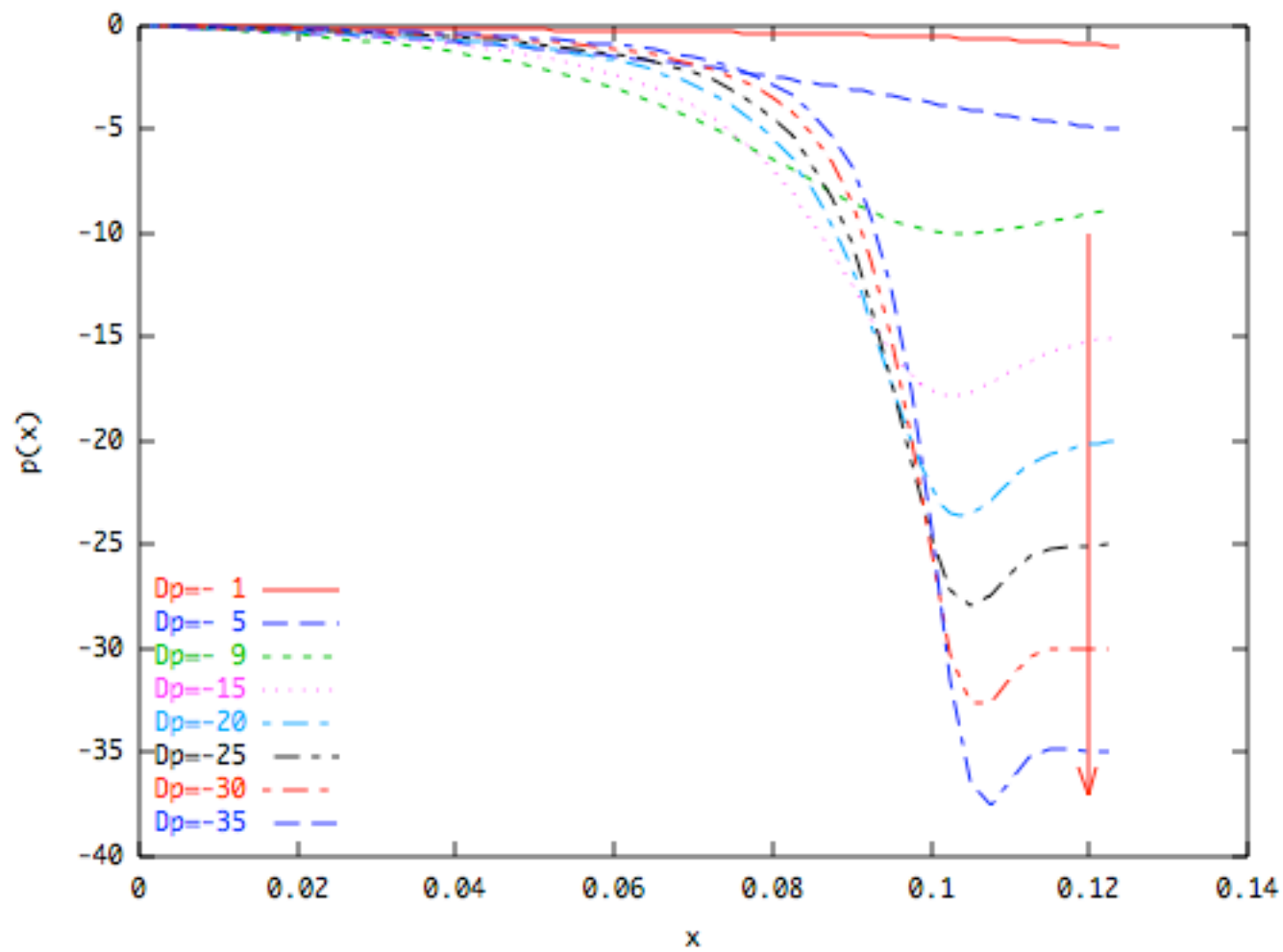
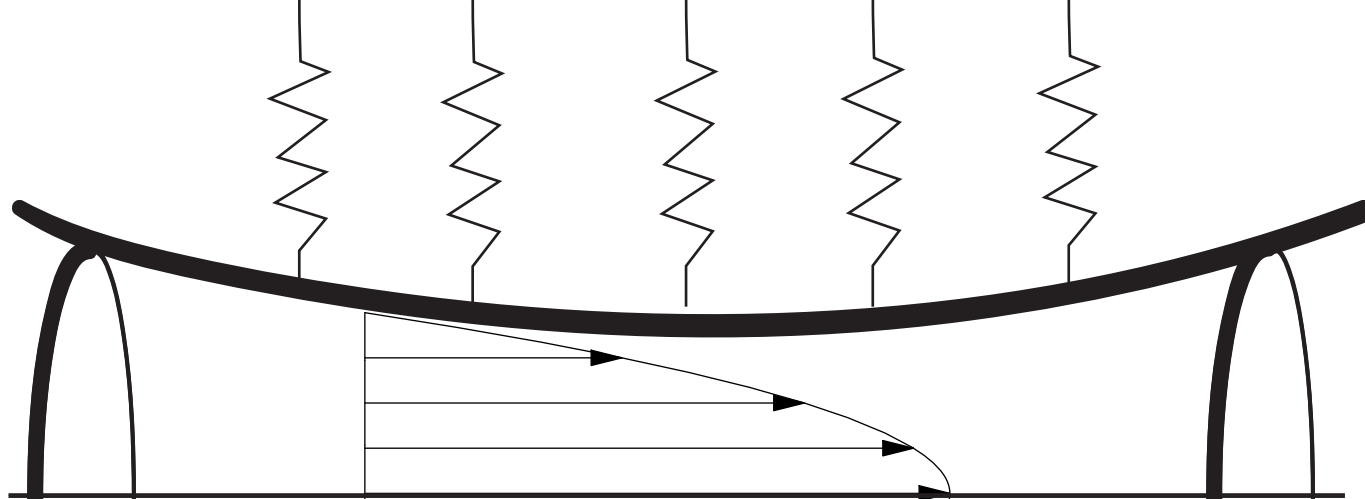
Collapsible tube



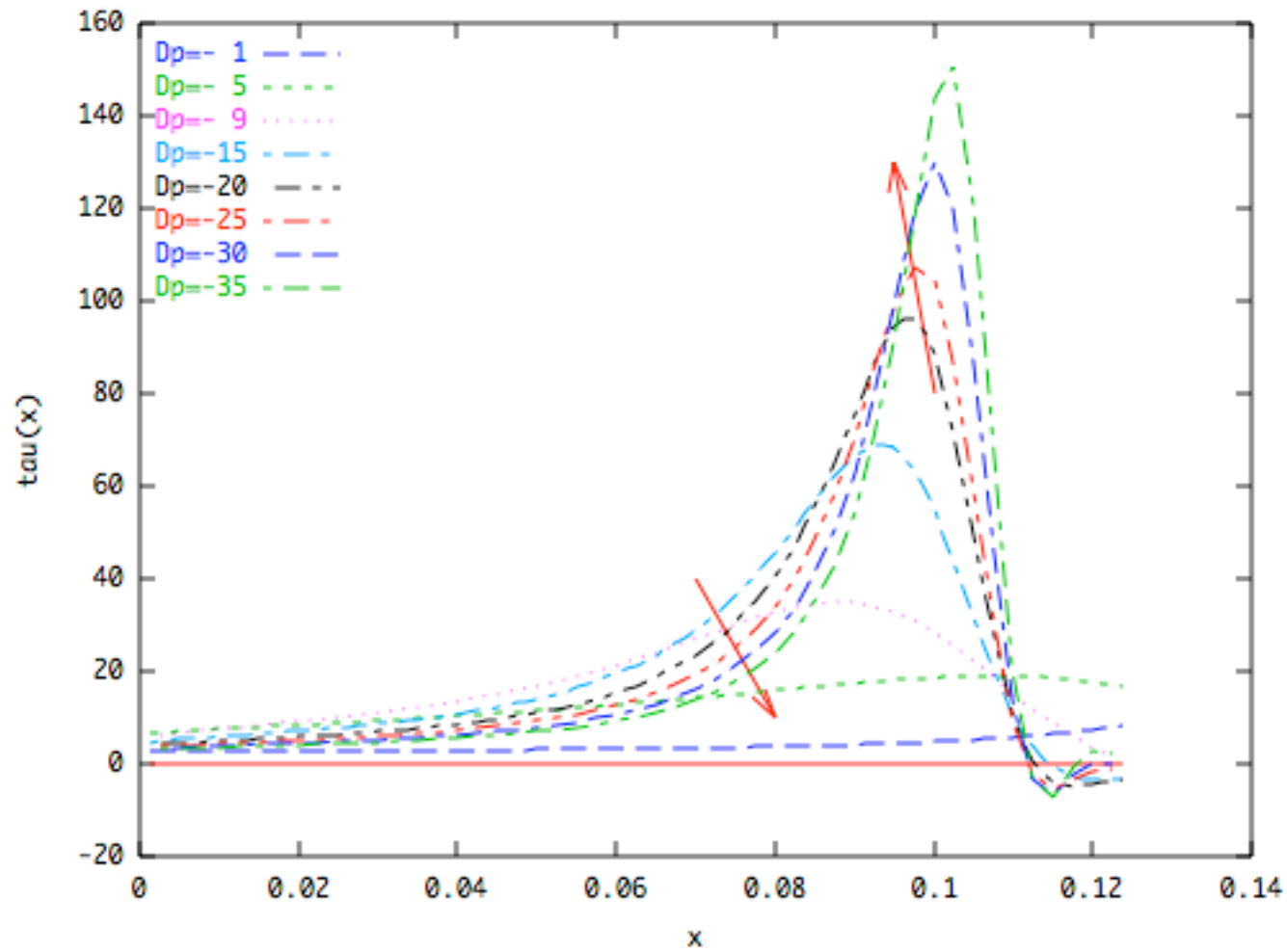
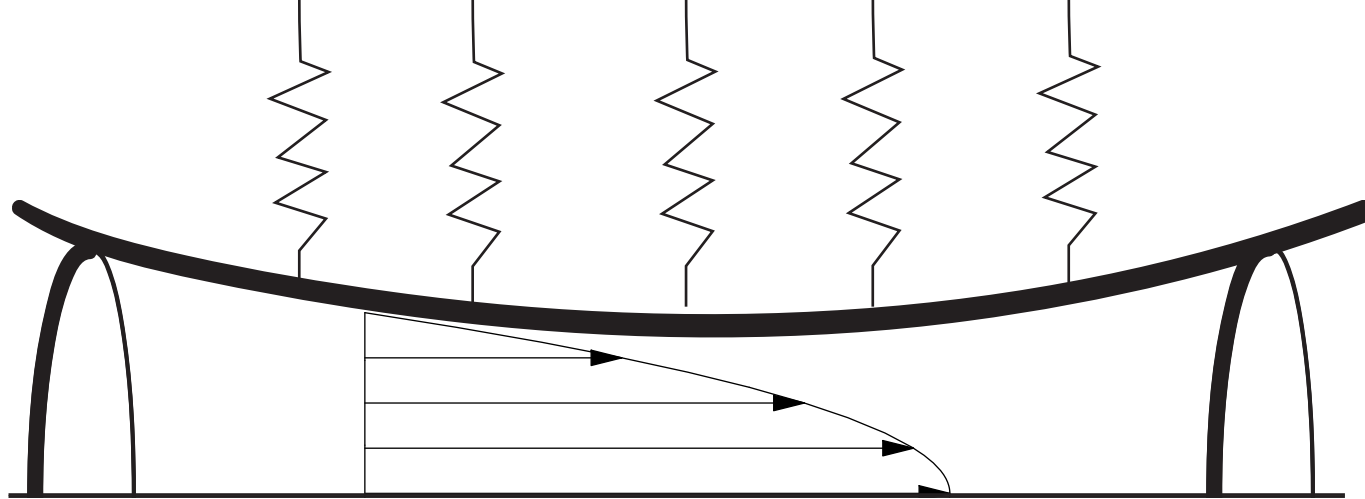
$R^n$  gives  $p^{n+1}$



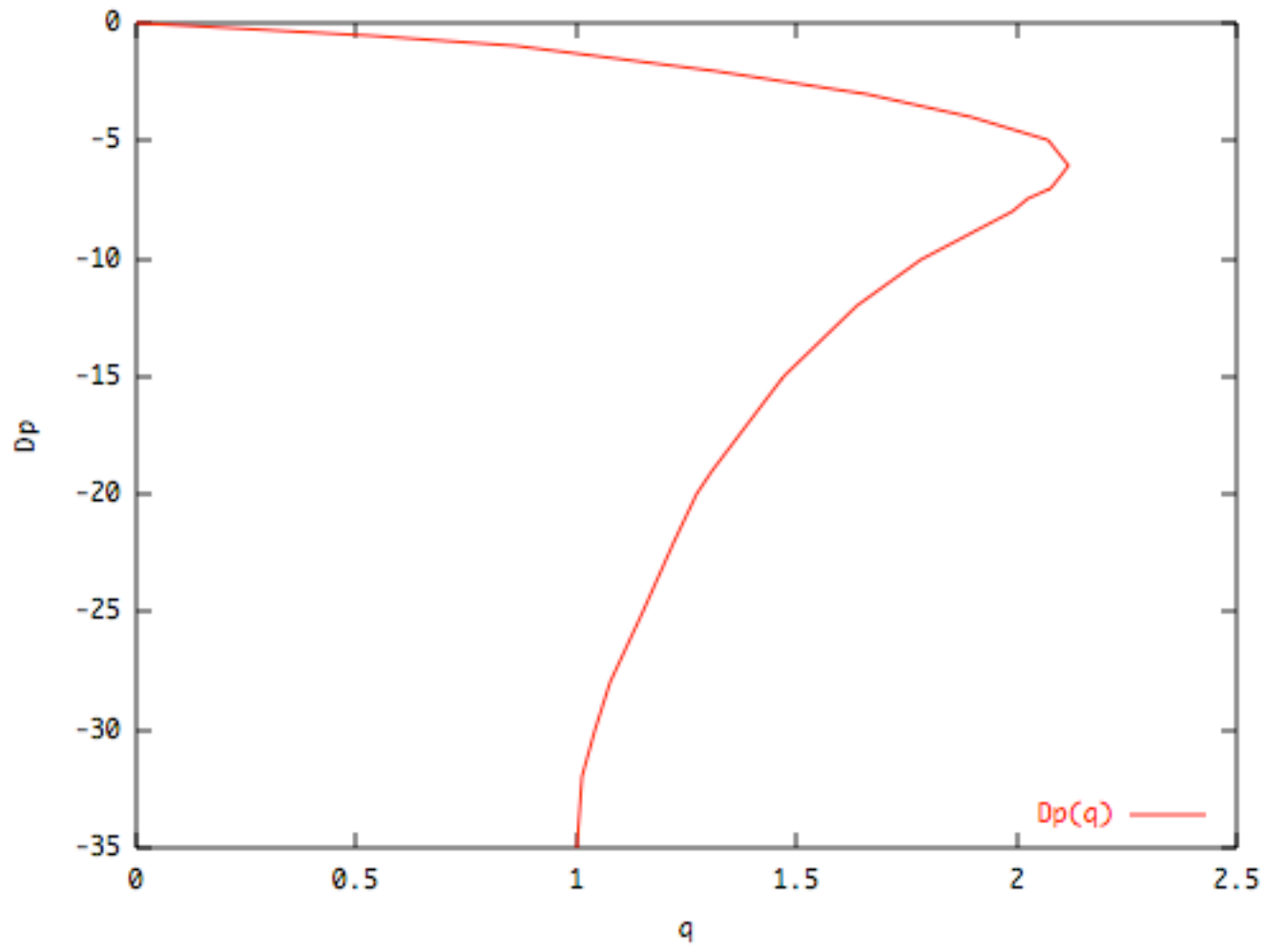
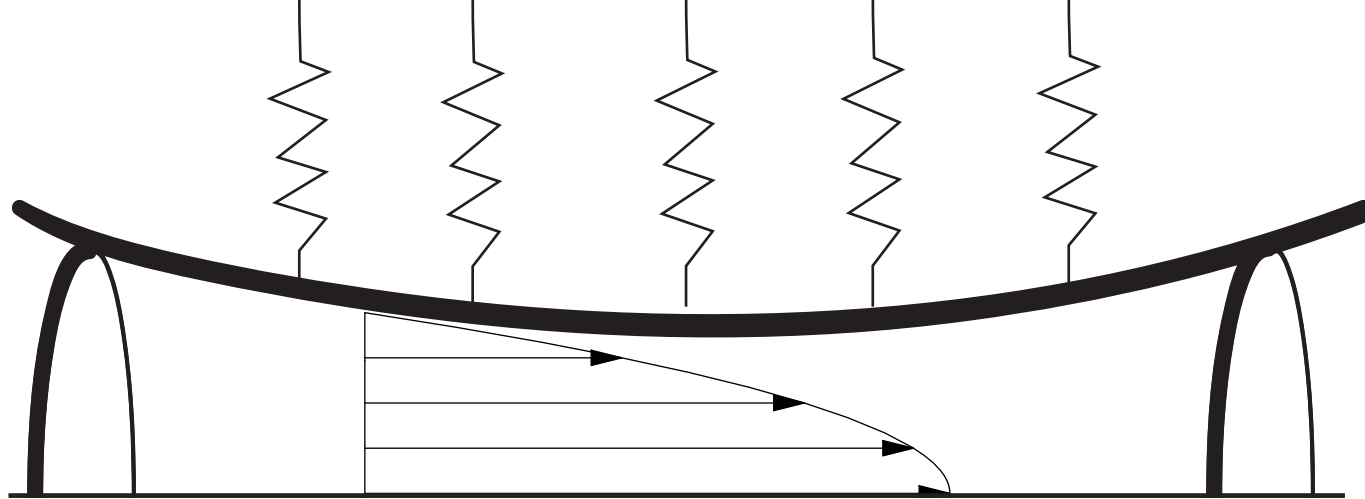
$$R^n \text{ gives } p^{n+1} \longrightarrow p^{n+1} = k(R^{n+1} - 1)$$



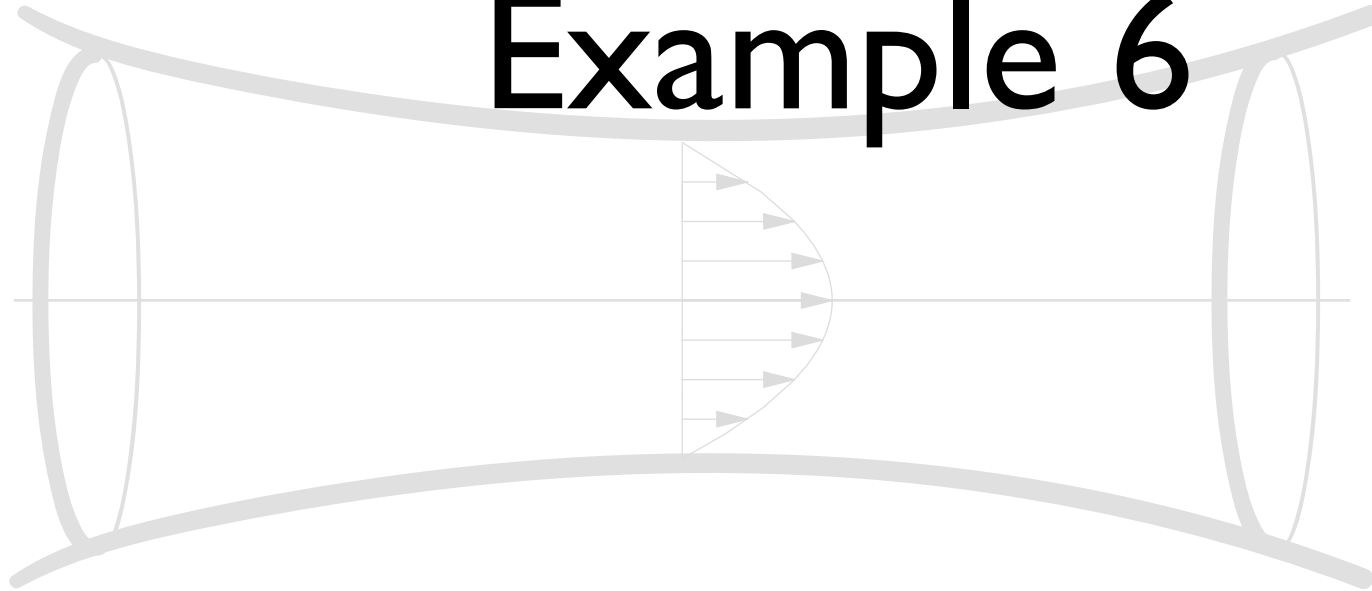




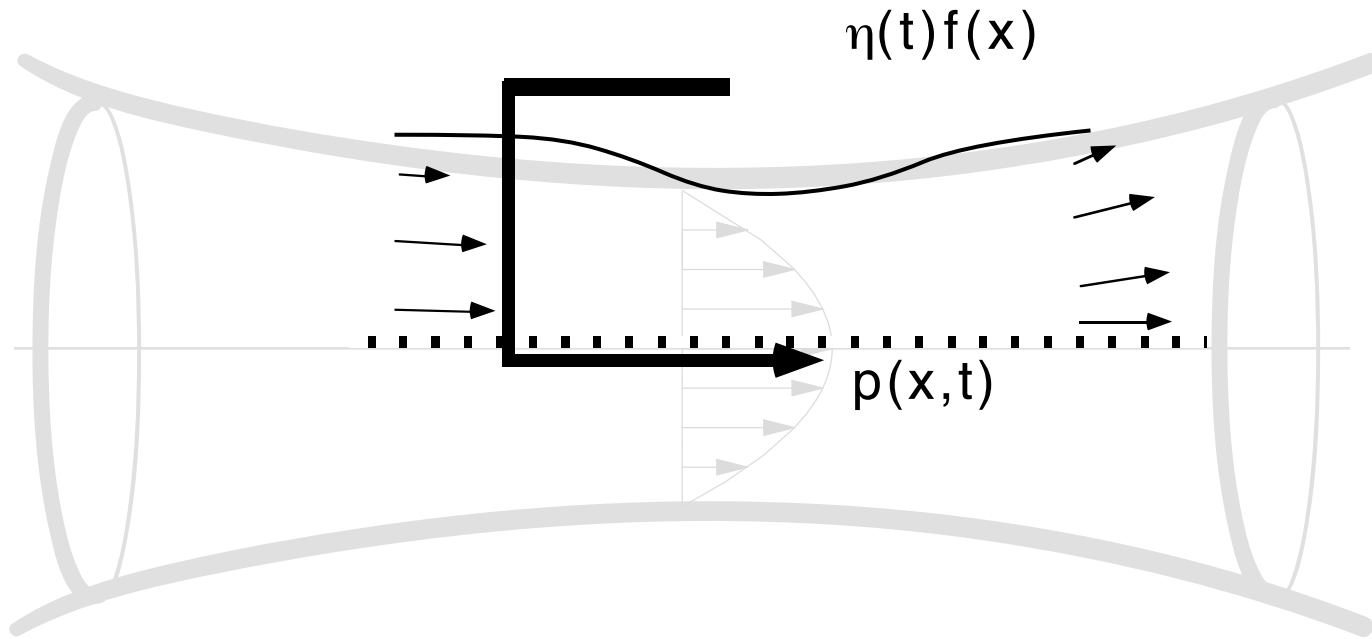


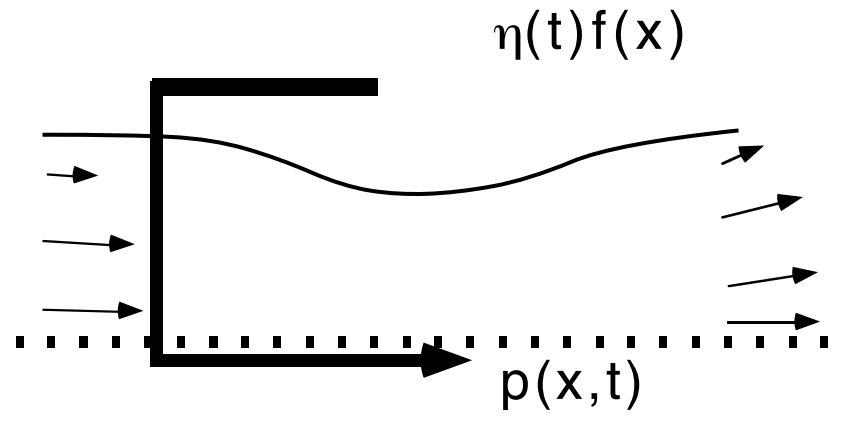


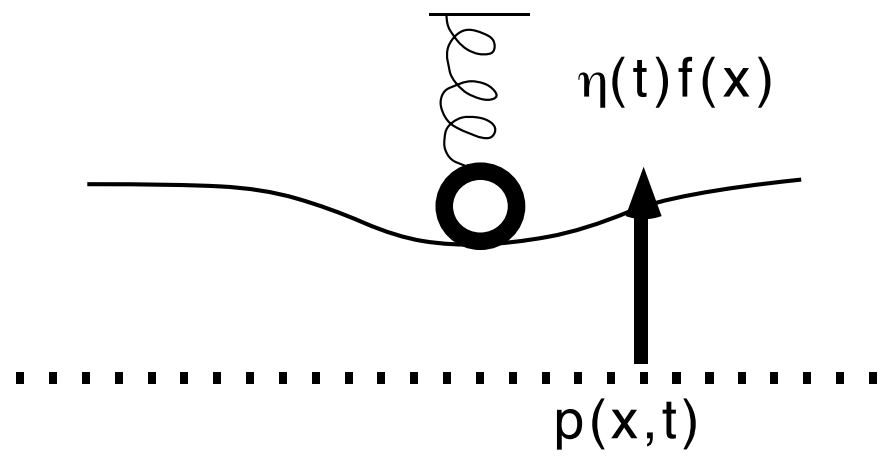
# Example 6

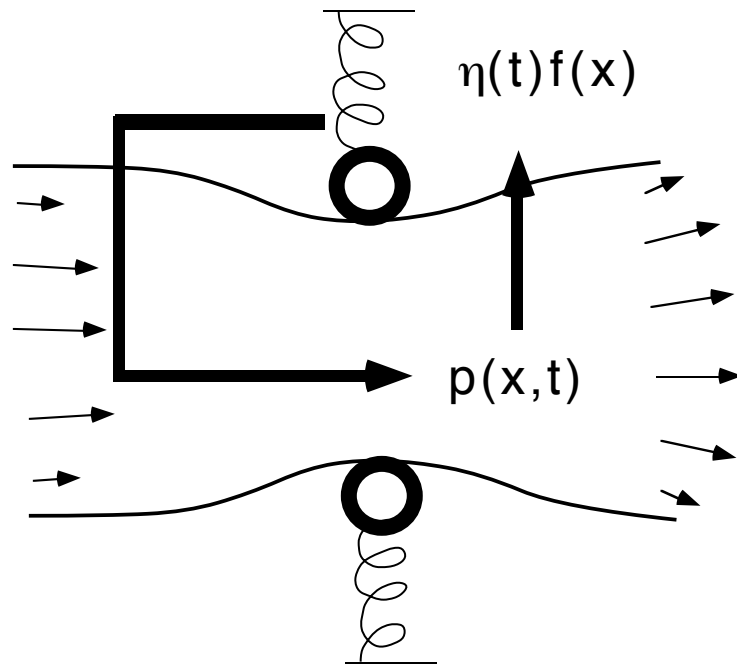


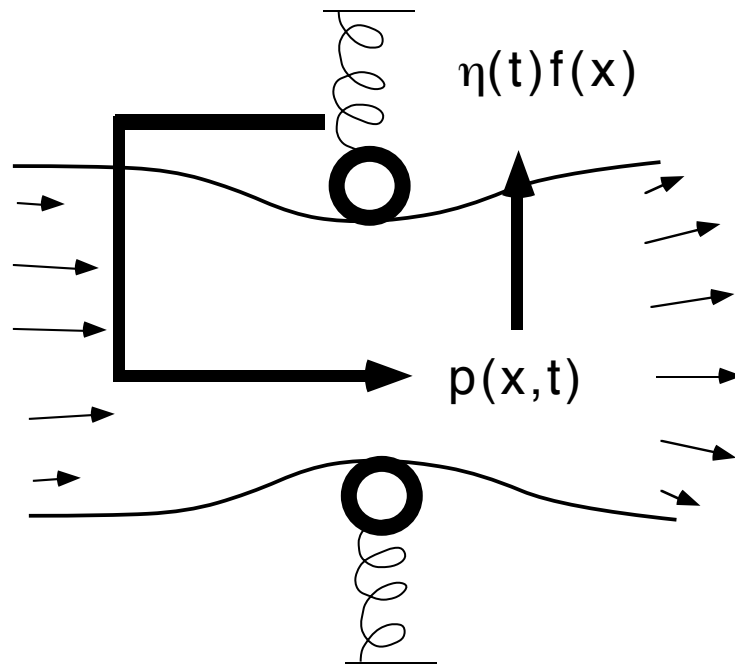
- flow with elastic wall with mass (glottis)



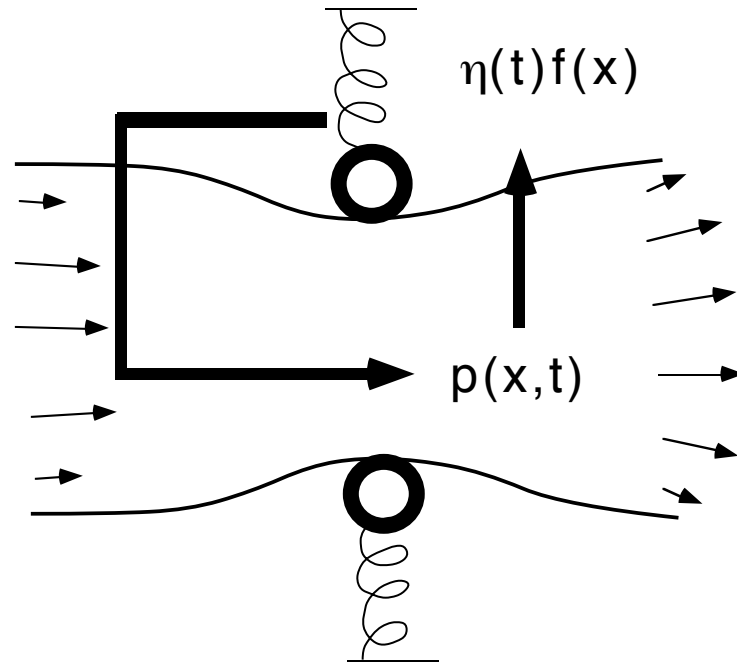






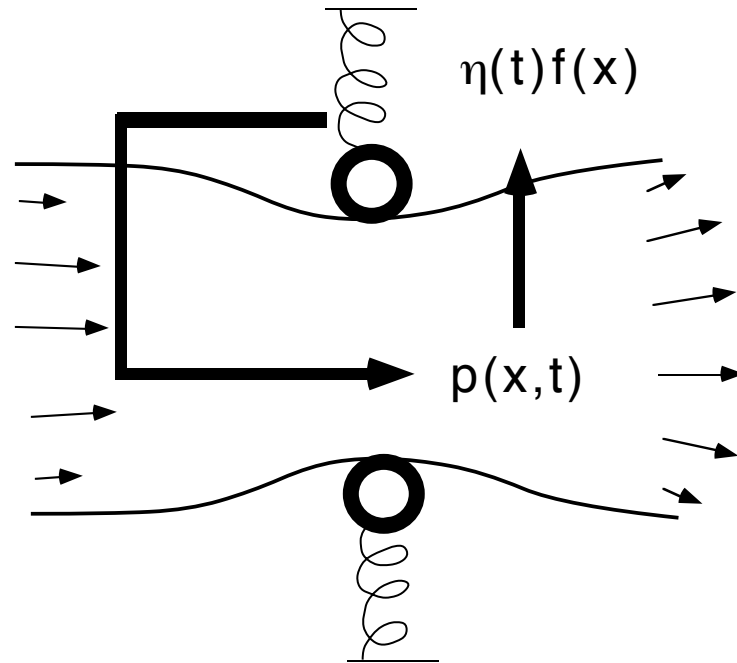


$$\mu \frac{\partial^2 \eta}{\partial t^2} + k\eta = -p$$



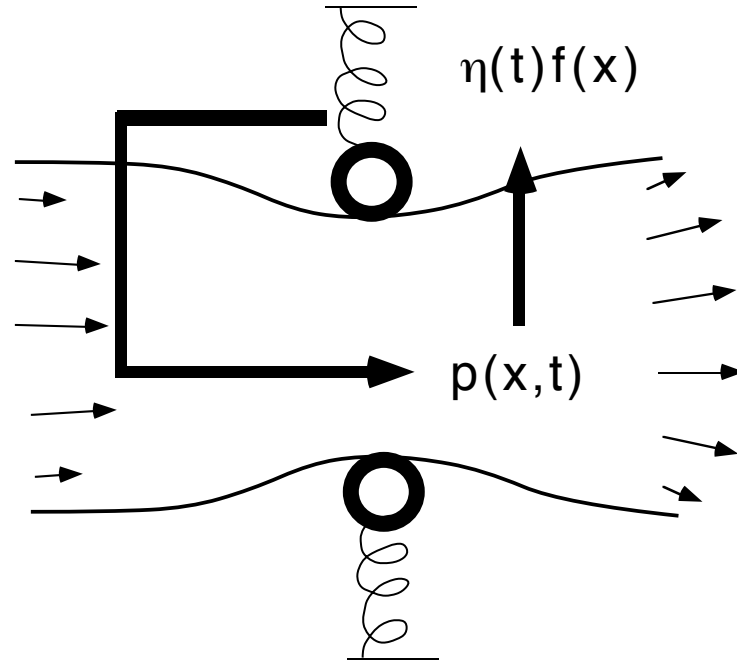
Newmark method for the spring:  
prediction/ correction





Newmark method for the spring:  
prediction/ correction

$$\eta^n, \frac{\partial \eta^n}{\partial t} \xrightarrow{\text{fluid}} p$$

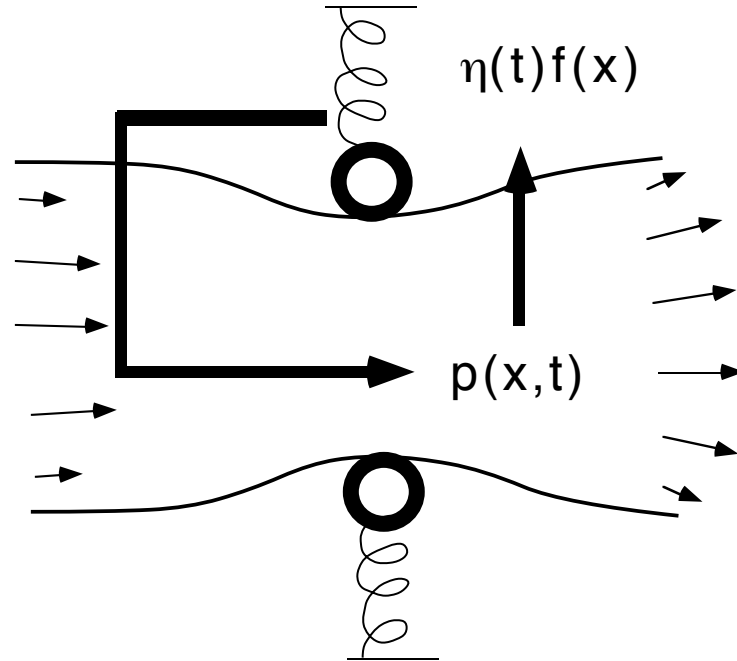


Newmark method for the spring:  
prediction/ correction

$$\eta^n, \frac{\partial \eta^n}{\partial t} \xrightarrow{\text{fluid}} p$$

$$\eta^e, \frac{\partial \eta^e}{\partial t}$$

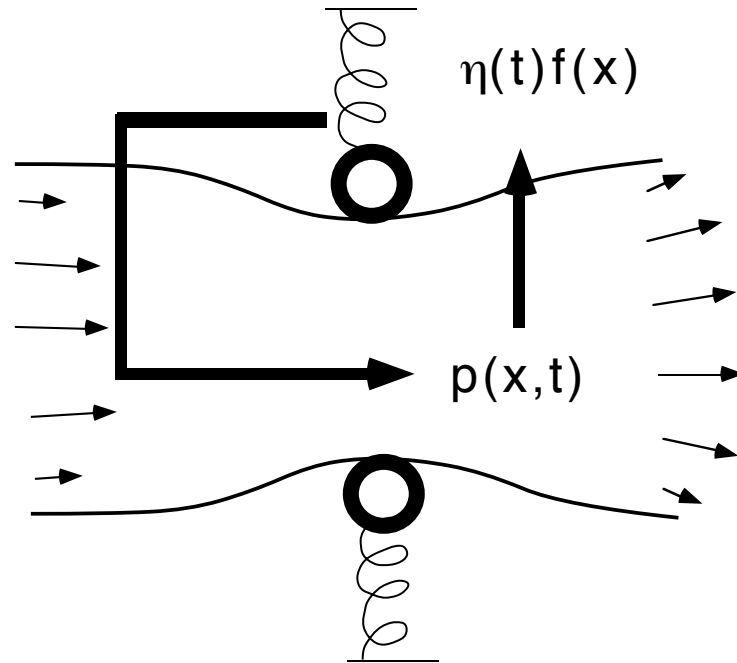
spring-prediction



Newmark method for the spring:  
prediction/ correction

$$\eta^n, \frac{\partial \eta^n}{\partial t} \xrightarrow{\text{fluid}} p \quad \eta^e, \frac{\partial \eta^e}{\partial t} \xrightarrow{\text{fluid}} p^e$$

spring-prediction



Newmark method for the spring:  
prediction/ correction

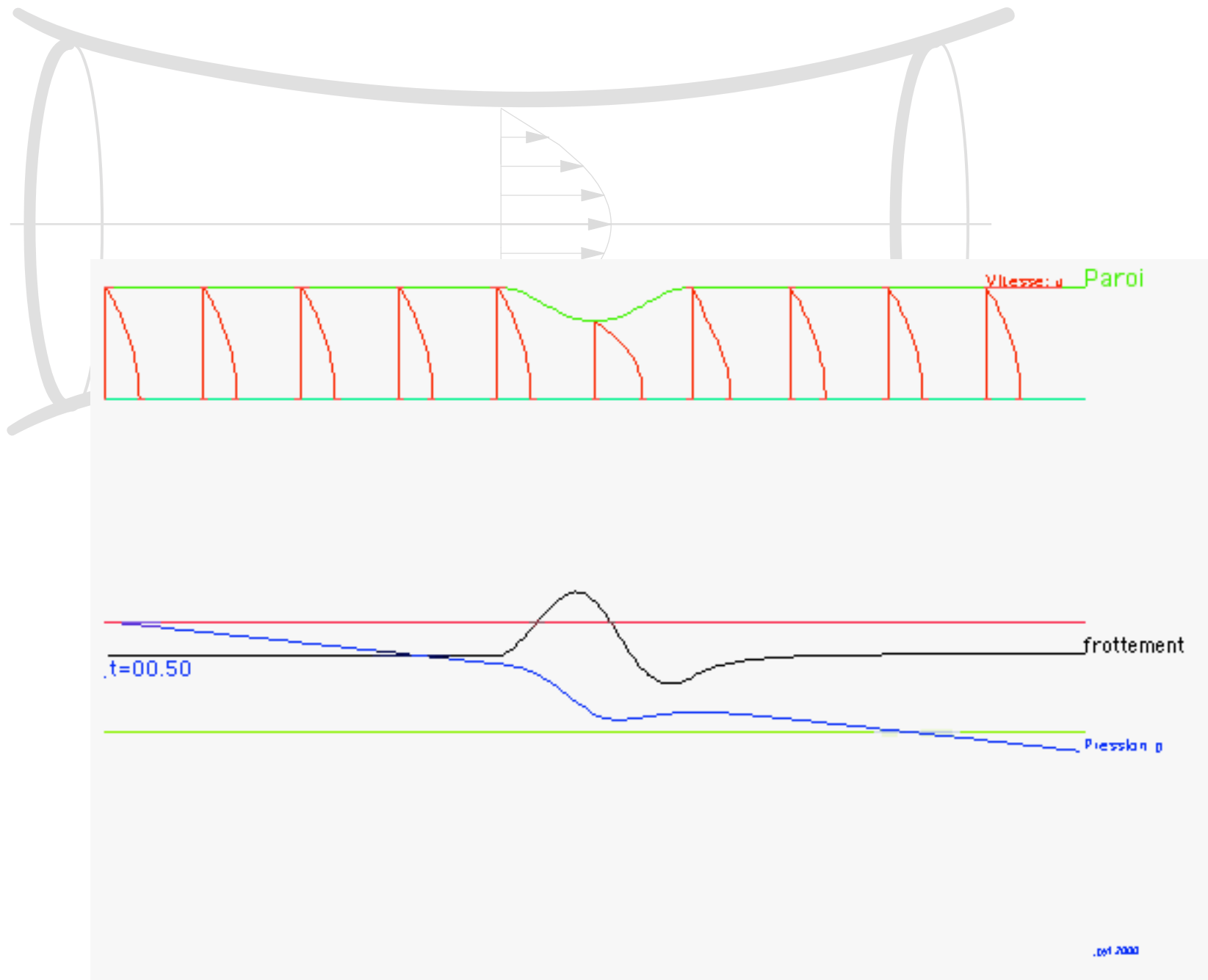
$$\eta^n, \frac{\partial \eta^n}{\partial t} \xrightarrow{\text{fluid}} p$$

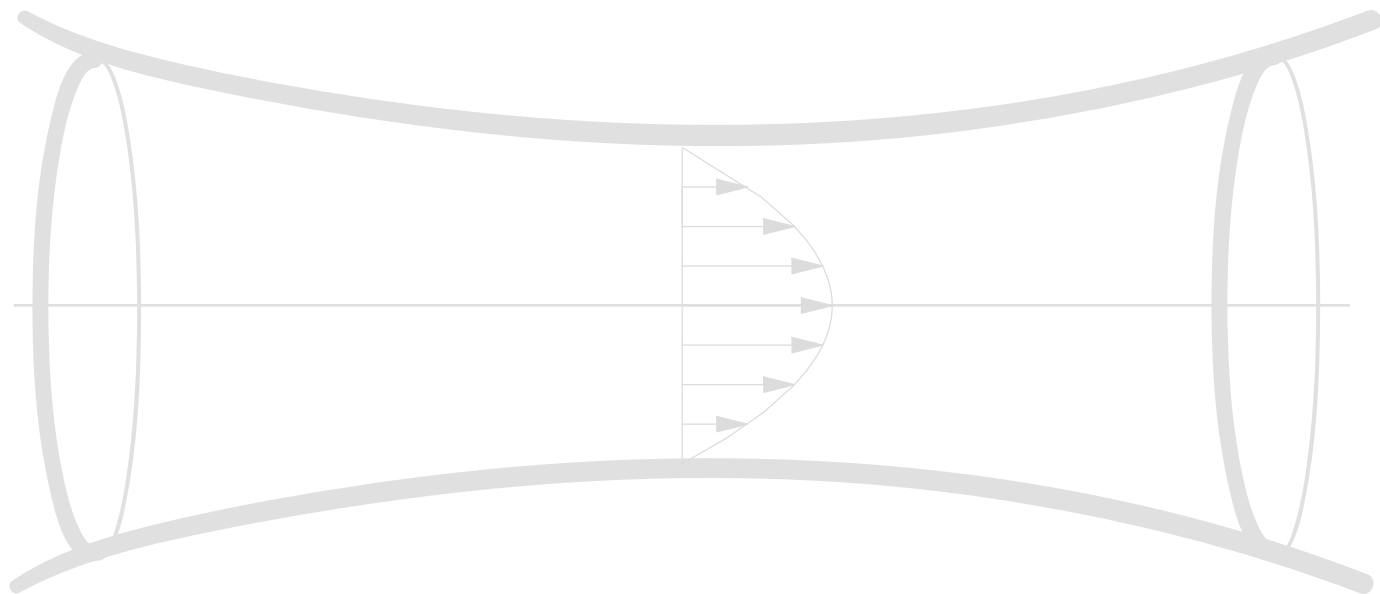
$$\eta^e, \frac{\partial \eta^e}{\partial t} \xrightarrow{\text{fluid}} p^e$$

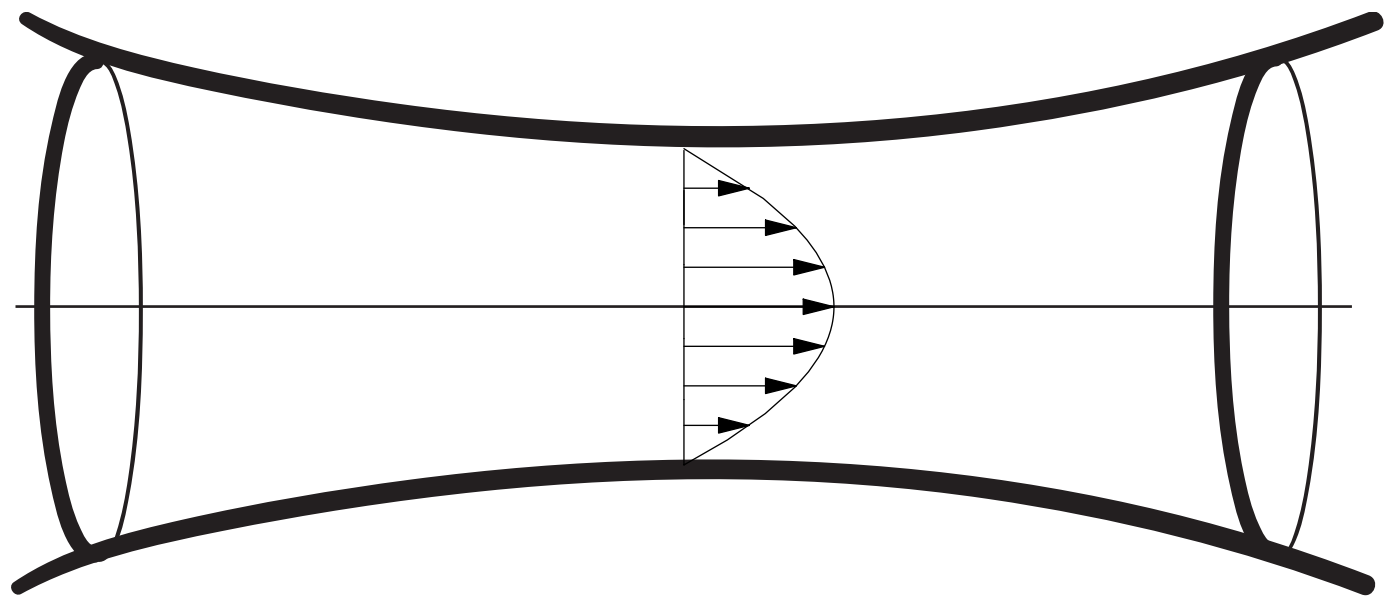
$$\eta^{n+1}, \frac{\partial \eta^{n+1}}{\partial t}$$

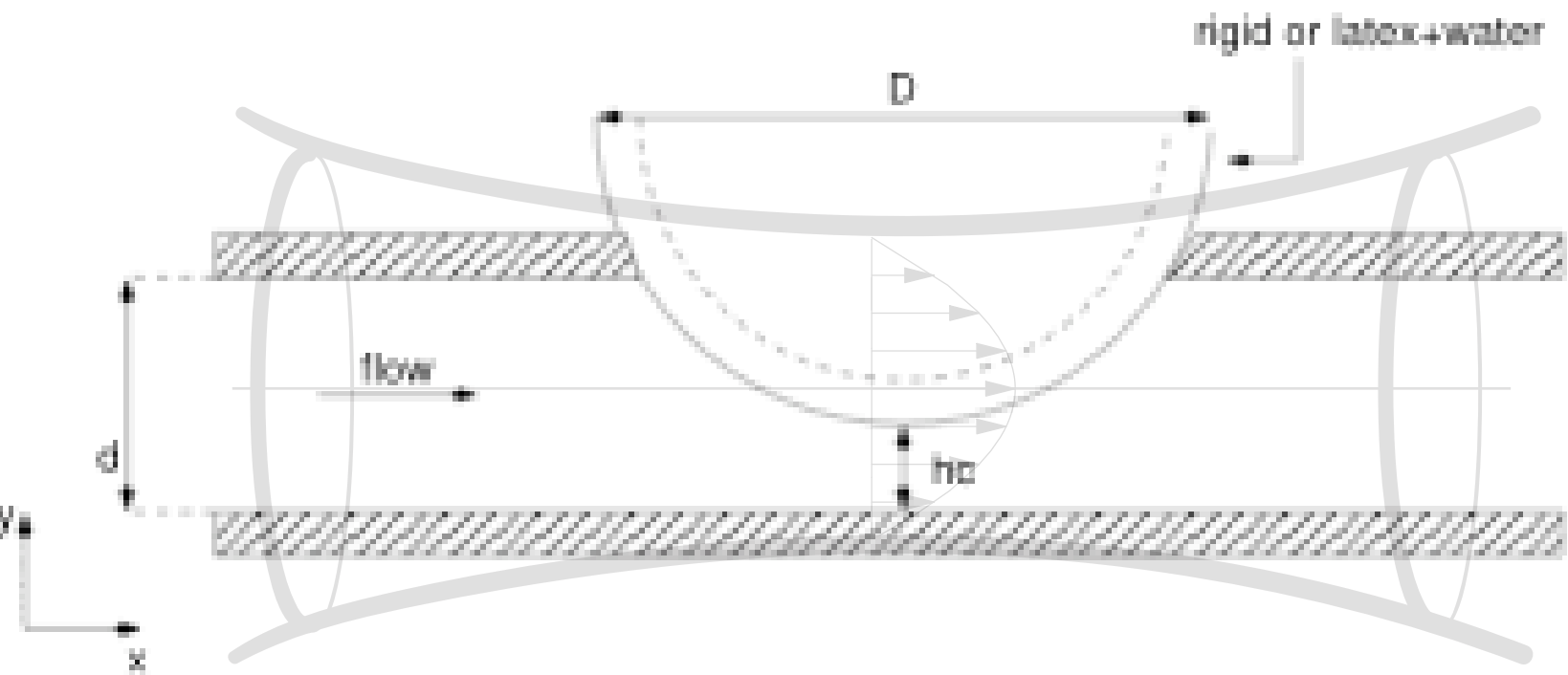
spring-prediction

spring-correction

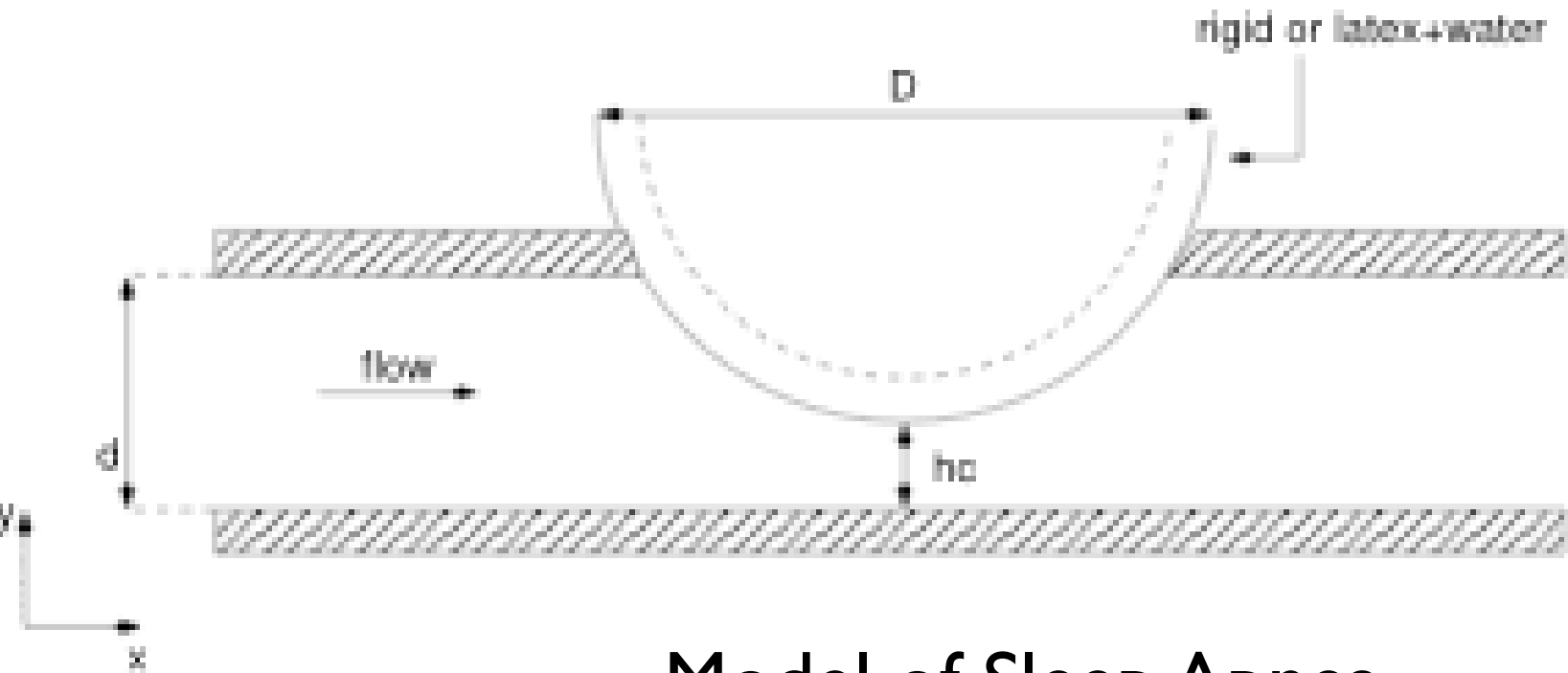




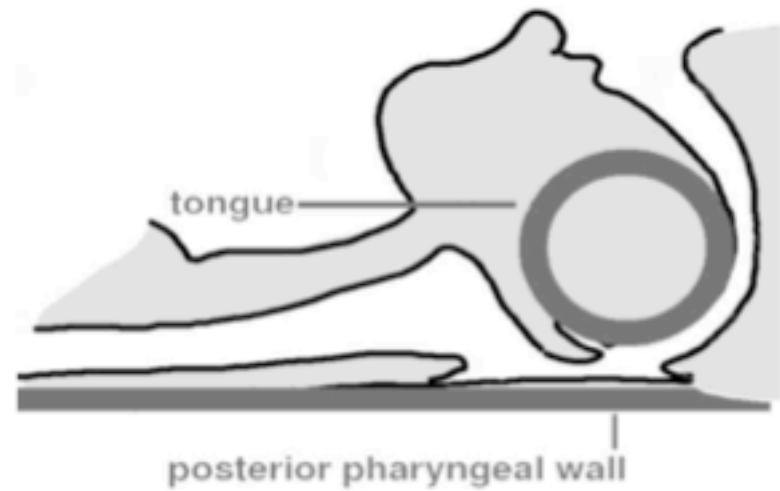
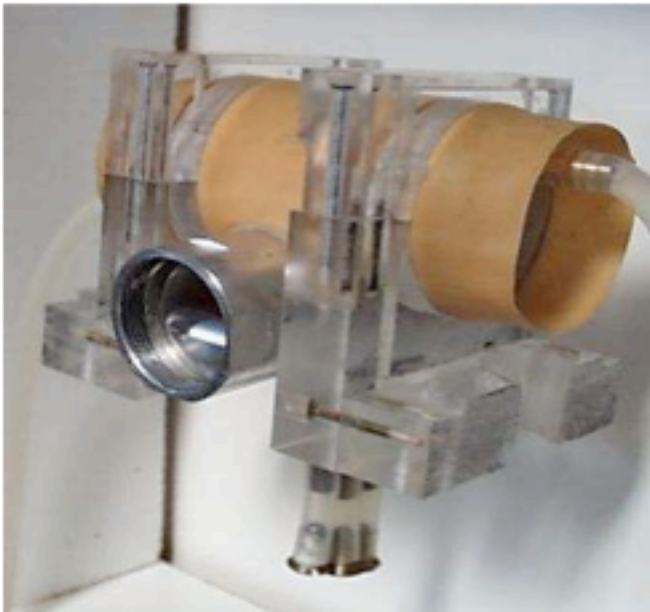


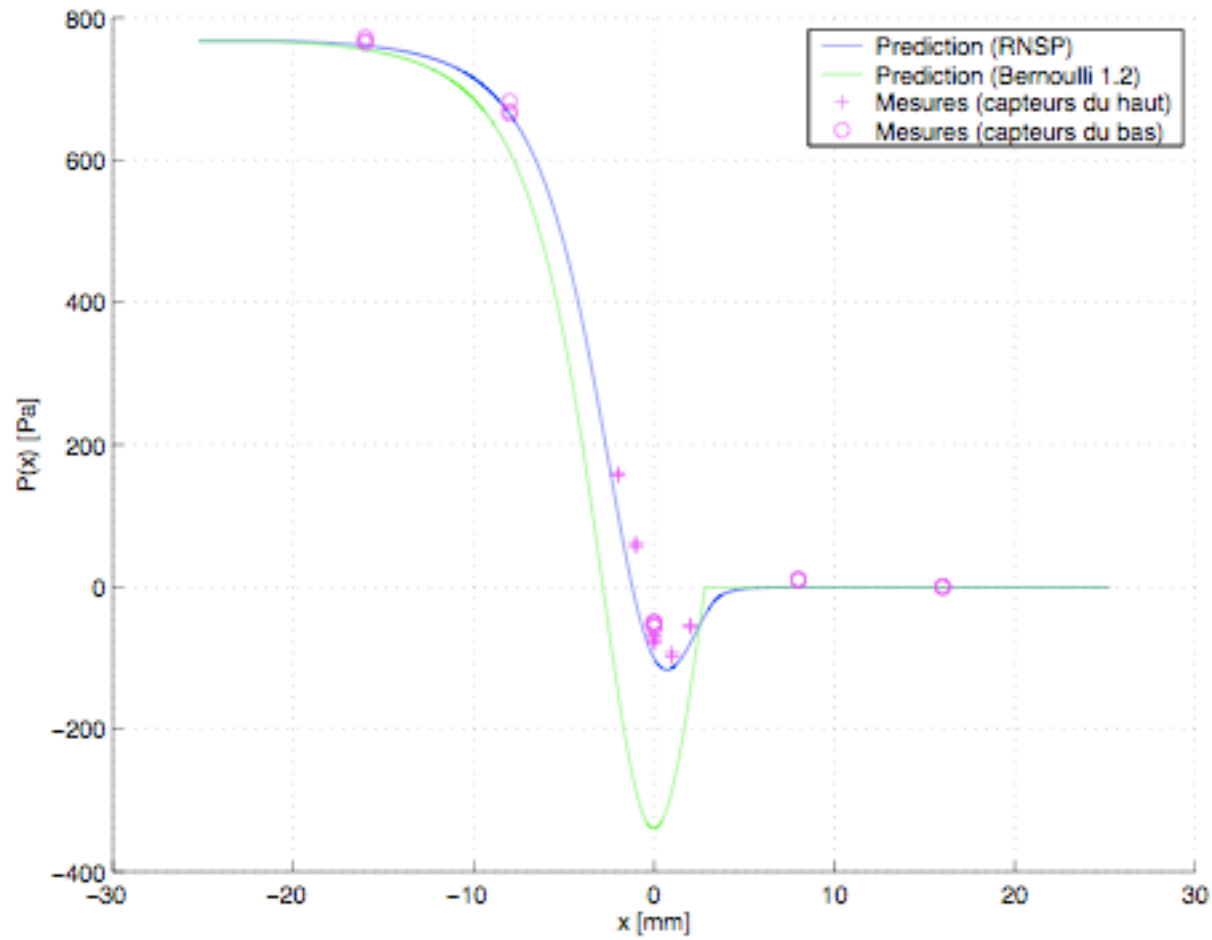
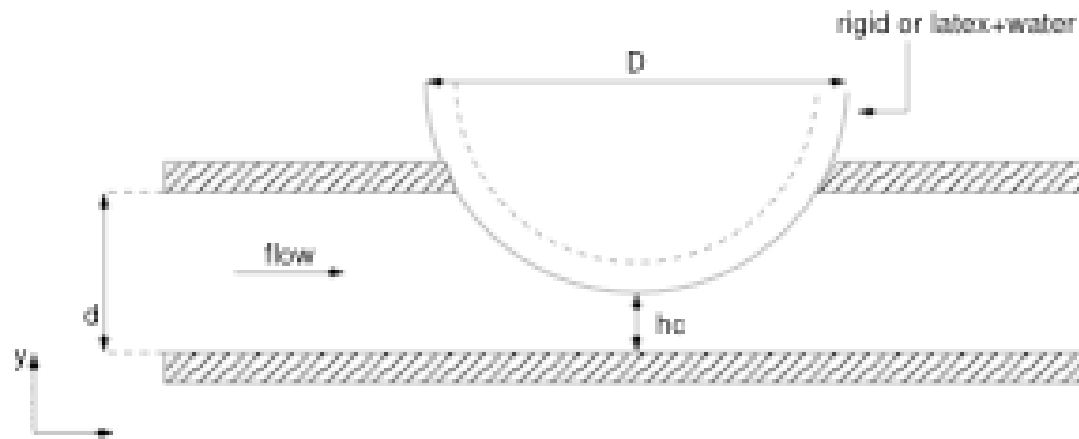




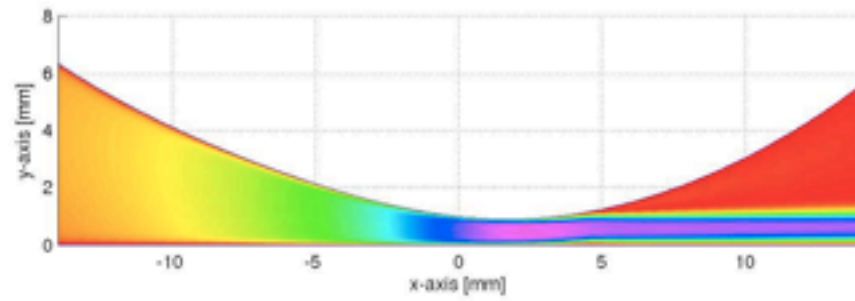


## Model of Sleep Apnea

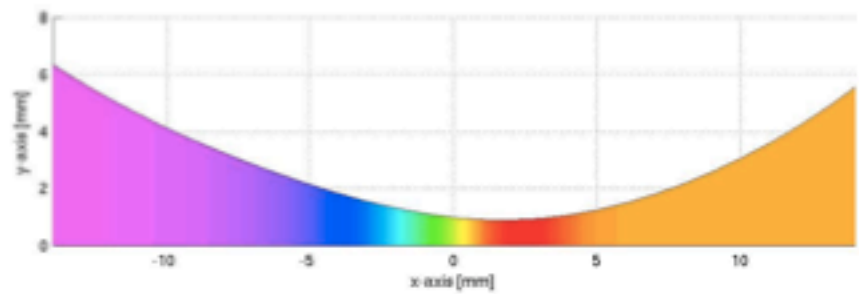




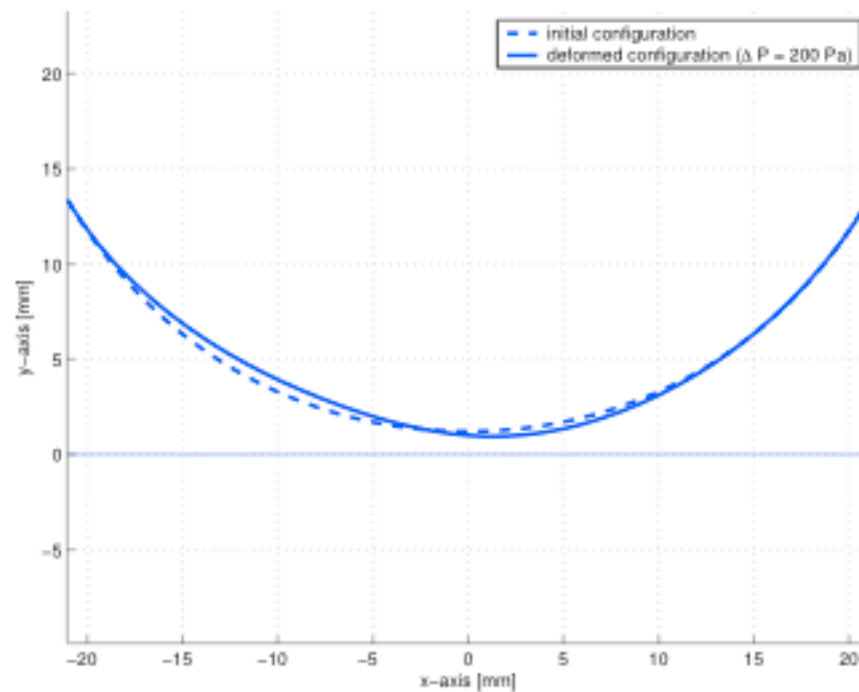
rigid case



(a)



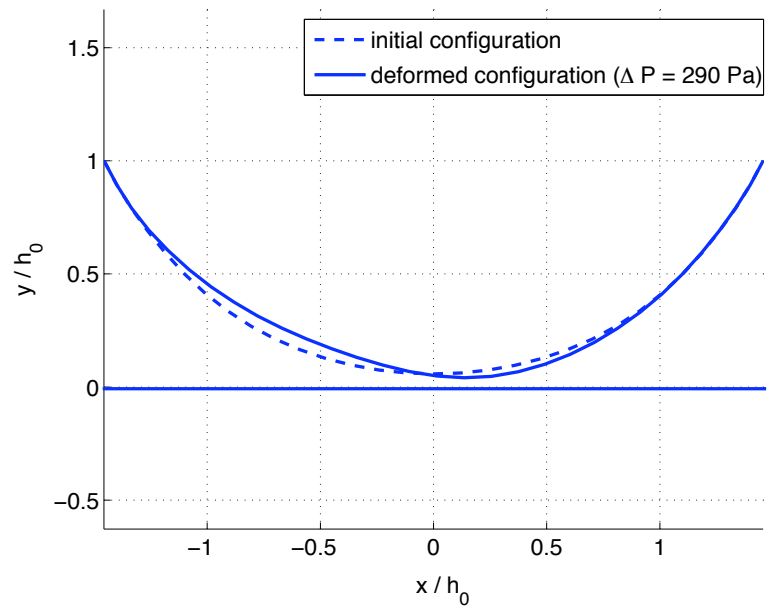
(b)



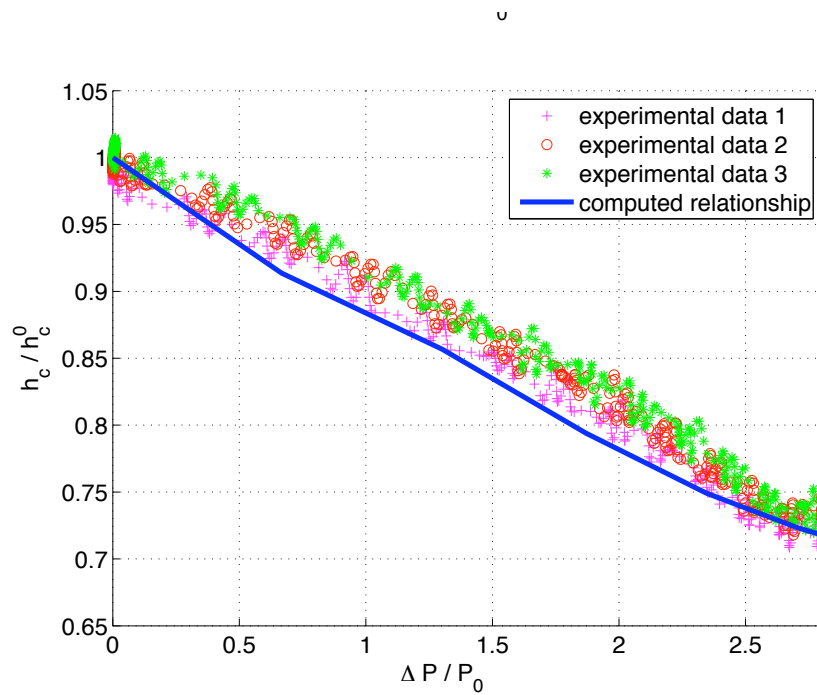
(c)

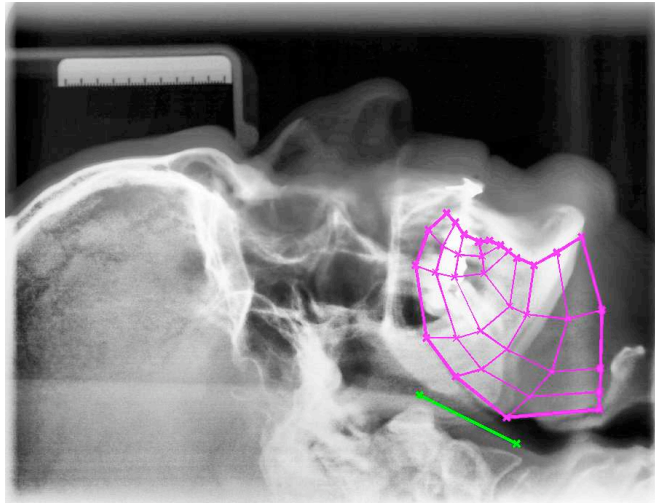
RNSP + Ansys

elastic wall

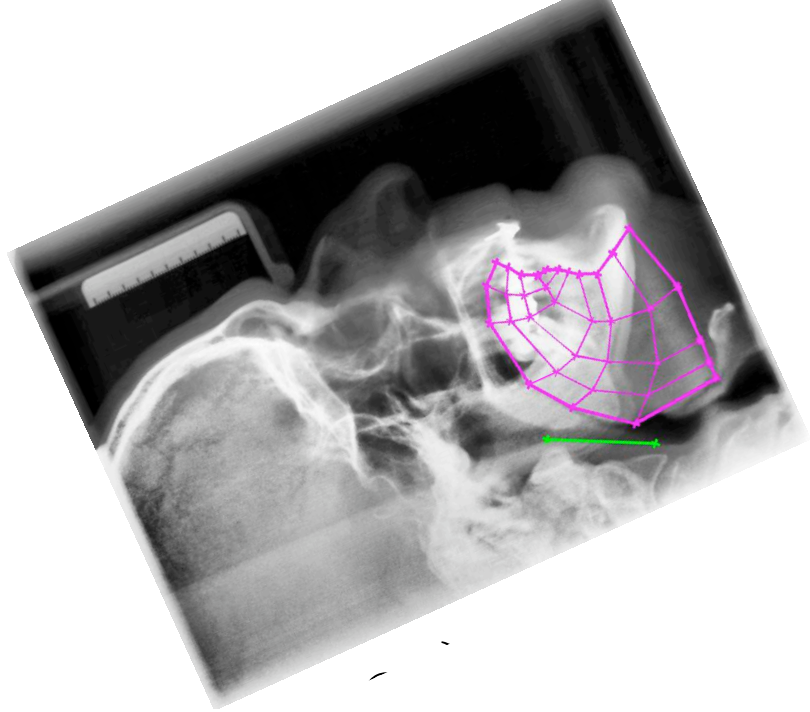


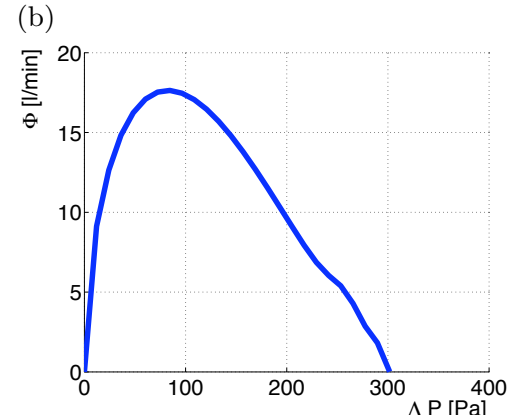
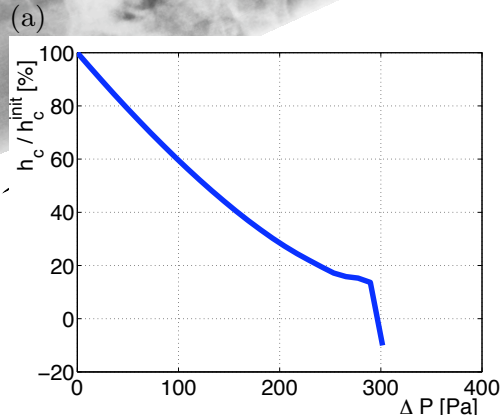
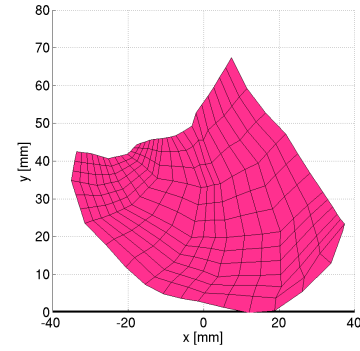
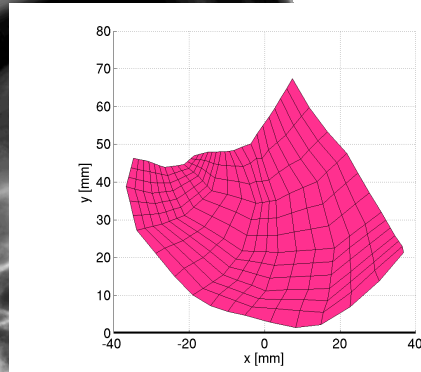
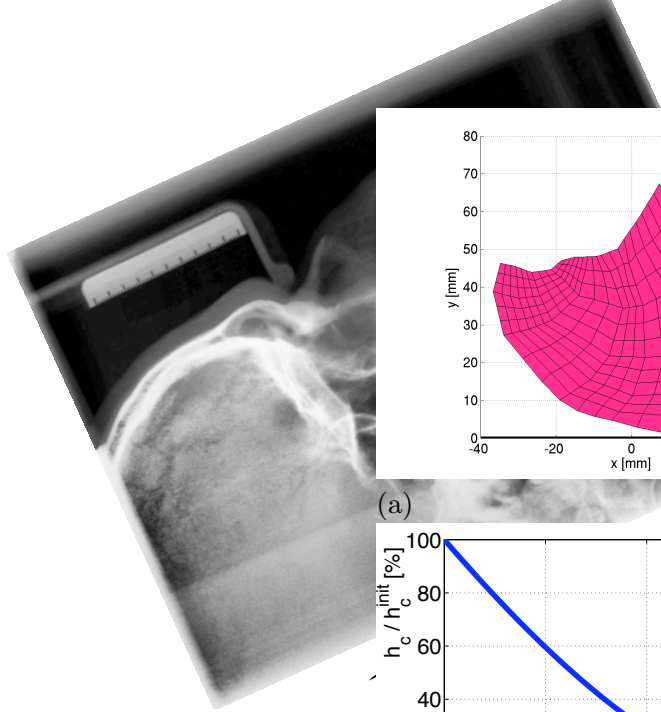
*Simulation of a fluid-structure interaction, for  $\Delta P = 290$  Pa,  $P_{ext} = 400$  Pa and  $h_c = 0.87$  mm.*





1 2 3





(c)

(d)

A. Van Hirtum, X. Pelorson & P.-Y. Lagr e (2005):

"In-vitro validation of some flow assumptions for the prediction of the pressure distribution during obstructive sleep apnea",  
Medical & biological engineering & computing, no 43(1) pp. 162-171.

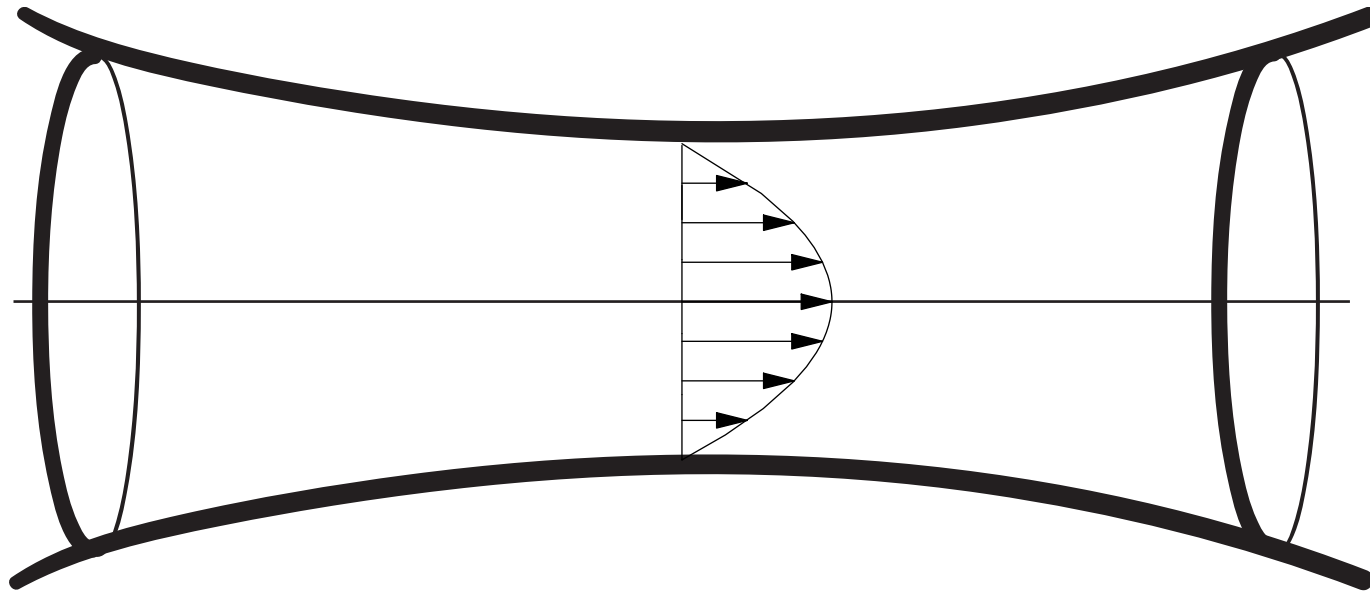
F. Chouly, A. Van Hirtum, X. Pelorson, Y. Payan, and P.-Y. Lagr e:

"An attempt to model Obstructive Sleep Apnea Syndrome: preliminary study" subm.

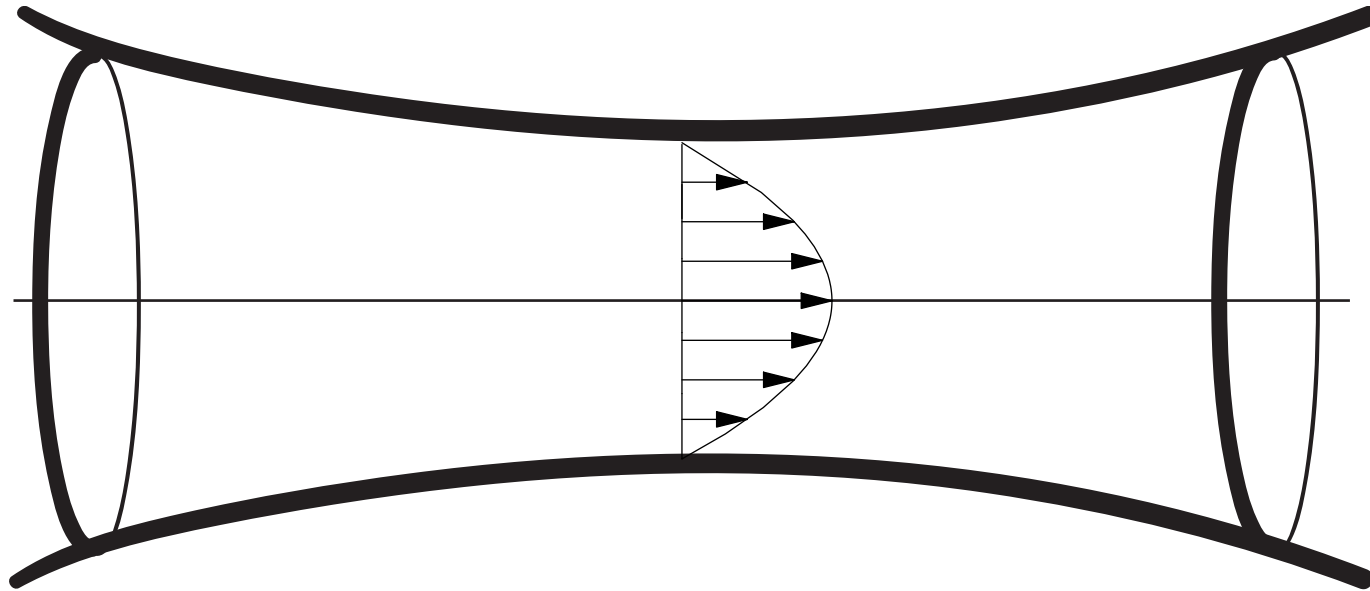
- 3D? Unsteady...



# Integral resolution

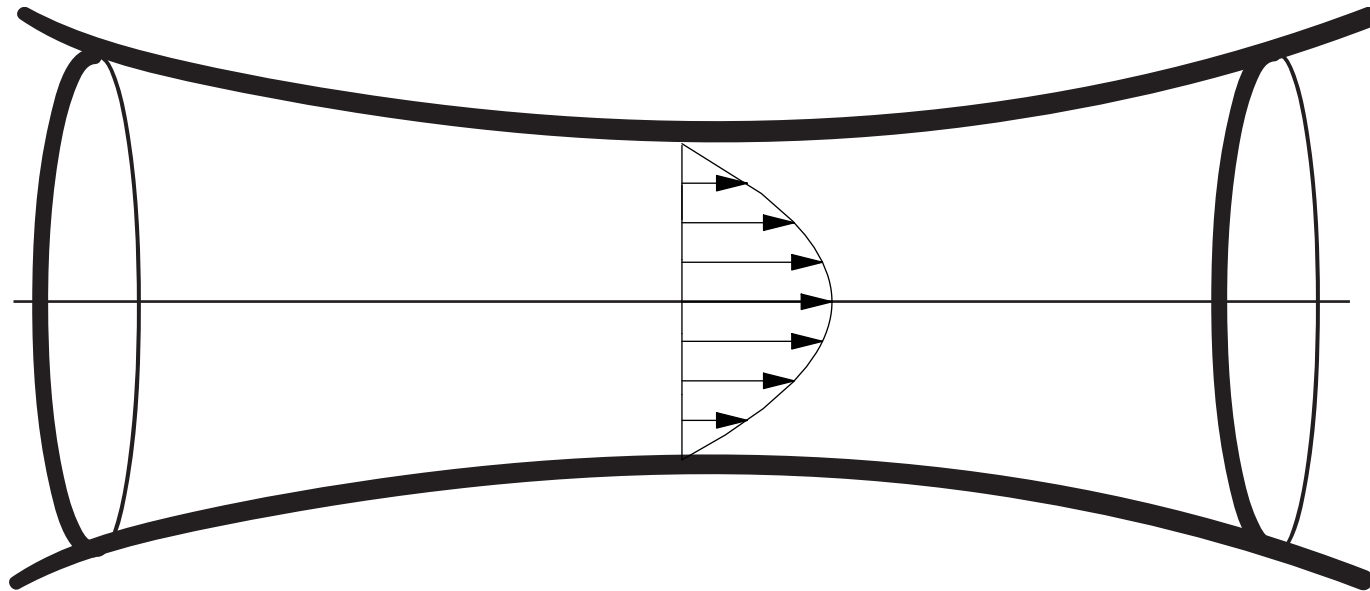


# Integral resolution

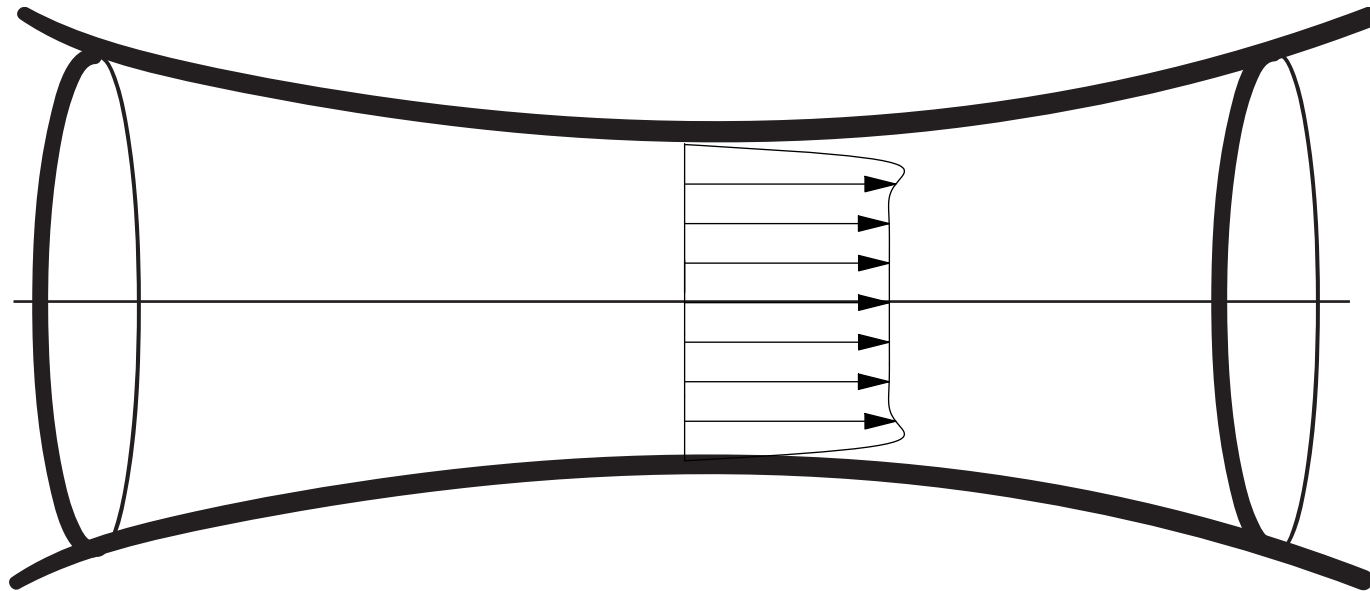


- integral system (ID) is included in RNSP
- we compute a more real profile

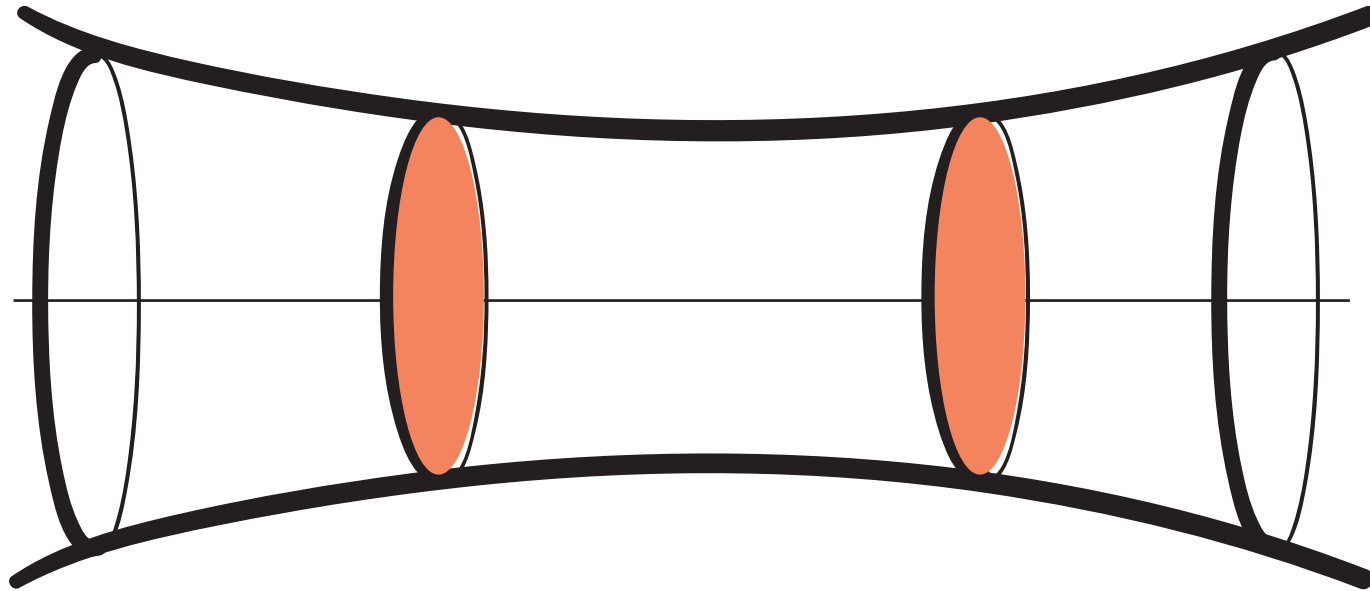
# Integral resolution



# Integral resolution

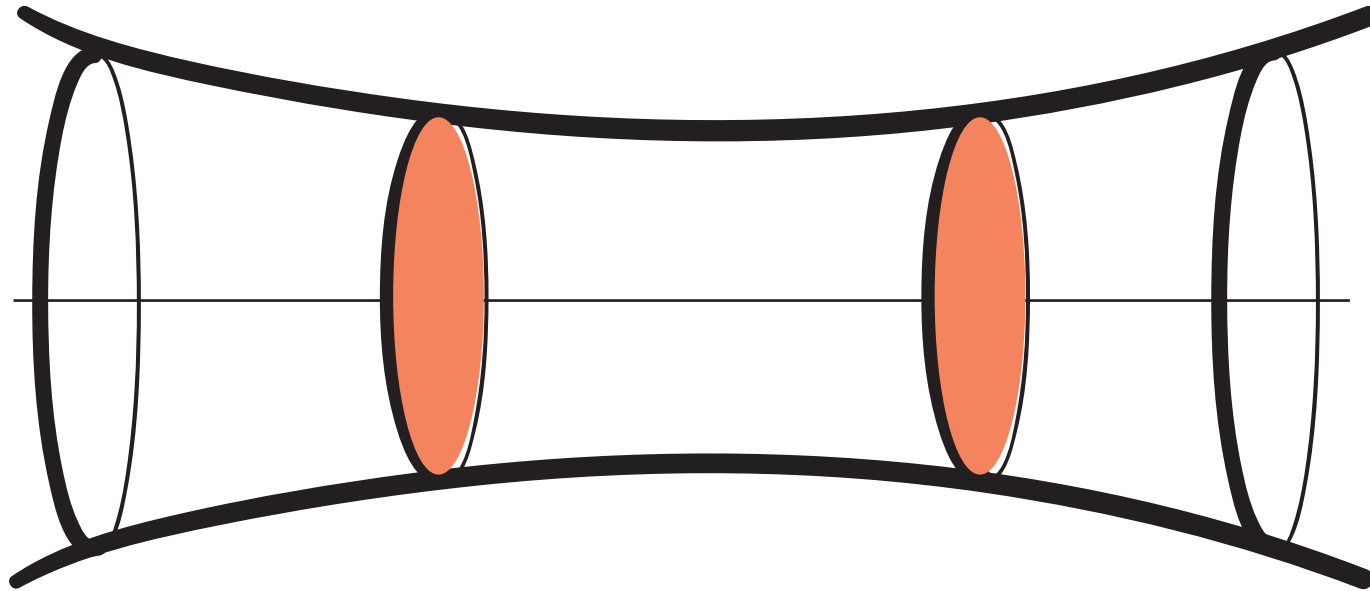


# Integral resolution



$$Q = \int_0^R 2\pi r u dr$$

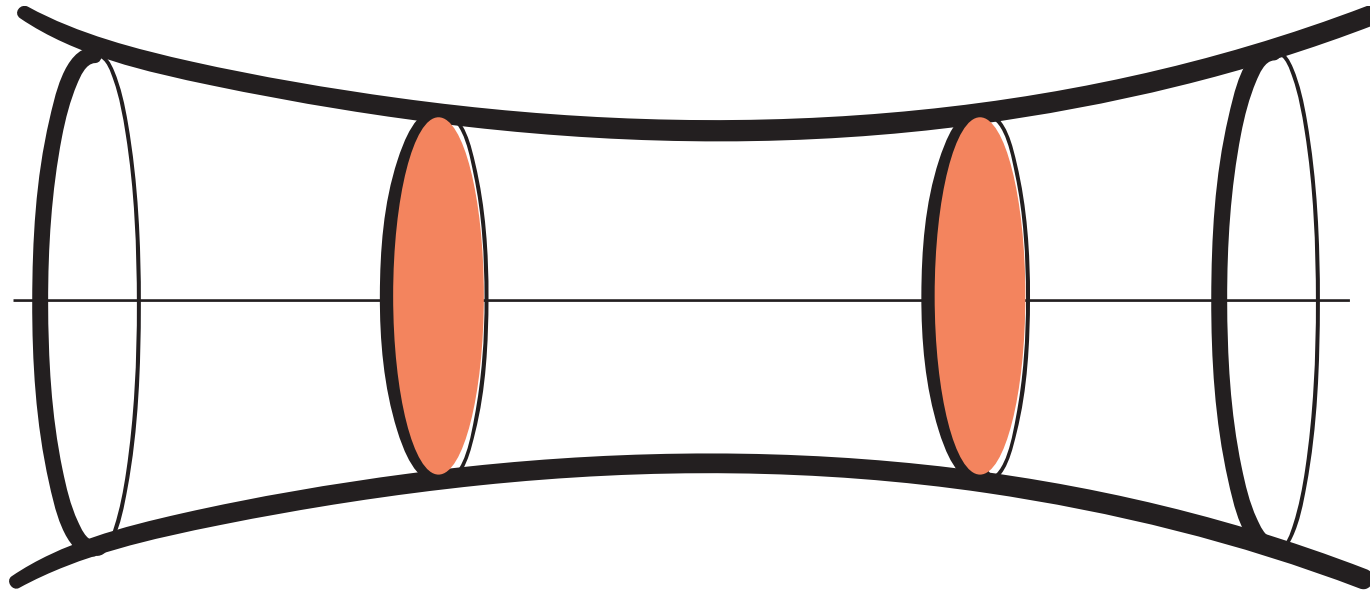
# Integral resolution



$$Q = \int_0^R 2\pi r u dr$$

$$\frac{\partial u}{\partial x} + \frac{\partial r v}{r \partial r} = 0$$

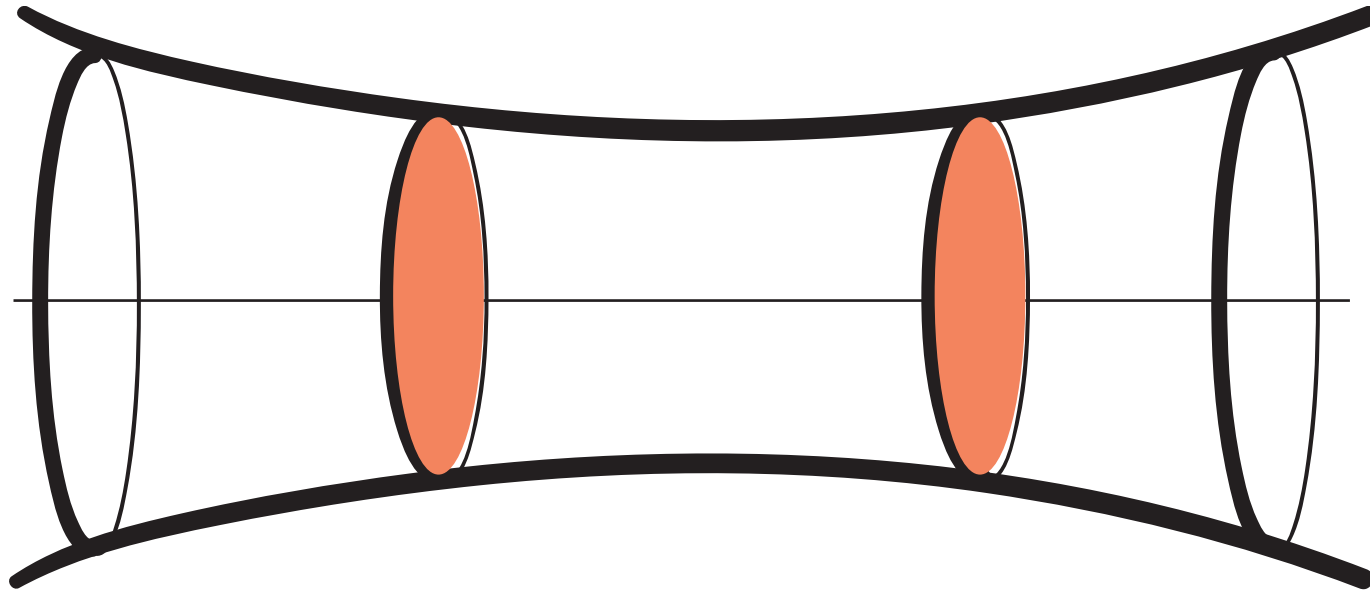
# Integral resolution



$$Q = \int_0^R 2\pi r u dr$$

$$\int_0^R 2\pi r dr \cdot \left( \frac{\partial u}{\partial x} + \frac{\partial r v}{r \partial r} \right) = 0$$

# Integral resolution

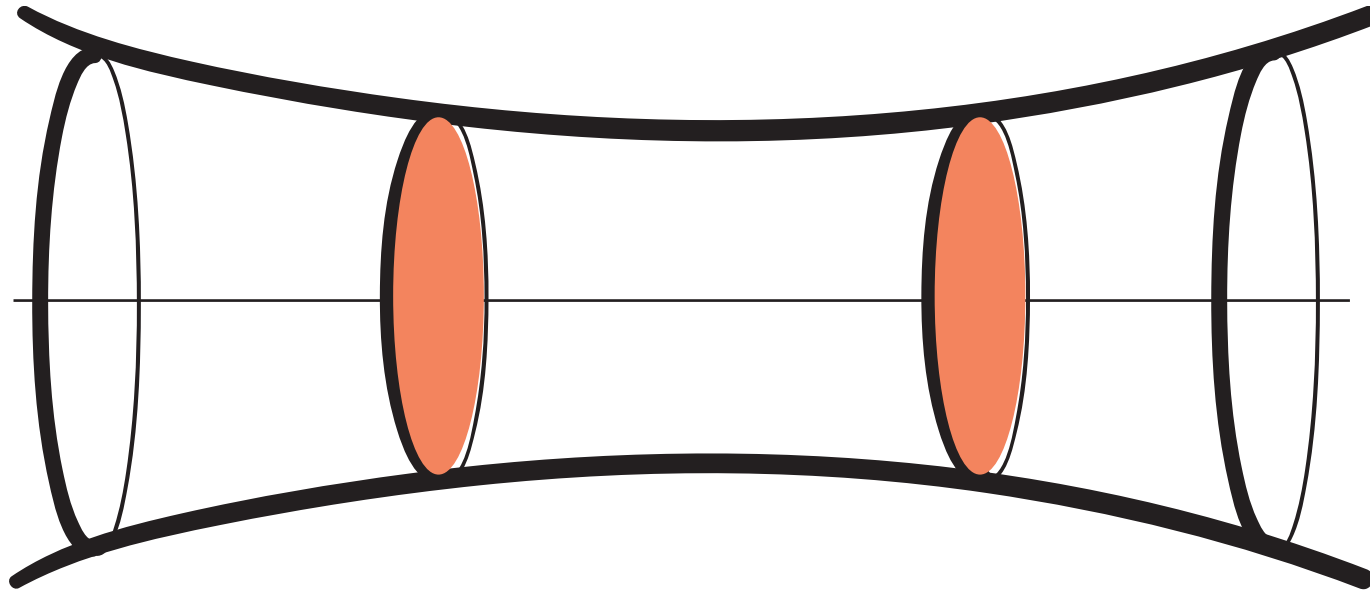


$$Q = \int_0^R 2\pi r u dr$$

$$\int_0^R 2\pi r dr \cdot \left( \frac{\partial u}{\partial x} + \frac{\partial r v}{r \partial r} \right) = 0 \rightarrow \frac{\partial(2\pi R^2)}{\partial t} + \frac{\partial Q}{\partial x} = 0$$



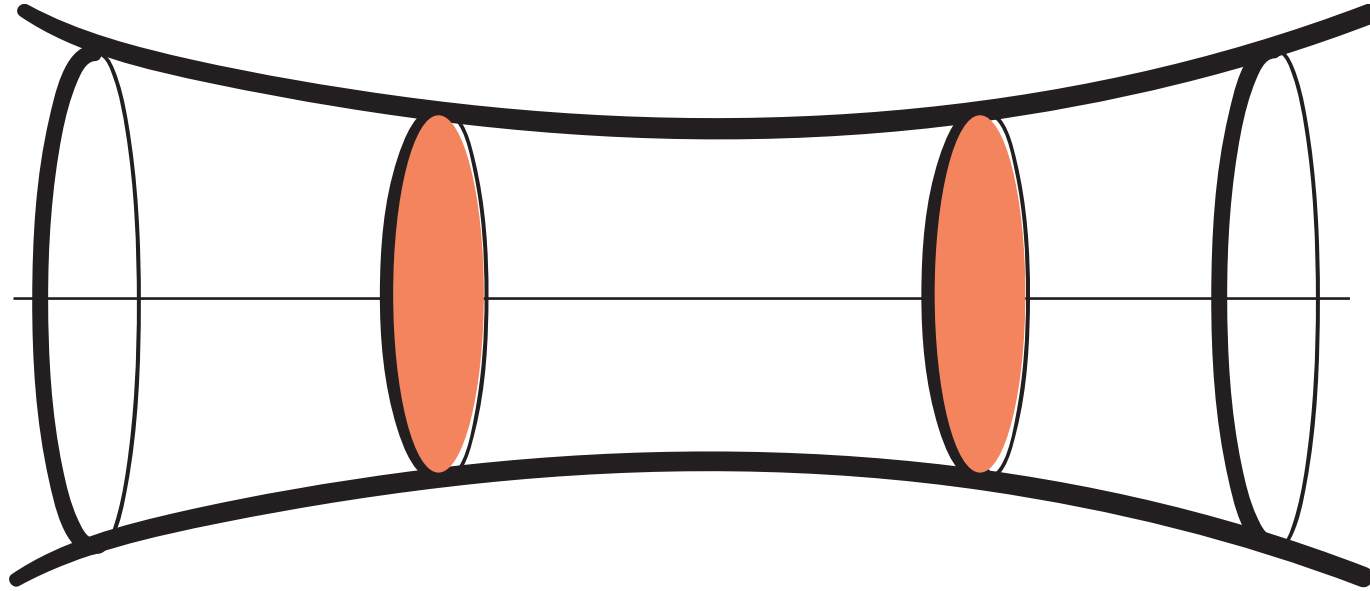
# Integral resolution



$$Q = \int_0^R 2\pi r u dr$$

$$\frac{\partial(2\pi R^2)}{\partial t} + \frac{\partial Q}{\partial x} = 0$$

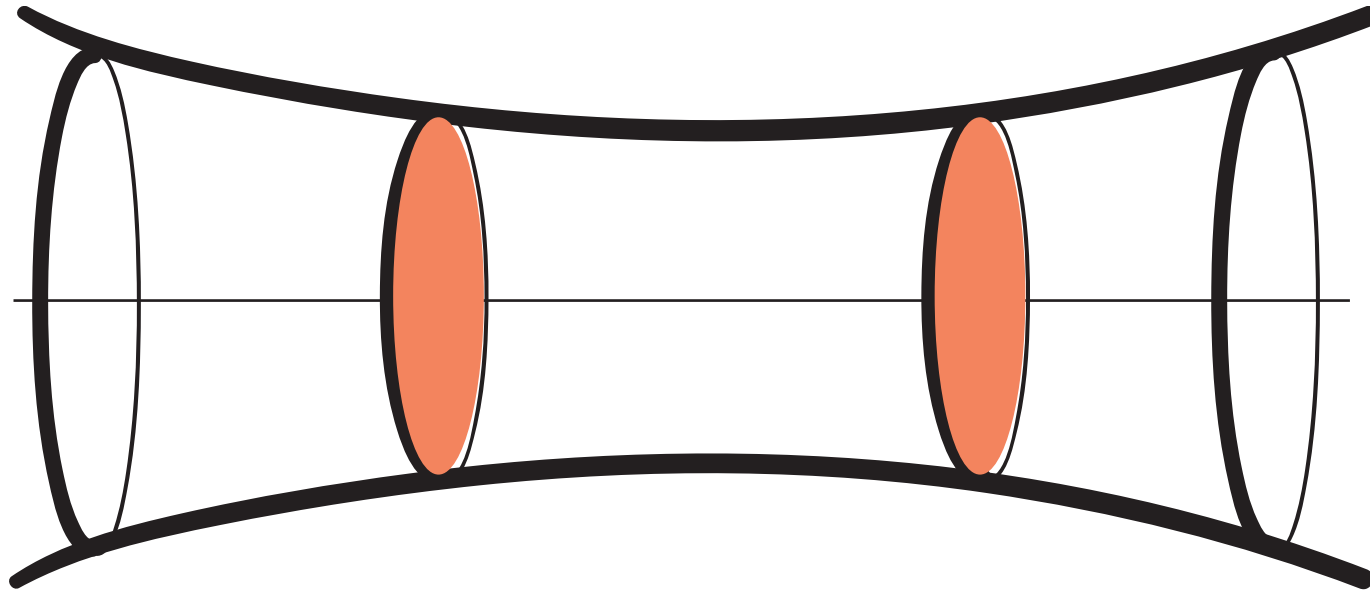
# Integral resolution



$$Q = \int_0^R 2\pi r u dr$$

$$\tau = \frac{\partial u}{\partial r}$$

# Integral resolution



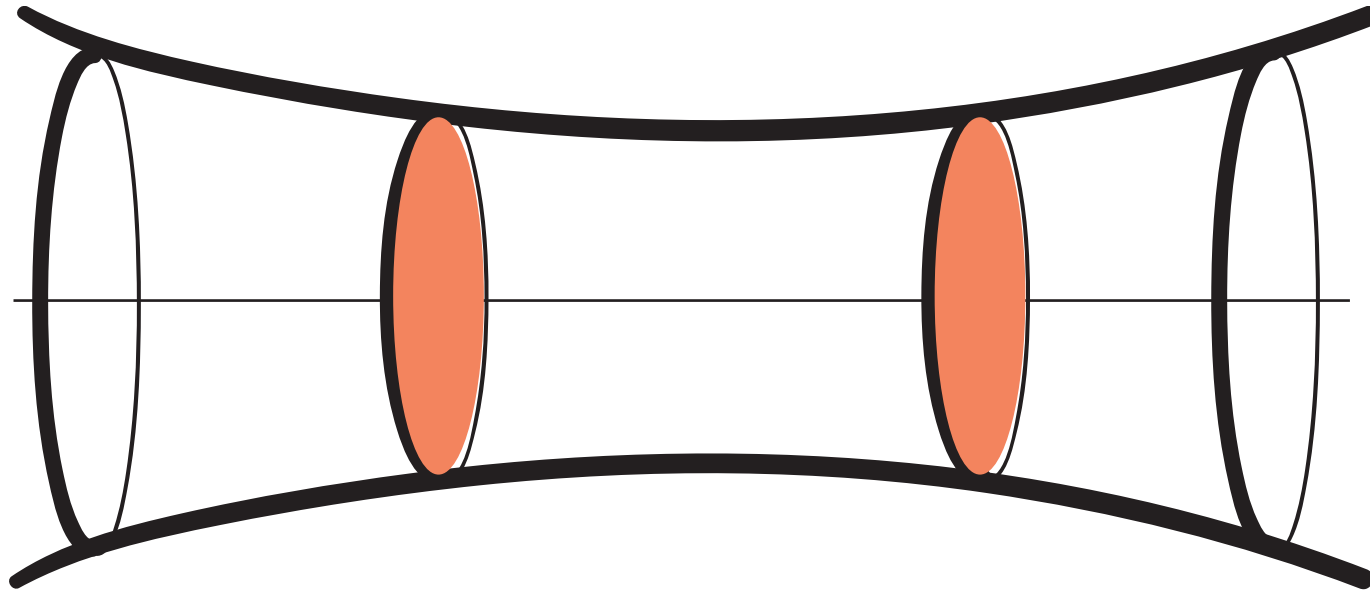
$$Q = \int_0^R 2\pi r u dr$$

$$Q_2 = \int_0^R 2\pi r u^2 dr$$

$$\tau = \frac{\partial u}{\partial r}$$

$$\int \left( \frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + v \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r} \right)$$
$$0 = -\frac{\partial p}{\rho \partial r}$$

# Integral resolution



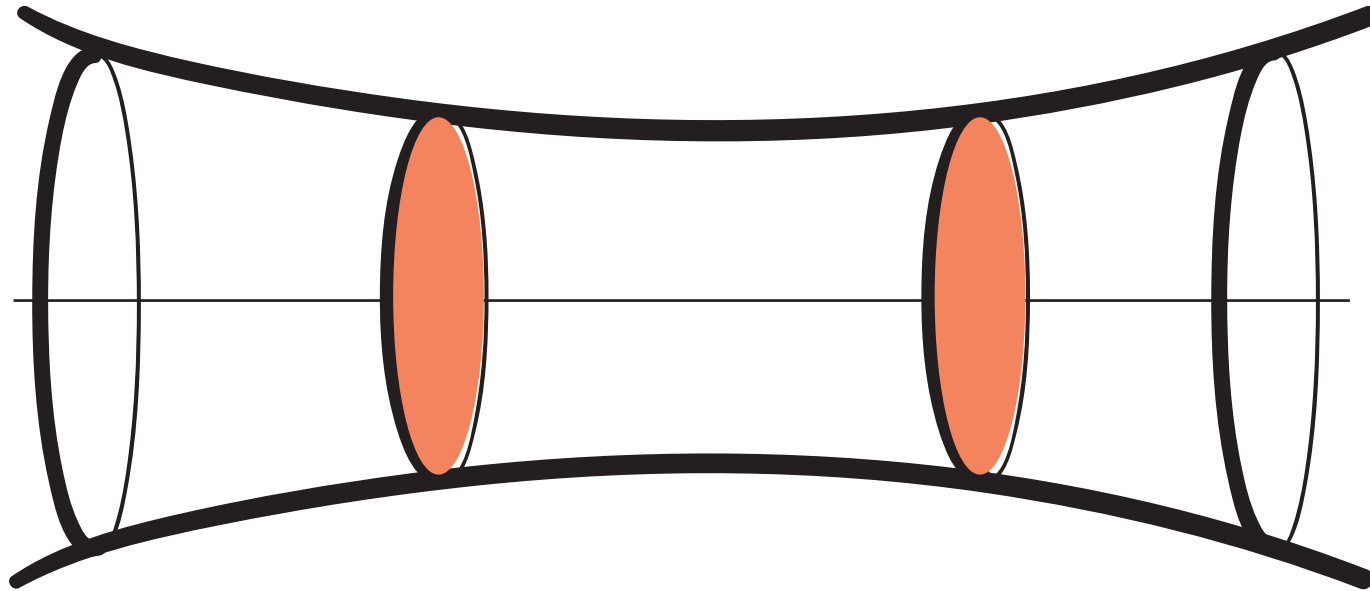
$$Q = \int_0^R 2\pi r u dr$$

$$Q_2 = \int_0^R 2\pi r u^2 dr$$

$$\tau = \frac{\partial u}{\partial r}$$

$$\frac{\partial Q}{\partial t} + \frac{\partial Q_2}{\partial t} = - (2\pi R^2) \frac{\partial p}{\partial x} - \tau$$

# Integral resolution 1D equations



$$Q = \int_0^R 2\pi r u dr$$

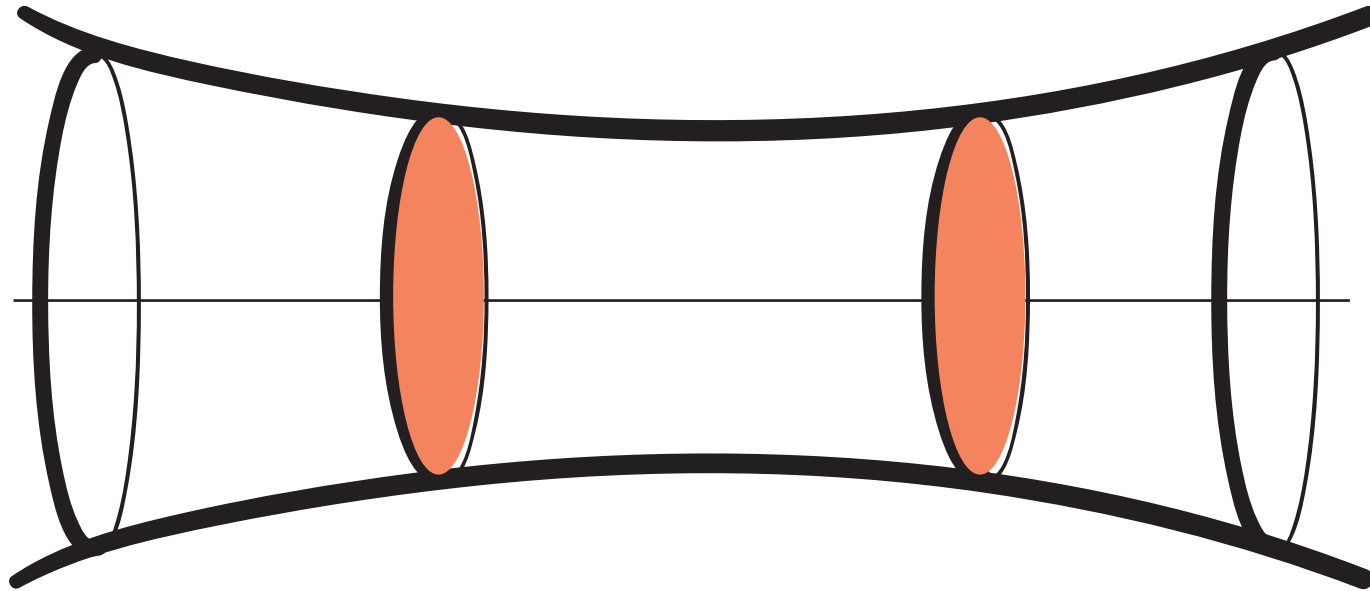
$$Q_2 = \int_0^R 2\pi r u^2 dr$$

$$\tau = \frac{\partial u}{\partial r}$$

$$\frac{\partial(2\pi R^2)}{\partial t} + \frac{\partial Q}{\partial x} = 0$$

$$\frac{\partial Q}{\partial t} + \frac{\partial Q_2}{\partial t} = -(2\pi R^2) \frac{\partial p}{\partial x} - \tau$$

# Integral resolution 1D equations

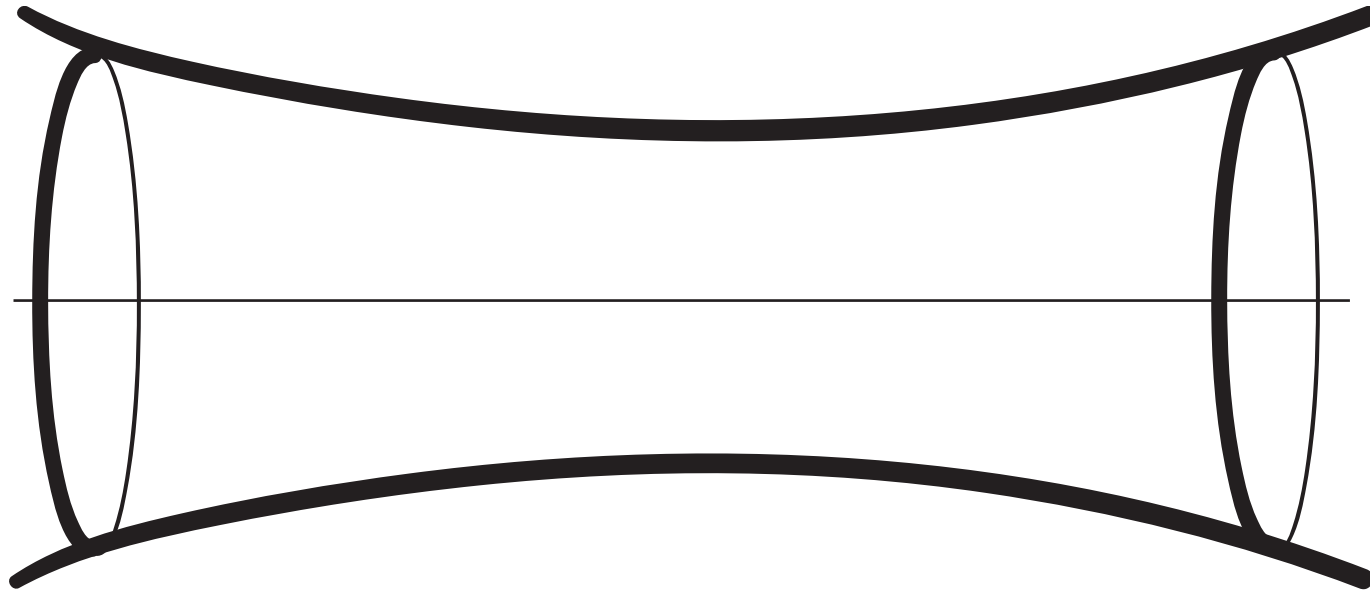


$$Q = \int_0^R 2\pi r u dr \quad Q_2 = \int_0^R 2\pi r u^2 dr \quad \tau = \frac{\partial u}{\partial r}$$

$$\frac{\partial(2\pi R^2)}{\partial t} + \frac{\partial Q}{\partial x} = 0 \quad \frac{\partial Q}{\partial t} + \frac{\partial Q_2}{\partial t} = -(2\pi R^2) \frac{\partial p}{\partial x} - \tau$$

relation between pressure and Radius  $p = k(R - R_0)$

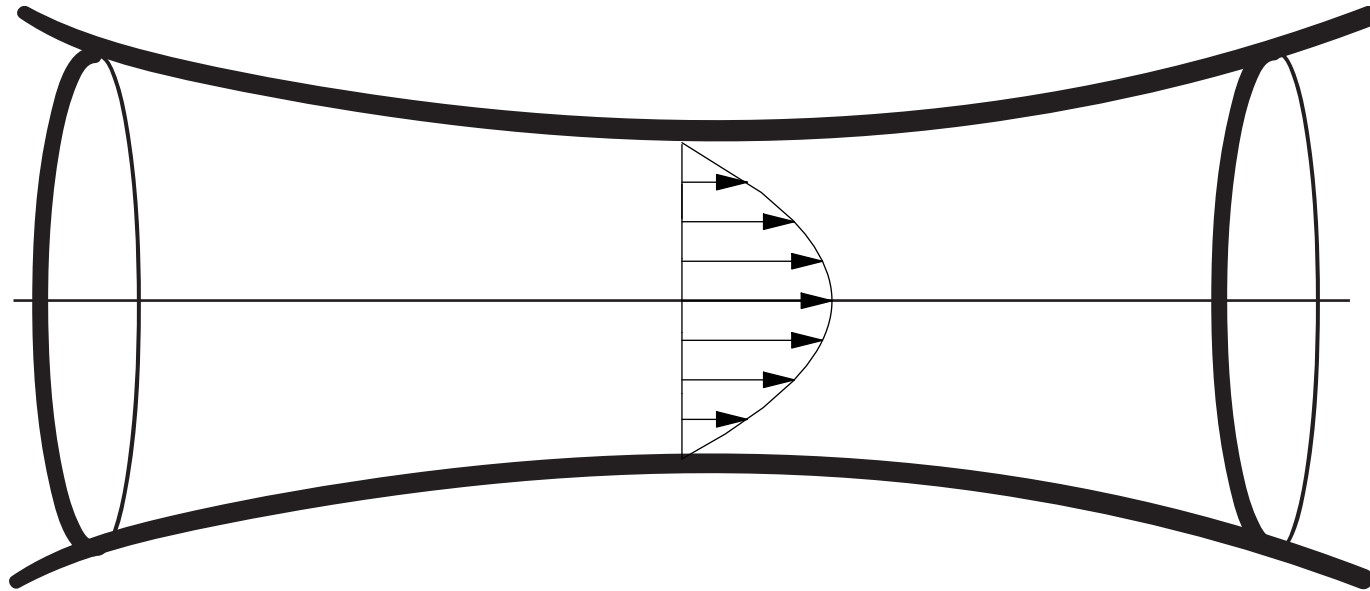
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gives  $Q_2$  as function of  $Q$  and  $\tau$  as function  $Q$

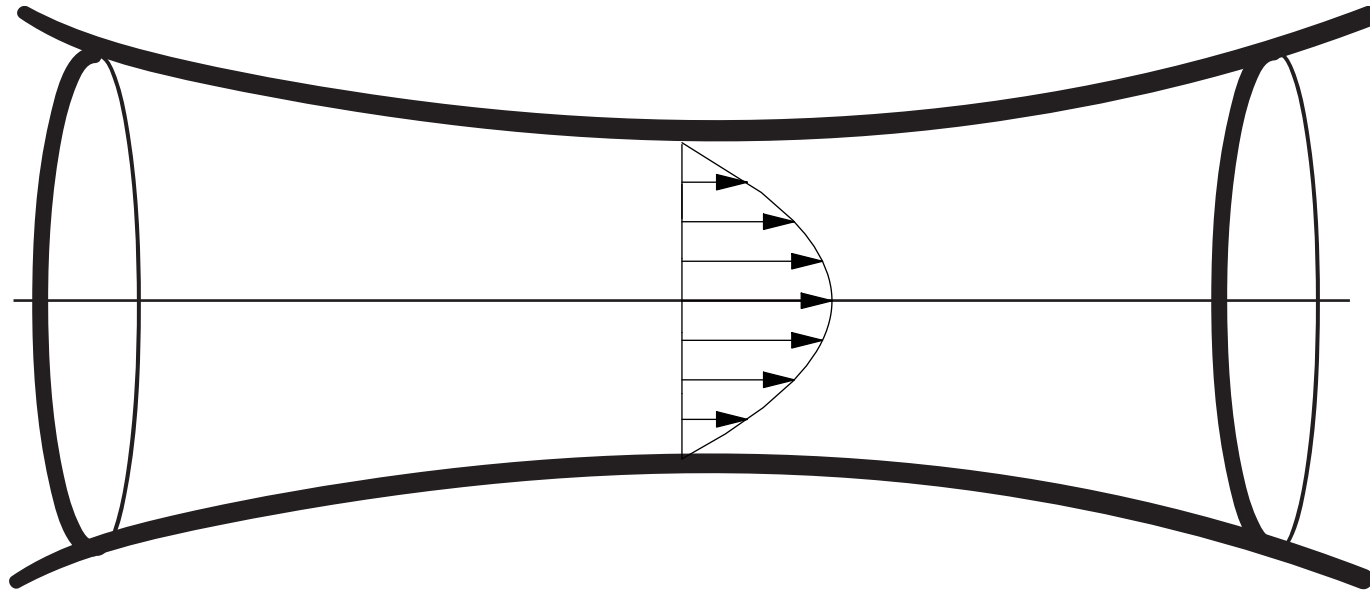
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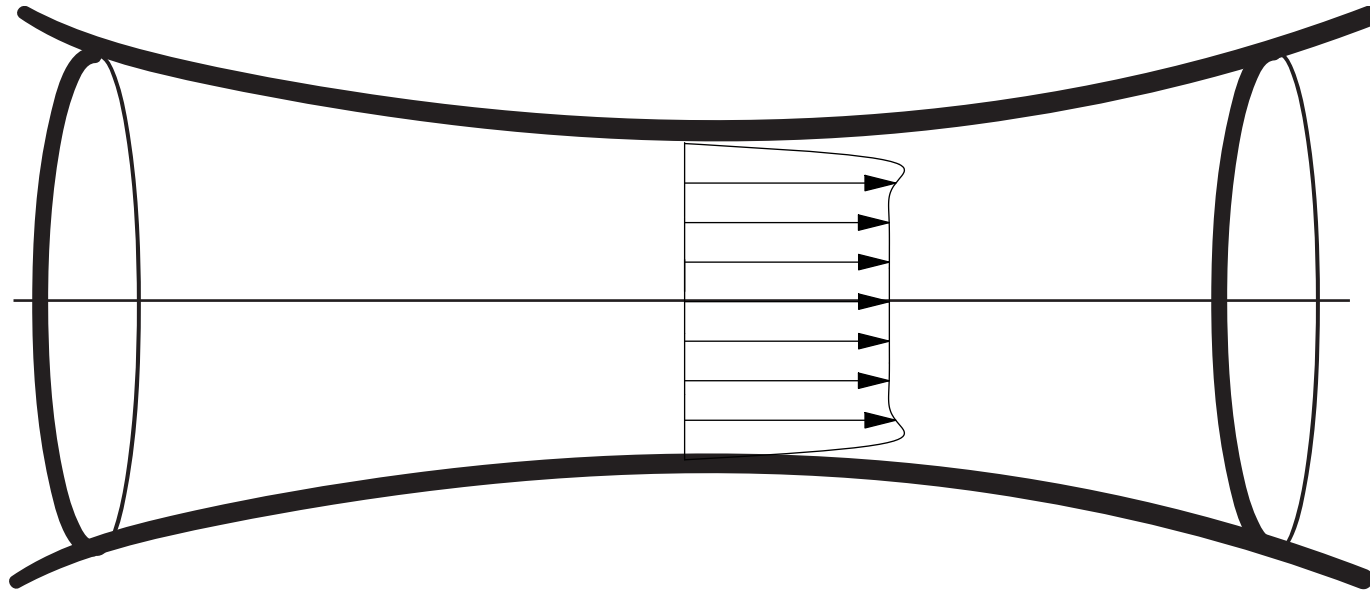
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$$Q_2 = \int_0^R 2\pi r u^2 dr \quad \tau = \frac{\partial u}{\partial r}$$

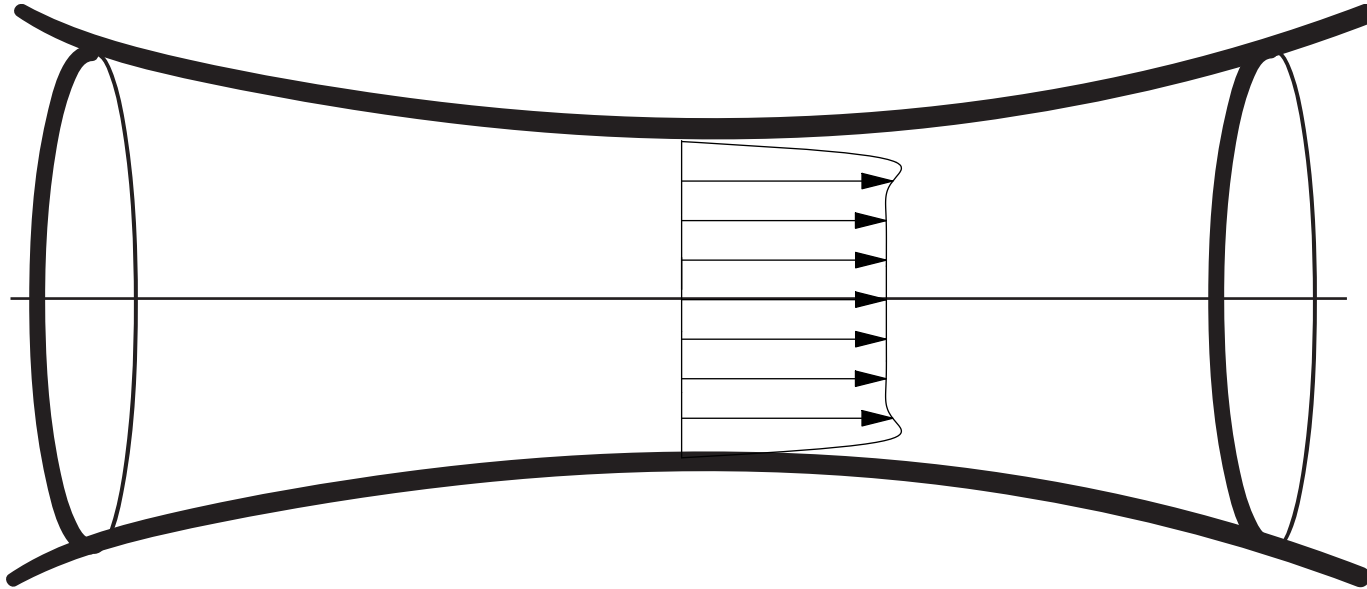
$$Q_2 = \left(\frac{4}{3}\right) \frac{Q^2}{\pi R^2} \quad \tau = (8\pi) \frac{Q}{\pi R^2}$$

# Integral resolution 1D equations



$$Q_2 = \int_0^R 2\pi r u^2 dr \quad \tau = \frac{\partial u}{\partial r}$$

# Integral resolution 1D equations



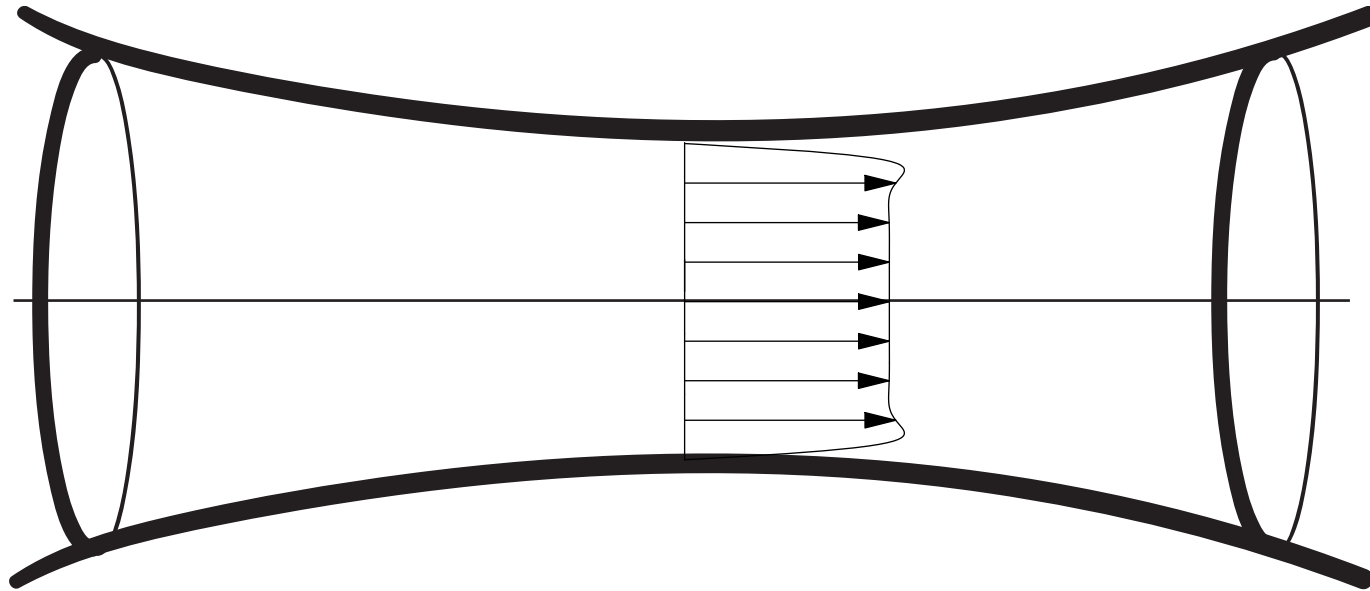
$$Q_2 = \int_0^R 2\pi r u^2 dr$$

$$\tau = \frac{\partial u}{\partial r}$$

$$Q_2 = \frac{Q^2}{\pi R^2}$$

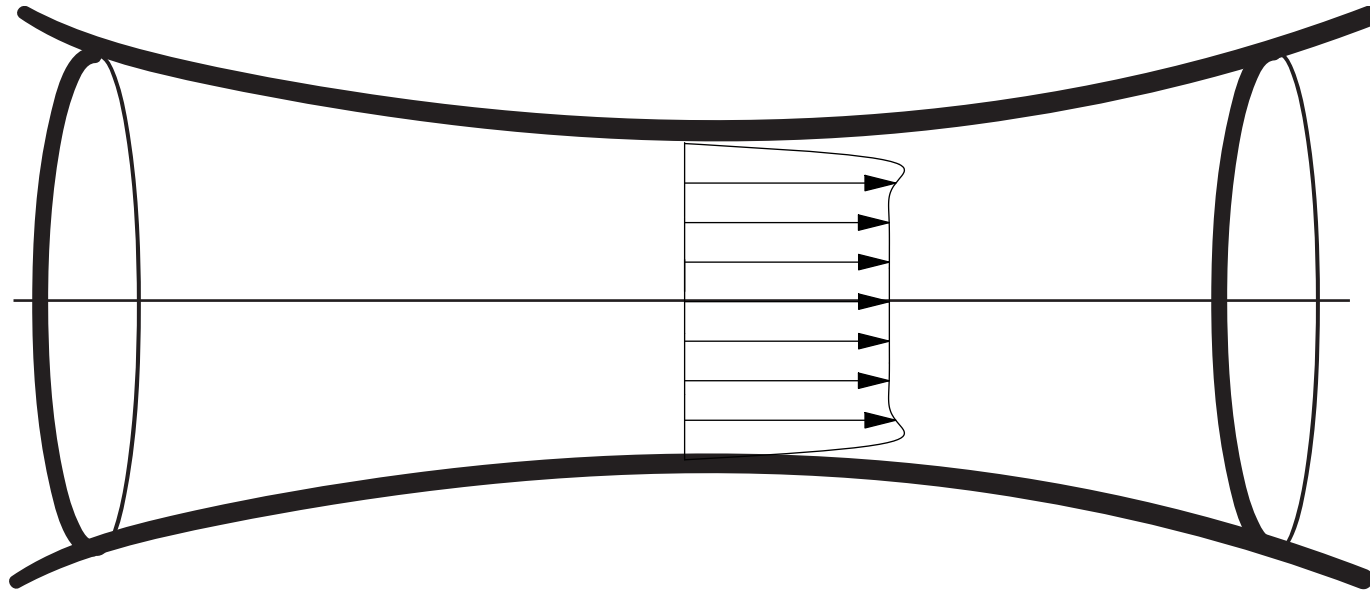
$$\tau = F(Q)$$

# Integral resolution 1D equations



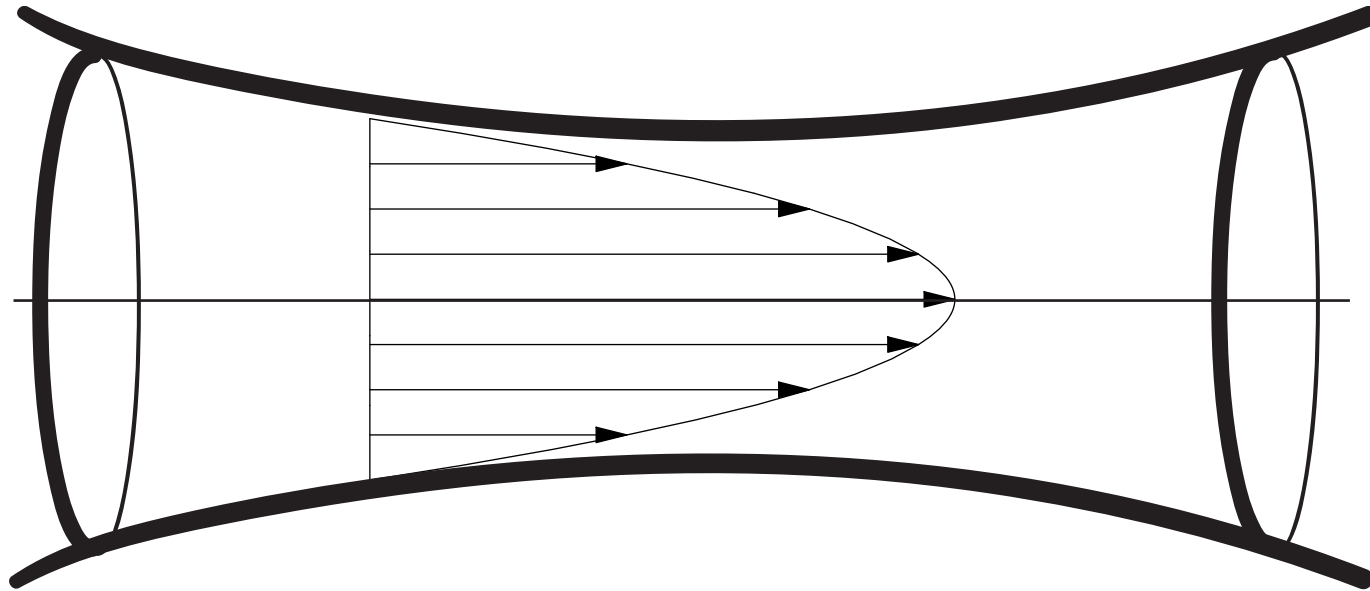
need of profile

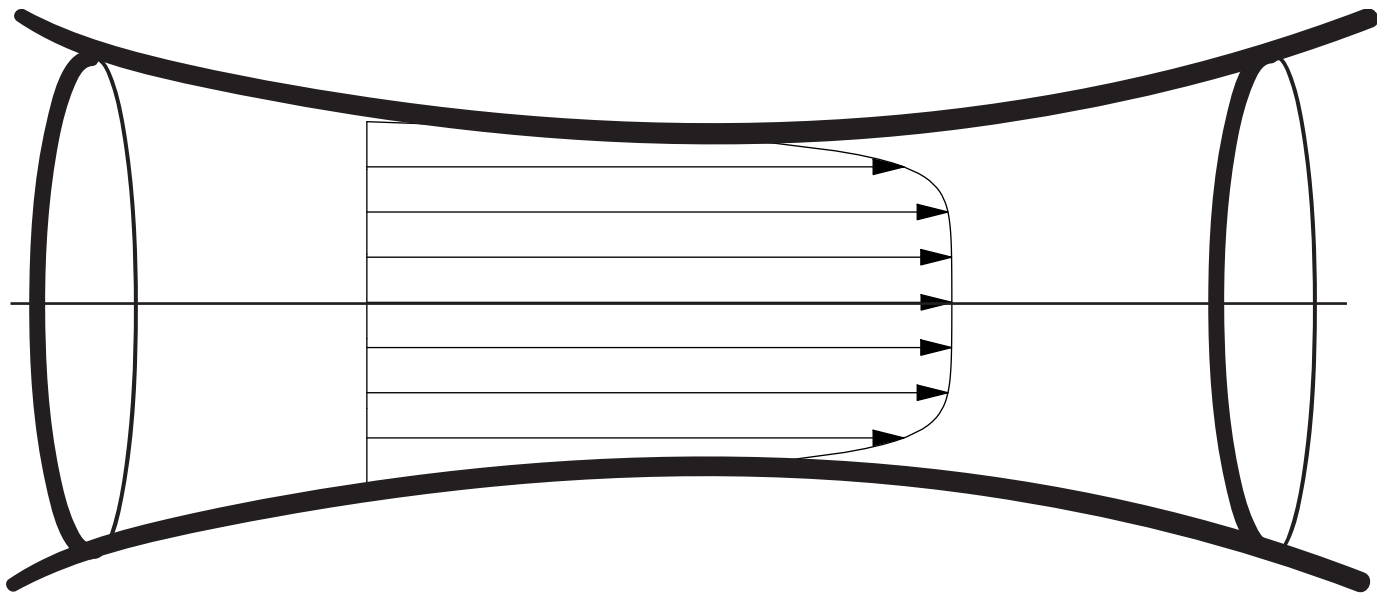
# Integral resolution ID equations

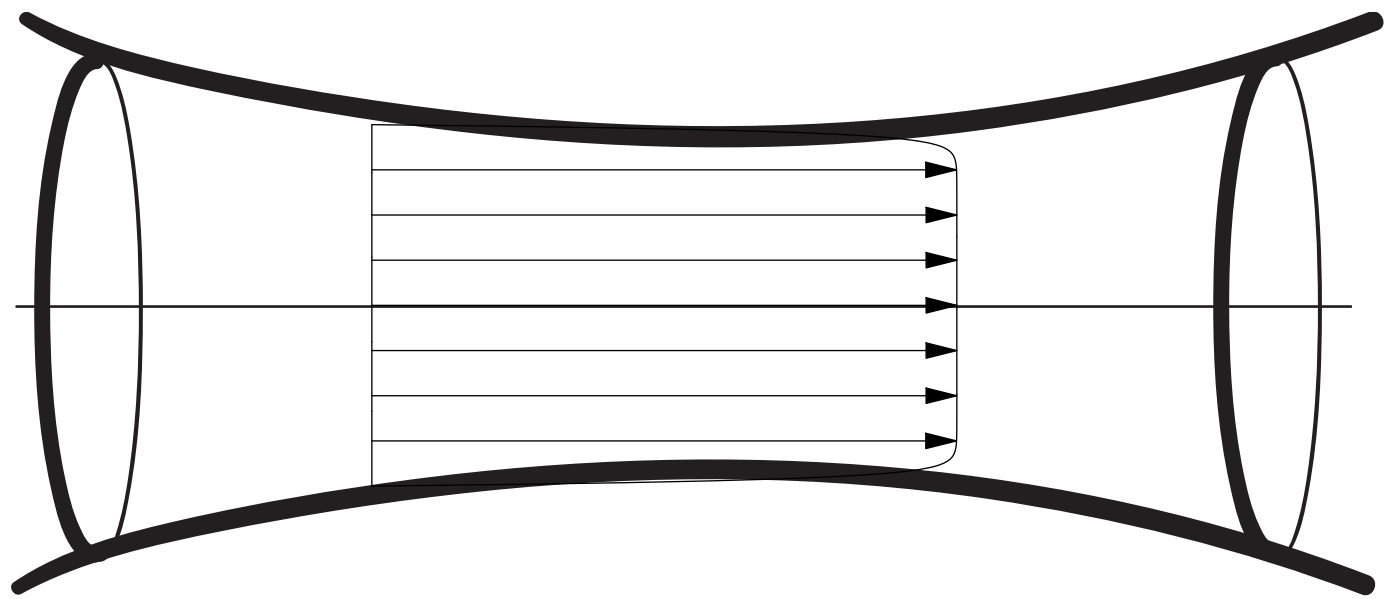


“usual” ID equations are a simplification of RNSP

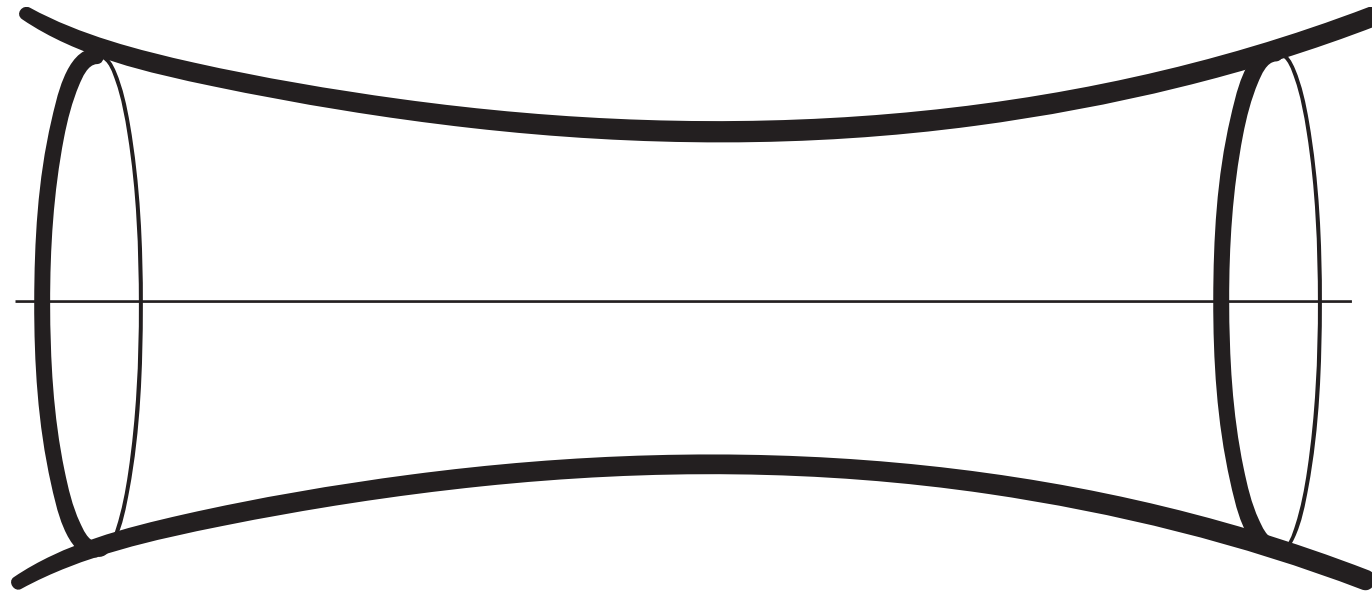
# Choice of profiles



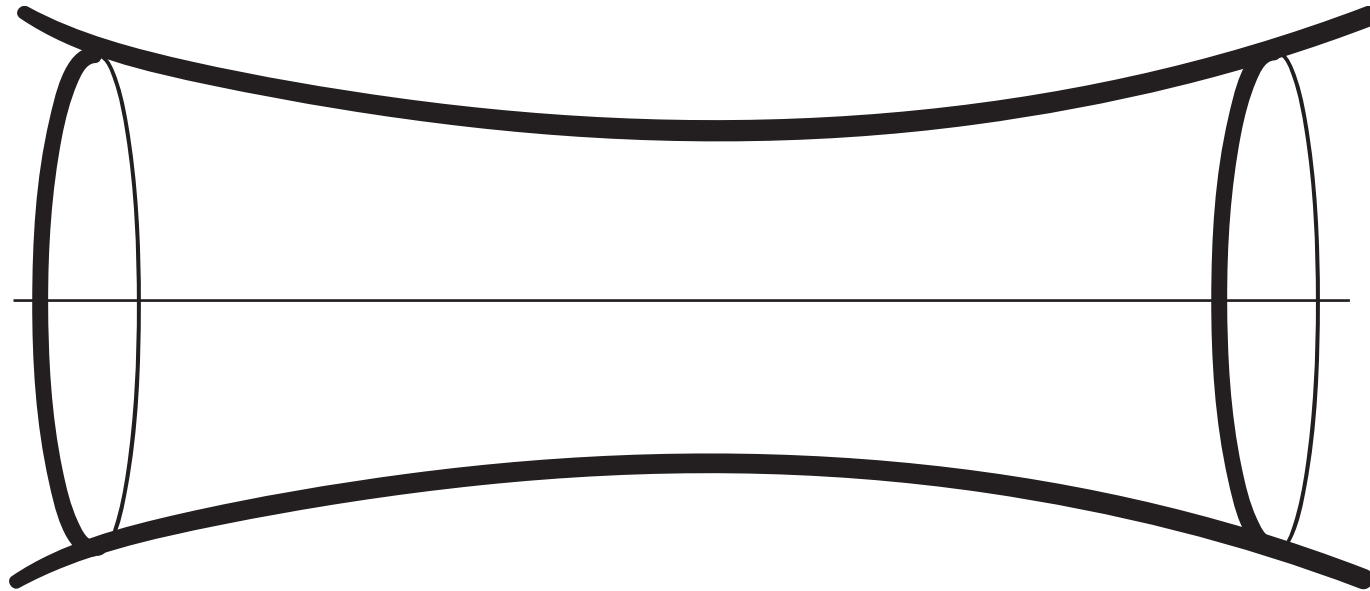








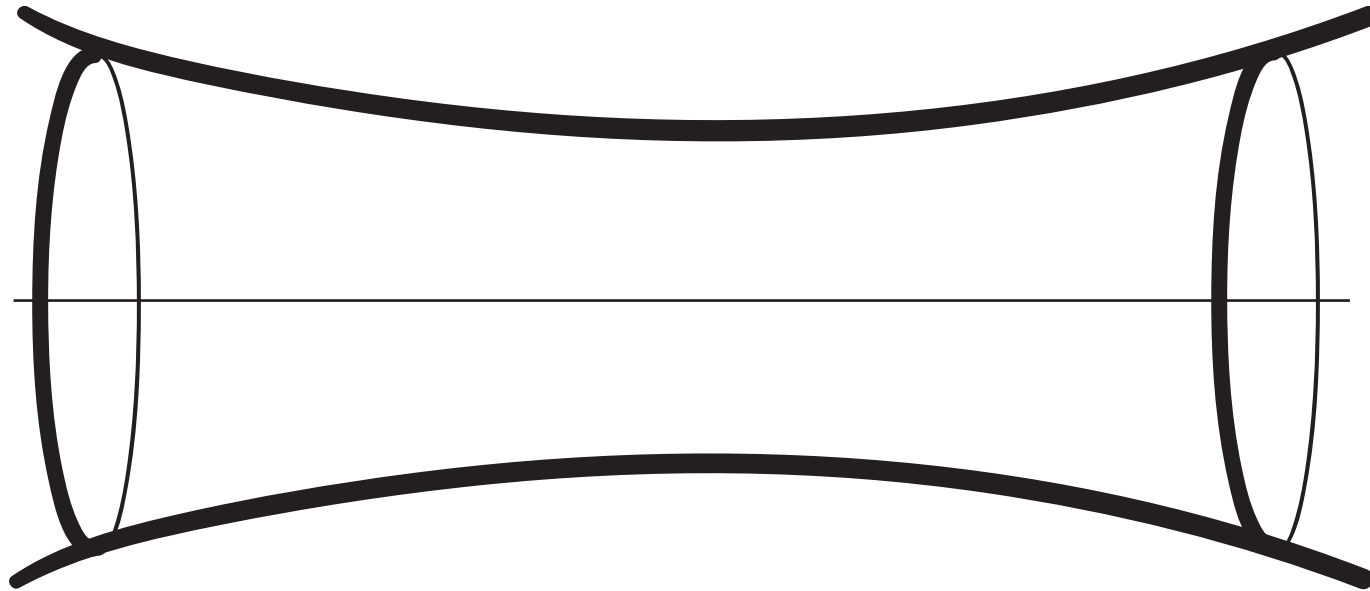
Choice of the family of simple profiles



Choice of the family of simple profiles

In an unsteady flow it is natural to use Womersley

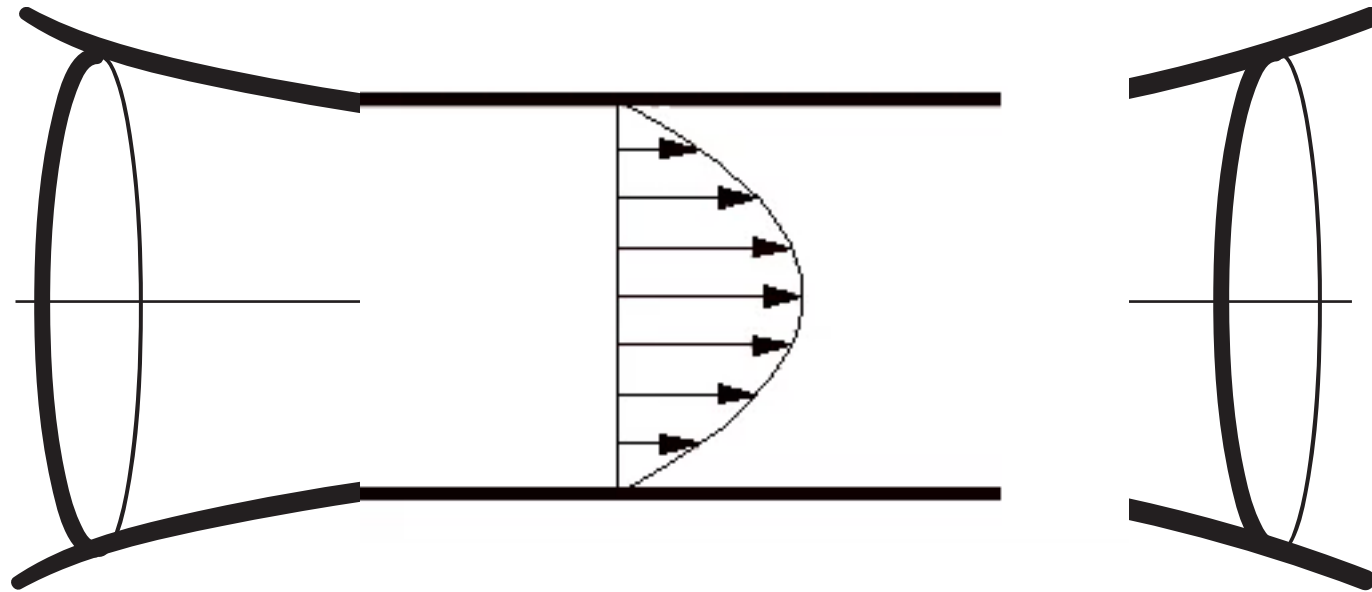
$$\frac{\partial u}{\partial t} + u \frac{\partial}{\partial x} u + v \frac{\partial}{\partial r} u = -\frac{\partial p}{\rho \partial x} + v \frac{\partial}{r \partial r} r \frac{\partial u}{\partial r}$$
$$0 = -\frac{\partial p}{\rho \partial r}$$



Choice of the family of simple profiles

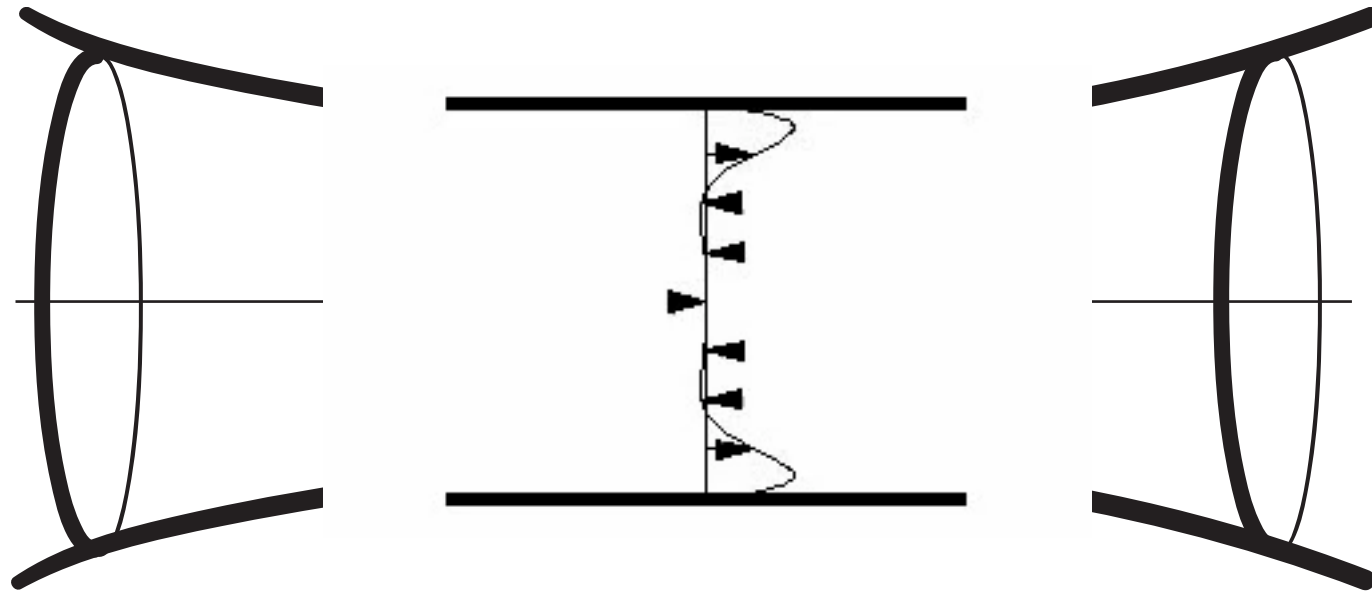
In an unsteady flow it is natural to use Womersley

Womersley profiles are solution of RNSP



Choice of the family of simple profiles

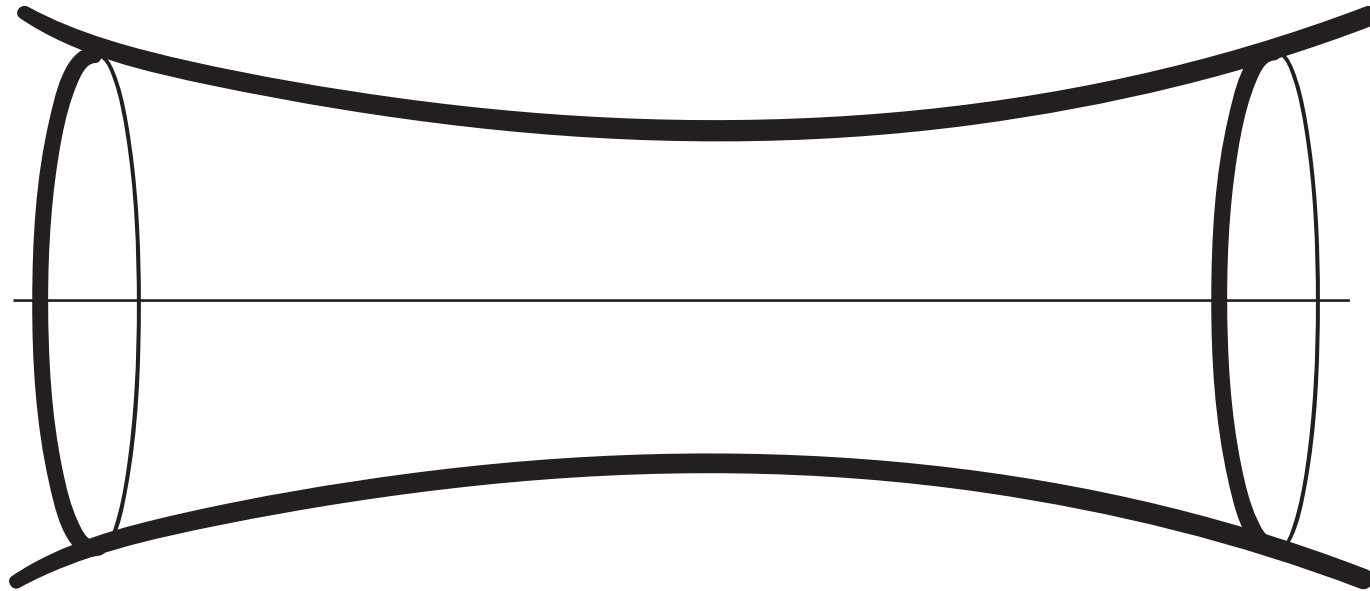
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Choice of the family of simple profiles

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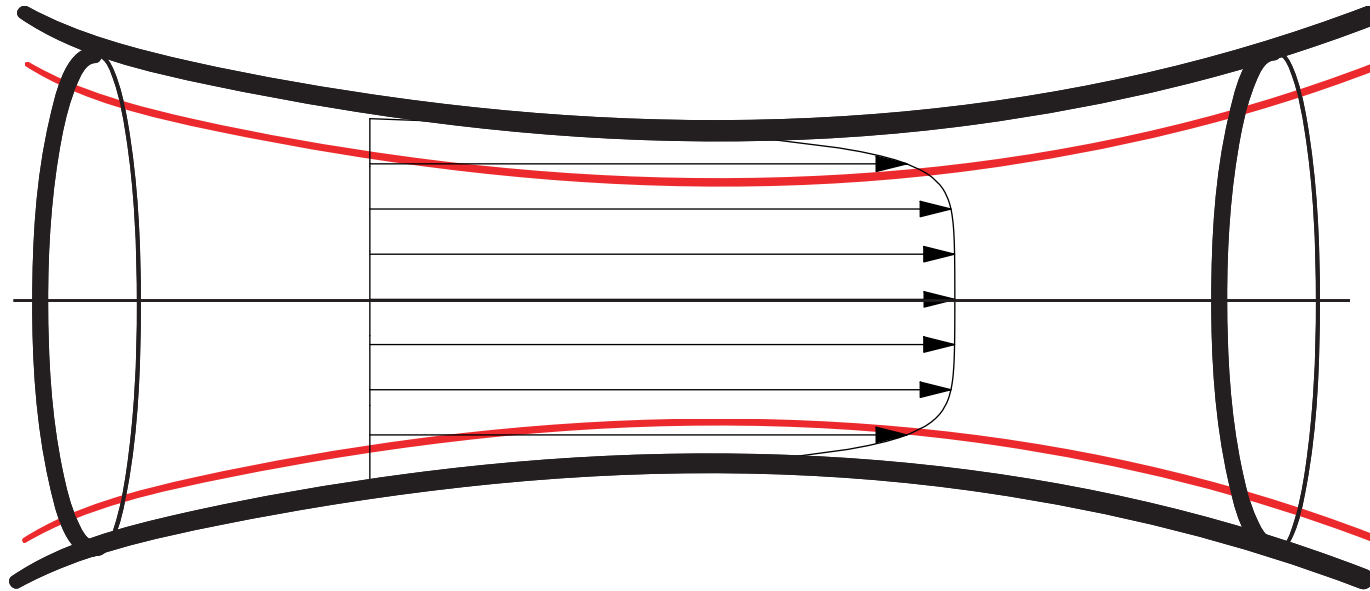
# Integral resolution



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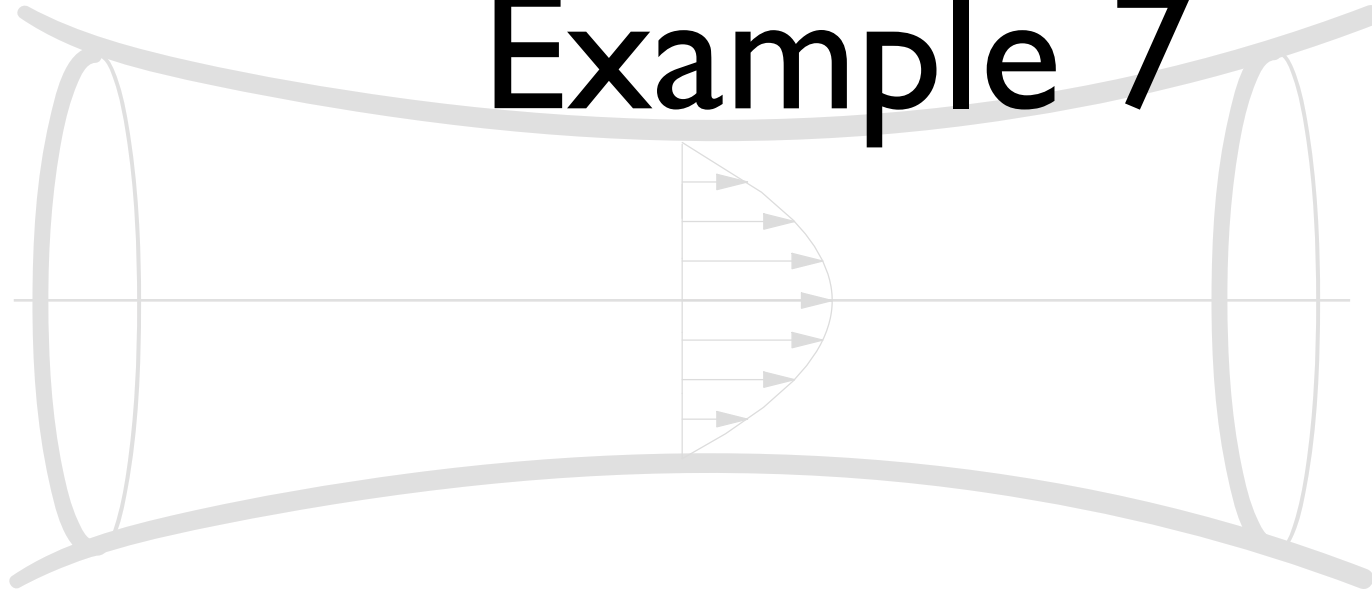
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# Integral resolution



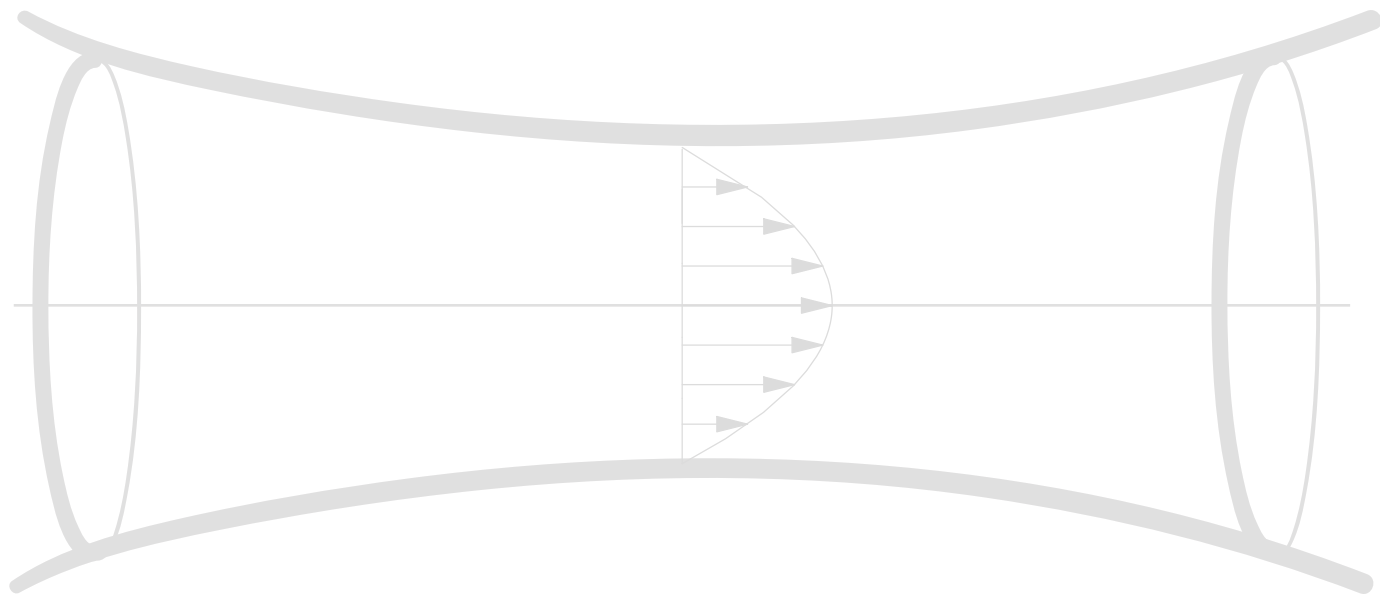
Numerical resolution:  
finite differences

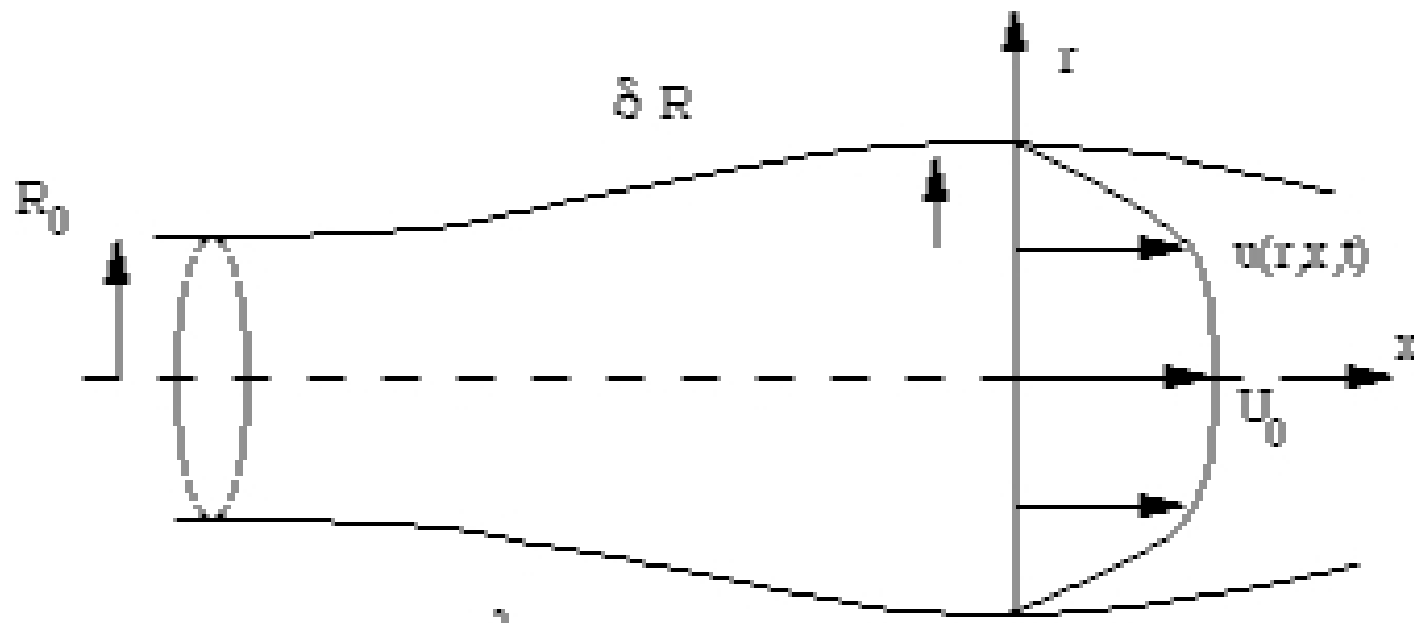
# Example 7



**flow in arteries**







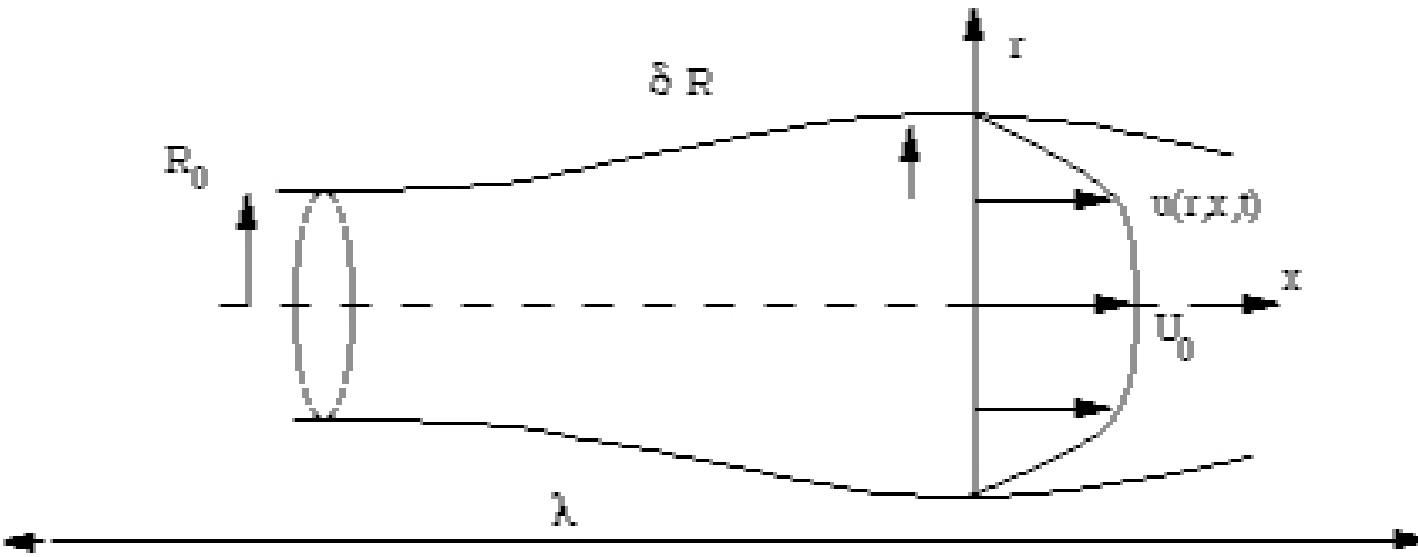
$$\frac{\partial}{\partial x}u + \frac{\partial}{r\partial r}rv = 0,$$

$$\frac{\partial u}{\partial t} + \varepsilon_2(u\frac{\partial}{\partial x}u + v\frac{\partial}{\partial r}u) = -\frac{\partial p}{\partial x} + \frac{2\pi}{\alpha^2 r}\frac{\partial}{\partial r}(r\frac{\partial}{\partial r}u), 0 = -\frac{\partial p}{\partial r}.$$

$$\varepsilon_2 = \frac{\delta R}{R_0}, \quad \alpha = R_0\sqrt{\frac{2\pi/T}{\nu}}$$

introducing wall elasticity:  $p(x, t) = k(R(x, t) - R_0)$

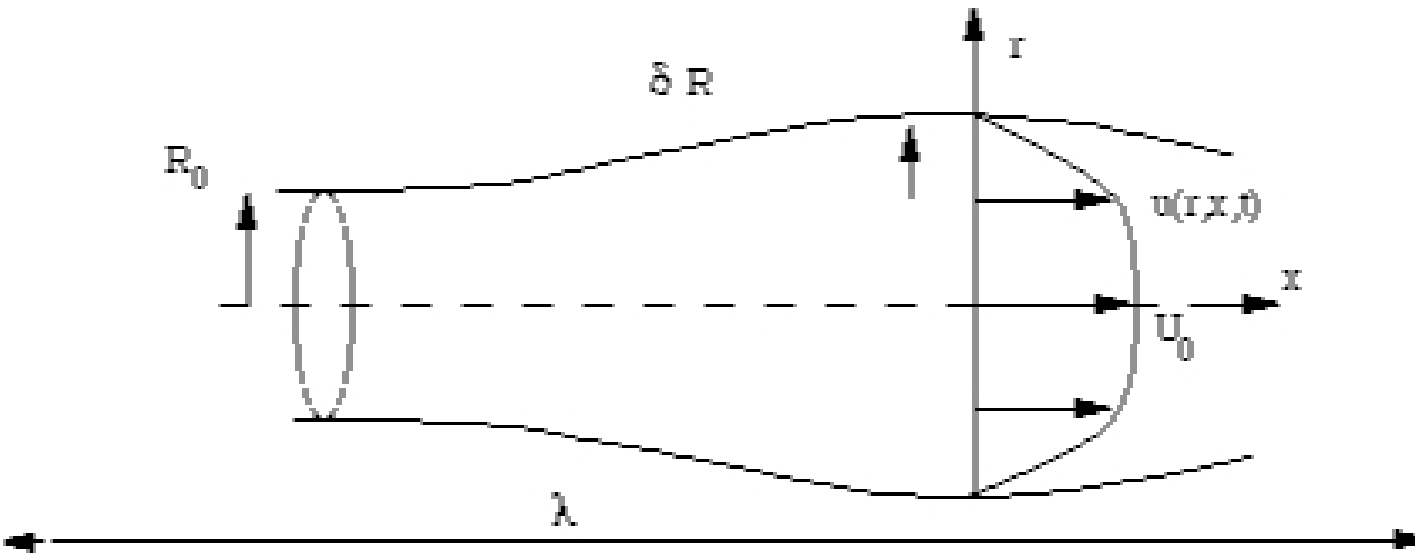
+ The boundary conditions: here hyperbolic ( $R(x_{in}, t)$  and  $R(x_{out}, t)$ ) given



weak coupling

$$\frac{u^{n+1} - u^n}{\Delta t} + NL(u^n) = -\frac{\partial p^n}{\rho \partial x} + \nu \frac{\partial}{\partial r} r \frac{\partial u^{n+1}}{\partial r}$$

$$\nu^{n+1}(R^n) = - \int_0^{R^n} \frac{r}{R^n} \frac{\partial u^{n+1}}{\partial x} dr$$

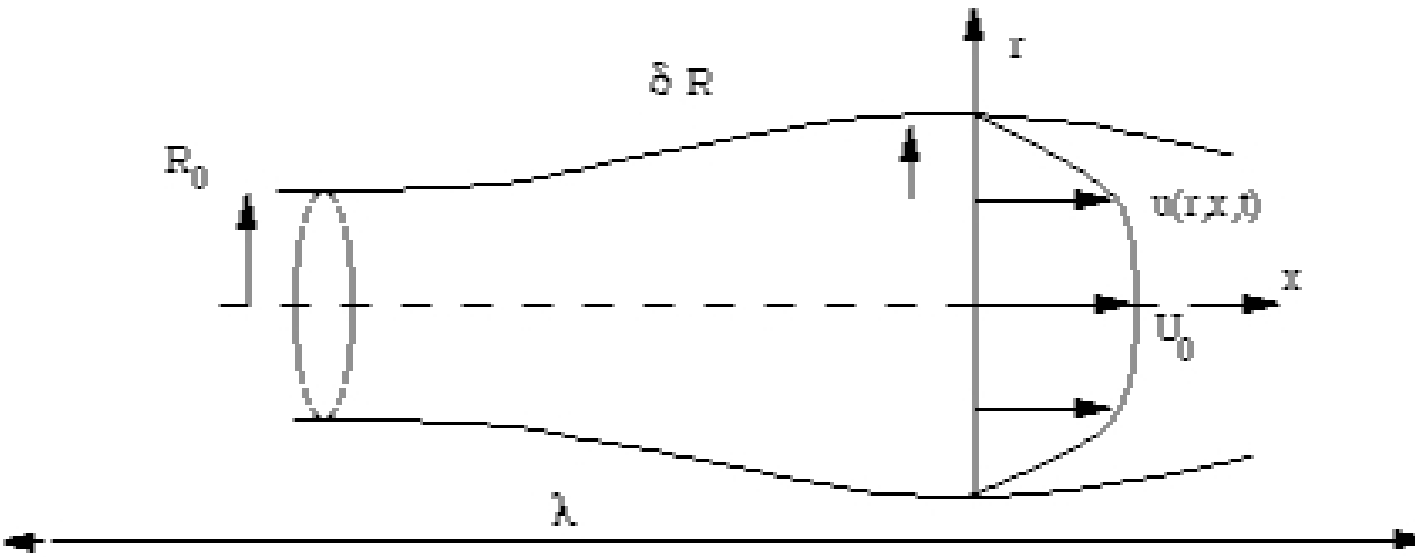


weak coupling

$$\frac{u^{n+1} - u^n}{\Delta t} + NL(u^n) = -\frac{\partial p^n}{\rho \partial x} + v \frac{\partial}{r \partial r} r \frac{\partial u^{n+1}}{\partial r}$$

$$v^{n+1}(R^n) = - \int_0^{R^n} \frac{r}{R^n} \frac{\partial u^{n+1}}{\partial r} dr$$

$$R^{n+1} = R^n + v^{n+1}(R^n) \Delta t$$

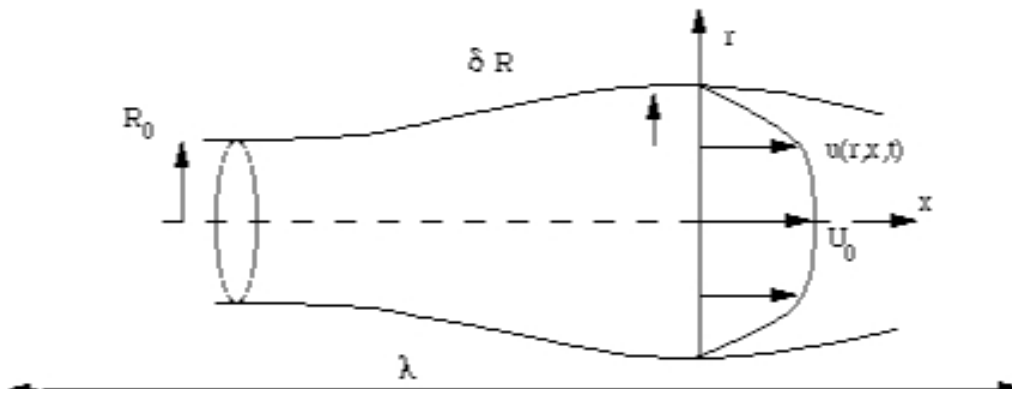


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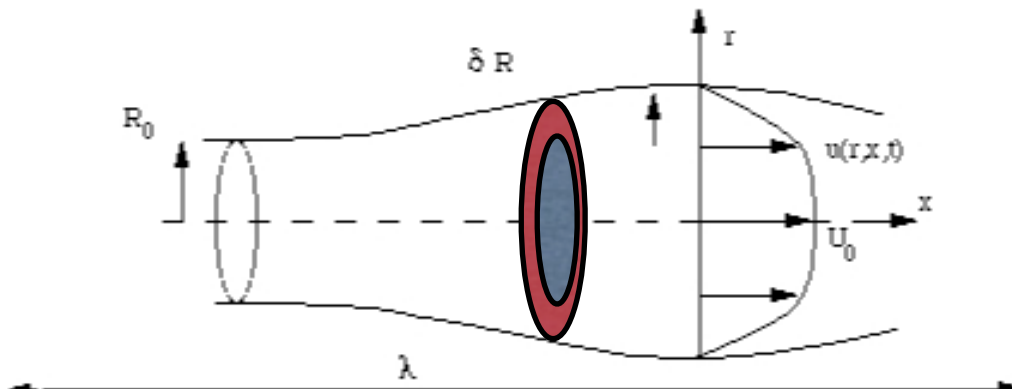
$$R^{n+1} = R^n + v^{n+1}(R^n) \Delta t \quad p^{n+1} = k(R^{n+1} - R_0)$$



## Flow in an elastic artery: integral relations

- new integral equations: adapting Von Kármán integral methods

The key is to integrate the equations with respect to the variable  $\eta = r/R$  from the centre of the pipe to the wall ( $0 \leq \eta \leq 1$ ).



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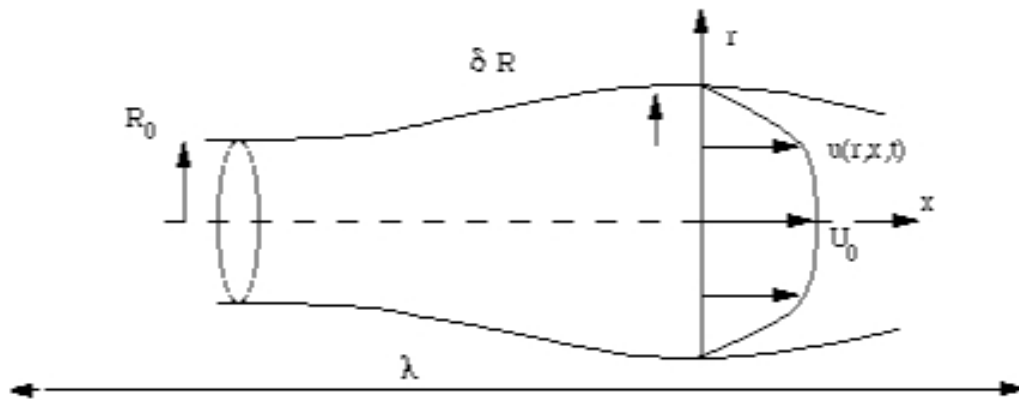
The key is to integrate the equations with respect to the variable  $\eta = r/R$  from the centre of the pipe to the wall ( $0 \leq \eta \leq 1$ ).

-  $U_0$ , the velocity along the axis of symmetry,

-  $q$  a kind of loss of flux ( $\delta_1$ ),

-  $\Gamma$  a kind of loss of momentum flux ( $\delta_2$ ):

$$U_0(x, t) = u(x, \eta = 0, t), \quad q = R^2(U_0 - 2 \int_0^1 u\eta d\eta) \quad \& \quad \Gamma = R^2(U_0^2 - 2 \int_0^1 u^2 \eta d\eta).$$



## Flow in an elastic artery: integral relations

$$\frac{\partial R^2}{\partial t} + \varepsilon_2 \frac{\partial}{\partial x} (R^2 U_0 - q) = 0, \quad R = 1 + \varepsilon_2 h.$$

Integrating RNSP, with the help of the boundary conditions, we obtain the equation for  $q(x, t)$ :

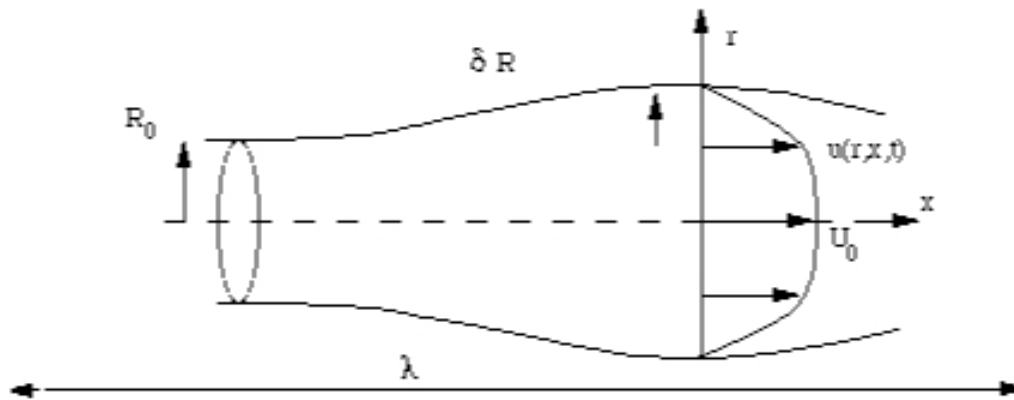
$$\frac{\partial q}{\partial t} + \varepsilon_2 \left( \frac{\partial}{\partial x} \Gamma - U_0 \frac{\partial}{\partial x} q \right) = -2 \frac{2\pi}{\alpha^2} \tau, \quad \tau = \left( \frac{\partial u}{\partial \eta} \right) \Big|_{\eta=1} - \left( \frac{\partial^2 u}{\partial \eta^2} \right) \Big|_{\eta=0}.$$

From the same equation evaluated on the axis of symmetry (in  $\eta = 0$ ), we obtain an equation for the velocity along the axis  $U_0(x, t)$ :

$$\frac{\partial U_0}{\partial t} + \varepsilon_2 U_0 \frac{\partial U_0}{\partial x} = -\frac{\partial p}{\partial x} + 2 \frac{2\pi}{\alpha^2} \frac{\tau_0}{R^2}, \quad \tau_0 = \left( \frac{\partial^2 u}{\partial \eta^2} \right) \Big|_{\eta=0}.$$

Boundary conditions ( $h(x_{in}, t)$  and  $h(x_{out}, t)$ ) given



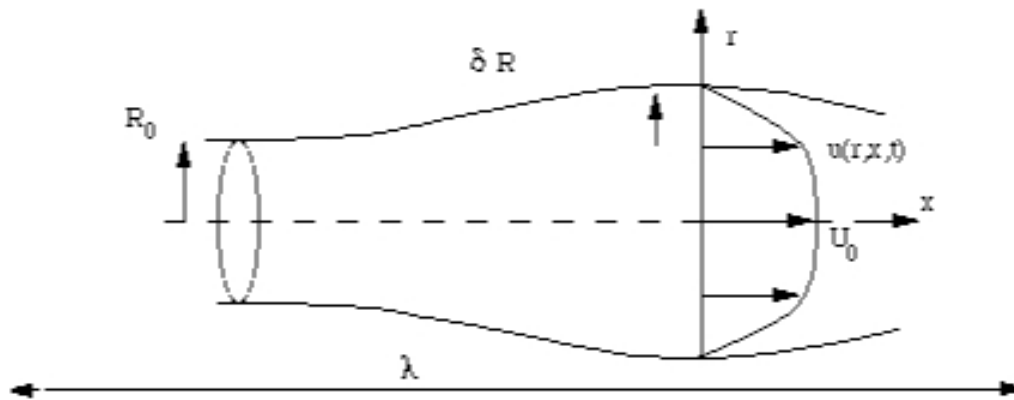


## Closure

The two previous relations introduced the values of the friction in  $\eta = 0$ , the axis of symmetry:  $((\frac{\partial^2 u}{\partial \eta^2})|_{\eta=0})$  and the skin friction in  $\eta = 1$ , at the wall:  $((\frac{\partial u}{\partial \eta})|_{\eta=1})$ .

- Information has been lost here, so we need a closure relation between  $(\Gamma, \tau, \tau_0)$  and  $(q, R, U_0)$ .

- we have to imagine a velocity profile and deduce from it relations linking  $\Gamma, \tau$  and  $\tau_0$  and  $q, U_0$  et  $R$ .

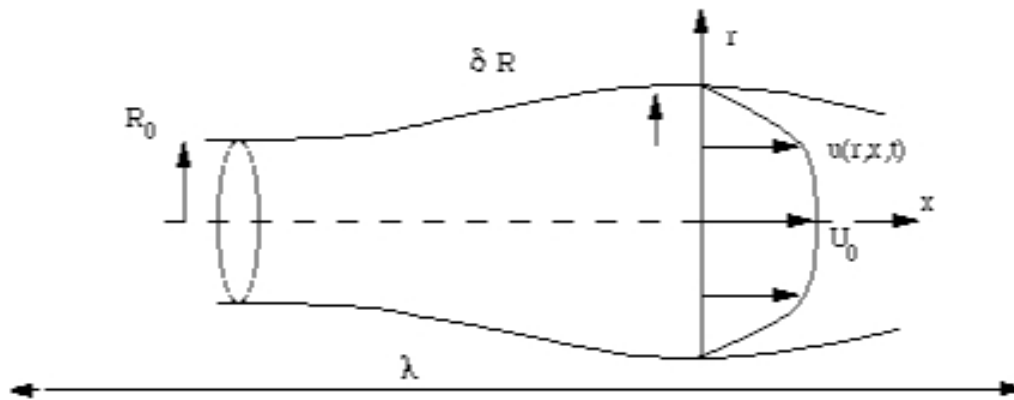


## Closure: Womersley

- the most simple idea is to use the profiles from the analytical linearized solution given by Womersley (1955) for

$$(j_r + ij_i) = \left( \frac{1 - \frac{J_0(i^{3/2}\alpha\eta)}{J_0(i^{3/2}\alpha)}}{1 - \frac{1}{J_0(i^{3/2}\alpha)}} \right).$$

- assume that the velocity distribution in the following has the same dependence on  $\eta$ . It means that we suppose that the fundamental mode imposes the radial structure of the flow.

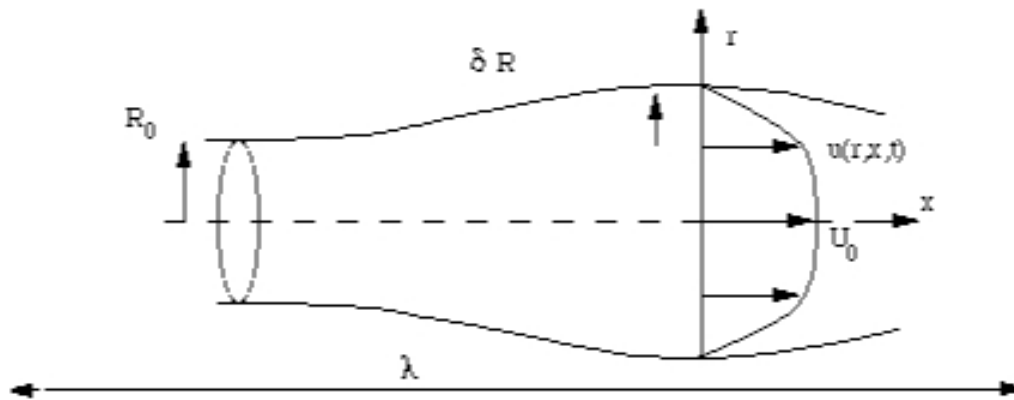


## The coefficients of closure

- by integration/ derivation, we obtain:

$$\Gamma = \gamma_{qq} \frac{q^2}{R^2} + \gamma_{qu} q U_0 + \gamma_{uu} R^2 U_0^2, \quad \tau = \tau_q \frac{q}{R^2} + \tau_u U_0 \quad \tau_0 = \tau_{0q} \frac{q}{R^2} + \tau_{0u} U_0.$$

The coefficients  $((\gamma_{qq}, \gamma_{qu}, \gamma_{uu}), (\tau_q, \tau_u), (\tau_{0q}, \tau_{0u}))$  are only functions of  $\alpha$ .



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The coefficients  $((\gamma_{qq}, \gamma_{qu}, \gamma_{uu}), (\tau_q, \tau_u), (\tau_{0q}, \tau_{0u}))$  are only functions of  $\alpha$ .

$$\begin{aligned} \gamma_{uu} = & 1 - \int j_i^2 / (\int j_i)^2 - (2 \int j_r j_i) / \int j_i - \int j_r^2 + \\ & + (2 \int j_i^2 \int j_r) / (\int j_i)^2 + (2 \int j_i j_r \int j_r) / \int j_i - \\ & - (\int j_i^2 (\int j_r)^2) / (\int j_i), \end{aligned}$$

$$\tau_{0u} = \partial_{\eta}^2 j_{r\eta=0} + \partial_{\eta}^2 j_{i\eta=0} / \int j_i - (\partial_{\eta}^2 j_{i\eta=0} \int j_r) / \int j_i.$$

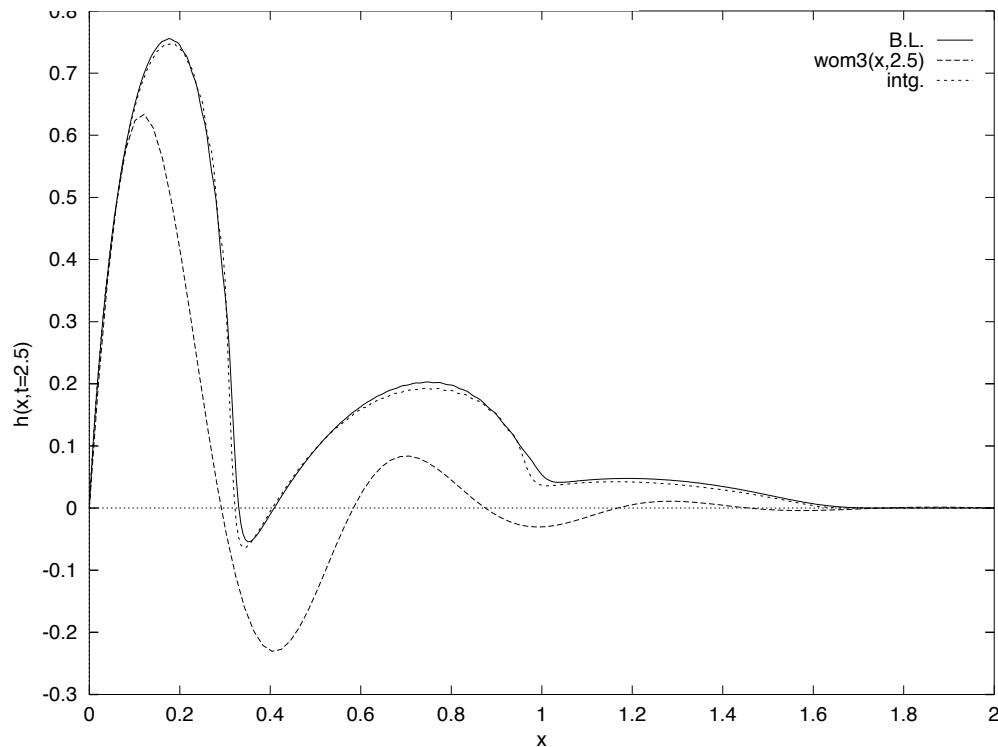
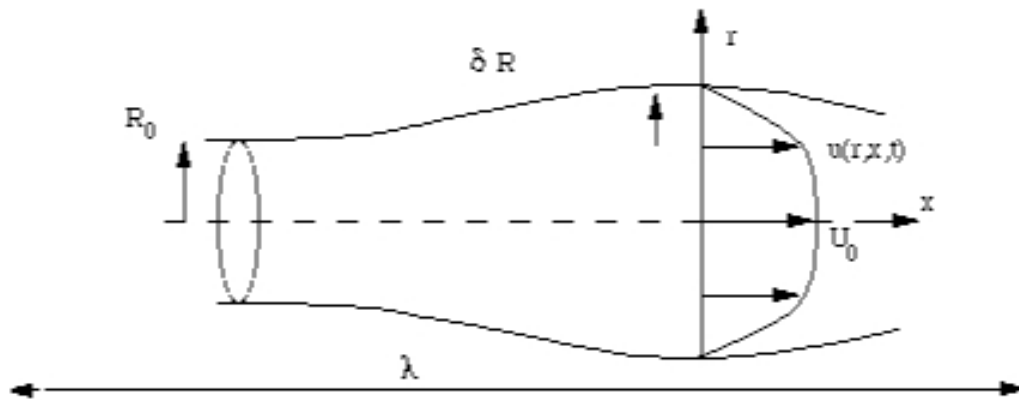
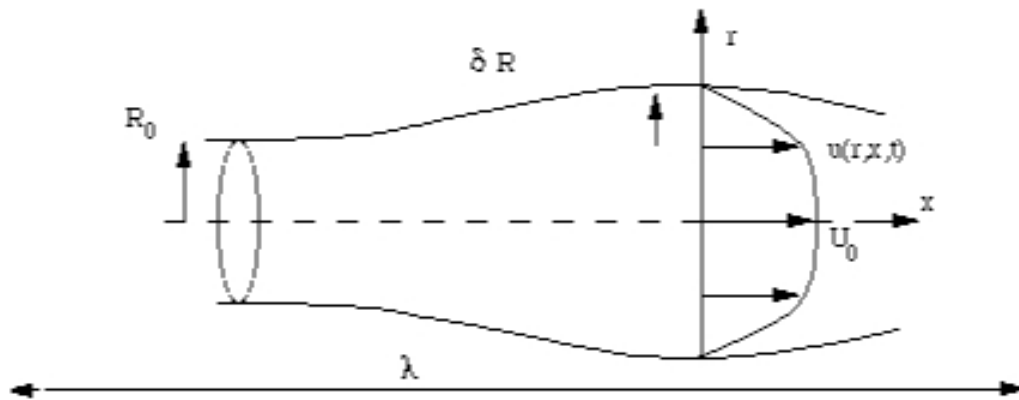


Figure 1: The displacement of the wall ( $h(x, t = 2.5)$ ) as a function of  $x$  is plotted here at time  $t = 2.5$ . The dashed line ( $wom3(x,2.5)$ ) is the Womersley solution (reference), the solid line (B.L.) is the result of the Boundary Layer code and the dots (intg) are the results of the integral method ( $\alpha = 3$ ,  $k_1 = 1$ ,  $k_2 = 0$  and  $\varepsilon_2 = 0.2$ ).

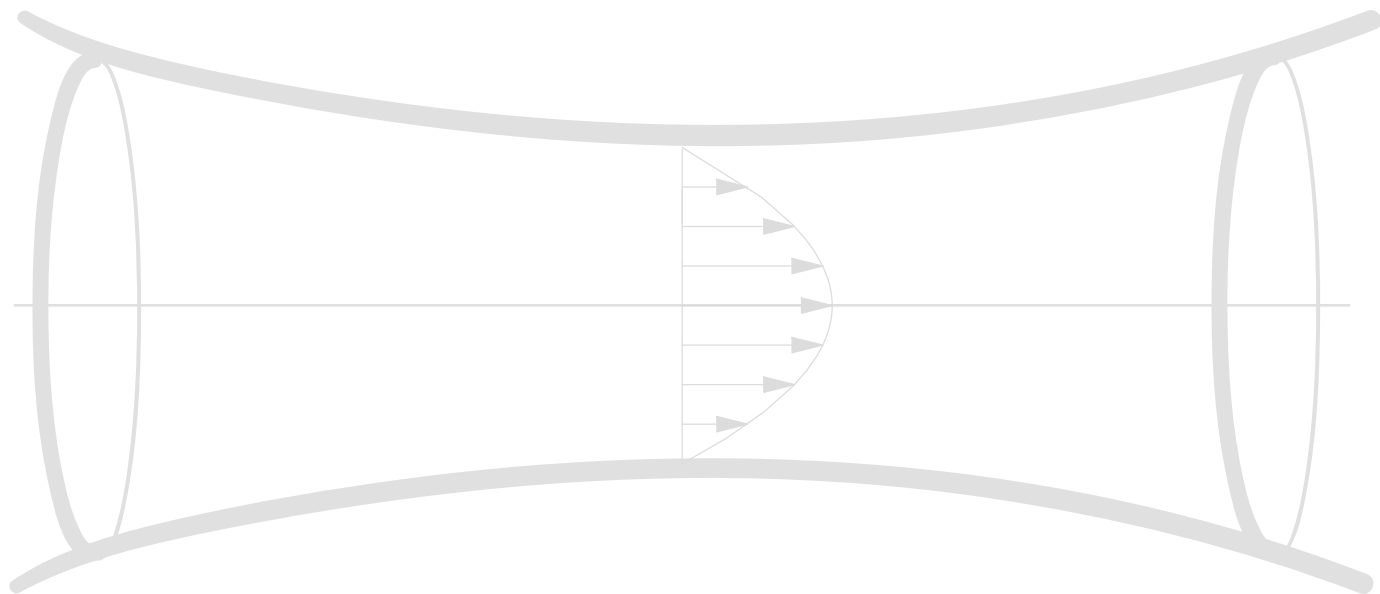


P.-Y. Lagrée (2000):

"An inverse technique to deduce the elasticity of a large artery ",  
 European Physical Journal, Applied Physics 9, pp. 153-163

Lagrée P.-Y and Rossi M. (1996):

"Etude de l'écoulement du sang dans les artères: effets nonlinéaires et dissipatifs",  
 C. R. Acad. Sci. Paris, t322, Série II b, p401- 408, 1996.



# Conclusion

A decorative background featuring a gray frame with curved top and bottom edges. In the center, there is a diagram of a velocity profile, showing a semi-circular shape with horizontal arrows of varying lengths pointing to the right, representing a parabolic flow profile.

- starting from Navier Stokes
- set of simple equations RNSP
- set of more simple equations Integral





# Conclusion

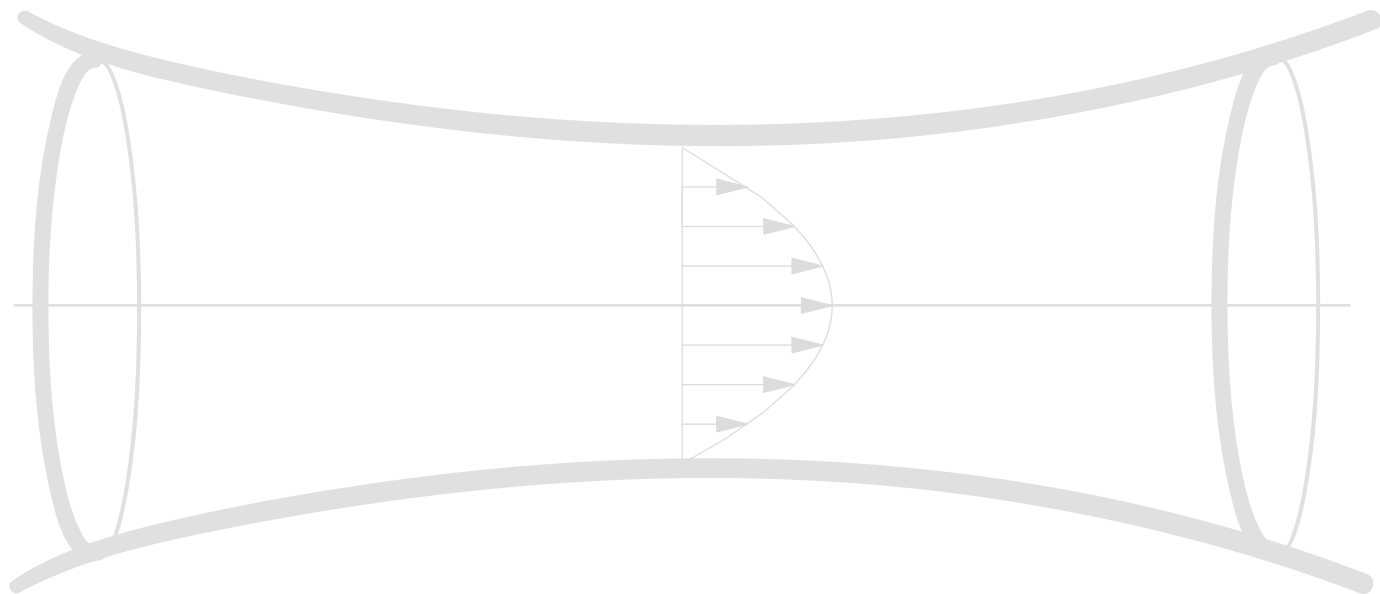
- starting from Navier Stokes
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# Conclusion



- starting from Navier Stokes
- set of simple equations RNSP
- set of more simple equations Integral
- Good agreement with full Navier Stokes
- “explain” the features of the flow
- boundary conditions for full NS
- real time simulation



F. Chouly, A. Van Hirtum, X. Pelorson, Y. Payan, and P.-Y. Lagrée:  
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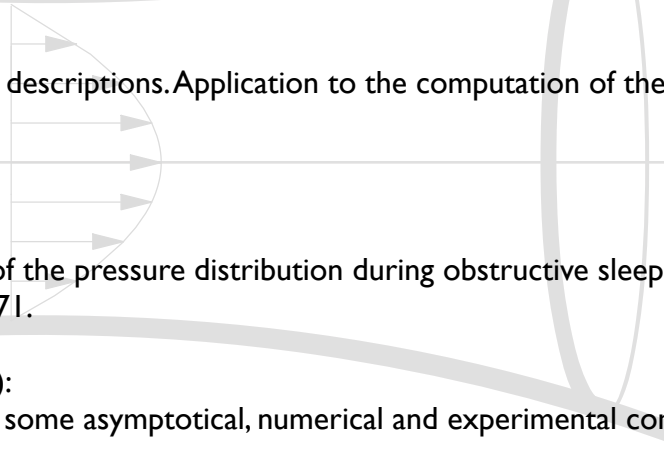
M. Deverge, X. Pelorson, C. Vilain, P.-Y. Lagrée, F. Chentouf, J. Willems & A. Hirschberg (2003):  
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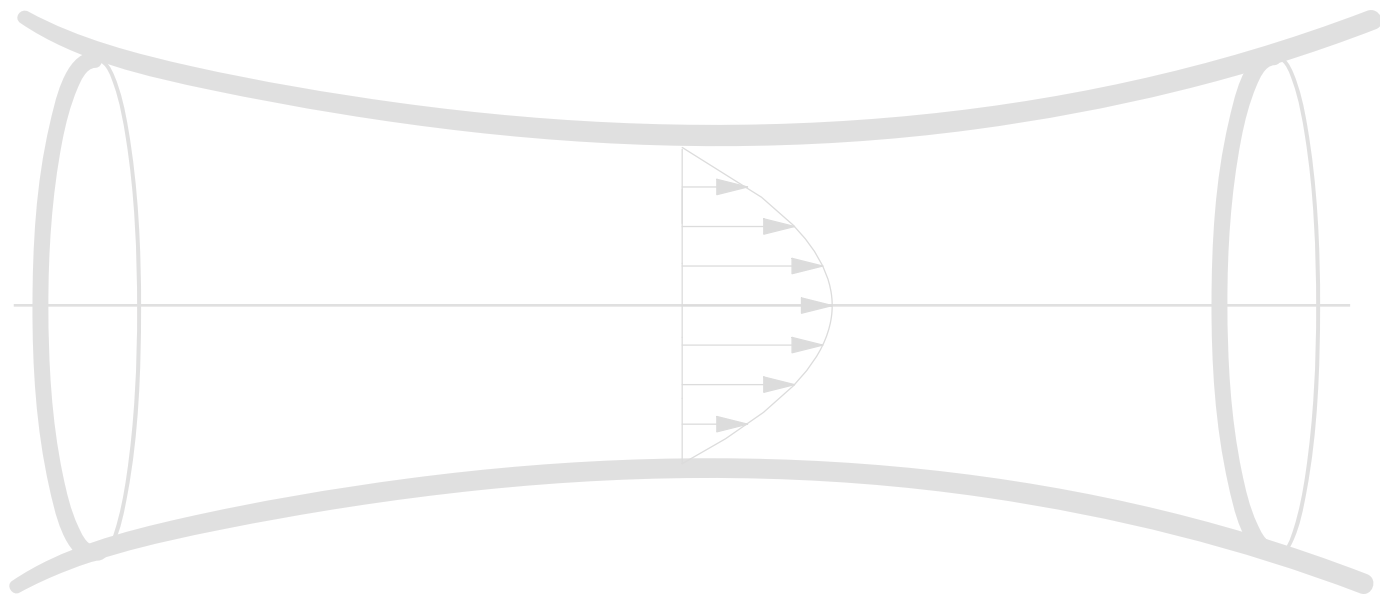
S. Lorthois, P.-Y. Lagrée, J.-P. Marc-Vergnes & F. Cassot. (2000):  
"Maximal wall shear stress in arterial stenoses: Application to the internal carotid arteries",  
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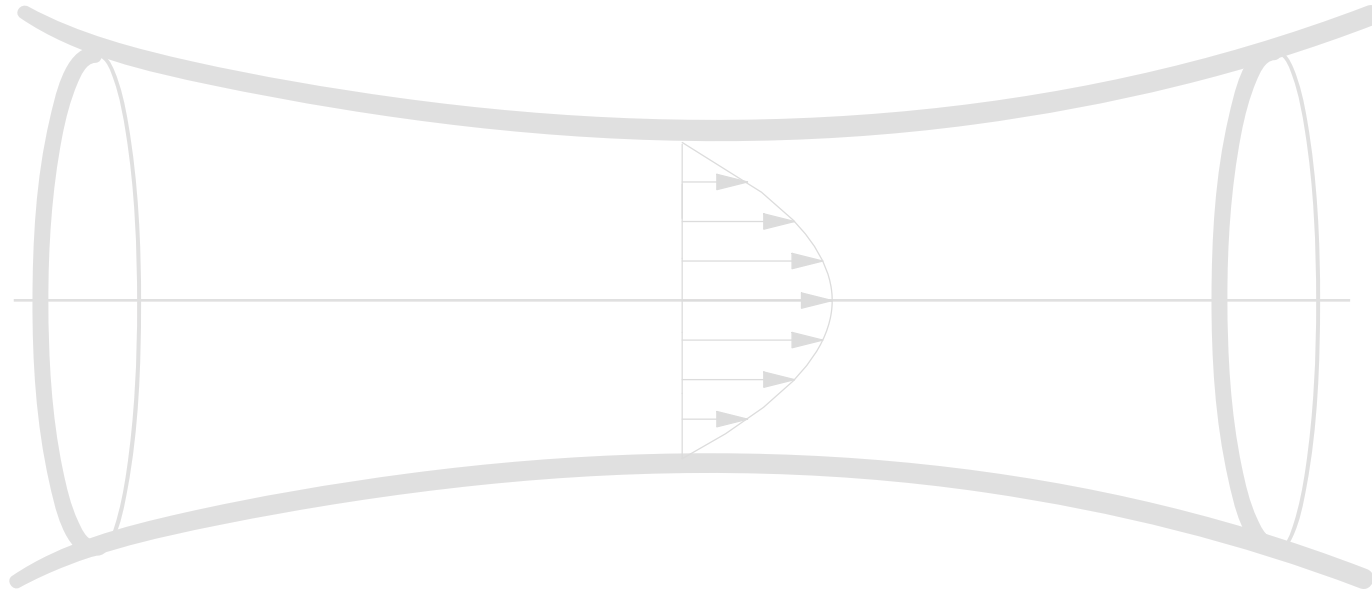
Lorthois S. & Lagrée P.-Y. (2000):  
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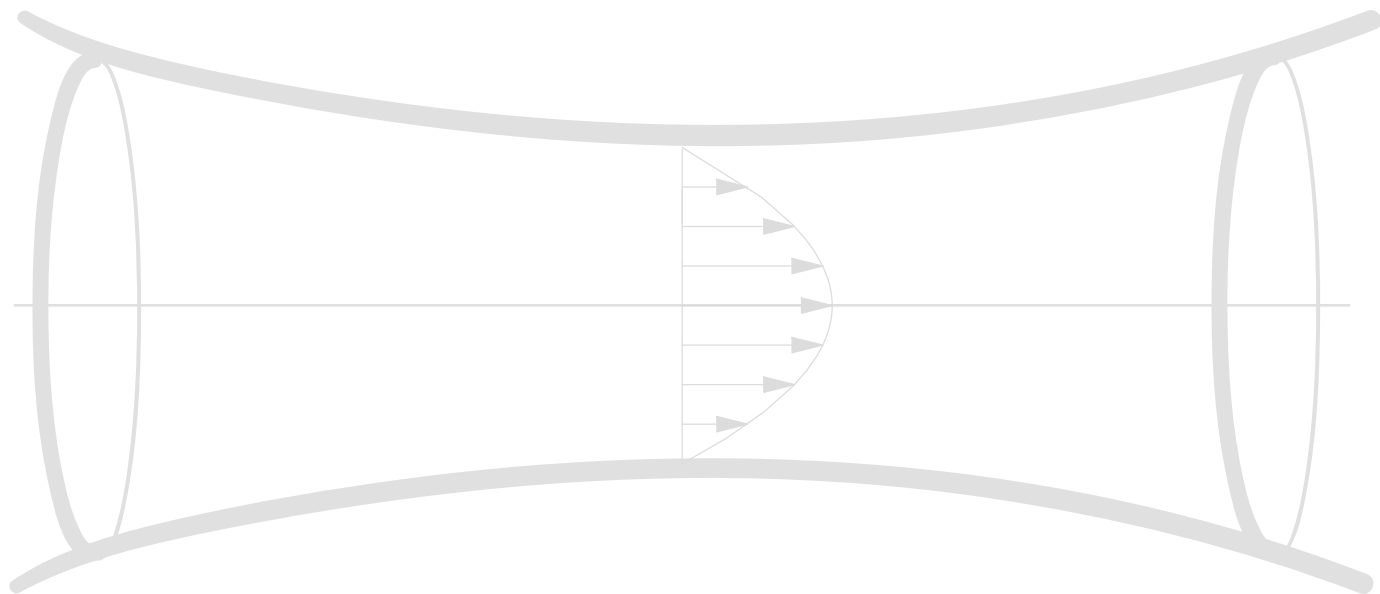
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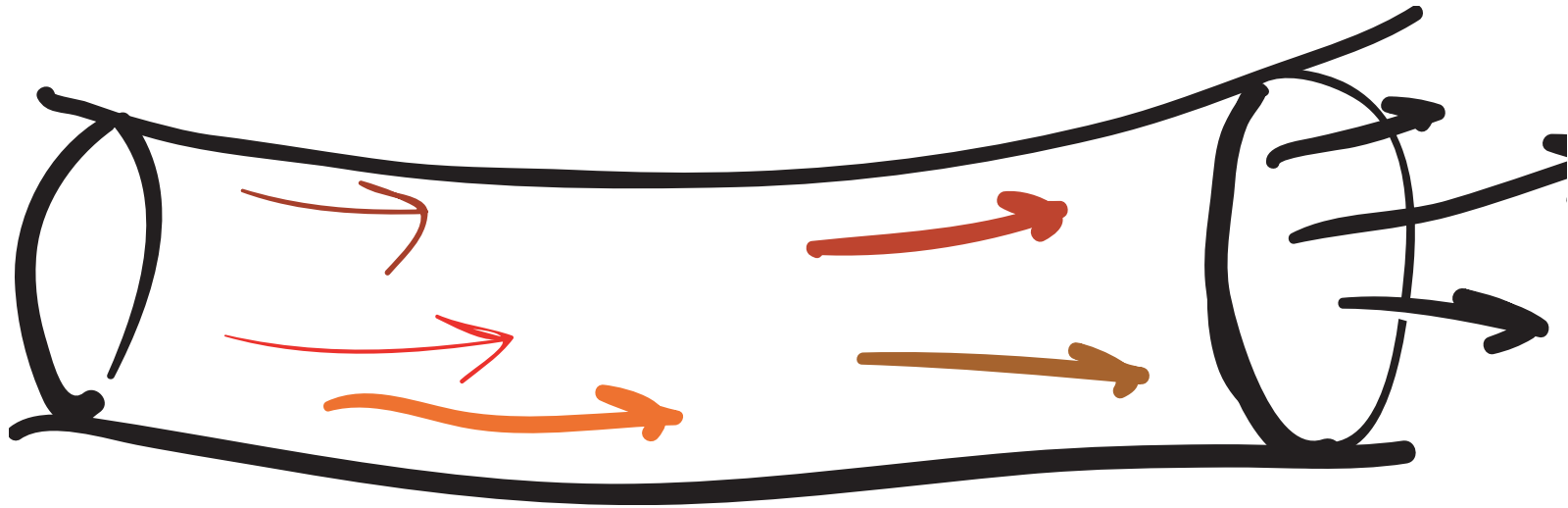






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- Updated version may be found here.





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